

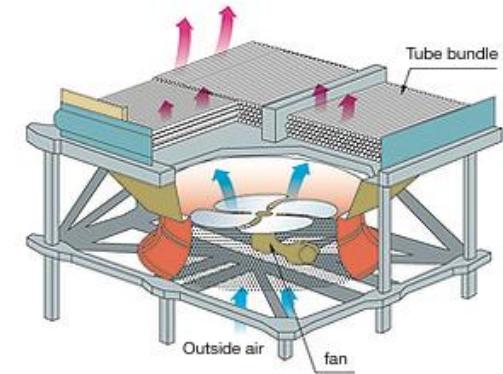
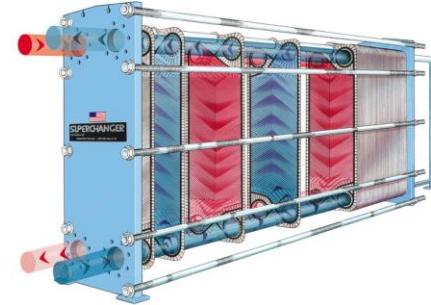
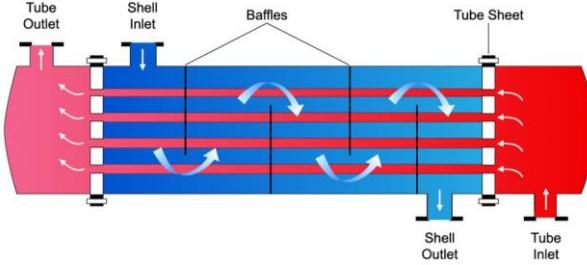
Heat Pump Systems

Summary W12

Prof. J. Schiffmann

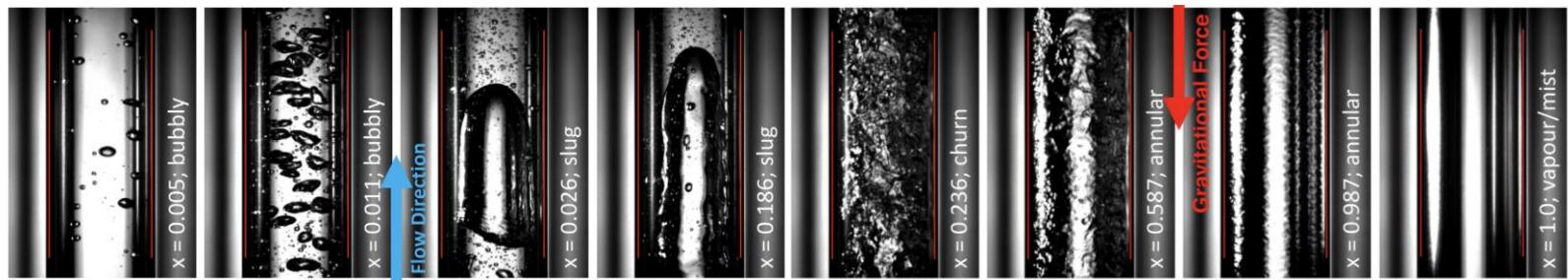
Heat Exchanger Function

- Heat exchangers transfer heat from hot fluid to cold fluid
 - Temperature difference between hot and cold composite should be as low as possible
 - Pressure drops should be as low as possible
 - Heat exchanger area should be as small as possible to reduce cost
- Different types depending on fluids and capacity



Heat Exchanger Function

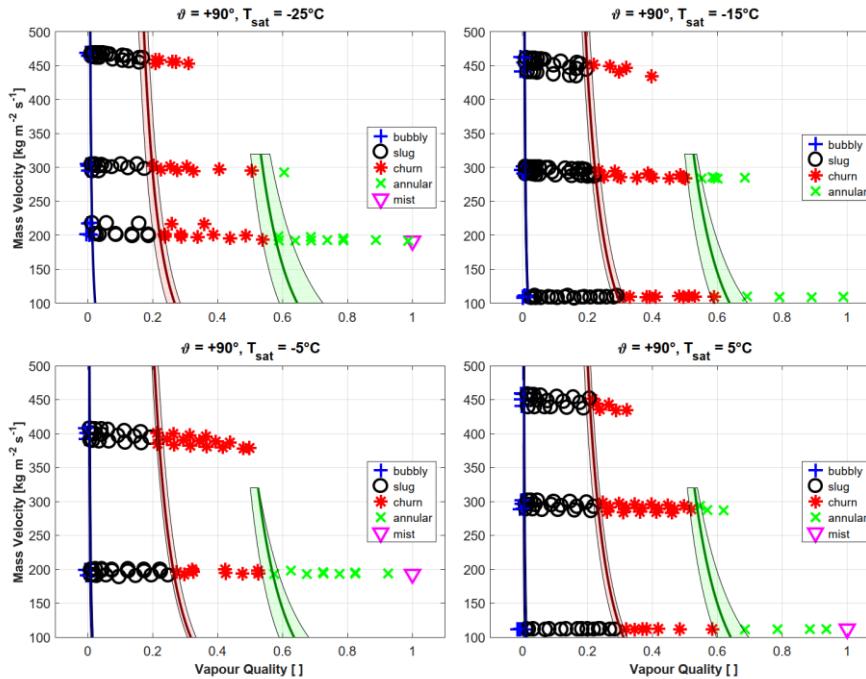
- Heat transfer occurs primarily through convection from fluid to wall and through conduction across wall
- Convection is usually limiting factor → challenging to model, particularly for two-phase flows
- Convective heat transfer strongly dependent on flow patterns



D. Schmid, B. Verlaart, P. Petagna, R. Revellin, J. Schiffmann. **Flow Pattern Observations and flow pattern map for adiabatic two-phase flow of carbon-dioxide in vertical upward and downward direction.** Experimental Thermal and Fluid Sciences, vol. 131, 110526, 2022.

