

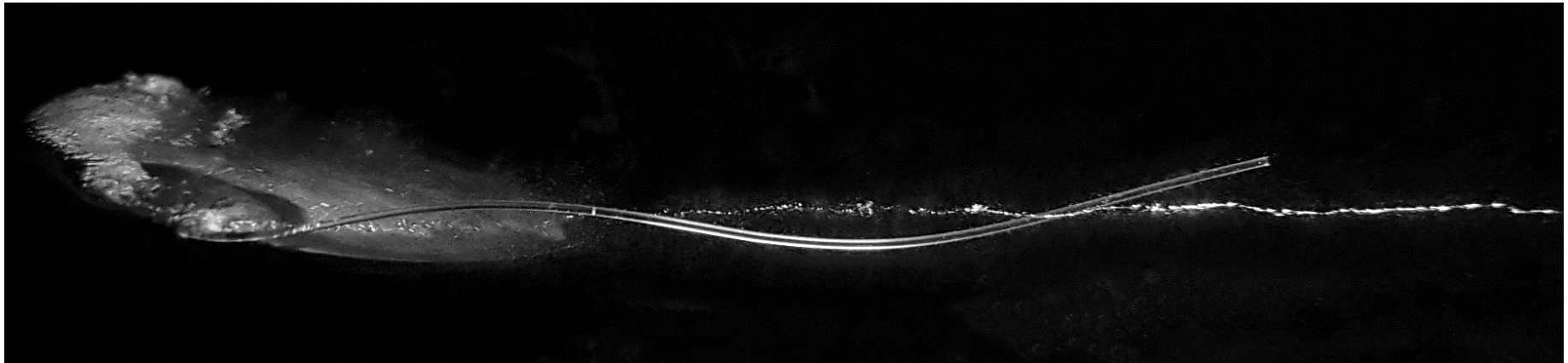
ÉCOLE POLYTECHNIQUE FÉDÉRALE DE LAUSANNE

SGM - 6<sup>th</sup> & 8<sup>th</sup> Semester, Fall 2024

CAVITATION AND INTERFACE PHENOMENA

## Chapter 5: Vortex Cavitation

### *5.3: Flow Control for Tip Vortex Cavitation Mitigation*



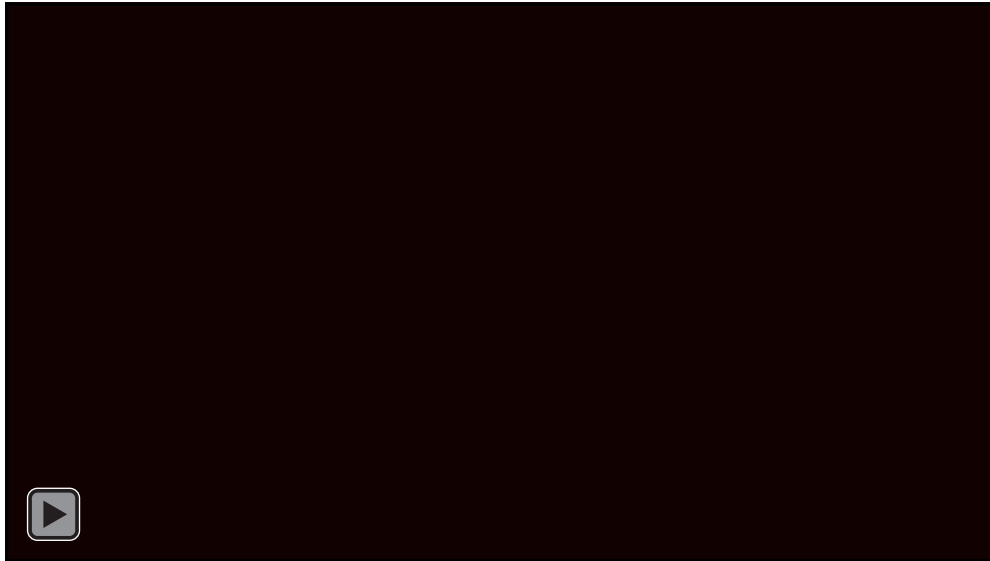
**By Dr Ali Amini**

*Dr Mohamed FARHAT   Assistants: A. Sache, Th. Berger*

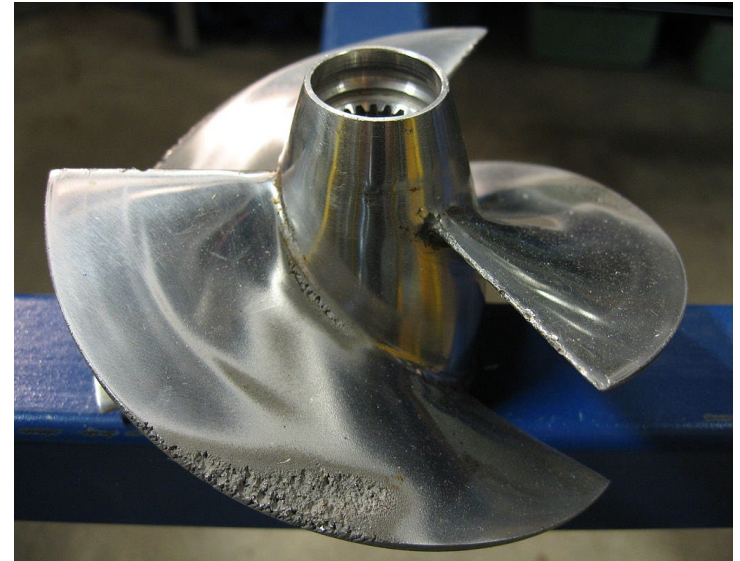
*EPFL – Cavitation Research Group, Avenue de Cour 33 bis, 1007 Lausanne*

## Tip vortex cavitation consequences

- **TVC** may occur in **axial** hydraulic machines
  - Risk of severe erosion in stationary and rotating parts



**Visualization of TVC in an axial turbine**

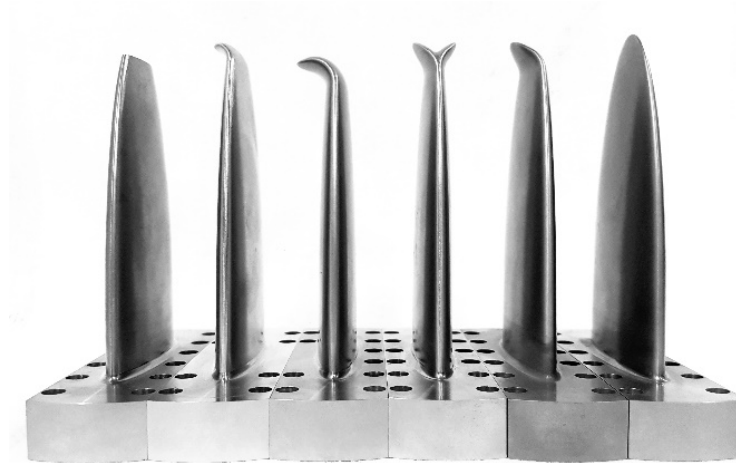


**Erosion of the blade tip in a propeller**

## TVC Mitigation techniques

- Passive or active injections
  - Water or viscoelastic polymer solutions
- Adding artificial roughness to the tip
- Bulbous tips
- ...

# Suppressing Tip Vortex Cavitation by Winglets



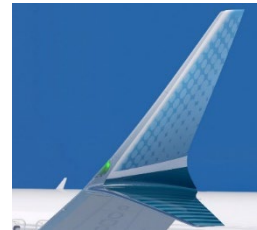
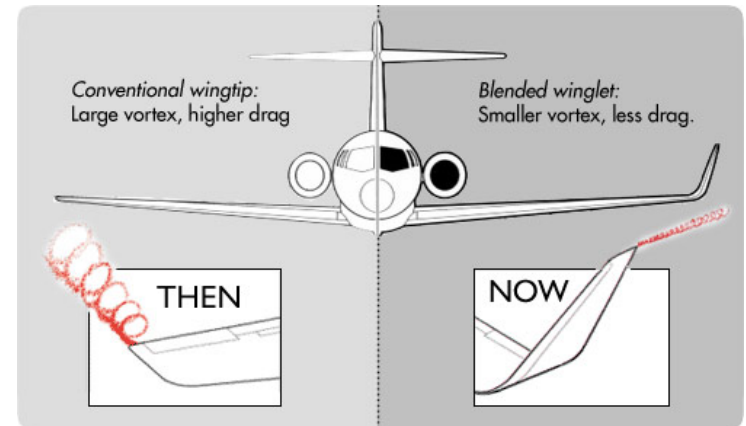
**Ali Amini**, Martino Reclari, Takeshi Sano, Masamichi Iino, and Mohamed Farhat. "Suppressing tip vortex cavitation by winglets." *Experiments in Fluids* 60, no. 11 (2019): 159.



# Introduction

## Tip vortices are a source of concern in aeronautics

- Lift-induced drag & flight hazards
- ✓ A common remedy is appending winglets to wingtips
  - Widely used in commercial airplanes
  - A large variety of winglets design
  - No unique solution exists
  - Each winglet has to be carefully designed based on the objectives and constraints.



