

Outline

1. Intro+kinematics
2. Dynamics+Dimensional Analysis
3. Low Reynolds number flow/ Stokes drag
4. Stokes drag-sedimentation-reversibility-
Hele Shaw-Pipe Flows
5. Lubrication
6. Unsteady flows
7. Boundary layers

D'Alembert's paradox

Inviscid flow theory does not provide a good prediction of the drag on a flat plate

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

$$\left(u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y} \right) u = -\frac{1}{\rho} \frac{\partial p}{\partial x} + v \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) u$$

$$\left(u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y} \right) v = -\frac{1}{\rho} \frac{\partial p}{\partial y} + v \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) v$$

Singular limit as $\text{Re} \rightarrow \infty$

$$\overline{\text{div}} \, \underline{\underline{U}} = 0$$

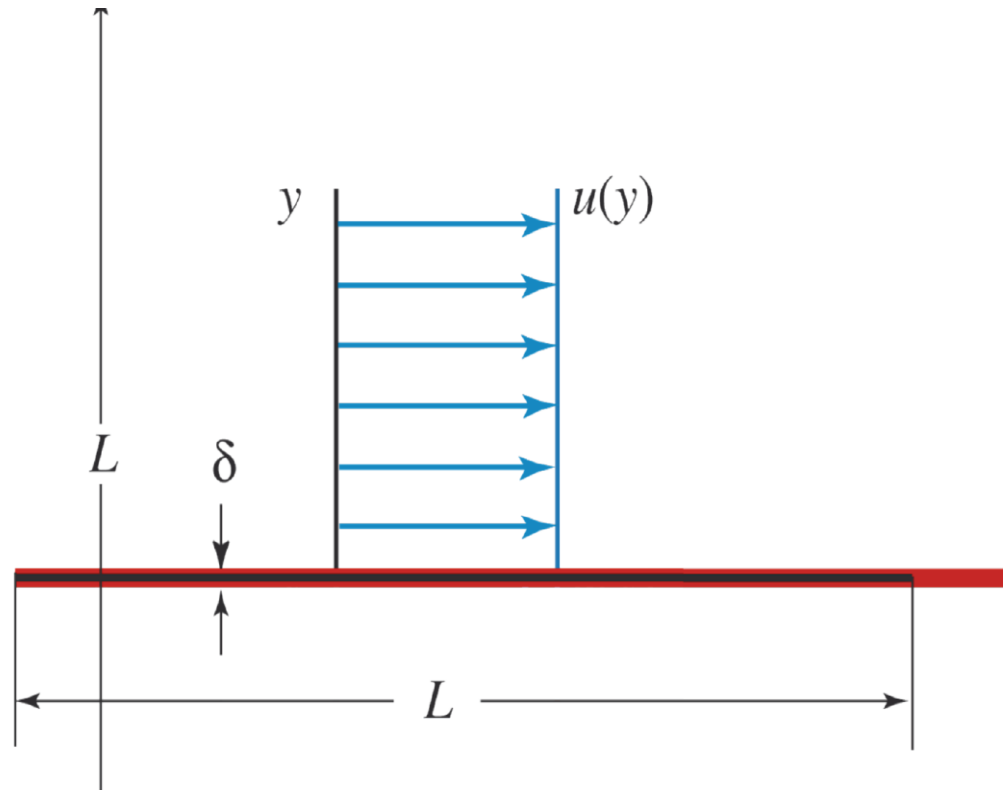
$$\underline{\underline{\text{grad}}} \underline{\underline{U}} \cdot \underline{\underline{U}} = -\underline{\underline{\text{grad}}} \underline{\underline{p}} + \cancel{\frac{1}{\text{Re}} \Delta \underline{\underline{U}}}$$

$$\underline{\underline{U}} \cdot \underline{\underline{n}} = 0$$

$$\cancel{\underline{\underline{U}}_T} = 0$$

D'Alembert's paradox

Inviscid flow theory does not provide a good prediction of the drag on a flat plate