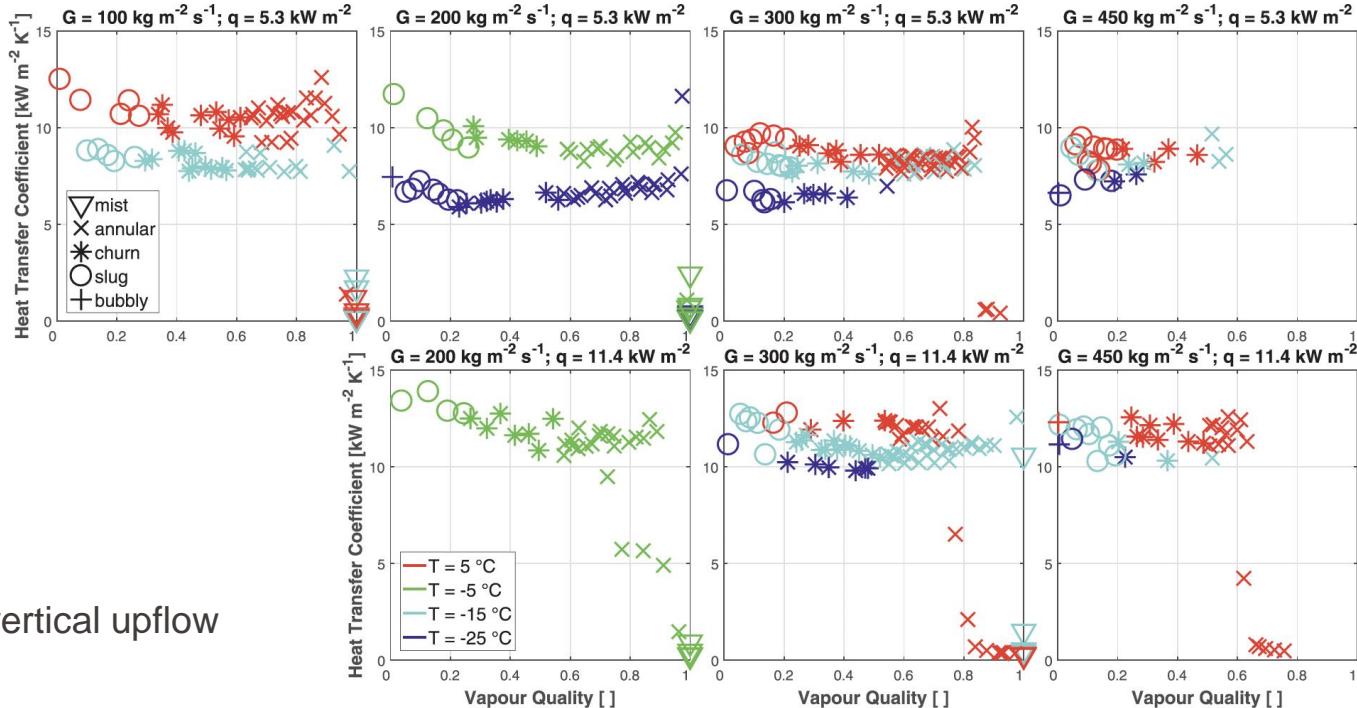
Flow Pattern Maps

CO₂, vertical upflow



- Flow patterns as a function of vapor quality, saturation temperature, and mass velocity

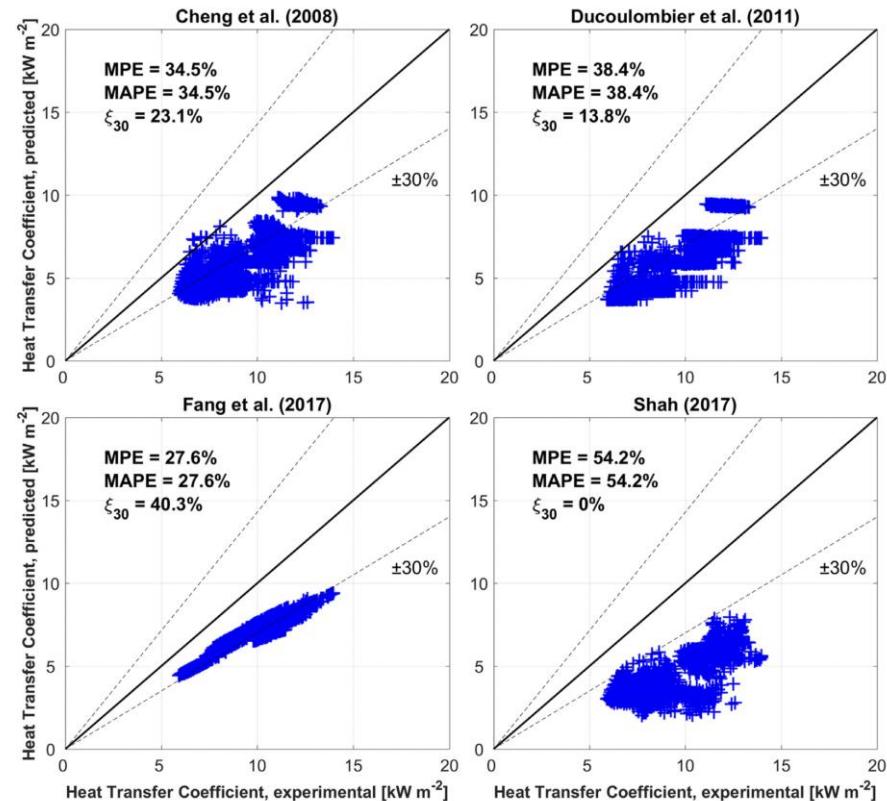
Convection Coefficient Maps



- Heat transfer coefficient as a function of vapor quality, saturation temperature, and mass velocity

Performance of Typical Correlations

- Available correlations usually not very accurate
- Heat transfer coefficient highly dependent on fluid, mass-flux, geometry