# Introduction

**Anti-cavitation lips** already exist, but ...

- A survey on **44** projects performed at **LMH** during the last **15 years** revealed that
    - **Only 27%** → **All on the suction side**
  - Usually have simple geometries
- ✓ A step towards **real winglet designs**
- Simple and generic geometries
  - Physical aspects of the flow

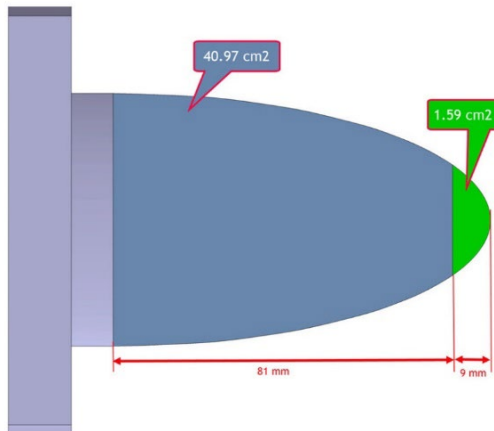
source: [www.andritz.com](http://www.andritz.com)



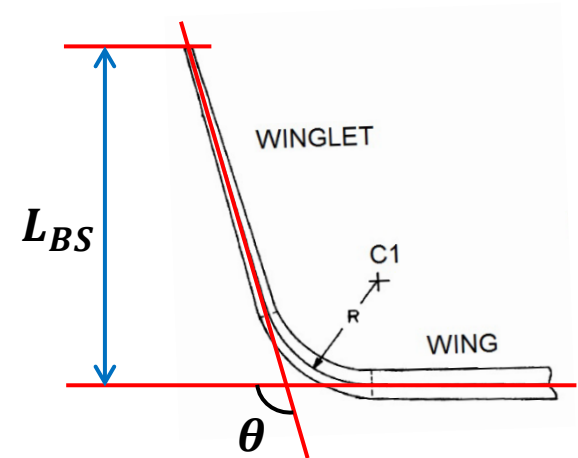
# Experimental Setup

The winglets are realized by **non-planar extensions** of the **original** section at various angles

- Design variables:  $\theta$  &  $L_{BS}$
- Smooth transition of the geometry



The affected area is **max. 3.7%** of the whole lifting surface.



Dihedral angle:  $\theta = 0^\circ, \pm 45^\circ, \pm 90^\circ$

Bent section length:  $L_{BS} = 0.05S$  &  $0.1S$

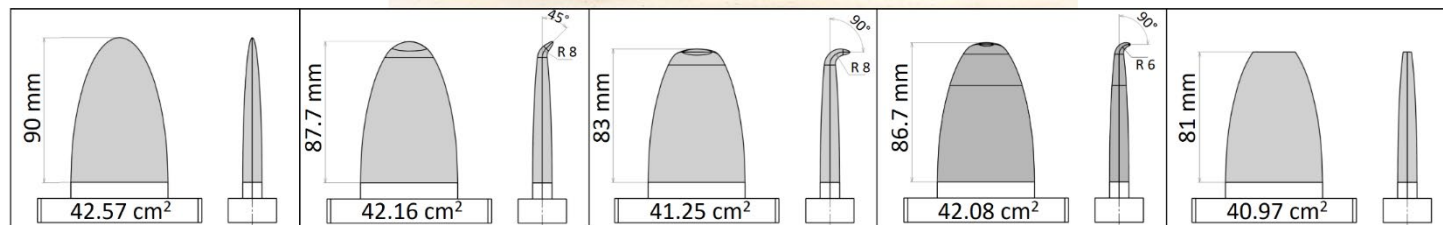
$S$ : span of the baseline hydrofoil (90 mm)

# Experimental Setup

## Manufactured hydrofoils from bronze



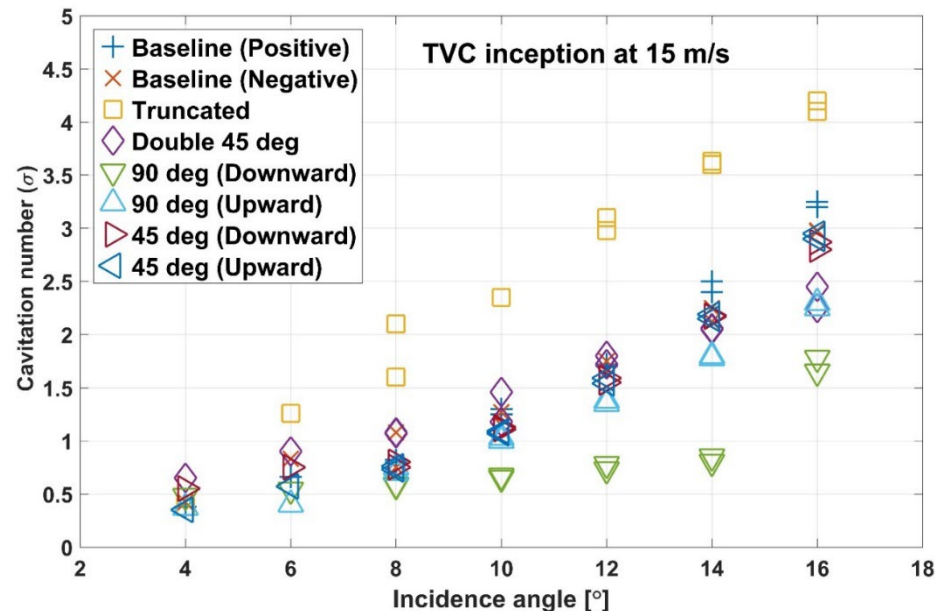
***Upward &  
Downward winglets***



## TVC inception results

Tests performed at  $U_\infty = 15 \text{ m/s}$  and fully saturated water for **10%-bent winglets**.

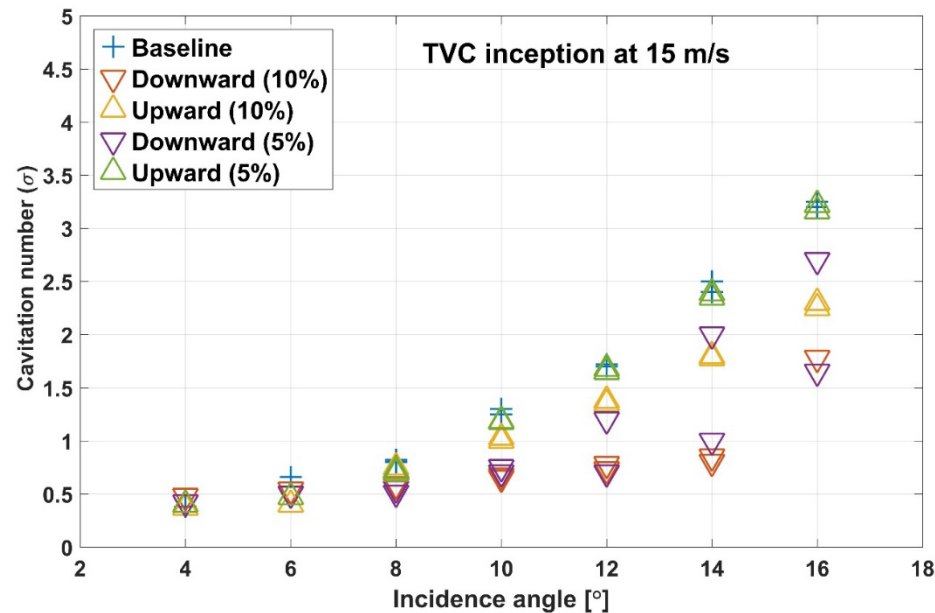
- The **90° winglet bent toward the pressure side** shows an outstanding performance.



## TVC inception results

Tests performed at  $U_\infty = 15 \text{ m/s}$  and fully saturated water for **10%-bent winglets**.

➤ The **5%-bent** winglets are **less effective** in TVC mitigation.

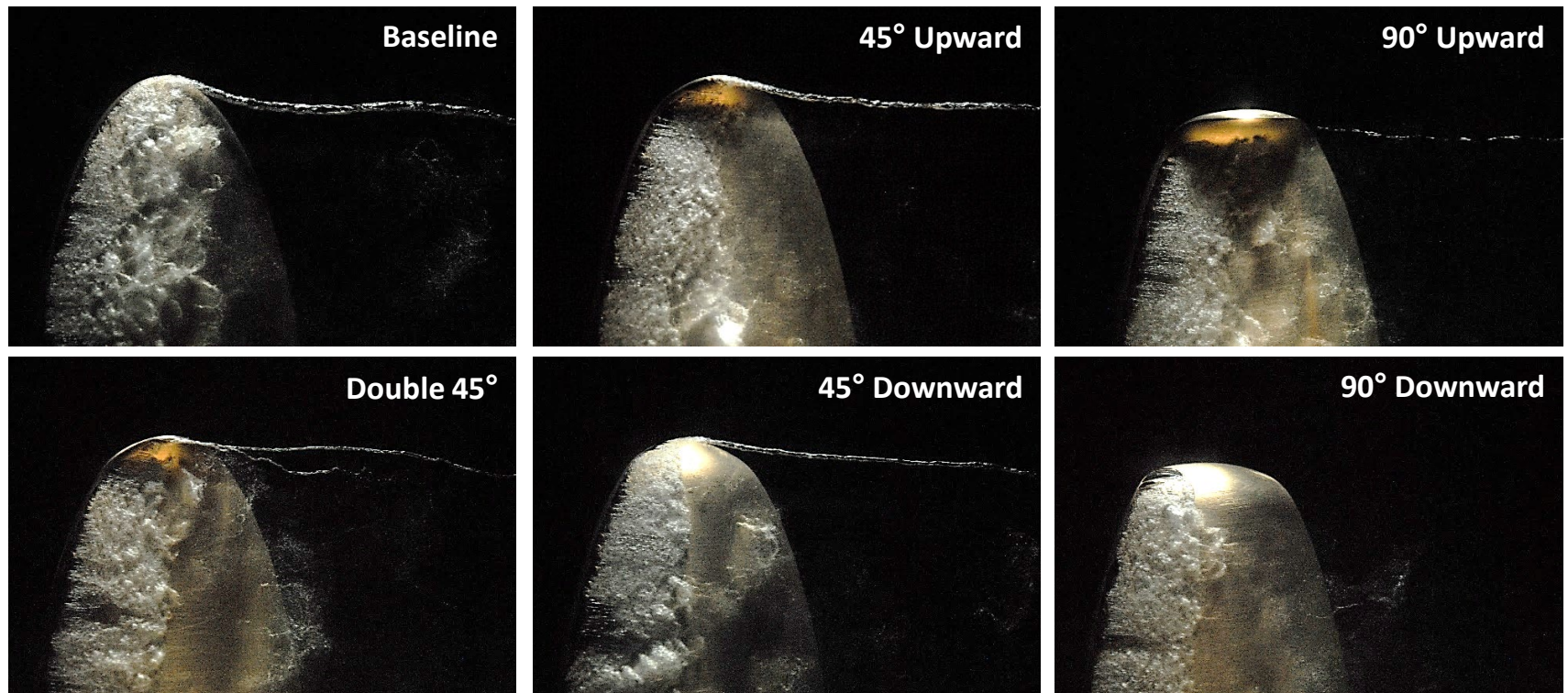




# Flow visualizations

Effect of various winglets on TVC ( $Re = 600,000$ )