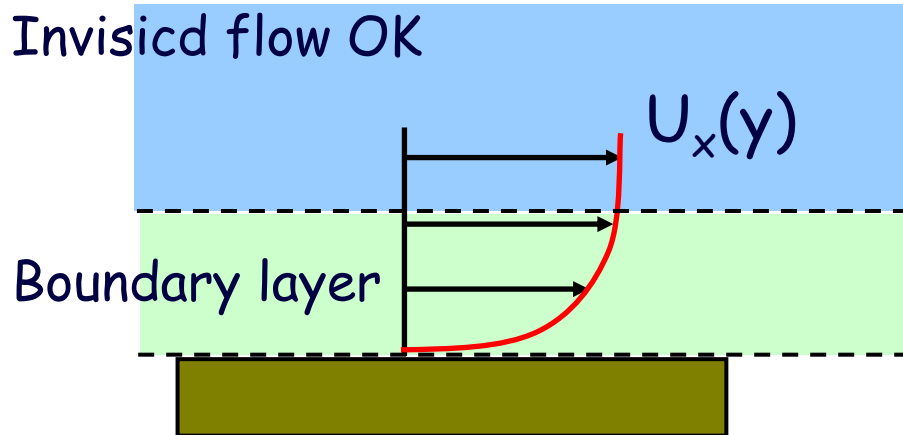
D'Alembert's paradox

Obviously, a moving body experiences a drag force in real life.

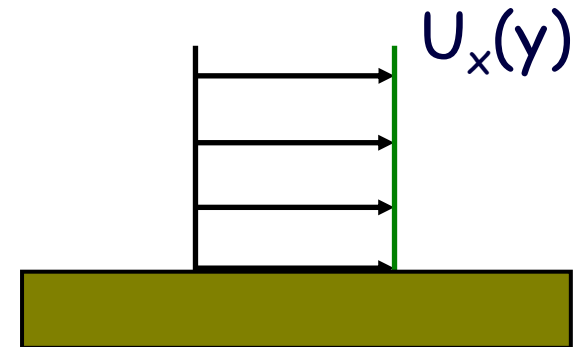
In order to resolve this paradox, we need to account for the viscous friction, **even at high Re** , that should not be neglected in a **boundary layer located close to the obstacle**. Inviscid flow theory can still be applied far from the obstacle

Viscous friction

y (normal)
 x (tangential)



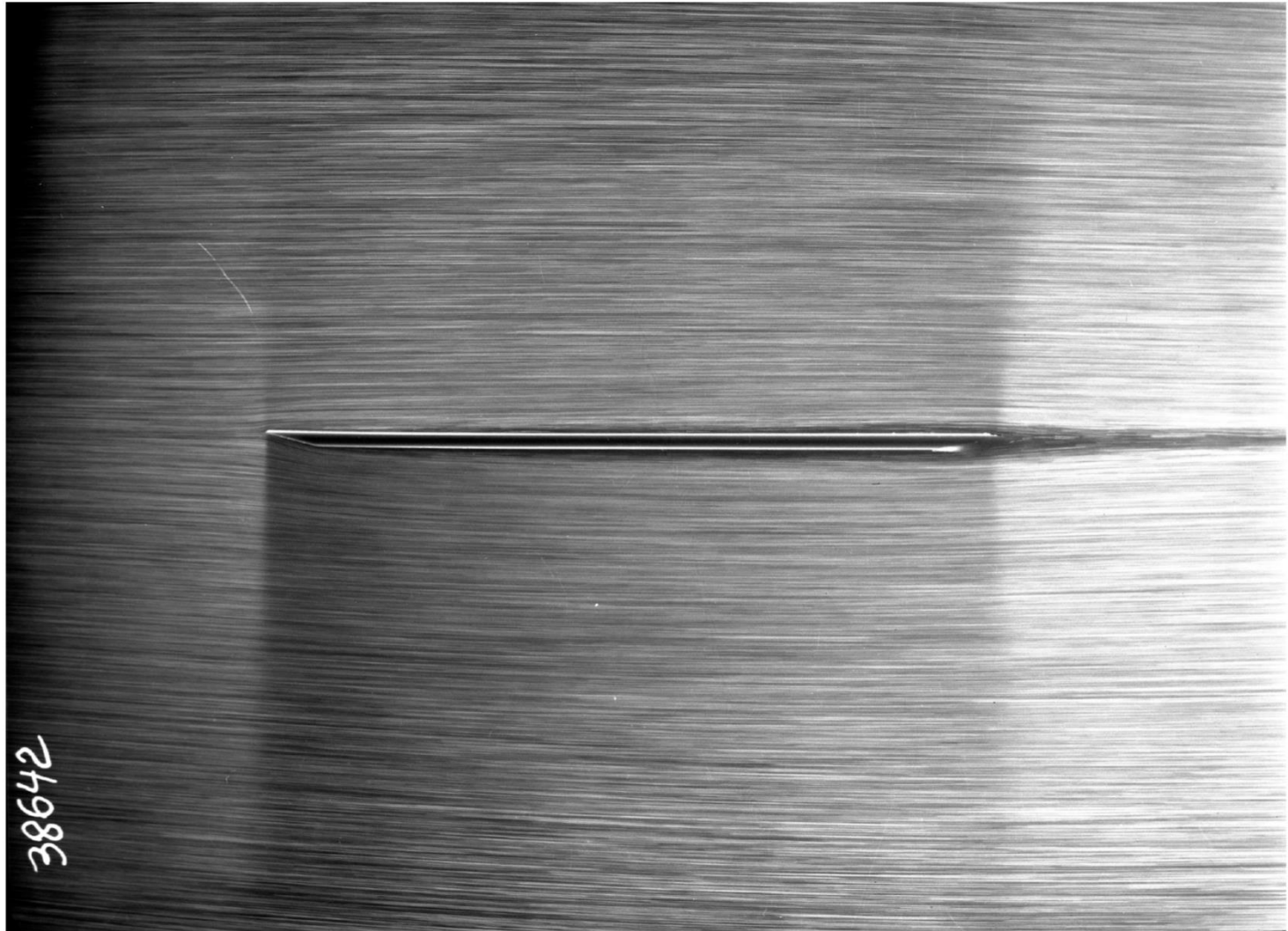
Real flow: the viscous friction decelerates the fluid close to the wall (no slip)