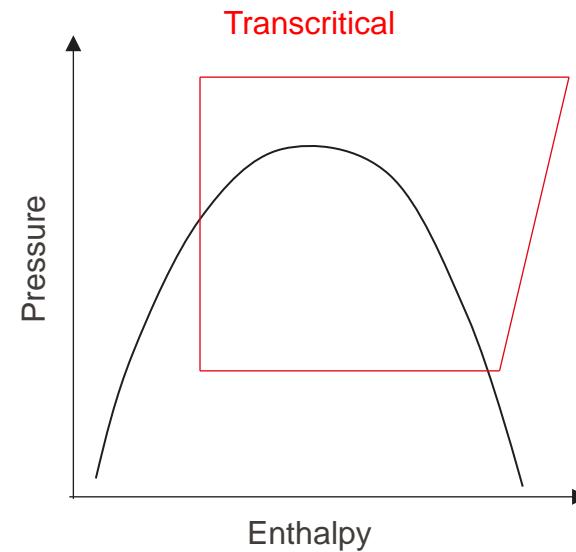
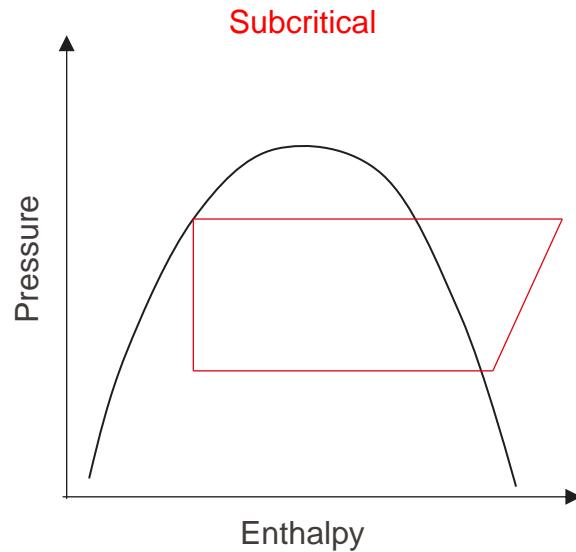
Heat Pump Systems

Transcritical
Heat Pumps &
Ejectors

Prof. J. Schiffmann

Transcritical Heat Pumps

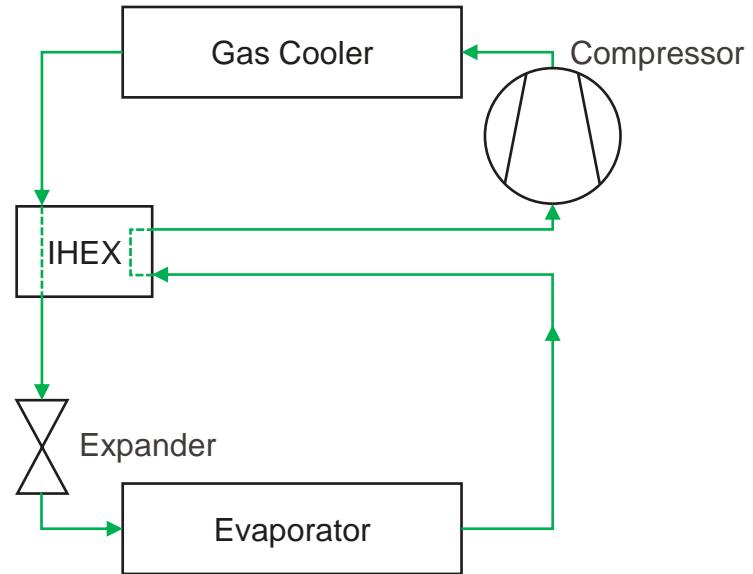
- Transcritical operation means hot source is above and cold source below critical point
- Typical applications are heat pumps with CO_2 as working fluid



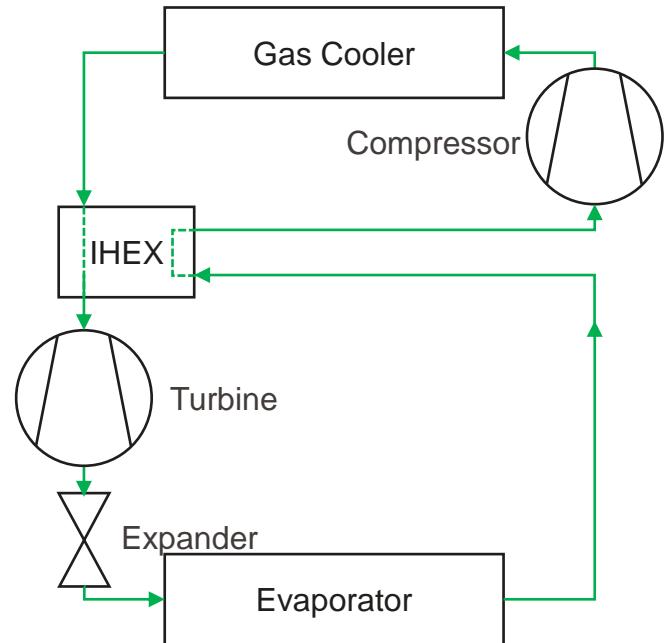
- CO₂ is interesting working fluid for heat pumps
 - Natural refrigerant, non-toxic, non-flammable, excellent thermal properties
 - High volumetric heat capacity → compact machines
 - High pressures (critical pressure 73.8 bar)
- Typical hot source temperatures lead to transcritical operation for CO₂
 - Heat rejection with large temperature glide
 - Suffer from high expansion losses

- Possible mitigation of high expansion losses
 - Internal heat exchanger → suction superheater cycle
 - Supercritical turbine expander
 - Ejector cycles

- Internal heat exchanger configuration (suction superheater cycle)
 - Improves COP and capacity by ~5-15%
 - Leads to higher compressor discharge temperatures (~10K)

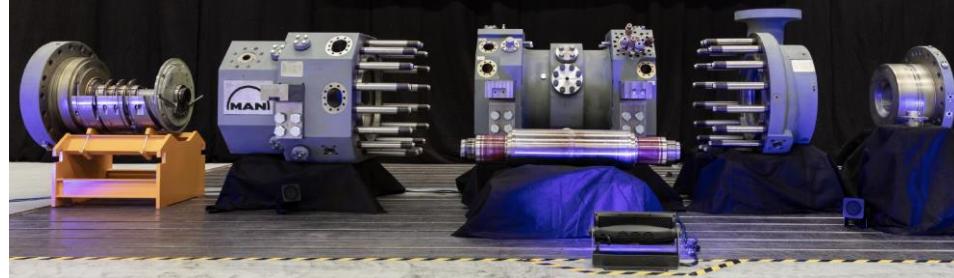


- Supercritical turbine expander
 - Turbine expander recovers work up to saturation line
 - Two-phase expansion continued in valve
 - Internal heat exchanger improves COP
 - Supercritical turboexpander improves COP by 20-30%



Transcritical CO₂ Heat Pumps

- Supercritical turbine expander
 - For a district heating heat pump with a heating power of 35MW
 - Turbine on same rotor as compressor
 - Hermetic design → no leakage
 - Supported on magnetic bearings
 - Oilfree solution

