

- Assume the following specifications:
 - Power $Q = 3000 \text{ MW(th)}$
 - Moderator/fuel ratio $V_{\text{H}_2\text{O}}/V_{\text{F}} = 1.9$
 - Inlet coolant temperature $T_i = 290 \text{ C}$
 - Outlet coolant temperature $T_{\text{out,max}} = 330 \text{ C}$
- Find:
 - Linear heat rate q'_{max} assuming a fuel conductivity $k_{\text{F}} = 2.25 \frac{\text{W}}{\text{m}\cdot\text{K}}$ and a safety margin factor of 0.6
 - Surface heat flux q''_{max} using a DNBR of 1.3 and a q''_{crit} of 150 W/cm^2
 - Fuel radius
 - Lattice pitch
 - Core volume and dimensions
 - Core-averaged power density
 - Number of fuel elements
 - Coolant mass flow rate
 - Mean coolant velocity.

- Linear heat rate q'_{\max} assuming a fuel conductivity $k_F = 2.25 \frac{W}{m \cdot K}$ and a safety margin factor of 0.6
 - Let's assume $T_{melt} = 2973 K$ and $T_{co} = 290 C = 563K \rightarrow \Delta T_{\max} = T_{melt} - T_{co} = 2410 K$
 - We can neglect the temperature jump in gap, clad and coolant $\rightarrow \Delta T_{\max} = \sim \frac{1}{4\pi k_f} q'_{\max}$
 - Therefore, $q'_{\max} = 4\pi k_f \cdot \Delta T_{\max} \sim 68 \frac{kW}{m} = 680 \frac{W}{cm}$
 - We then apply a safety margin of 0.6 to reach a more realistic value of $q'_{\max} = 680 \cdot 0.6 = 400 \frac{W}{cm}$
- Surface heat flux q''_{\max} using a DNBR of 1.3 and a q''_{crit} of 150 W/cm^2
 - We can use the definition of DNBR = $\frac{q''_{\text{crit}}}{q''_{\max}} = 1.3$
 - Therefore $q''_{\max} = \frac{q''_{\text{crit}}}{1.3} \sim 115 \frac{W}{cm^2}$

- Fuel radius

- We can use the equivalence between linear heat rate and heat flux at the rod outer surface

$$q''_{\max} \cdot 2\pi r_f \cdot dz = q'_{\max} \cdot dz$$

- Therefore $r_f = \frac{q'_{\max}}{2\pi q''_{\max}} = 0.56 \text{ cm}$

- Lattice pitch

- Assuming a square lattice with pitch $p \rightarrow V_m = p^2 - \pi r_f^2$ and $V_f = \pi r_f^2$
 - Knowing the moderator to fuel volume ratio

$$\frac{V_m}{V_f} = \frac{p^2 - \pi r_f^2}{\pi r_f^2} = \frac{p^2}{\pi r_f^2} - 1 = 1.9 \rightarrow p^2 = 2.9 \cdot \pi r_f^2 \rightarrow p = 1.70$$

- Core volume and dimensions

- To estimate core volume and dimensions we assume the core to be cylindrical with radius R and height going from $-H/2$ to $H/2$ (total height H)
- We start from the power distribution in a cylindrical core (see lecture on thermal analysis) which follows a modified bessel function along the radius r , and a cosine function along the axial coordinate z :

$$q'''(r, z) = q'''(0,0) \cdot J_0\left(\frac{2.405}{R}r\right) \cdot \cos\left(\frac{\pi}{H}z\right) = q'''_{max} \cdot J_0\left(\frac{2.405}{R}r\right) \cdot \cos\left(\frac{\pi}{H}z\right)$$

- If we integrate the power density over the whole volume ($dV = 2\pi r \cdot dr \cdot dz$) we get total core power

$$\int_V q'''(r, z) dV = q'''_{max} \cdot \int_0^R J_0\left(\frac{2.405}{R}r\right) \cdot 2\pi r \cdot dr \int_{-\frac{H}{2}}^{\frac{H}{2}} \cos\left(\frac{\pi}{H}z\right) dz$$

- The two integrals are equal to

$$\int_0^R J_0\left(\frac{2.405}{R}r\right) \cdot 2\pi r \cdot dr = \pi R^2 \cdot \frac{1}{2.32}$$

$$\int_{-\frac{H}{2}}^{\frac{H}{2}} \cos\left(\frac{\pi}{H}z\right) dz = H \cdot \frac{1}{1:57}$$

- Substituting the two integrals

$$\int_V q''''(r, z) dV = Q = q'''_{max} \cdot \pi R^2 \cdot \frac{1}{2.32} H \cdot \frac{1}{1:57} = q'''_{max} \cdot \frac{1}{2.32} \cdot \frac{1}{1:57} \cdot V$$

- We can substitute the maximum linear heat rate for the maximum power density in the unit element of the lattice

$$q'''_{max} \cdot p^2 \cdot dz = q'_{max} dz \rightarrow q'''_{max} = q'_{max} / p^2$$

- And finally obtain the volume:

$$V = \frac{Q \cdot 2.32 \cdot 1.57}{q'_{max} / p^2} = 77.4 \text{ m}^3 = 7.74 \cdot 10^7 \text{ cm}^3$$

- To obtain the radius and height we assume a H/2R ratio = 1

$$V = \pi R^2 H = 2\pi R^3 = 77.4 \text{ m}^3$$

$$R = 2.3 \text{ m and } H = 4.6 \text{ m}$$

- Core-averaged power density
 - Going back to the formula

$$Q = q'''_{max} \cdot \frac{1}{2.32} \cdot \frac{1}{1.57} \cdot V$$

- We find that:

$$\overline{q'''} = \frac{Q}{V} = q'''_{max} \cdot \frac{1}{2.32} \cdot \frac{1}{1.57} = 3.87 \cdot 10^7 \frac{W}{m^3}$$

- From this we can also find the physical meaning of the $C_r = 2.32$ and $C_z = 1.57$ factors, which indeed prove to be the **radial and axial power peaking factor** (i.e. the ratio between maximum and average power)

$$2.32 \cdot 1.57 \cdot \overline{q'''} = q'''_{max} \text{ or } C_r C_z = \frac{q'''_{max}}{\overline{q'''}}$$

- Number of fuel elements

- A single fuel element has volume $V_{element} = \pi r^2 H$
- Therefore the number of fuel elements N is

$$N = \frac{V_{core}}{V_{element}} = \frac{\pi R^2 H}{\pi r^2 H} = \frac{\pi R^2}{\pi r^2} \sim 58359$$

- Coolant mass flow rate

- Assuming water heat capacity at the operating temperature to be $c_p = 6.4 \cdot 10^3 \frac{J}{kgC}$ and knowing water change of temperature to be $320-290=30$ C, then

$$\dot{m} = \frac{Q}{c_p \Delta T} = 15.6 \frac{t}{s}$$

- Mean coolant velocity

- The total flow area for the water is $A_{tot} = N A_m = N(p^2 - \pi r_f^2)$
- Because the volumetric flow rate $\dot{m} = \rho A_{tot} \bar{v}$, thus assuming a density of $0.7 \text{ g/cm}^3 = 700 \text{ kg/m}^3$

$$\bar{v} = \frac{\dot{m}}{\rho A_{tot}} = 2 \frac{m}{s}$$