Inviscid flow: the fluid slips on the wall

Écoulement le long d'une plaque plane

$Re_L = 10^4$



Werlé (1974)

The boundary layer

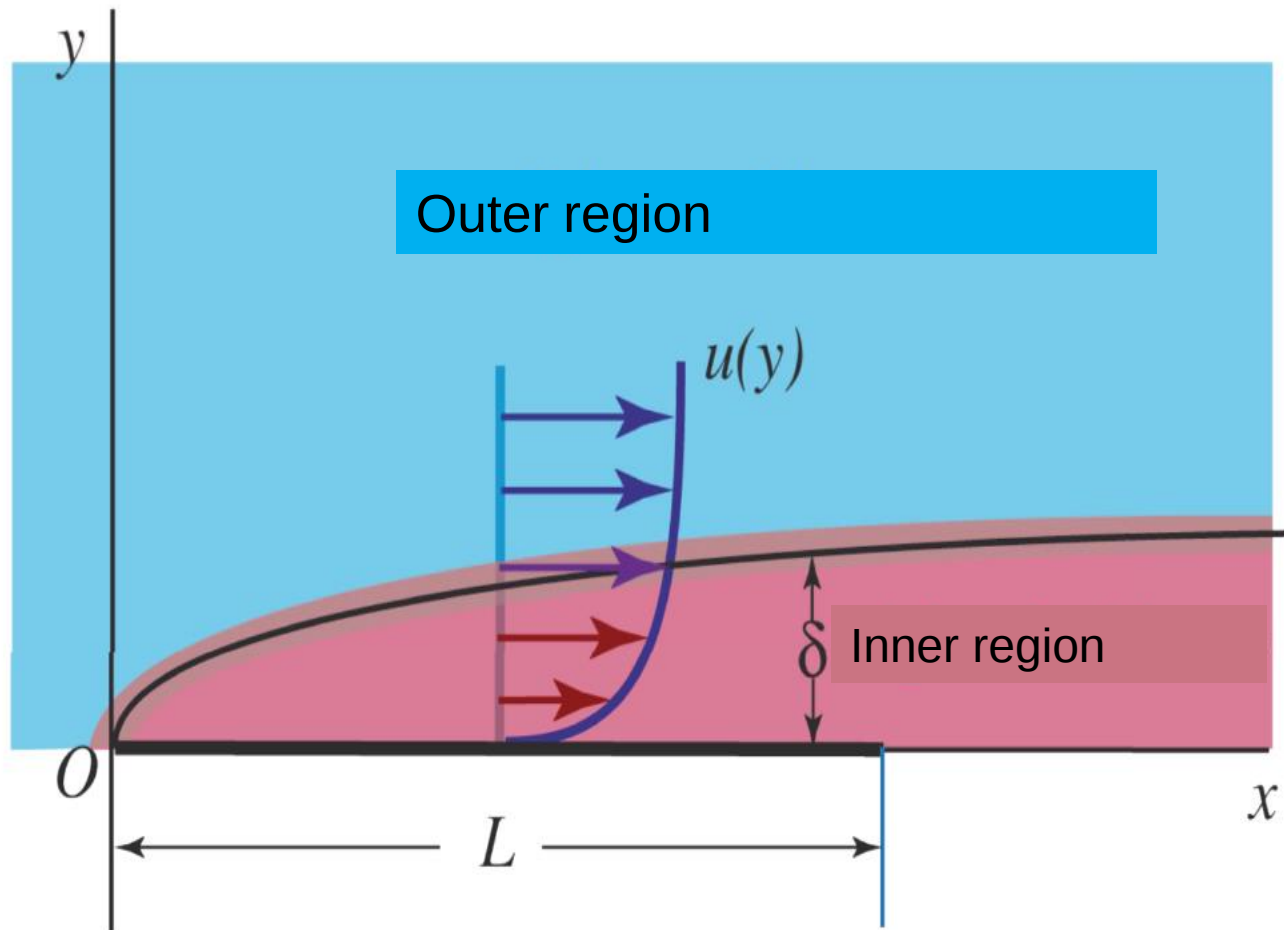
The boundary layer is a region where the velocity (streamwise and normal) is reduced due to no slip condition and friction.