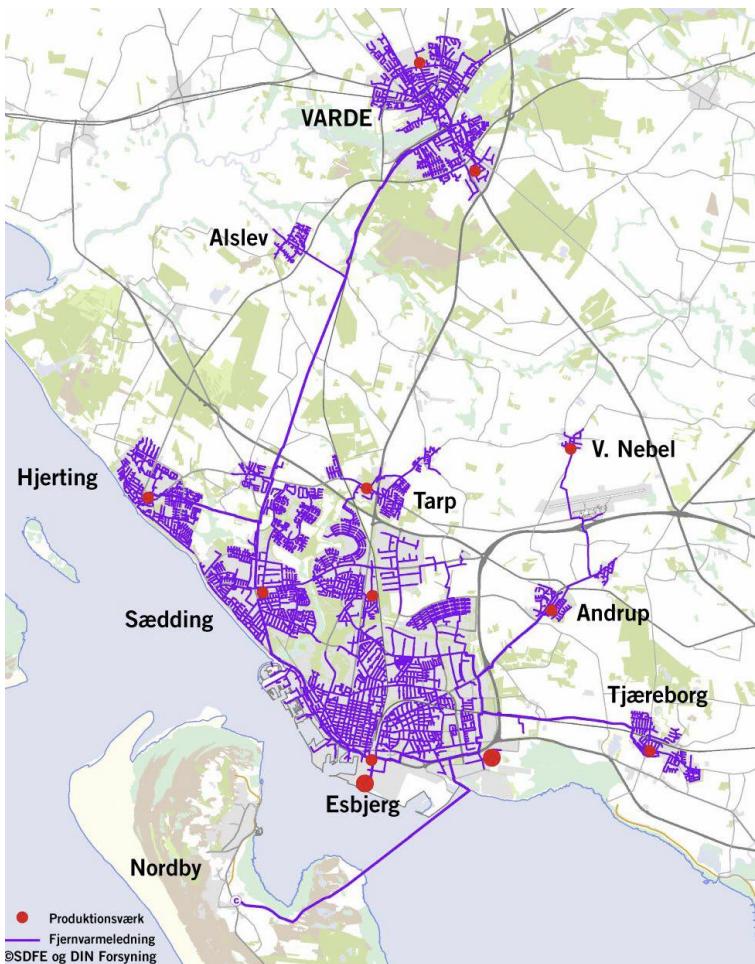


Parameter	Unit	Max. Value
Motor Active Power	MW _{el}	10.5
Heating Duty	MW _{th}	35
Cooling Duty	MW _{th}	25
CO ₂ Pressure	bar	140
CO ₂ Temperature	°C	130
Min. CO ₂ Evaporation Temperature	°C	-2



Transcritical CO₂ Heat Pumps

- Climate-neutral district heating for 100'000 inhabitants in Esbjerg (DK)
 - 280'000 MWh/y CO₂-neutral district heating
→ sufficient for 25.000 households
 - 100'000 t CO₂ savings per year
→ emission of approx. 55.000 cars
 - 60 MWth power
→ 50m swimming pool to boil in <4h
 - 4000 Liter/s sea water demand
→ 15 bathtubs per second



Transcritical CO₂ Heat Pumps

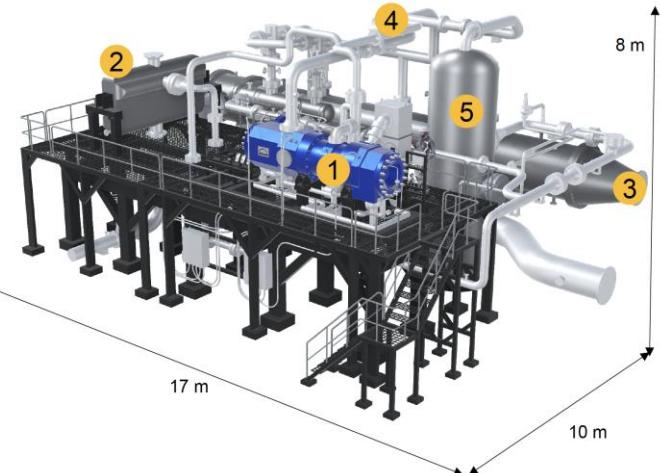
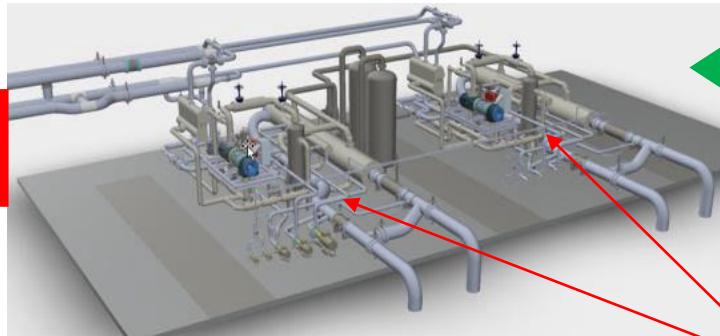
- Two units provide CO₂ free 60MW_{th}



Green energy

Seawater source
1-20°C

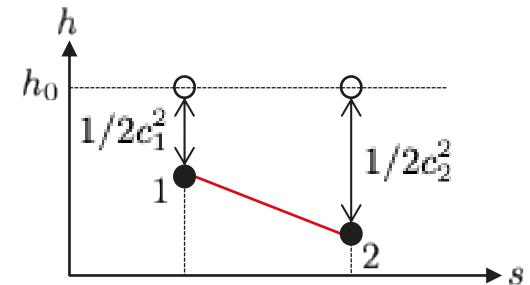
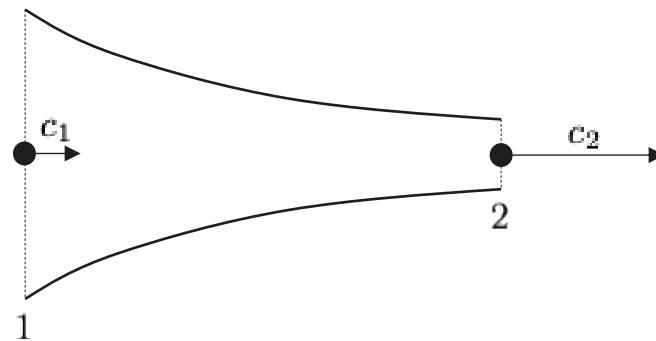
District heat
60MW_{th}
60-90°C



- Mitigation of high expansion losses
 - Internal heat exchanger
 - Supercritical turbine expander
 - Ejector

What Is an Ejector?

