

Numerical Methods in Biomechanics

ME-484

Alexandre Terrier

EPFL - Laboratory of Biomechanical Orthopedics

Content - schedule

- W01: Organization, introduction and examples
- W02: External lecturers
- W03: Partial Differential Equations
- W04: Solid mechanics in numerical biomechanics
- W05: Fluid mechanics in numerical biomechanics**
- W06: Midterm project presentations
- W07: The Finite Element Method (FEM), and its extensions
- W08: Midterm evaluation
- W09: Multiphysics, coupling, Comsol
- W10: Multiphysics example 1
- W11: Multiphysics example 2
- W12: Multiphysics example 3
- W13: Final project presentation

Fluid mechanics

- Eulerian description (reference frame)
 - Fluid properties (velocity, pressure, ...) considered at “fixed” points in space
 - Quantities not “followed”, as in Lagrangian description (solid)
- No moving boundaries (of the domain)

Navier-Stokes

Momentum conservation $\rho \frac{\partial \mathbf{u}}{\partial t} + \rho(\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \boldsymbol{\tau}] + \mathbf{F}$

Mass conservation $\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$

State equation $p = p(\rho)$

Constitutive equation $\boldsymbol{\tau} = \mu \dot{\boldsymbol{\gamma}} - \frac{2}{3} \mu (\nabla \cdot \mathbf{u}) \mathbf{I}$

5 dependent variables ($\mathbf{u}$, p , ρ) for 5 PDE

Navier-Stokes

Momentum conservation

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho (\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \boldsymbol{\tau}] + \mathbf{F}$$

inertial acceleration



0 if stationary flow

Navier-Stokes

Momentum conservation

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho (\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \boldsymbol{\tau}] + \mathbf{F}$$

convection



Navier-Stokes is nonlinear

Navier-Stokes

Momentum conservation

$$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho (\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \boldsymbol{\tau}] + \mathbf{F}$$

pressure gradient

Navier-Stokes

volume forces

Momentum conservation $\rho \frac{\partial \mathbf{u}}{\partial t} + \rho(\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \boldsymbol{\tau}] + \mathbf{F}$

Navier-Stokes

viscous stress tensor

Momentum conservation $\rho \frac{\partial \mathbf{u}}{\partial t} + \rho(\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \tau] + \mathbf{F}$

Constitutive equation $\tau = \mu \dot{\gamma} - \frac{2}{3} \mu (\nabla \cdot \mathbf{u}) \mathbf{I}$

Shear strain rate $\dot{\gamma} = (\nabla \mathbf{u} + (\nabla \mathbf{u})^T)$

Shear rate $\dot{\gamma} = |\dot{\gamma}| = \sqrt{\frac{1}{2} \dot{\gamma} : \dot{\gamma}}$

Dynamic viscosity μ

Kinematic viscosity $\frac{\mu}{\rho}$

Navier-Stokes

Momentum conservation $\rho \frac{\partial \mathbf{u}}{\partial t} + \rho(\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \boldsymbol{\tau}] + \mathbf{F}$

Constitutive equation $\boldsymbol{\tau} = \mu \dot{\gamma} - \frac{2}{3} \mu (\nabla \cdot \mathbf{u}) \mathbf{I}$

Shear strain rate $\dot{\gamma} = (\nabla \mathbf{u} + (\nabla \mathbf{u})^T)$

Conservation of mass $\frac{\partial \rho}{\partial t} + \nabla \cdot (\rho \mathbf{u}) = 0$

0 if **incompressible** ($\rho = \text{cte}$) $\Rightarrow \nabla \cdot \mathbf{u} = 0$

$\rho \frac{\partial \mathbf{u}}{\partial t} + \rho(\mathbf{u} \cdot \nabla) \mathbf{u} = \nabla \cdot [-p \mathbf{I} + \underbrace{\mu(\nabla \mathbf{u} + (\nabla \mathbf{u})^T)}_{\mu \nabla^2 \mathbf{u}}] + \mathbf{F}$

Navier-Stokes

- Typical boundary conditions:
 - Walls no slip: $\mathbf{u} = 0$
 - Inlet: $\mathbf{u}$ or p fixed
 - Outlet: $\mathbf{u}$ or p fixed
- Typical initial conditions
 - $\mathbf{u}(t_o) = \mathbf{u}_0 = 0$
 - $p(t_o) = p_0 = 1 \text{ [atm]}$
 - $\rho(t_o) = \rho = \text{constant}$