It is a region where the streamwise velocity component is much larger than the normal $U_x \gg U_y$

It is a region of large shear, where the normal gradient of the streamwise velocity is large.

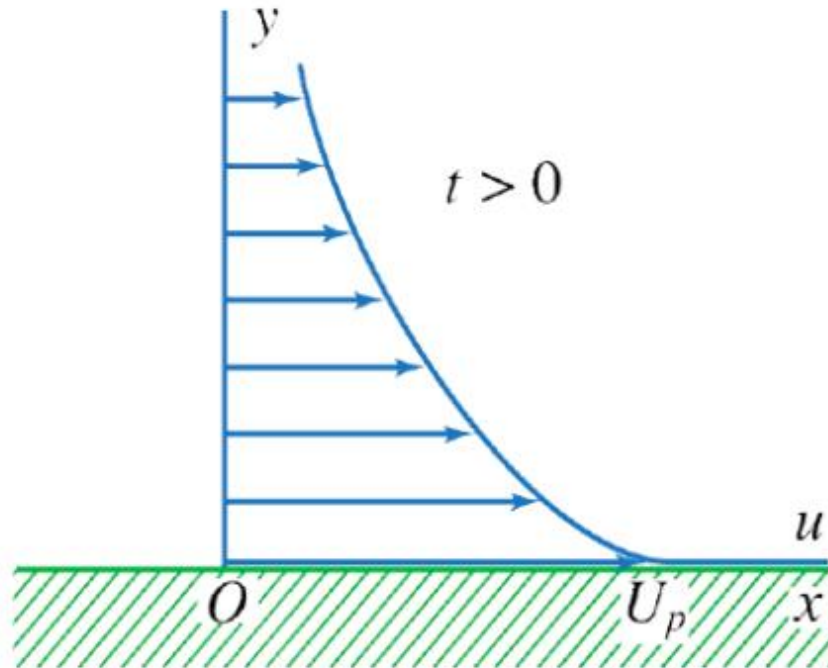
$\frac{\partial U_x}{\partial y}$ is not negligible even if $U_x \ll 1$

Boundary layer



Fundamental question:
What is the thickness of the boundary layer?