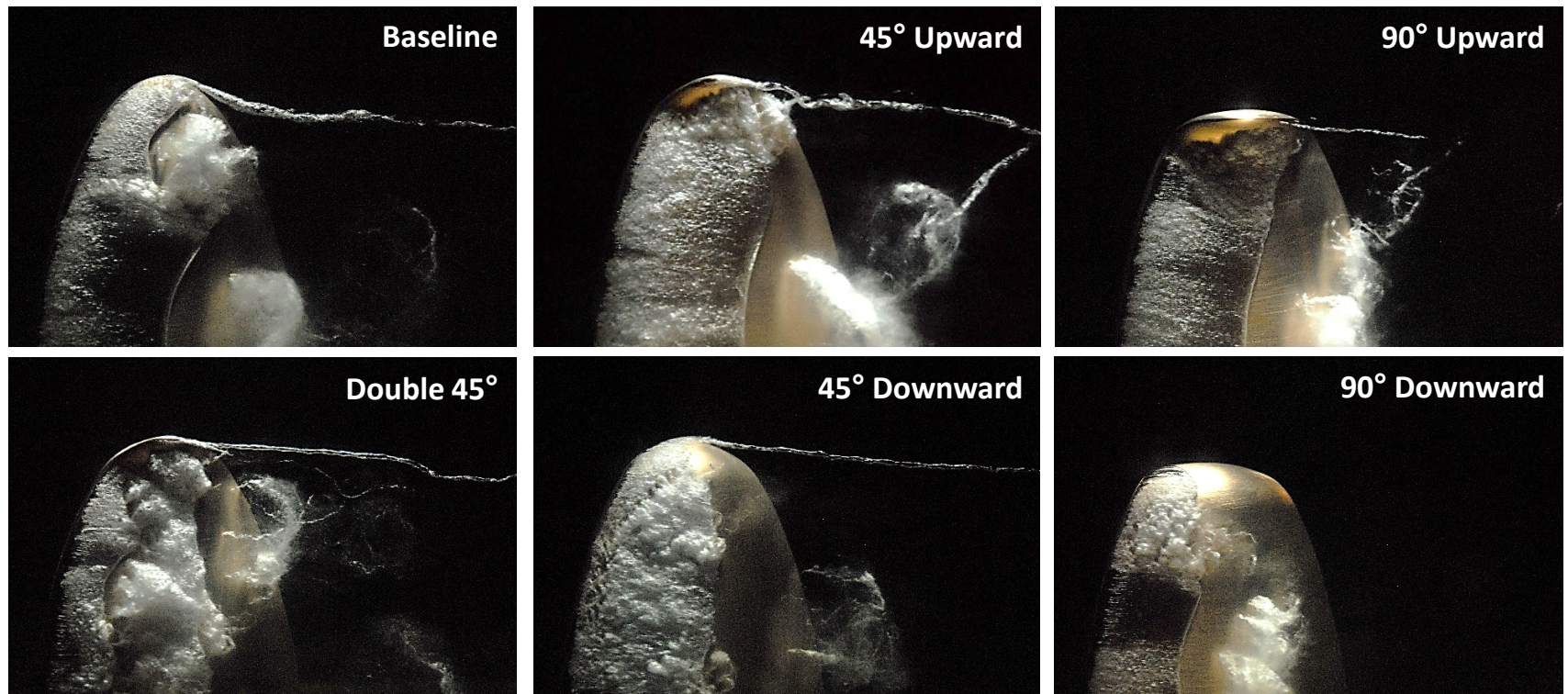
Test conditions:  $U_\infty = 10 \text{ m/s}$ ,  $\alpha = 12^\circ$ ,  $\sigma = 1.2$



# Flow visualizations

Effect of various winglets on TVC ( $Re = 900,000$ )

Test conditions:  $U_\infty = 15 \text{ m/s}$ ,  $\alpha = 12^\circ$ ,  $\sigma = 1.2$

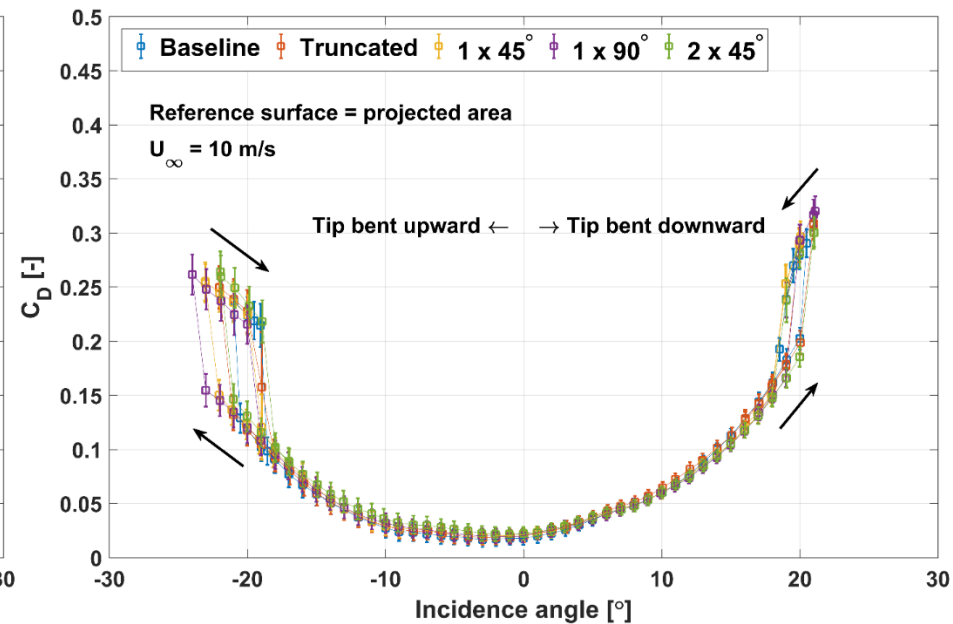
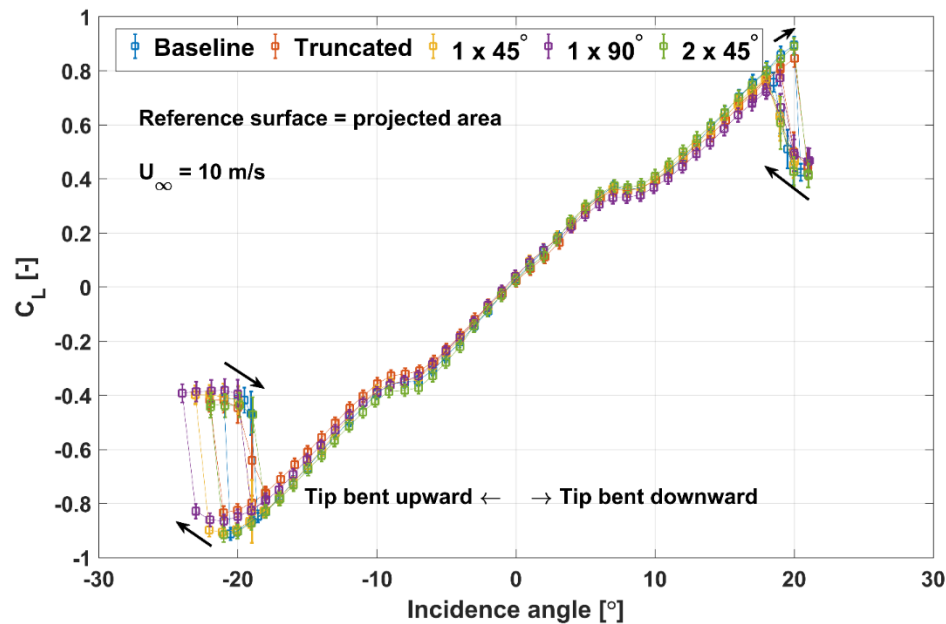




# Lift & drag measurements

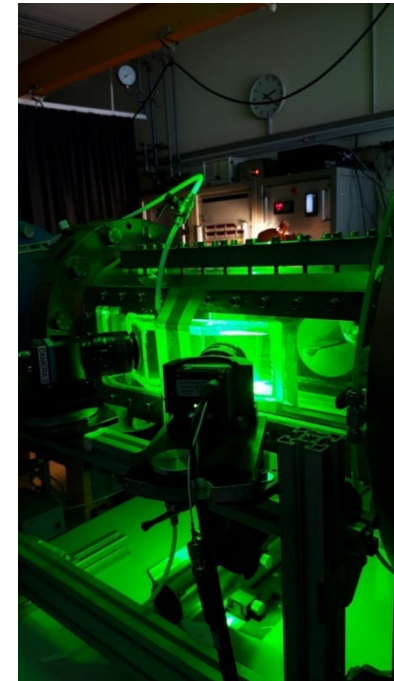
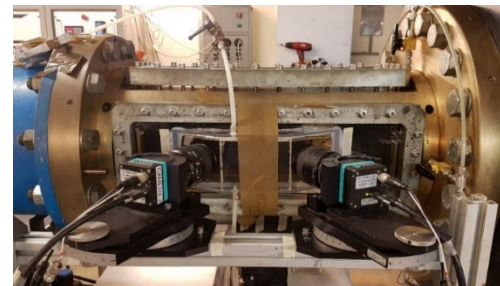
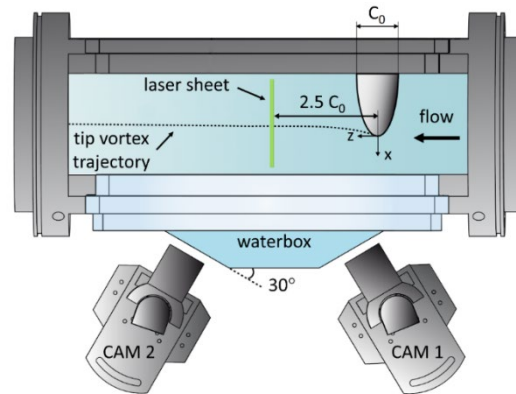
Measurements performed at 10 m/s to avoid cavitation