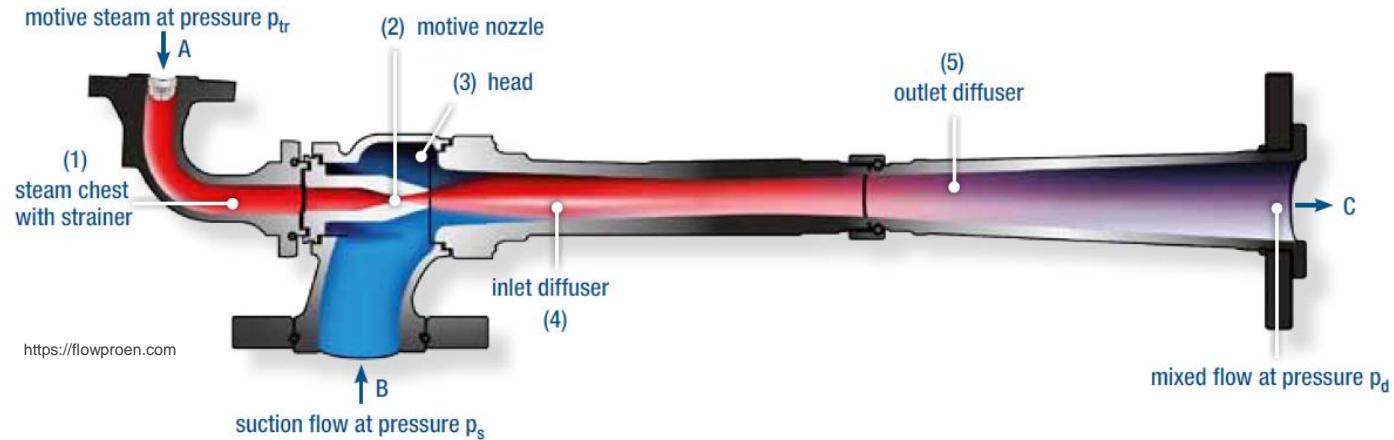
- Static pressure in channel drops with flow velocity



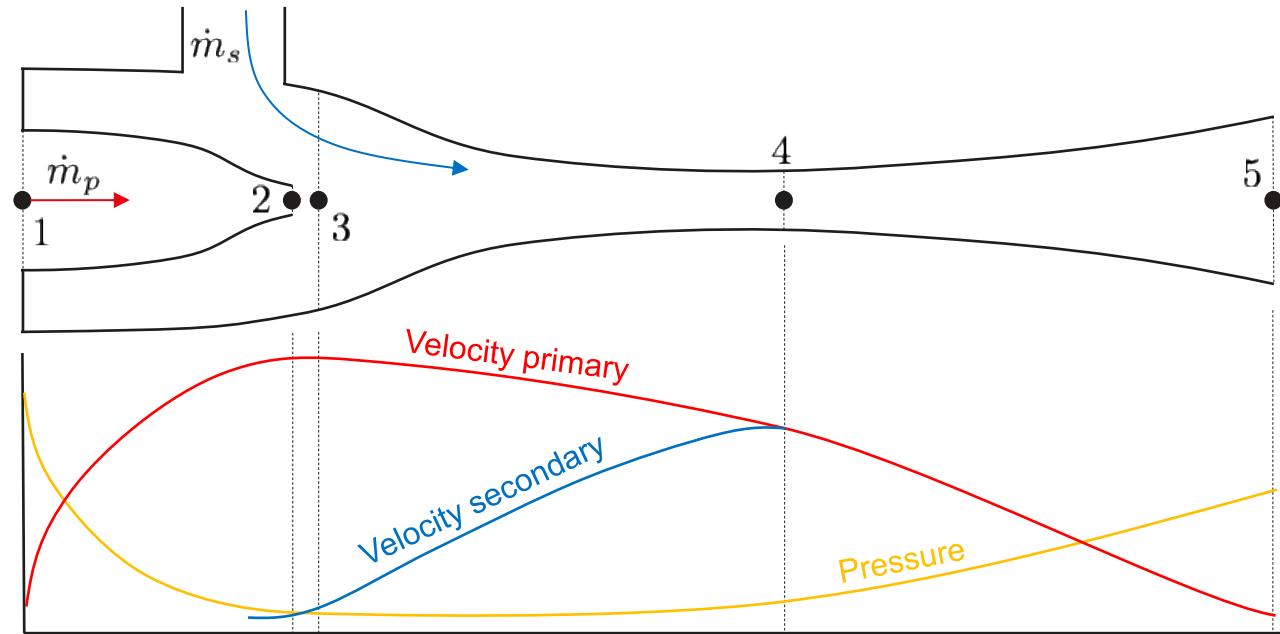
- Effect can be used to reduce pressure and/or accelerate gas
- High momentum flow at nozzle discharge can be used to accelerate other fluid at rest

What is an Ejector?

- Primary flow (motive fluid) accelerated in nozzle
- Reduced pressure used as suction for secondary gas
- High momentum of primary flow accelerates secondary flow
- Pressure recovered in diffuser



- Ejectors can be used to increase pressure of secondary flow



- Ejectors offer interesting features for heat pumps
 - Simple device → cheap
 - No moving parts
 - May be noisy
- Applications
 - Thermally driven heat pumps for cooling duty
 - Pre-compression in vapor compression cycles

- Mass balance

$$\dot{m}_p + \dot{m}_s = \dot{m}_3 = \dot{m}_4 = \dot{m}_5$$

- Energy balance

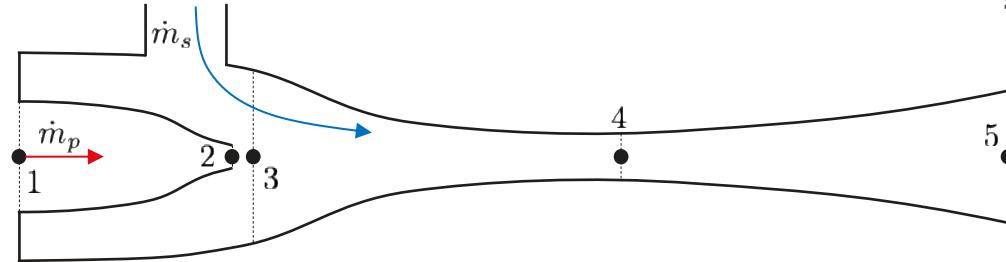
$$\dot{m}_p h_{0p} + \dot{m}_s h_{0s} = (\dot{m}_p + \dot{m}_s) h_{05}$$