Have a look at a similar experiment : an impulsively started plate (1st Stokes problem)



$$\frac{\partial u}{\partial t} = \nu \frac{\partial^2 u}{\partial y^2}$$

$$u(0, t) = U_p \quad t > 0$$

$$u(y, t) \rightarrow 0 \quad y \rightarrow +\infty$$

With a space-time analogy, dimensional analysis suggests to replace t by x/U in δ

$$\delta(t) \sim (vt)^{1/2}$$

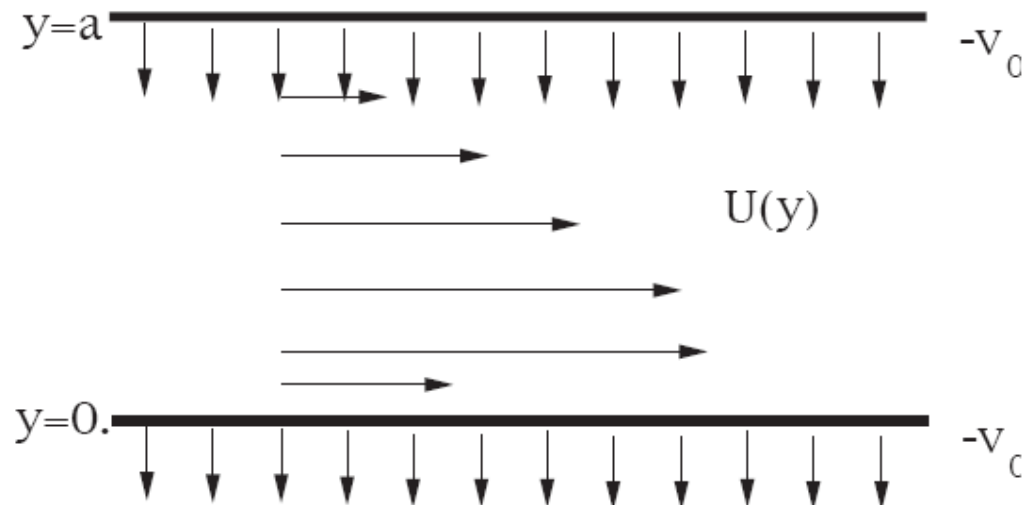
$$\Rightarrow \delta(x) \sim (vx/U)^{1/2}$$

$$\delta(x)/x \sim Re_x^{-1/2}$$

The method of matched asymptotic expansion



Escher



A 2D channel flow from left to right due to an imposed pressure gradient with suction at the lower wall and blowing at the upper.

We look for a streamwise independent solution

$$\vec{u}(x, y, z) = U(y) \vec{e}_x + V(y) \vec{e}_y$$

Continuity ensures that $V=V_0$

$$0 = -\frac{\partial p}{\partial y} \quad \text{so} \quad p(x) = -kx + P_0$$

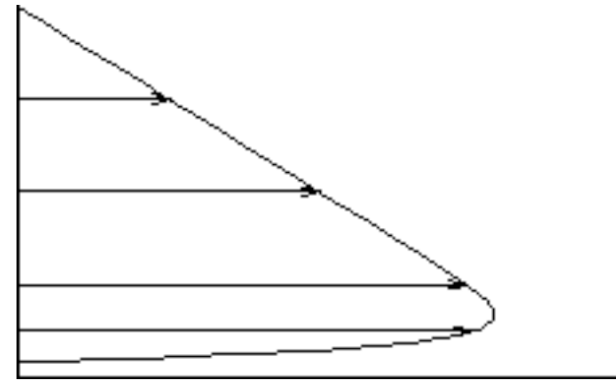
$$-v_0 \frac{\partial U}{\partial y} = \frac{k}{\rho} + \nu \frac{\partial^2 U}{\partial y^2}$$

$$U(0)=U(a)=0$$

$$U_{exact}(y) = \frac{ka}{\rho v_0} \left(-\frac{y}{a} + \left(\frac{1 - \exp(-v_0 y / \nu)}{1 - \exp(-v_0 a / \nu)} \right) \right)$$