- Almost **similar** hydrodynamic performances are obtained for all the hydrofoils



# Stereo-PIV setup

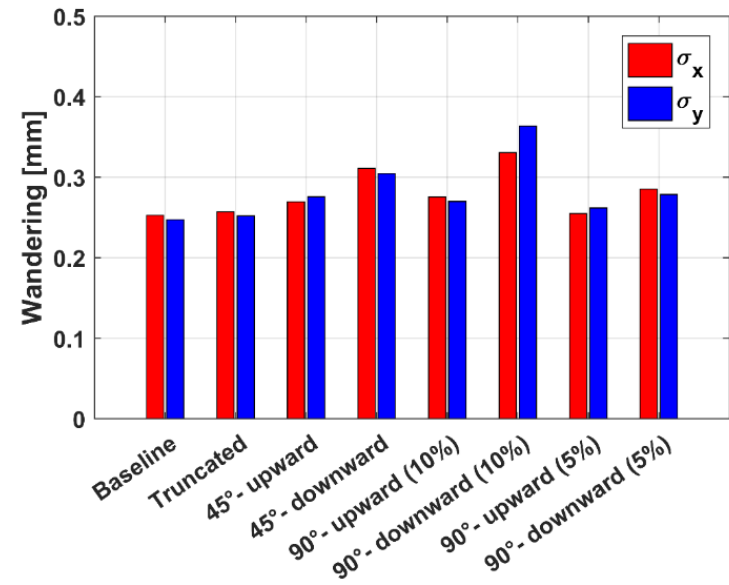
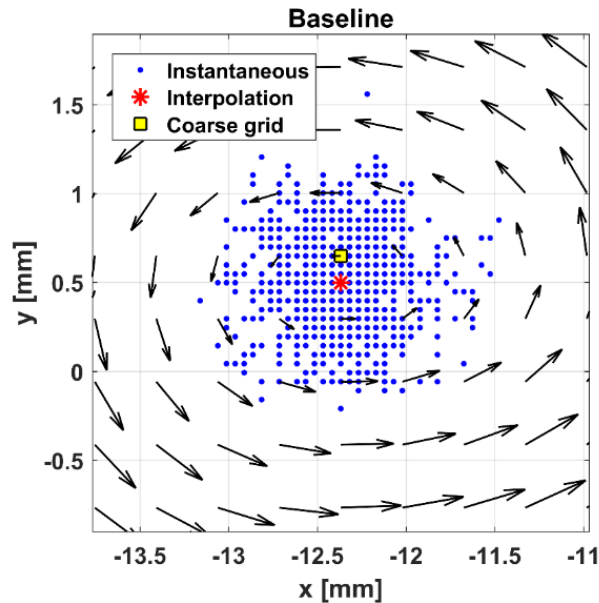
- Double-pulsed laser (135 mJ/pulse)
- Seeding particles
  - Hollow glass spheres
  - Average diameter of 10  $\mu\text{m}$
- 1000 image-pairs for each flow condition
- Vector-to-vector resolution of 0.35 mm
- Wandering motion correction
  - Center detection by Graftieaux algorithm
  - 2D cubic spline interpolation
- Vatistas vortex model curve-fit



# Velocimetry Results

## Wandering effect and its correction

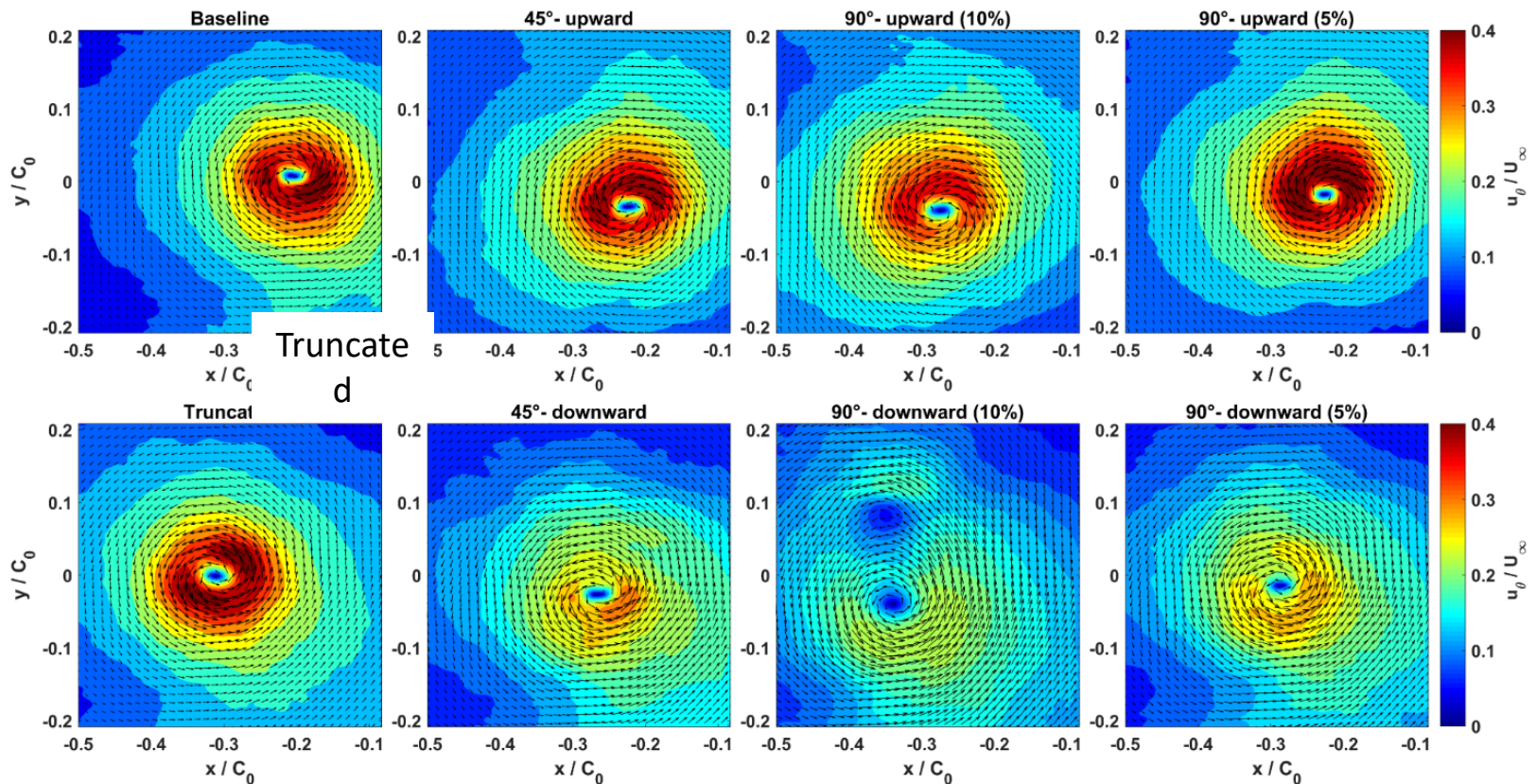
- Flow conditions:  $U_{\infty} = 10 \text{ m/s}$ ,  $\alpha = 12^{\circ}$
- 30 - 40% of variations between different hydrofoils
- Higher for downward configurations
- Sub-grid fluctuations



# Stereo-PIV results

## Contours of the **tangential** velocity

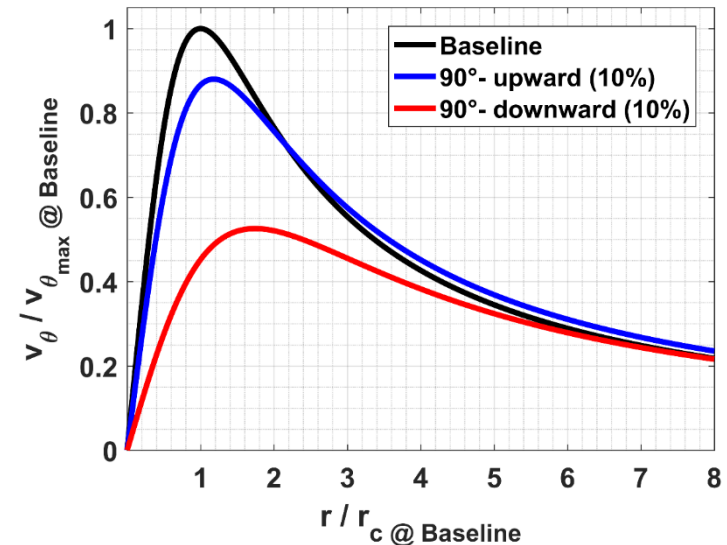
- Flow conditions:  $U_\infty = 10 \text{ m/s}$ ,  $\alpha = 12^\circ$



# Velocimetry Results

Comparison of the tangential velocity profiles:

- Flow conditions:  $U_\infty = 10 \text{ m/s}$ ,  $\alpha = 12^\circ$
- **90°-downward (10%)**
  - Outstanding suppression effects
  - Increasing the viscous radius ( $r_c$ ) by 70 %
  - Decreasing  $v_{\theta_{max}}$  to almost 50 %