$$h_0 = h + c^2/2$$

- Conservation of momentum

$$\dot{m}_p c_2 + \dot{m}_s c_3 = (\dot{m}_p + \dot{m}_s) c_4$$

$$\dot{m}_p c_2 = (\dot{m}_p + \dot{m}_s) c_4$$



$$\frac{\dot{m}_s}{\dot{m}_p} = \frac{c_2}{c_4} - 1$$

Working Principle

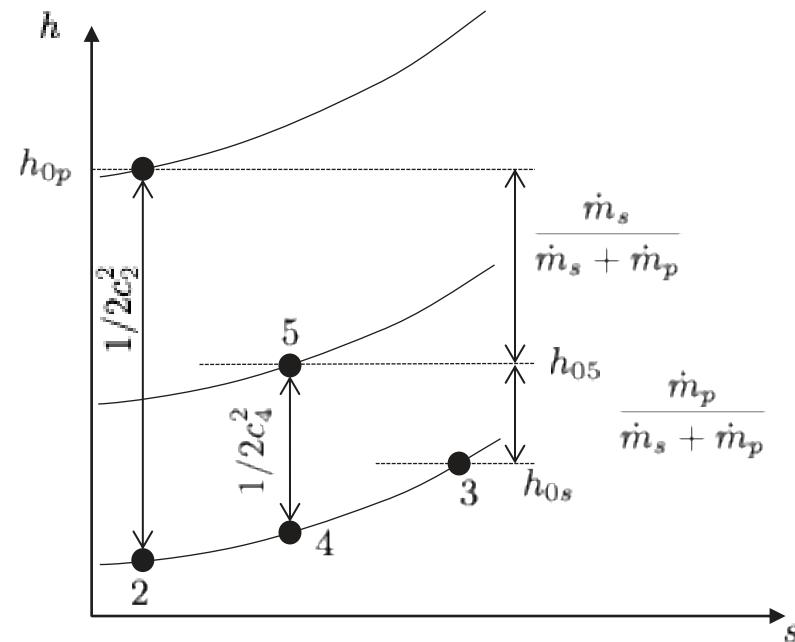
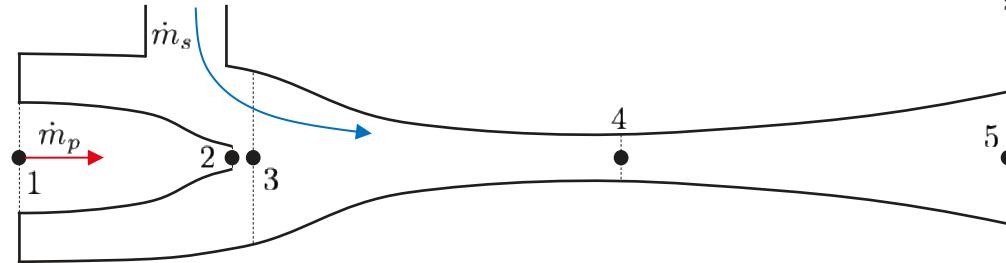
- In hs-diagram
 - Energy balance defines outlet enthalpy

$$\frac{\dot{m}_p h_{0p} + \dot{m}_s h_{0s}}{\dot{m}_p + \dot{m}_s} = h_{05}$$

- Momentum equation defines outlet pressure

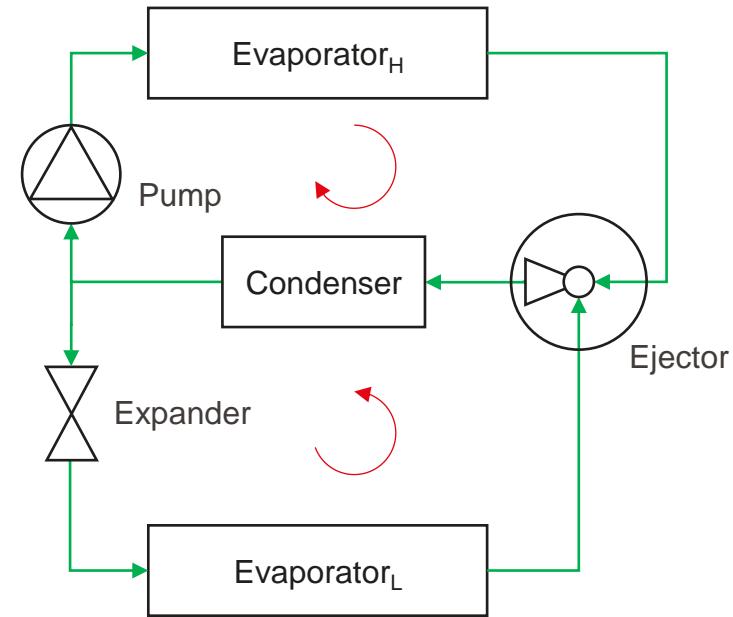
$$\frac{\dot{m}_s}{\dot{m}_p} = \alpha_e \frac{c_2}{c_4} - 1$$

- Assumes acceleration and diffusion occur isentropically ($\alpha_e = 1$)

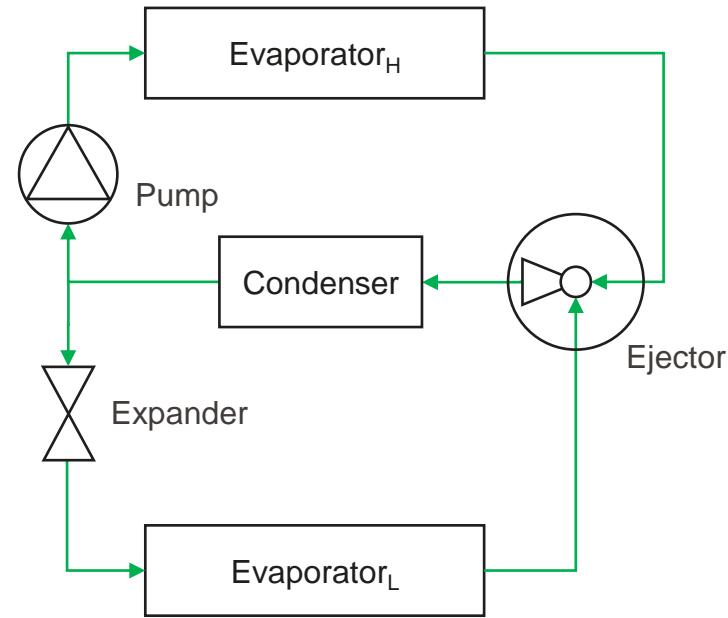


- Thermally Driven Refrigeration Heat Pump
- Driven by compressor discharge to pre-compress
- Pre-compression through expansion (booster)