Weak aspiration



strong aspiration

Nondimensionalization

$$y = a\bar{y} \quad u = U_0\bar{u}$$

$$- (v_0 a / \nu) \frac{\partial \bar{U}}{\partial \bar{y}} = k a^2 / (\rho \nu U_0) + \frac{\partial^2 \bar{U}}{\partial \bar{y}^2}$$

$$U_0 = k a^2 / \rho \nu$$

$$- (v_0 a / \nu) \frac{\partial \bar{U}}{\partial \bar{y}} = 1 + \frac{\partial^2 \bar{U}}{\partial \bar{y}^2}$$

Reynolds

$$\bar{U}(0) = \bar{U}(1) = 0$$

Light blowing/suction, $Re \ll 1$ $\varepsilon = Re$

$$\bar{U}'' + \varepsilon \bar{U}' + 1 = 0, \text{ and } \bar{U}(0) = \bar{U}(1) = 0$$

$$U = u_0 + \varepsilon u_1$$

$$u_0 = \frac{\bar{y}}{2} - \frac{\bar{y}^2}{2}$$

$$\bar{u}_1'' + 1/2 - \bar{y} = 0 \text{ and } \bar{u}_1(0) = \bar{u}_1(1) = 0$$

$$\bar{u}_1 = \frac{\bar{y}^3}{6} - \frac{\bar{y}^2}{4} + \frac{\bar{y}}{12}$$

Validation

$$\begin{aligned}\bar{U}_{exact} &= -\frac{-e^{\varepsilon}\bar{y} + \bar{y} + e^{\varepsilon} - e^{\varepsilon-\varepsilon\bar{y}}}{\varepsilon - \varepsilon e^{\varepsilon}} \\ &= \left(\frac{\bar{y}}{2} - \frac{\bar{y}^2}{2}\right) + \left(\frac{\bar{y}^3}{6} - \frac{\bar{y}^2}{4} + \frac{\bar{y}}{12}\right)\varepsilon .\end{aligned}$$

Large blowing/suction, $Re \gg 1$ $\varepsilon = 1/Re$

$$\varepsilon \bar{U}'' + \bar{U}' + 1 = 0; \text{ with } \bar{U}(0) = \bar{U}(1) = 0$$

$$\bar{U} = 1 - \bar{y}$$

$$\bar{U}(0) = 1,$$

Contradiction in $y=0$!

Singular perturbation!

Large blowing/suction, $Re \gg 1$ $\varepsilon = 1/Re$

$$\varepsilon \bar{U}'' + \bar{U}' + 1 = 0; \text{ with } \bar{U}(0) = \bar{U}(1) = 0.$$

Rescaling close to $y=0$

$$U_0 = ka/\rho/v_0$$

$$\tilde{U} = \bar{U} \text{ and } \bar{y} = \delta \tilde{y}, \text{ with } \delta \ll 1$$

$$\varepsilon \frac{1}{\delta^2} \frac{d^2 \tilde{U}}{d\tilde{y}^2} + \frac{1}{\delta} \frac{d\tilde{U}}{d\tilde{y}} + 1 = 0$$

Dominant balance

$$\varepsilon \frac{1}{\delta^2} = \frac{1}{\delta} \quad \delta = \varepsilon$$

$$\varepsilon \frac{1}{\delta^2} \frac{d^2 \tilde{U}}{d\tilde{y}^2} + \frac{1}{\delta} \frac{d\tilde{U}}{d\tilde{y}} + \cancel{1} = 0$$

$$\frac{d^2 \tilde{U}}{d\tilde{y}^2} + \frac{d\tilde{U}}{d\tilde{y}} = 0 \quad \tilde{U}(0) = 0$$