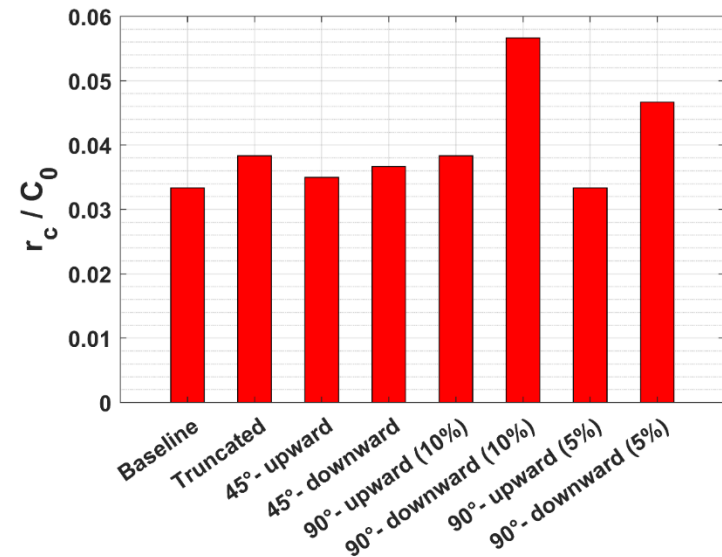
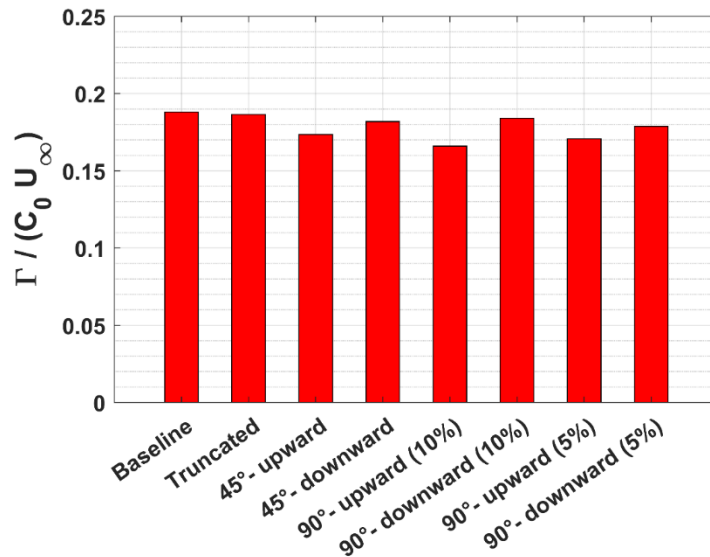


- ✓  $\Gamma$  remains constant while  $r_c$  increases
- ✓ **Viscous core thickening** is the dominant mechanism of TVC mitigation

# Velocimetry Results

Comparison of the tangential velocity profiles

- Same flow conditions:  $U_{\infty} = 10 \text{ m/s}$ ,  $\alpha = 12^{\circ}$
  - Azimuthally-averaged profiles
- $\Gamma$  remains constant while  $r_c$  increases
- **Viscous core thickening** is the dominant mechanism of TVC mitigation

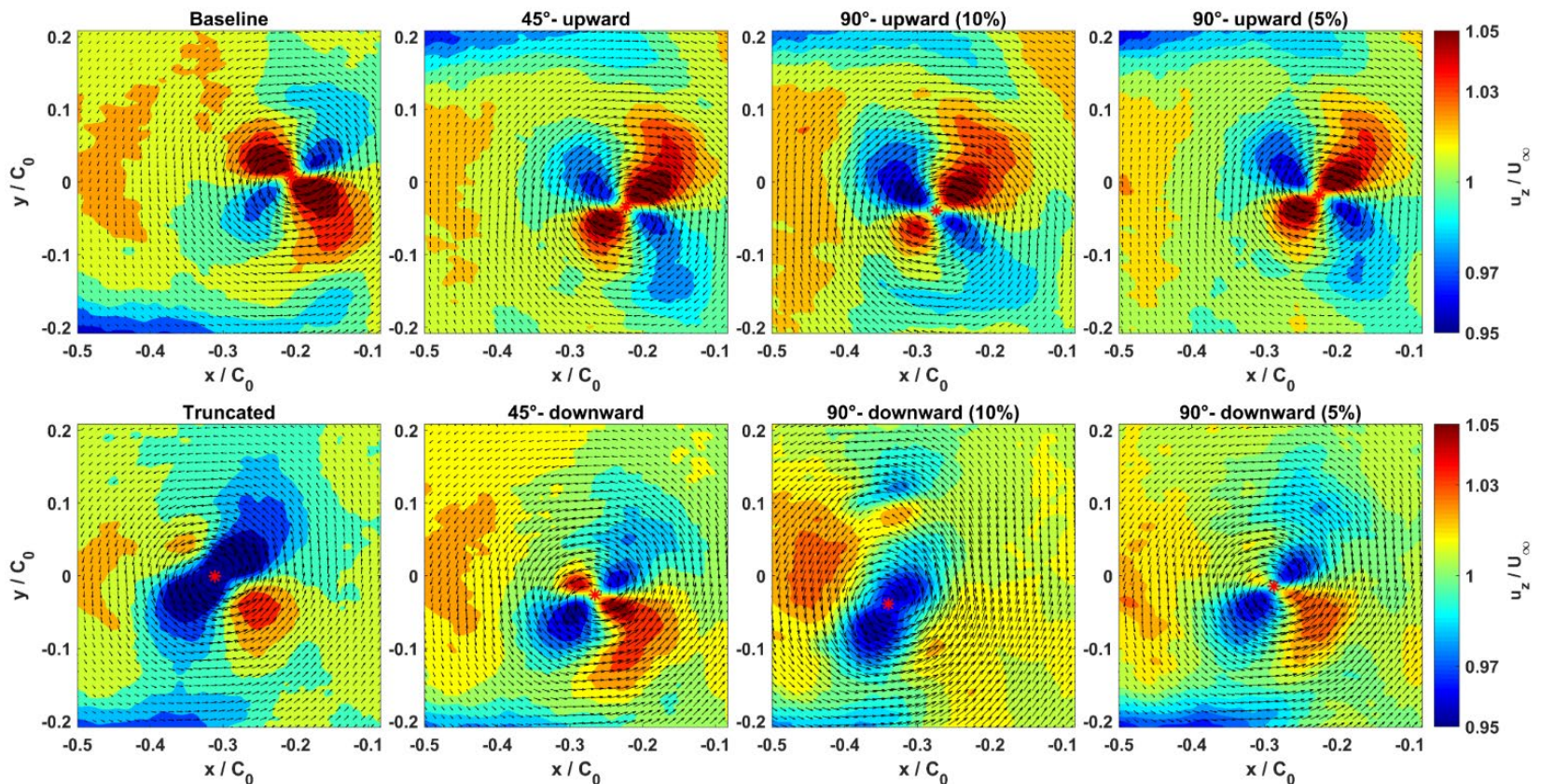




# Velocimetry Results

## Contours of the axial velocity

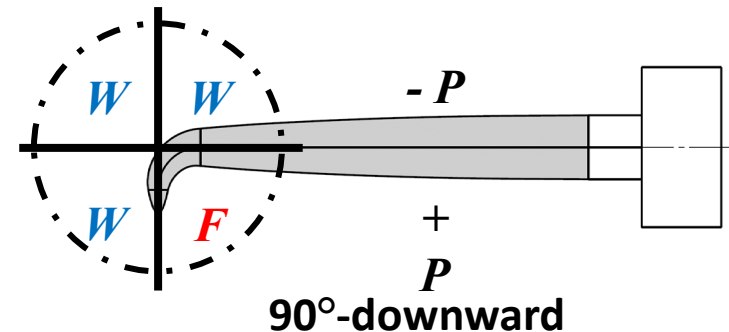
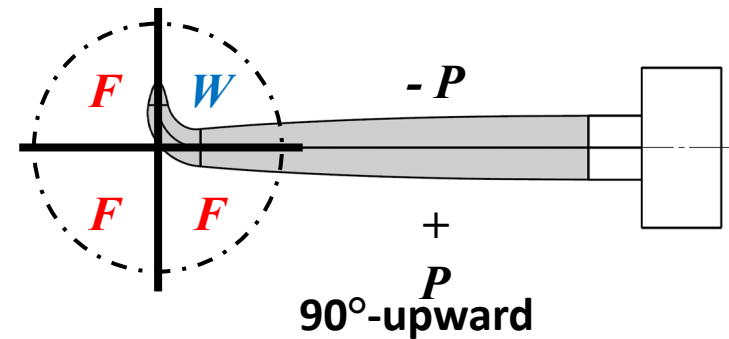
- Flow conditions:  $U_\infty = 10 \text{ m/s}$ ,  $\alpha = 12^\circ$



# Velocimetry Results: Discussion

The higher effectiveness of the downward configurations is due to the fact that:

- 1) A downward-facing winglet facilitates the entrainment of the **wake** into the vortex flow,
- 2) which, in turn, increases the momentum diffusion rates, and thereby,
- 3) smooths down the velocity profiles.

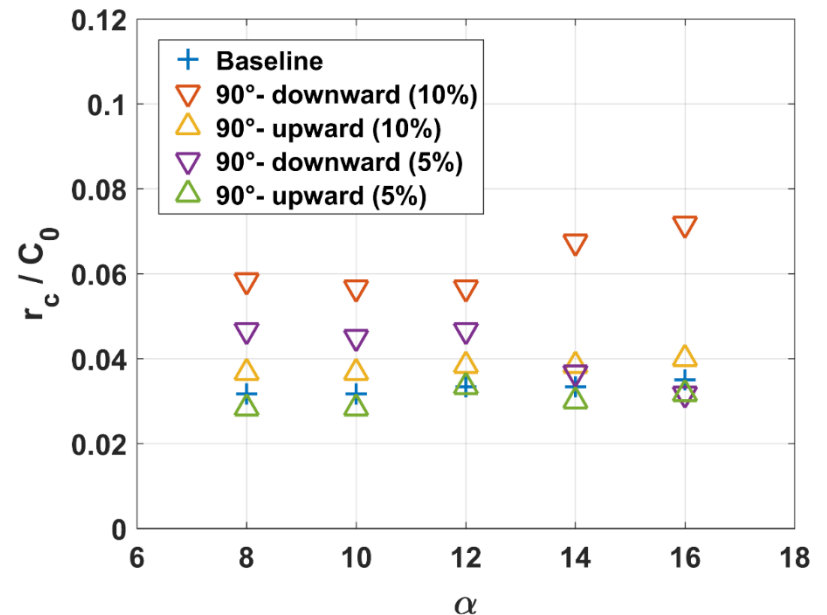
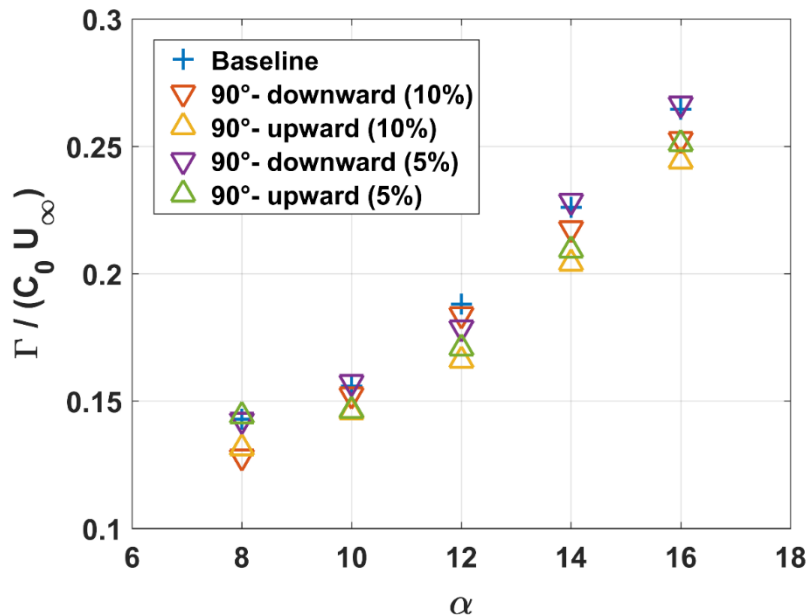