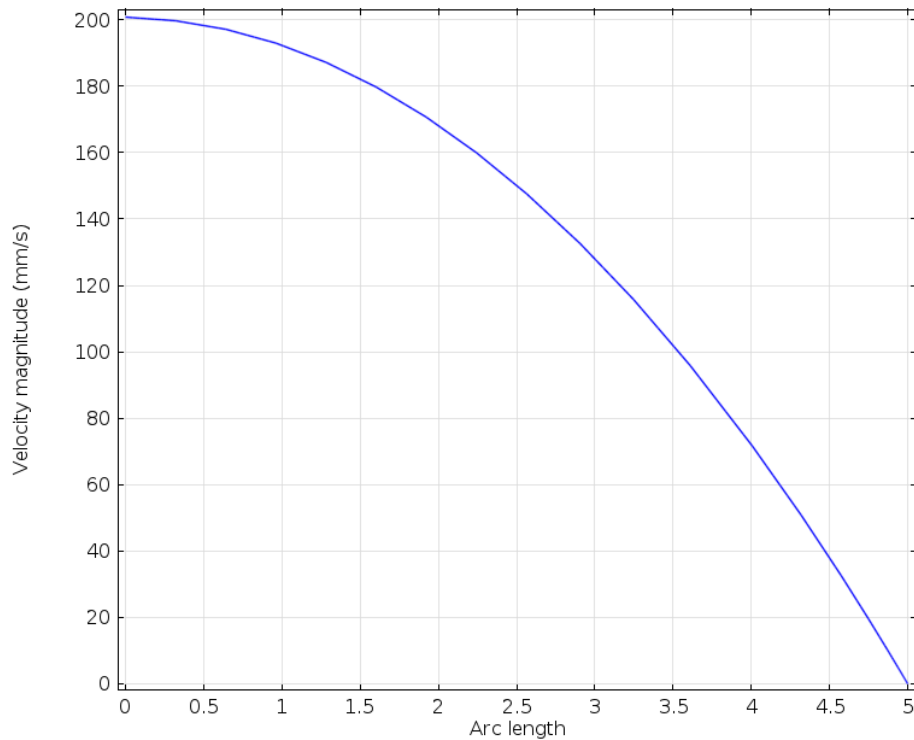
Simplified artery (Poiseuille) flow

- Cylinder (2D axisymmetric)
- Diameter: 10 mm
- Length: 50 mm
- $\rho = 10^3 \text{ kg/m}^3$ and $\mu = 3 \cdot 10^{-3} \text{ Pa}\cdot\text{s}$
- Inlet velocity: 100 mm/s (average laminar)
- Outlet (relative) pressure: 100 mmHg
- Stationary (laminar) flow
- Profiles of $\mathbf{u}$ and p