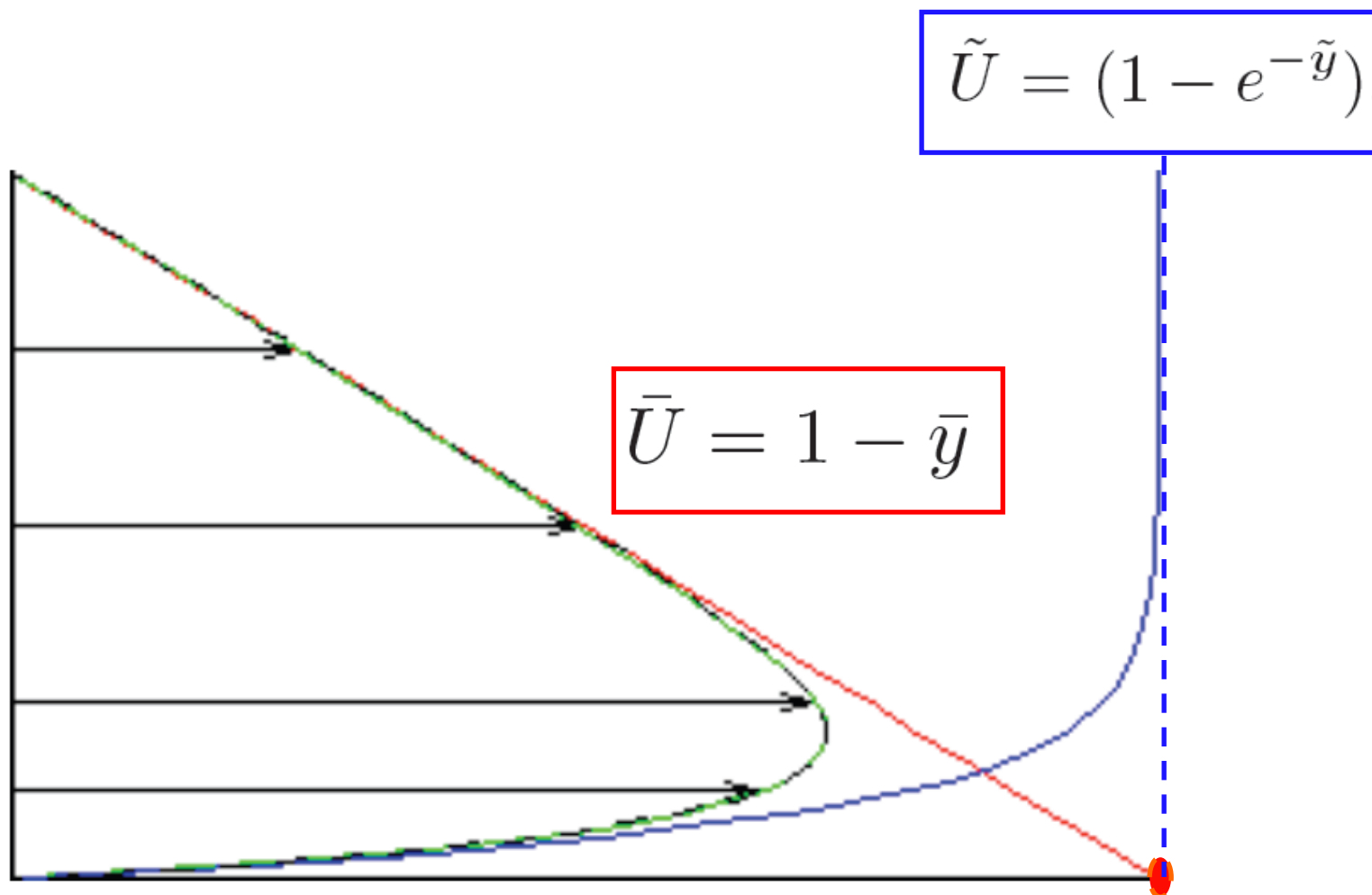
$$\tilde{U} = A(1 - e^{\tilde{y}})$$

Matching

$$\lim_{\bar{y} \rightarrow 0} \bar{U}(\bar{y}) = \lim_{\tilde{y} \rightarrow \infty} \tilde{U}(\tilde{y})$$