# Velocimetry Results

Effect of incidence angle on  $\Gamma$  and  $r_c$

- Flow conditions:  $U_\infty = 10 \text{ m/s}$
- Similar trends are conserved



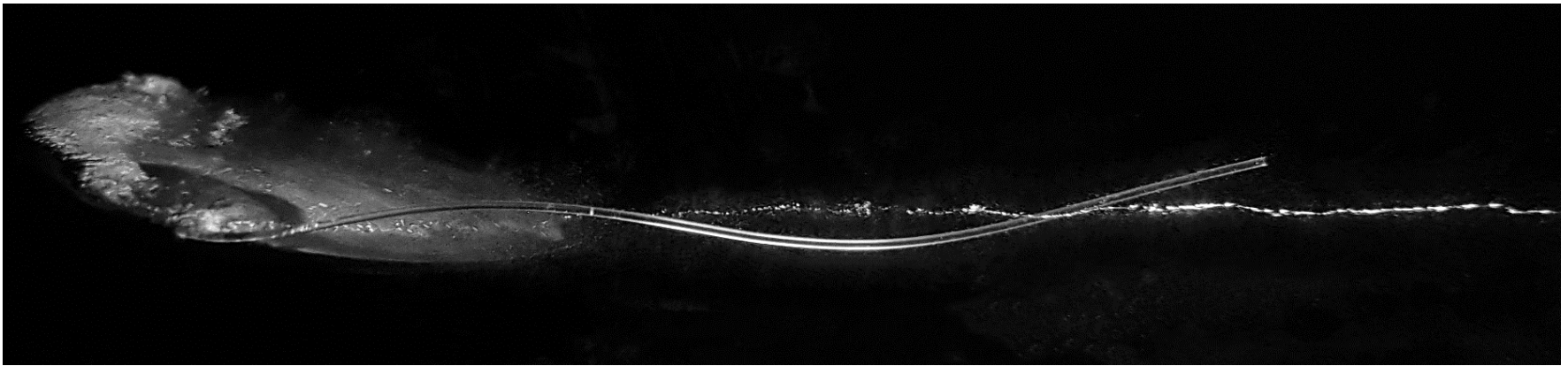
# Conclusion



## Effectiveness of nonplanar winglets in TVC suppression is investigated:

- Almost for all the flow conditions, the winglet-equipped hydrofoils perform better than the baseline hydrofoil in terms of delaying TVC.
- The hydrodynamic performances of the hydrofoils are **not** degraded by the winglets.
- For  $L_{BS} = 0.1S$ ,  $\theta = 90^\circ$  yielded much better results compared to  $\theta = 45^\circ$ .
- The negative dihedral angles (**downward**) are superior to the positive ones (**upward**), due to enhanced **wake entrainment** effects.
- Longer vertical sections outperform the shorter ones.
- Best configuration performance: **10%-bent 90°-downward**
  - Outstanding suppression (68% delay in inception)
  - Increasing the viscous core radius by 70%
  - Decreasing  $v_{\theta_{max}}$  to almost 50%

# Mitigating Tip Vortex Cavitation by a Flexible Trailing Thread

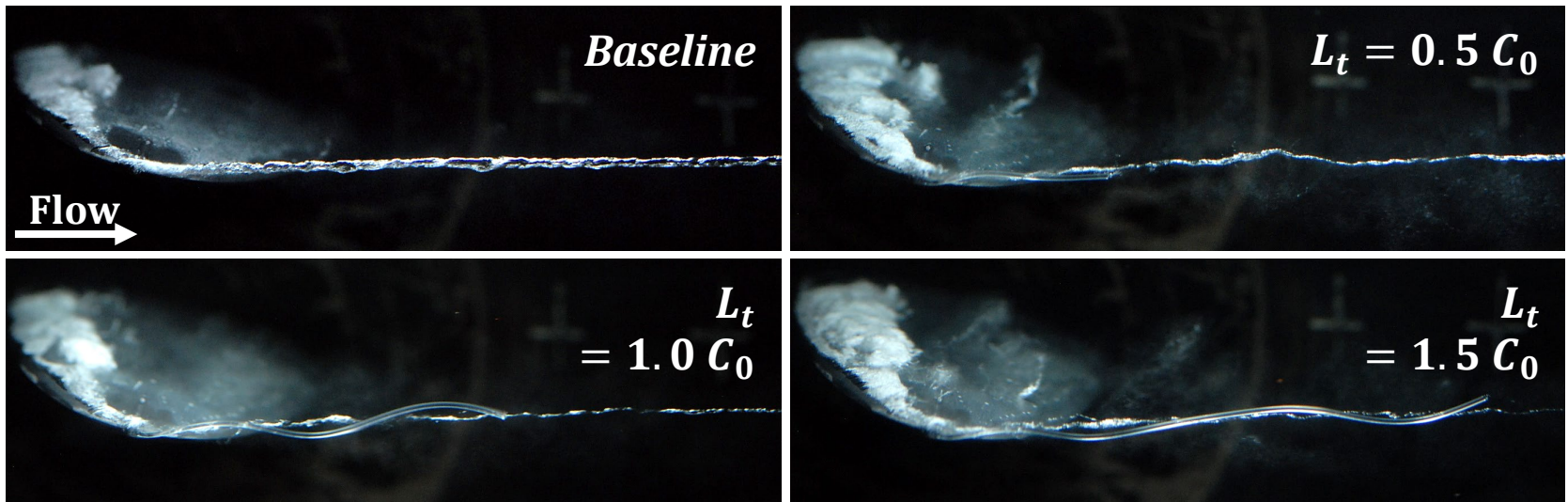


Ali Amini, Jeonghwa Seo, Shin Hyung Rhee, and Mohamed Farhat. "Mitigating tip vortex cavitation by a flexible trailing thread." *Physics of Fluids* 31, no. 12 (2019): 127103.

# TVC Suppression Mechanisms

## Interaction of a flexible trailing thread with the cavitating vortex flow

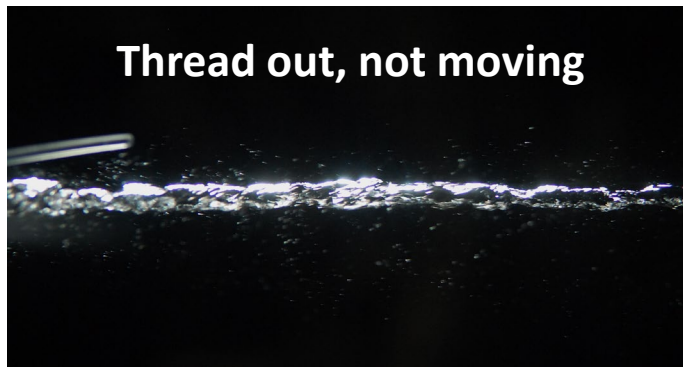
- Flow conditions:  $\alpha = 15^\circ$ ,  $U_\infty = 15$  m/s, and  $\sigma = 1.8$
- Effect of the thread length ( $L_t$ ) on TVC suppression (thread diameter:  $d_t = 0.7$  mm)
- $C_0$  is the root chord length of the hydrofoil ( $C_0 = 60$  mm)



## Analysis of the thread motion

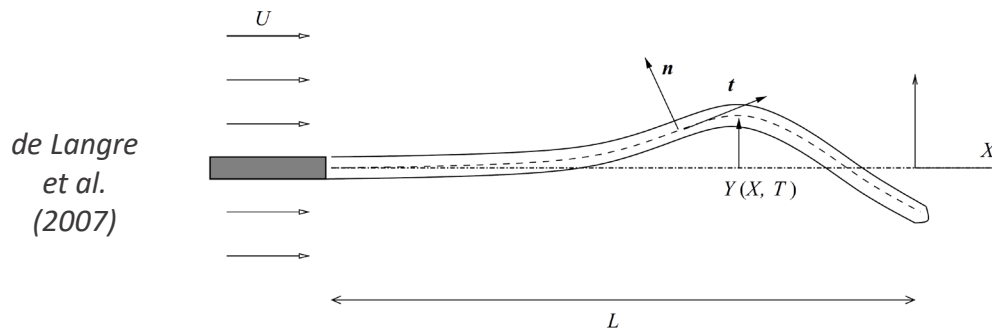
- The most rigid thread tested under the following conditions → **Two stable** states
  - Test conditions:  $\alpha = 10^\circ$ ,  $U_\infty = 10$  m/s,  $\sigma = 1.2$ ,  $d_t = 0.7$  mm and  $L_t = 0.5 C_0$
- Freestream velocity → Acceleration vs Deceleration → Static vs Dynamic response
- The thread should be **flexible enough** to **align** with the vortex and **interact** with it dynamically.

But, what does ***flexible enough*** mean?