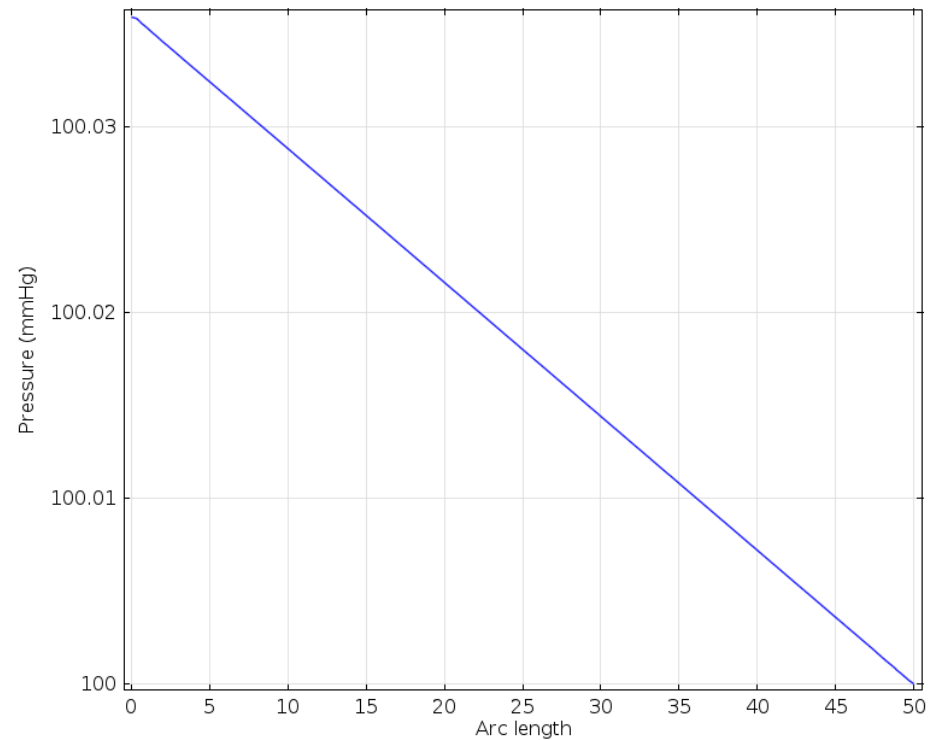
Simplified artery flow

Parabolic velocity profile
(fully developed)



Radial axis

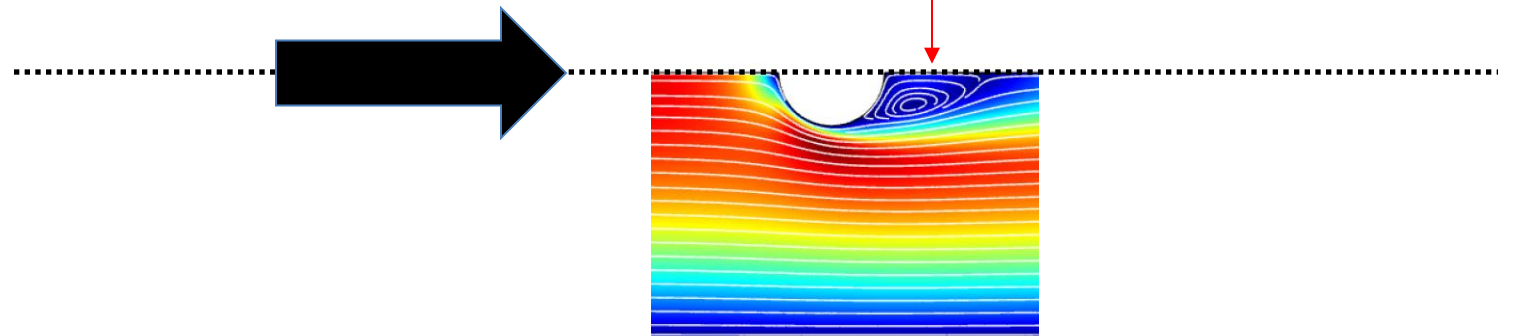
Linear pressure drop



Longitudinal axis

Simplified artery flow with obstacle

- Add spherical obstacle
 - Radius: 1 mm
 - Distance from inlet: 10 mm
- Superpose streamlines on $\mathbf{u}$
- Plot profiles of $\mathbf{u}$ and p
- Change inlet velocity



Shear stress

- Boundary with no slip condition
- Tangential stress applied by the fluid on the wall
- Proportional to viscosity and shear rate $\tau_{\text{wall}} = \mu \dot{\gamma}_{\text{wall}}$
- Arterial wall shear stress (~ 1 Pa)

Mach number

$$\text{Ma} = \frac{|\mathbf{u}|}{a}$$

(a : speed of sound)

- Characterize compressibility
- $\text{Ma} = 0$ for incompressible fluid
- Dimensionless number

Reynolds number

$$\text{Re} = \frac{\rho UL}{\mu} \quad \begin{array}{l} (U: \text{typical velocity}) \\ (L: \text{typical length}) \end{array}$$

- Represented the ratio inertia/viscosity forces
- Characterize turbulence (caused by inertia)
- Dimensionless number
- Local cell Reynolds: $\text{Re}^c = \frac{\rho |\mathbf{u}| h}{2\mu}$, (h : element size)

With last example

- Evaluate flow rate $Q = \iint_{\partial\Omega} \mathbf{u} \cdot \mathbf{ds}$
- Evaluate wall shear stress τ_{wall}
- Evaluate cell Reynolds number Rec
- Evaluate “cylinder” Reynolds number $Re = V \cdot D / \nu$
 - V : average velocity ($V = Q/S$, S : tube section)
 - D : cylinder diameter
 - ν : kinematic viscosity