- Requires a “driving cycle” to provide pressurized primary flow
- Cycle absorbs heat at high temperature
- Primary fluid used to compress secondary fluid in ejector

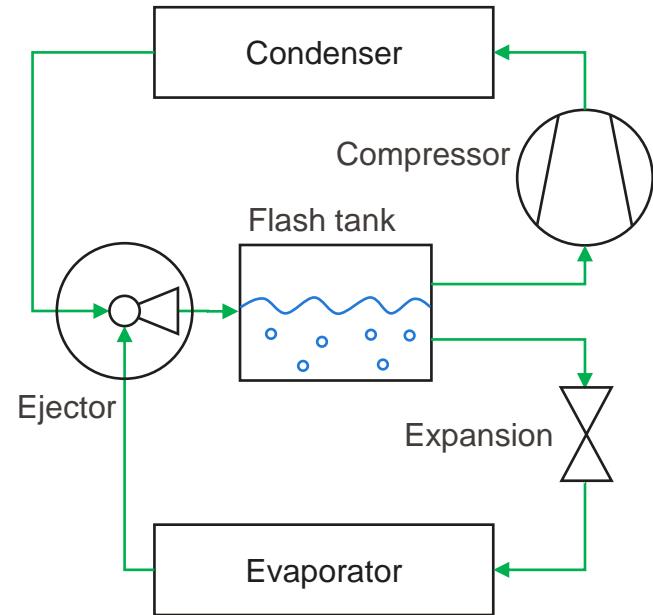


- Typical temperatures: $T_H = 80^\circ\text{C}$, $T_L = 5^\circ\text{C}$, $T_{\text{Cond}} = 30^\circ\text{C}$
- $\text{COP}_C = 0.13 - 0.5$



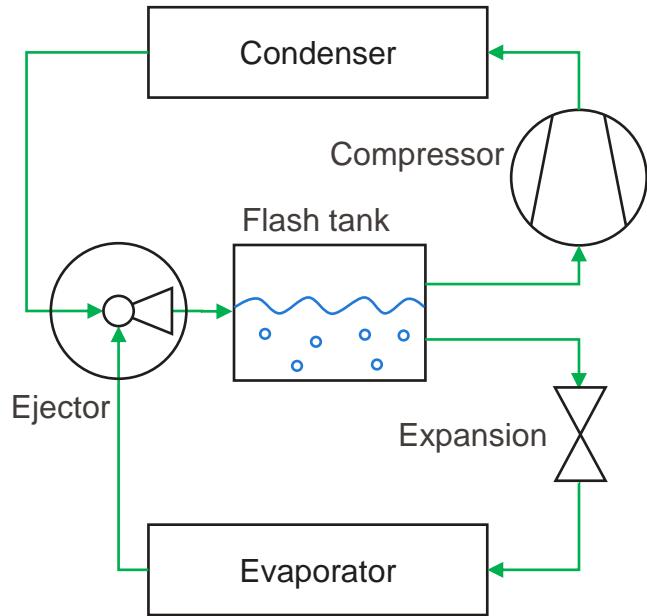
Pre-Compression Through Expansion

- Ejector driven by expansion process to pre-compress fluid
- Pre-compression reduces compressor work
- Interesting for cycles operated close or above critical point to reduce expansion losses
- Improvement of COP up to 10-20%
- Poor off-design performance