# Analysis of the thread motion

The lateral motion of a flexible beam retained straight in axial flow is analyzed.



flexural rigidity

fluid-induced forces

$$EI \frac{\partial^4 y}{\partial x^4} - \frac{\partial}{\partial x} \left( \Theta(x) \frac{\partial y}{\partial x} \right) + m \frac{\partial^2 y}{\partial t^2} = F_{fluid}$$

variation of local axial tension

Acceleration term  
 $m$ : mass per unit length

Non-dimensional velocity

$$u = \left( \frac{\rho A}{EI} \right)^{1/2} U_{\infty} L_t$$

# TVC Suppression Mechanisms

## ➤ Analysis of the thread motion

- Test conditions:  $d_t = 0.5$  mm,  $L_t = 1.5 C_0$ ,  $\alpha = 10^\circ$  and  $U_\infty = 10$  m/s

## ➤ Harmonic waves are travelling along the thread.

## ➤ The thread clearly encloses the vortex axis by **rotating** around it.



# TVC Suppression Mechanisms

## Dynamic interaction of the trailing thread and the vortex flow

- Flow conditions:  $\alpha = 12^\circ$ ,  $U_\infty = 10 \text{ m/s}$ , and  $\sigma = 1.4$
- Thread configuration:  $d_t = 7 \text{ mm}$  and  $L_t = 1.5 C_0$
- A complex interaction is observed

Whipping  
motion



Rotationa  
l motion



# Analysis of the Whipping Motion

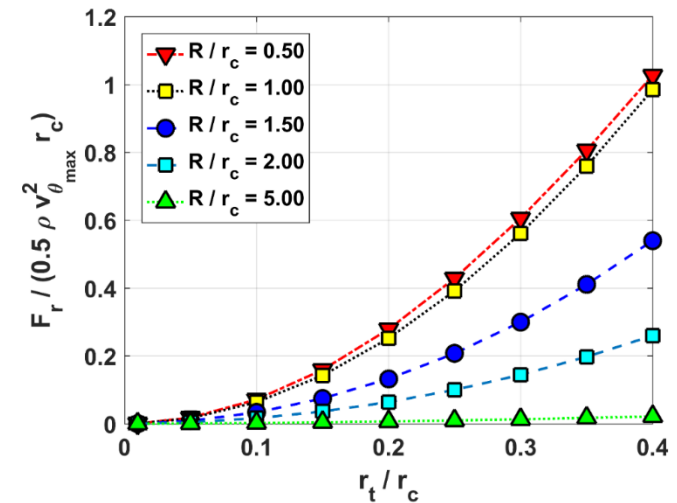
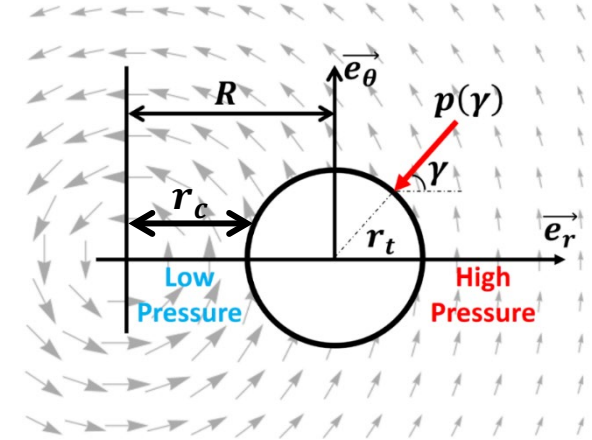
- A Lamb-Oseen vortex profile is considered:

$$v_{\theta}(r) = \frac{\Gamma}{2\pi r} \left(1 - e^{-1.256(r/r_c)^2}\right) \quad \text{and} \quad \frac{\partial p}{\partial r} = \rho \frac{v_{\theta}^2}{r}$$

- Calculating the **attraction force ( $F_r$ )** in one-way coupling for various thread diameters and eccentricities results in:

$$F_r = 2r_t \int_0^{\pi} p(\gamma) \cos(\gamma) d\gamma$$

- The radial force increases with the thread diameter, **however**, this increase becomes more significant as the thread gets closer and closer to the vortex axis.
- Away from the vortex axis, the relation is almost linear.
- Close to the axis  $\rightarrow F_r \propto d_t^2$



# Analysis of the Whipping Motion

## Modeling of the coincidence phase:

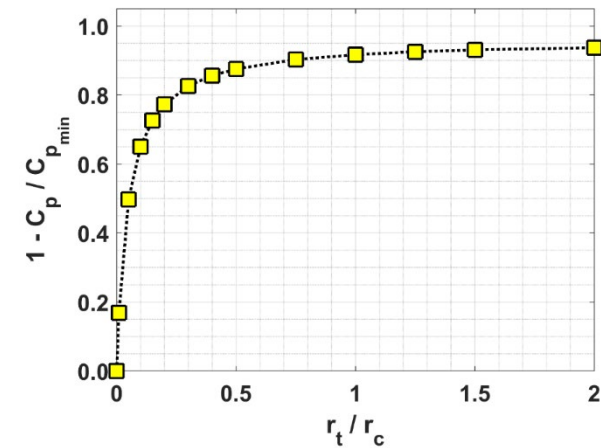
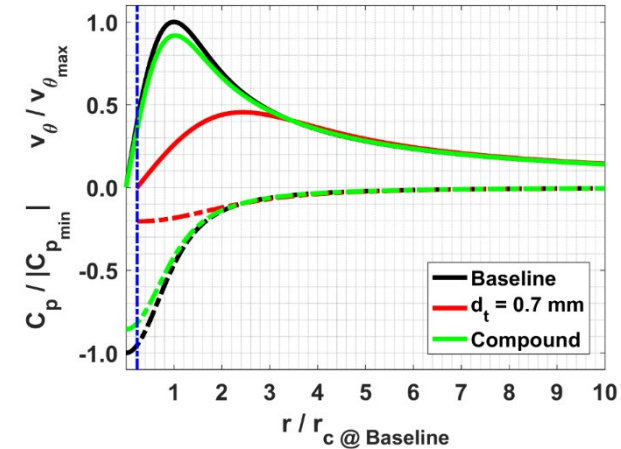
- The coincidence phase is **fast**, which implies that the **integral parameters** of the vortex should remain constant. ( $\Gamma_1 = \Gamma_2 = \Gamma$ )
- To find the **new** viscous core radius, we implement the conservation of angular momentum principle:

$$H_1 = \rho L \Gamma \int_0^\infty r \left( 1 - e^{-1.256(r/r_c)^2} \right) dr$$

$$H_2 = \rho L \Gamma \int_0^\infty \frac{r^2}{r - r_t} \left( 1 - e^{-1.256((r-r_t)/r_{c,2})^2} \right) dr$$

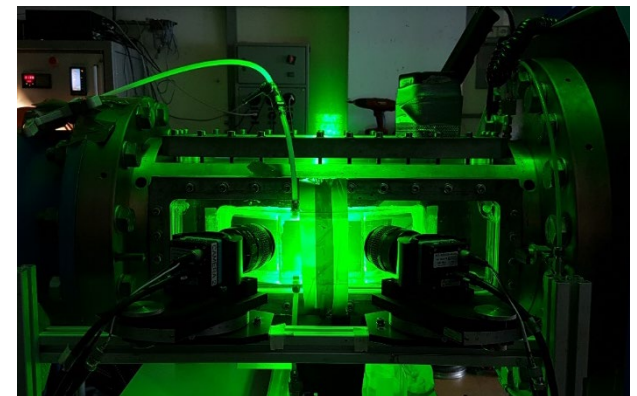
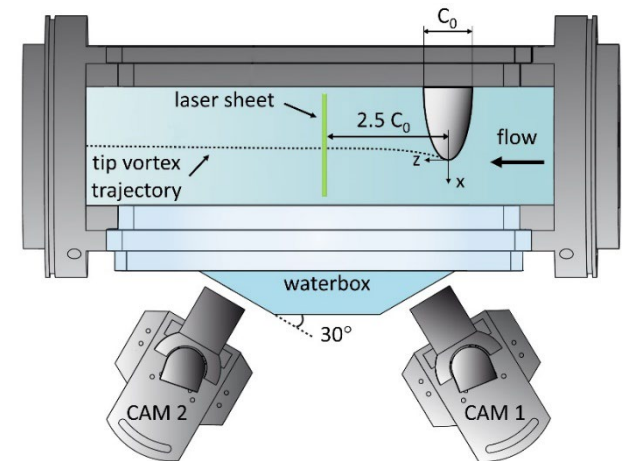
- The coincidence of the thread results in a considerable rise in the minimum pressure induced by the vortex.
- The pressure rise is almost proportional to  $r_t^{0.2}$

Arbitrary Lamb-Oseen vortex  
with  $r_c = 1.5$  mm



# Stereo-PIV setup

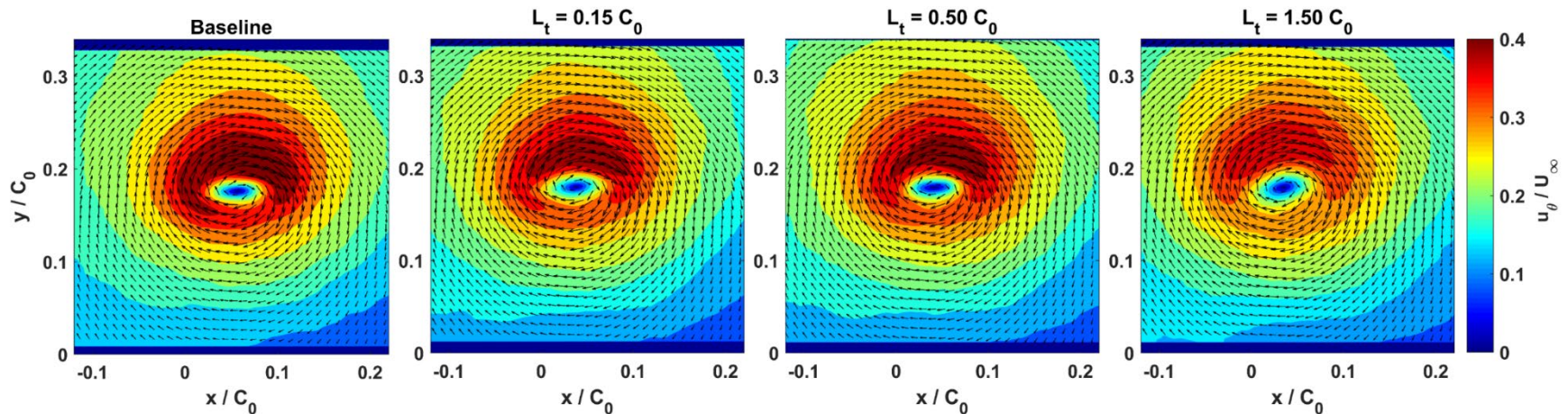
- Double-pulsed laser (135 mJ/pulse)
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  - Average diameter of 10  $\mu\text{m}$
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- Wandering motion correction
  - Center detection by Graftieaux algorithm
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# Velocity measurements