Flow types

- Incompressible ($\rho = \text{constant}$, $\nabla \cdot \mathbf{u} = 0$, $\text{Ma} = 0$)
- Laminar (streamline): layers sliding over one another
- Turbulent: ($\text{Re} \gtrsim 4000$)
- Non-Newton (fluid): viscosity depends on shear rate
- Euler (inviscid): no viscosity
- Stokes flow: no inertia (N-S becomes linear), $\text{Re} \ll 1$
- Irrotational: no vorticity ($\nabla \times \mathbf{u} = 0$)
- Non-isothermal: temperature dependent variable
- Poiseuille: $\Delta P = \frac{8\mu L Q}{\pi r^4}$

Non-newtonian blood flow

- Viscosity decreases with shear rate
 $0.1 < \dot{\gamma} < 1000 \text{ 1/s}$
- Viscosity increases with vessel diameter (< 1 mm)
Fahraeus-Lindqvist effect

Viscosity increases with hematocrit

Ht : 40→70% $\Rightarrow \mu$ approximately doubles

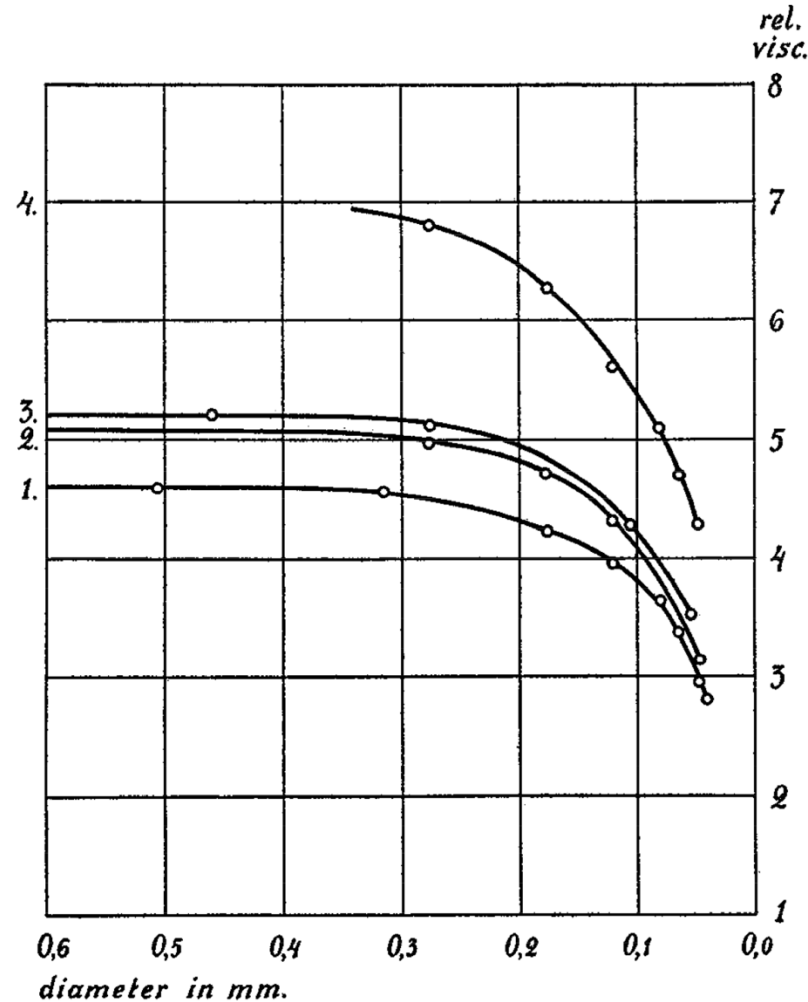
- Viscosity decreases with temperature ($\sim 2\%/^{\circ}\text{C}$)
cold foot / brain