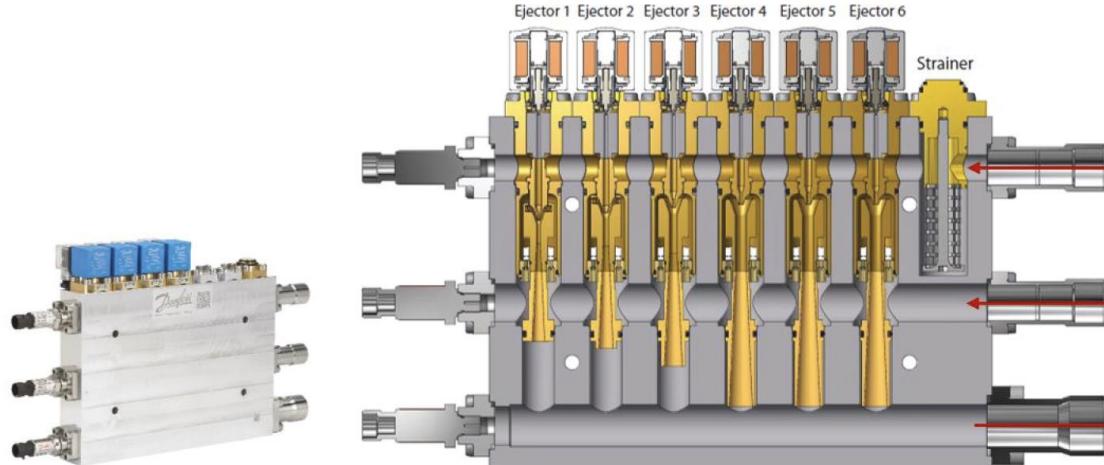
Pre-Compression Through Expansion

- In Ph-diagram



Pre-Compression Through Expander

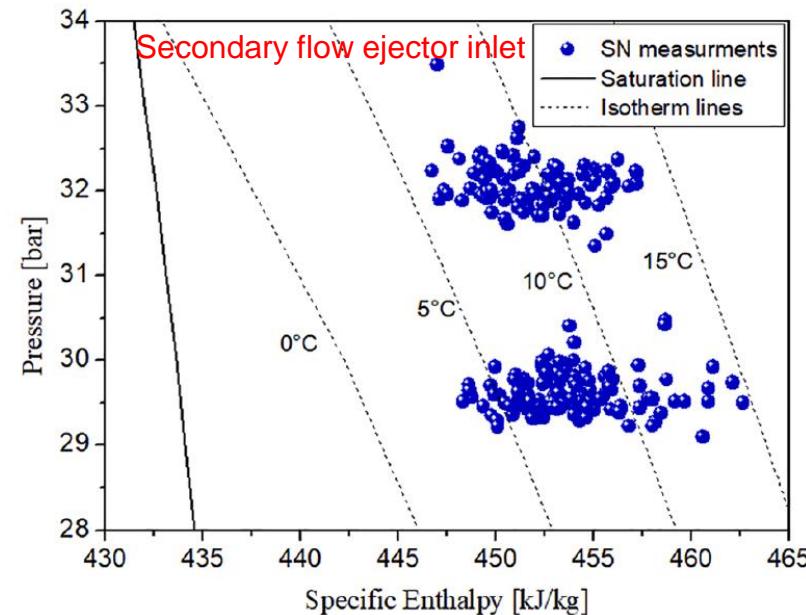
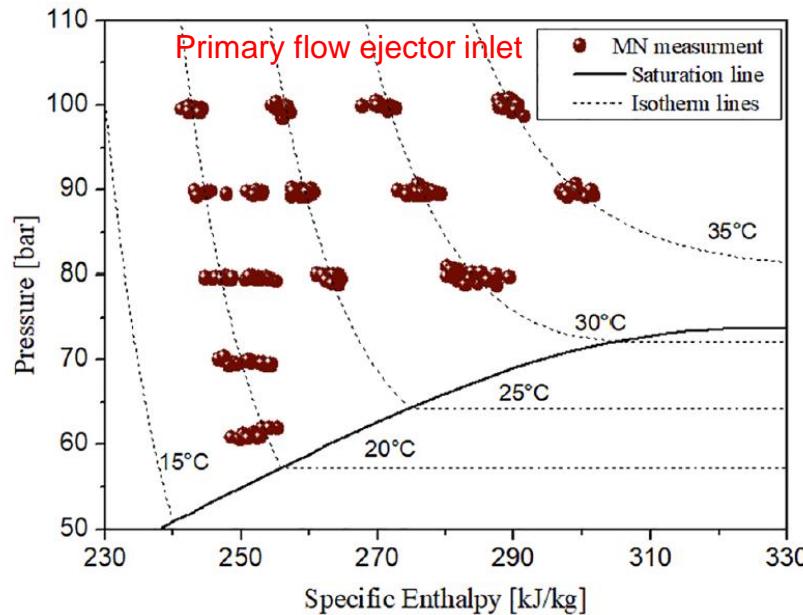
- Poor off-design performance → multi-ejector-packs
- Each ejector turned on/off depending on required mass flows



<https://assets.danfoss.com/documents/latest/397042/AI311723869490en-000202.pdf>

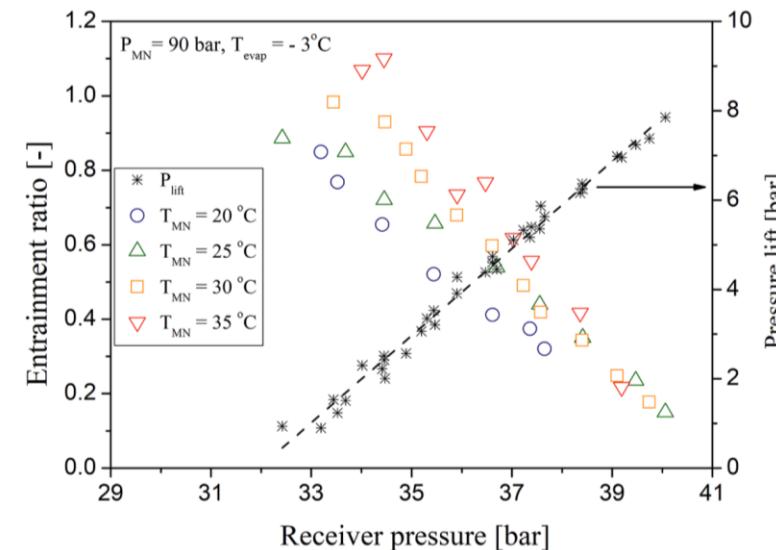
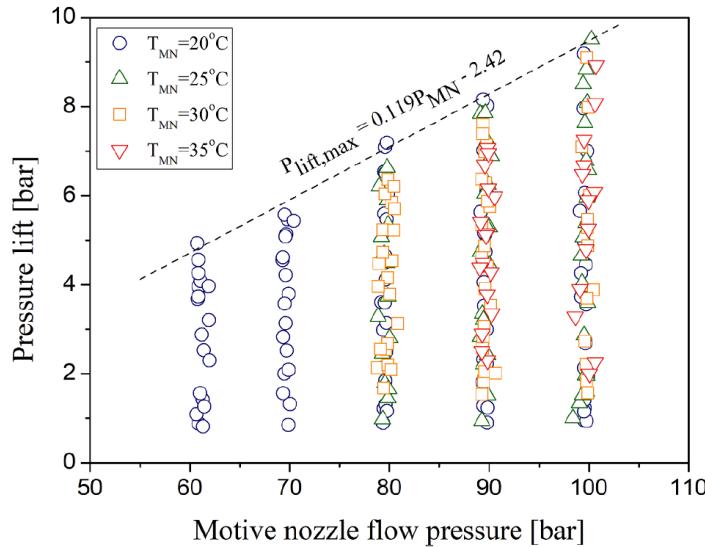
Experimental Data on Multi-Ejector-Pack in Transcritical CO₂ Heat Pump

- Ejector data points



Experimental Data on Multi-Ejector-Pack in Transcritical CO₂ Heat Pump

- Measured ejector performance
 - Maximum pressure rise 9.51 bar → low entrainment ratio
 - Ejector recovers up to 37% of expansion losses



- Transcritical heat pump cycles interesting if hot source requires large temperature rise
- CO₂ is interesting fluid (natural) but leads to high pressures
- Transcritical cycles suffer from high expansion losses → mitigation
 - Internal heat exchanger
 - Supercritical turbine expander
 - Ejector cycles

- Theory questions
- Ejector cycle heat pump

Heat Pump Systems

Exam

Prof. J. Schiffmann

- Date: 23.06.2023, 15.15 – 18.15h
- Place: BS 260, BS270, attributed seats
- Allowed support
 - Non-annotated cheat sheet provided on Moodle
 - Calculator
- Not allowed
 - Slides, exercises, corrections, books
 - Computer, pads, cell phone
 - Any other communication system

- Exam covers all topics seen in course
- Composed of theoretical (MCQ) and numerical problems to solve
- Preparation
 - Study course and summaries
 - Understand exercises and corrections