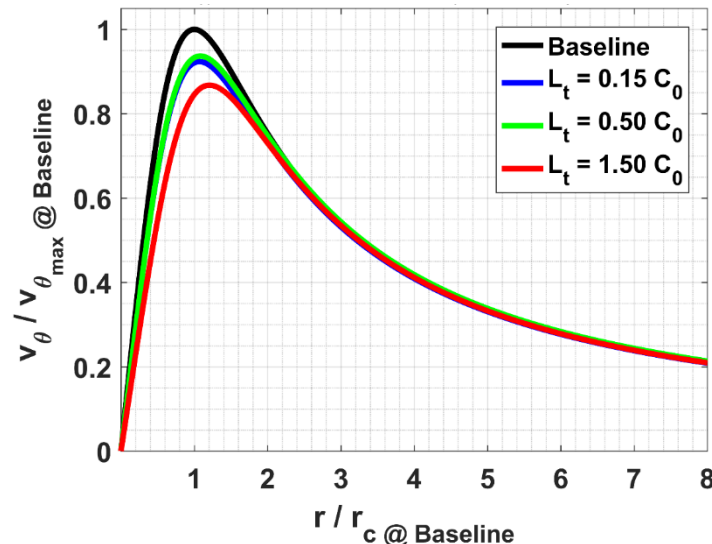
Contours of the **tangential** velocity:

- Flow conditions:  $U_\infty = 10 \text{ m/s}$ ,  $\alpha = 12^\circ$
- Thread configuration:  $d_t = 0.7 \text{ mm}$
- A clear **reduction** is observed in the magnitude of the tangential velocity.



## Azimuthally-averaged $v_\theta$ profiles

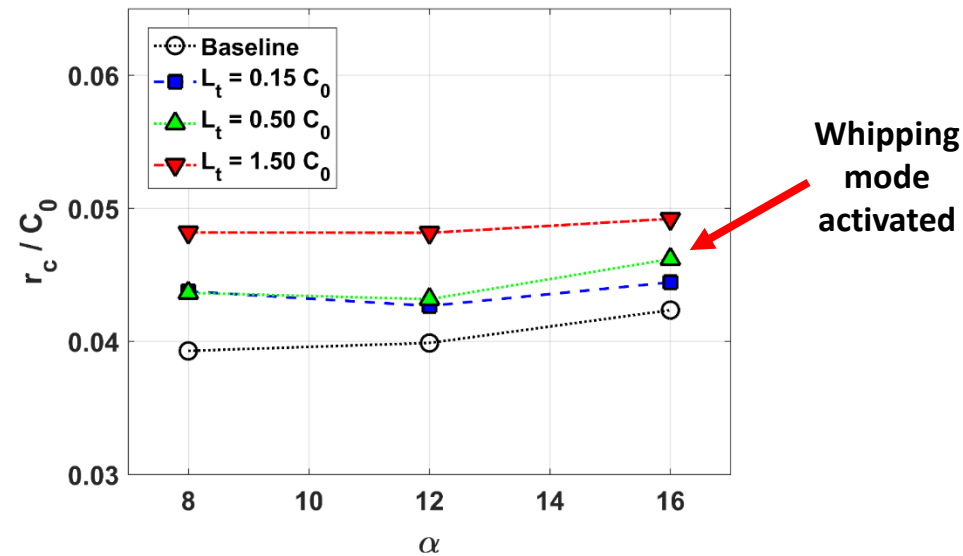
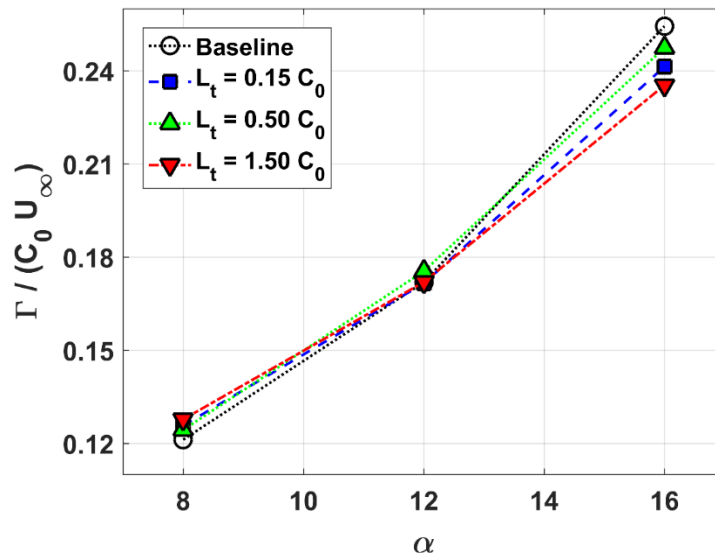
- Test conditions:  $\alpha = 12^\circ$ ,  $U_\infty = 10 \text{ m/s}$  and  $d_t = 0.7 \text{ mm}$ .
- A clear **reduction** is observed in the magnitude of the tangential velocity.
- In this test,  $L_t = 0.5 C_0$  is in **non-flapping** state.
  - The **winglet effect** for the rigid structures implies the **augmented turbulent mixing**.



# Velocity measurements

Effect of *incidence angle* on tip vortex parameters at various thread lengths.

- Flow conditions:  $U_\infty = 10 \text{ m/s}$  and non-cavitating regime
- Vortex intensity is conserved and TVC suppression is due to a **viscous core thickening**.



## Discussion:

Now, let's put all the **effective parameters** together:

- The extent of thread-vortex interaction  $\rightarrow u = \left(\frac{\rho A}{EI}\right)^{1/2} U_{\infty} L_t$
- The likelihood of whipping motion  $\rightarrow F_r \propto d_t^2$
- The pressure rise due to the whipping  $\rightarrow \Delta p \propto d_t^{0.2}$

➤ If we multiply the three terms together and scale the **thread diameter** with the **viscous core radius**, we get the following non-dimensional variable:

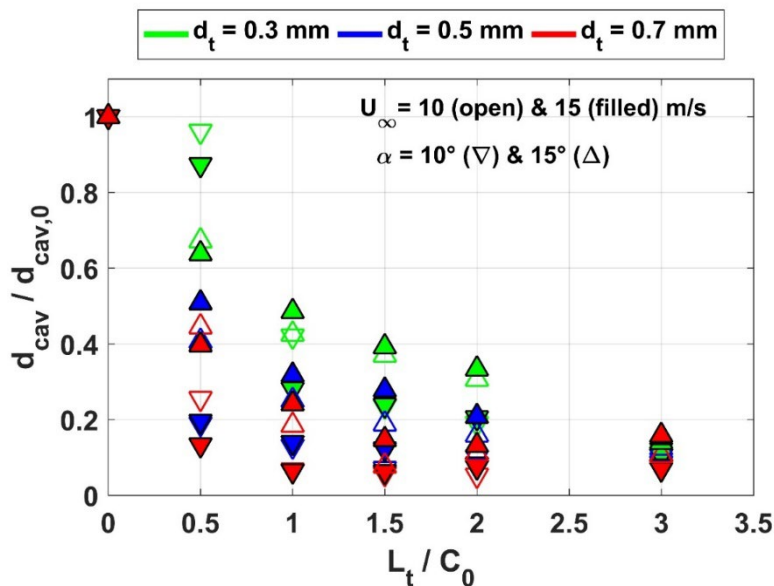
$$L^* = \frac{4}{r_c^{2.2}} \sqrt{\frac{\rho}{E}} U_{\infty} L_t d_t^{1.2}$$



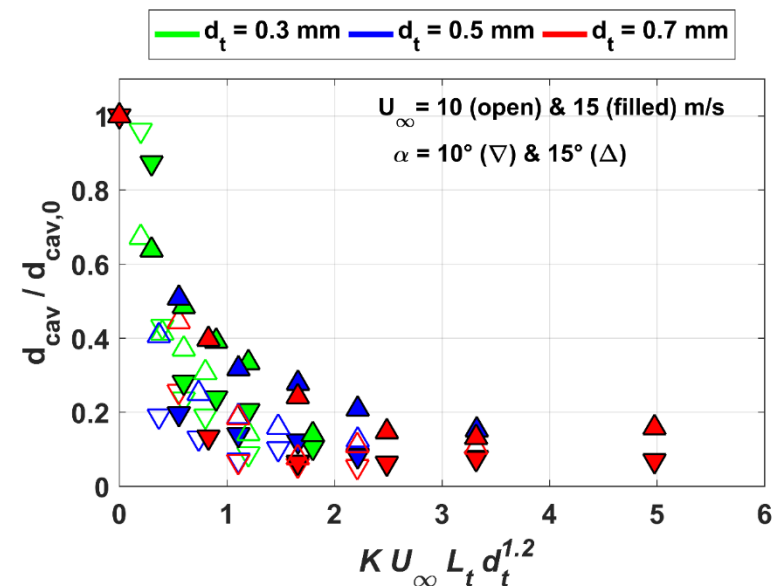
## Discussion:

### Plotting TVC suppression against the non-dimensional variable:

- The suppression effect is saturated beyond  $L^* \cong 2$  for all the configurations.



$$K = \frac{4}{r_c^{2.2}} \sqrt{\frac{\rho}{E}}$$





# Conclusion

## Effectiveness of a flexible thread in TVC mitigation

- A thread should be **flexible enough** to:
  - Get ***aligned*** with the vortex
  - Interact with it ***dynamically***
- **Two interaction/mitigation regimes:**
  - ***Rotational*** motion
  - ***Whipping*** motion
- **Viscous core thickening**