THE VISCOSITY OF THE BLOOD IN NARROW CAPILLARY TUBES

ROBIN FÅHRÆUS AND TORSTEN LINDQVIST

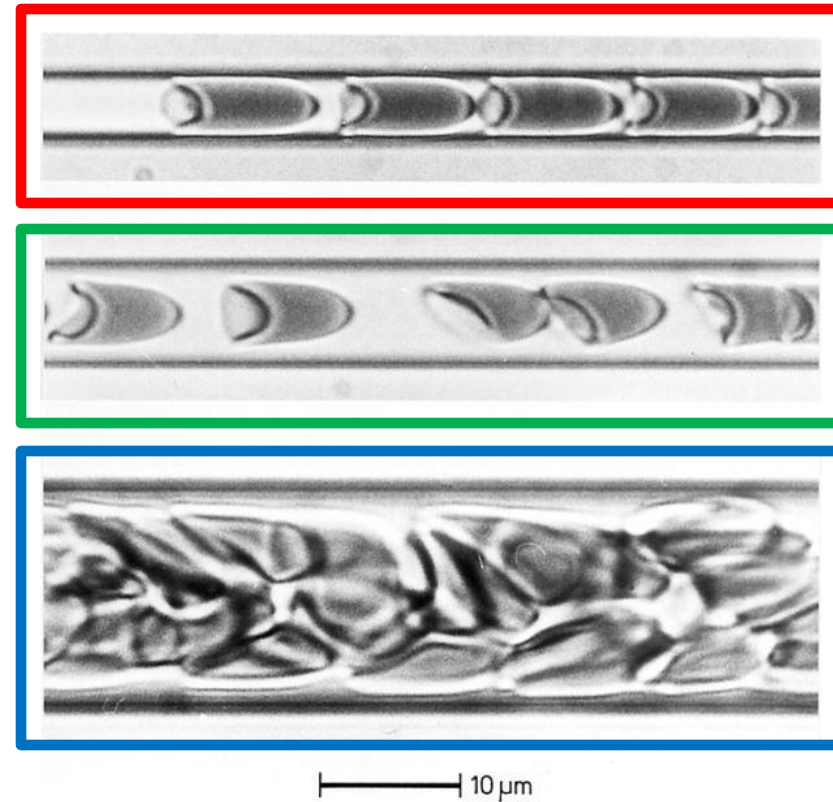
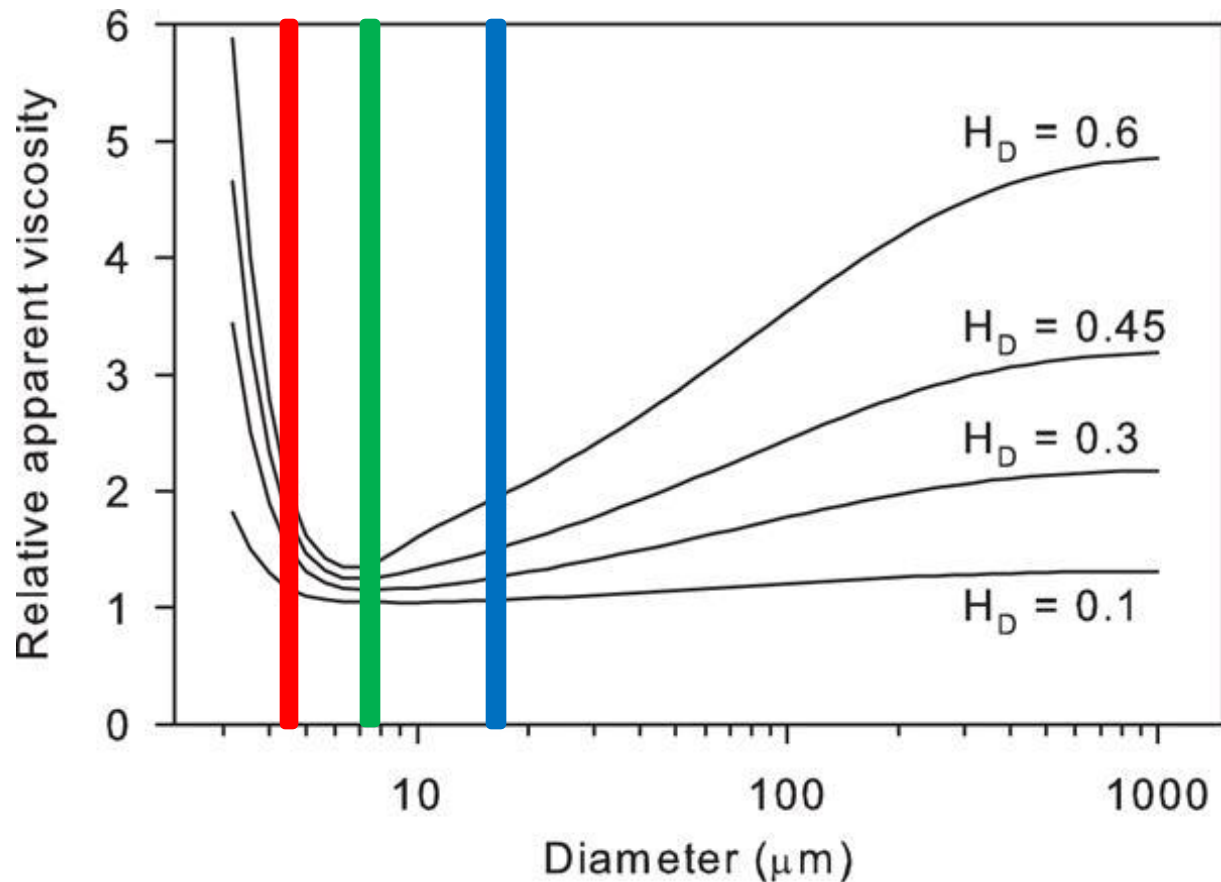
From the Pathological Institute, Uppsala, Sweden

Received for publication December 6, 1930

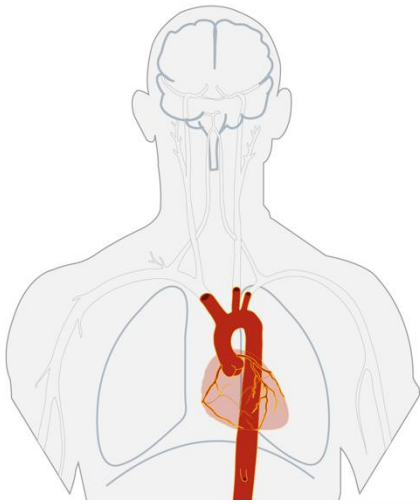


Relative viscosity
for 4 series with
different reference
plasma viscosity

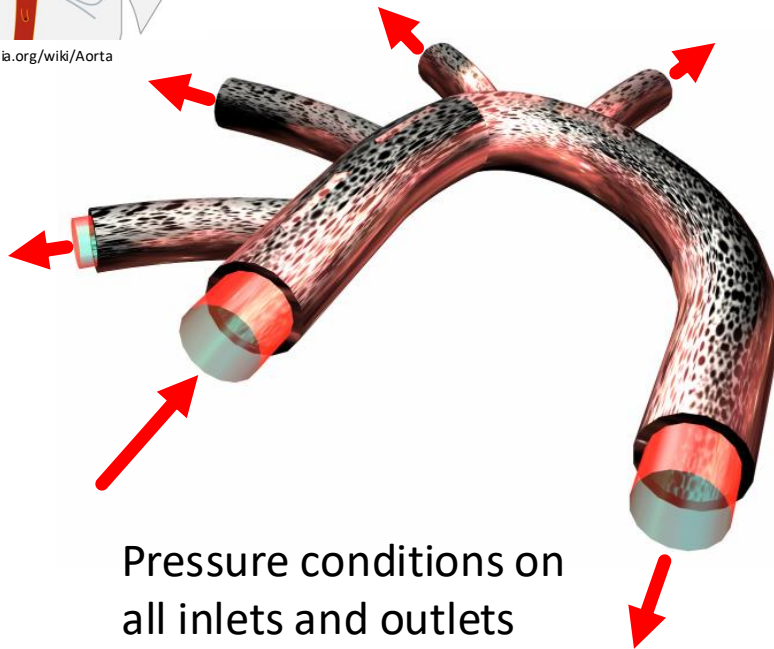
Viscosity vs diameter & hematocrit



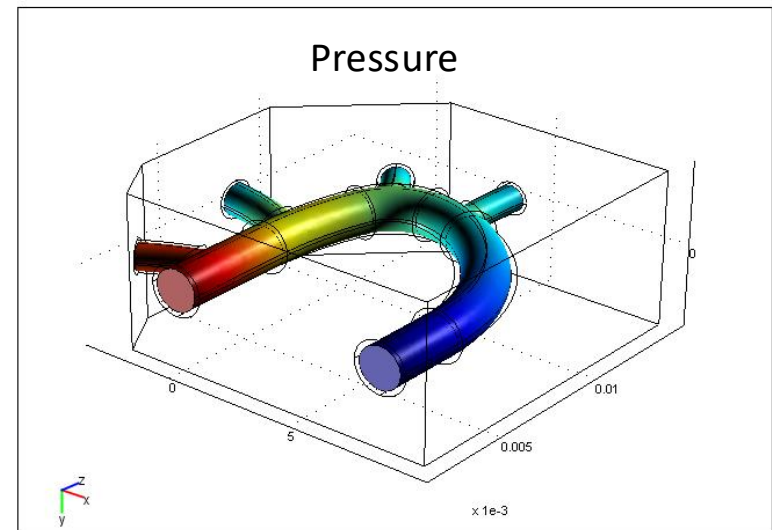
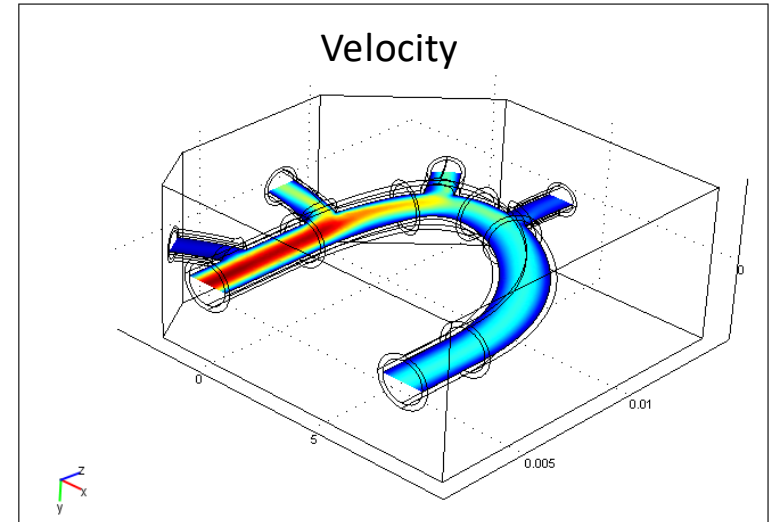
Dynamic flow in aorta



<https://en.wikipedia.org/wiki/Aorta>

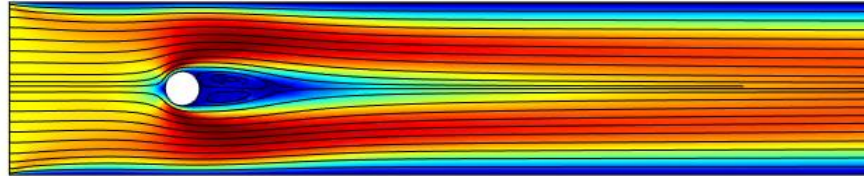


Pressure conditions on
all inlets and outlets

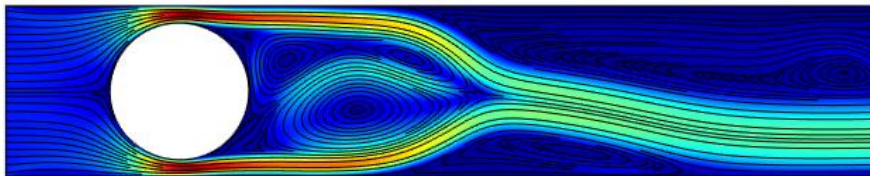
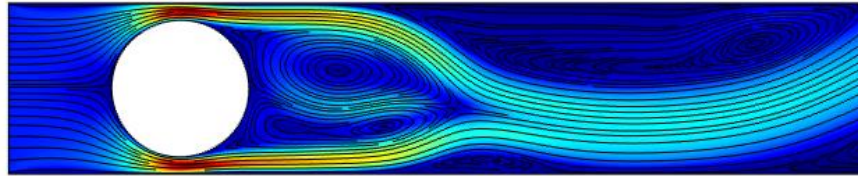


Turbulent flow

Laminar flow

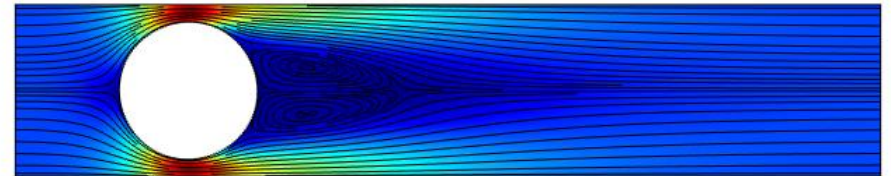


Turbulent flow



Refine mesh

Direct Numerical Simulation (DNS)

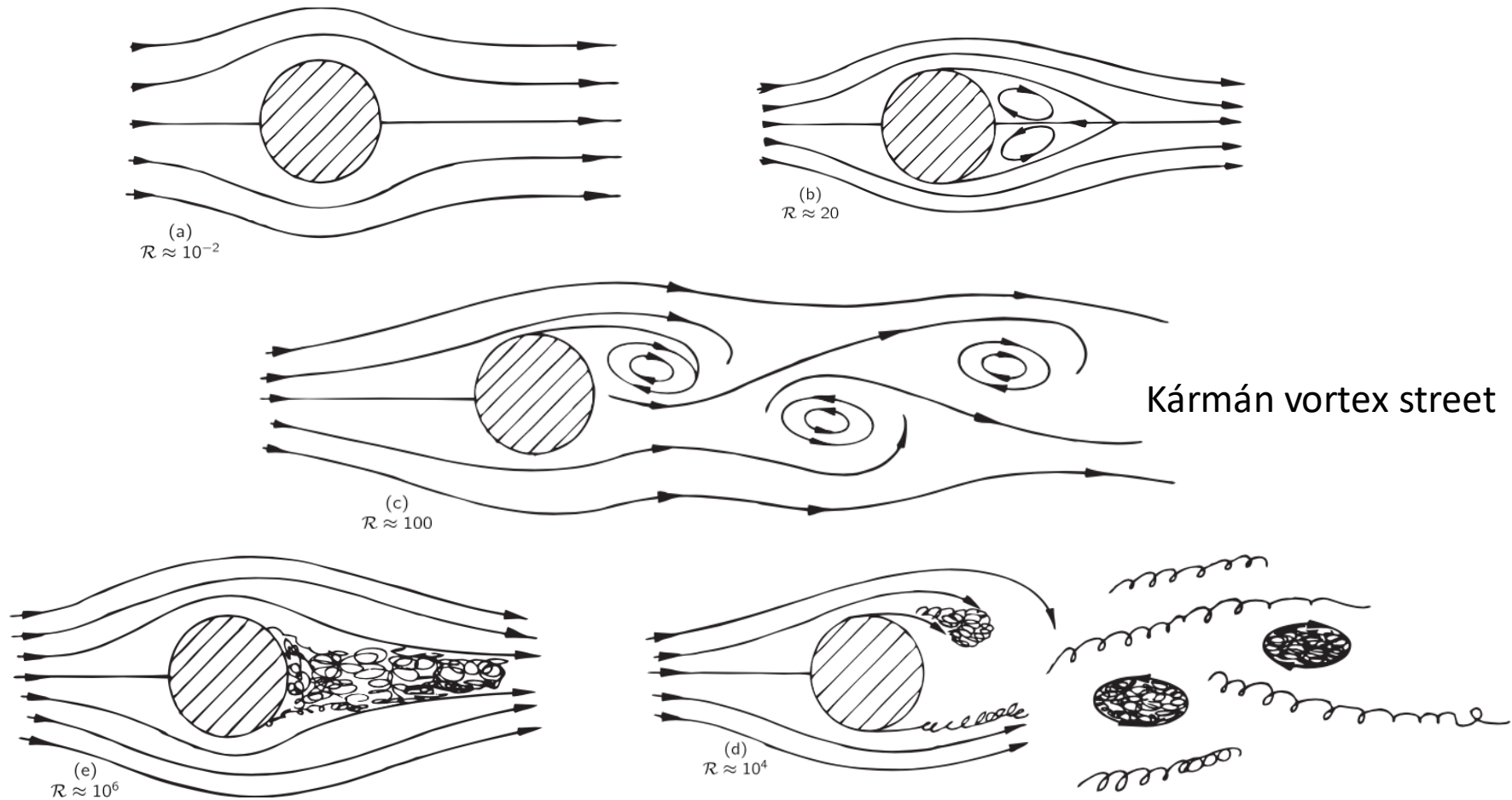


Turbulent model

Reynolds-Averaged Navier-Stokes (RANS)

Turbulent flow

Flow past a cylinder for various Reynolds numbers



References

- Fluid Mechanics for Engineers, Schobeiri, 2010
- Cardiovascular Mathematics, Modeling and simulation of the circulatory system, Luca Formaggia, Alfio Quarteroni and Alessandro Veneziani
- G.K. Batchelor, *An Introduction To Fluid Dynamics*, Cambridge University Press, 1967
- R.L. Panton, *Incompressible Flow*, 2nd ed., John Wiley & Sons, 1996
- The Feynman Lectures on Physics, 1964 (www.feynmanlectures.info)