$$\bar{U}(0) = 1 \quad \tilde{U}(\infty) = A$$

$$A = 1$$

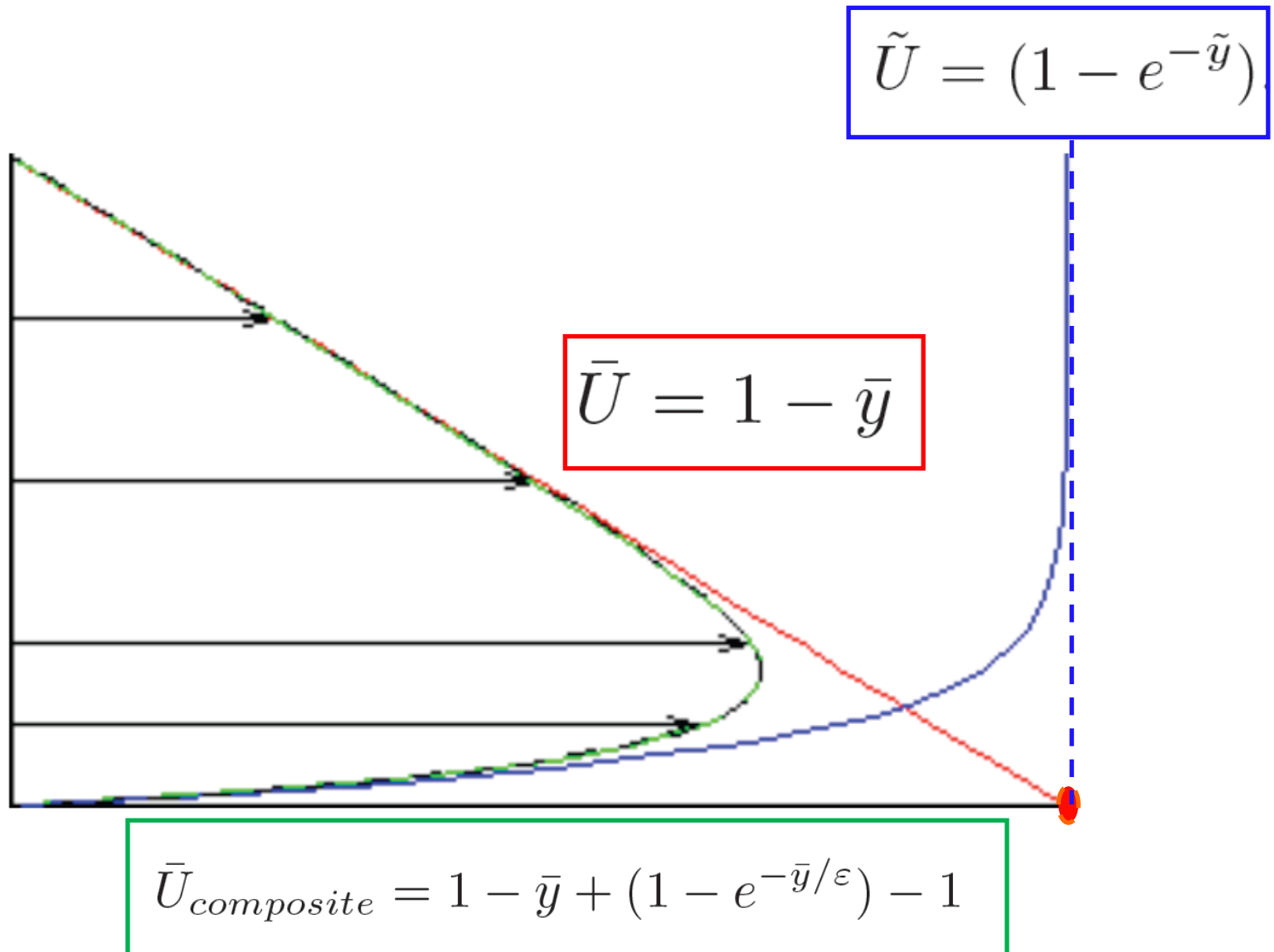


$$\bar{U}_{exact} \simeq -\varepsilon(\bar{y}/\varepsilon) + \frac{(1 - e^{\bar{y}/\varepsilon})}{(1 - 0)} \simeq (1 - e^{-\bar{y}/\varepsilon})$$

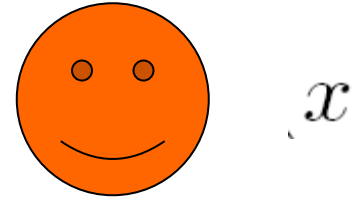
Composite expansion

$$\bar{U}_{composite} = \bar{U}(\bar{y}) + \tilde{U}(\bar{y}/\varepsilon) - \bar{U}(0)$$

$$\bar{U}_{composite} = 1 - \bar{y} + (1 - e^{-\bar{y}/\varepsilon}) - 1$$



Matching condition



$$\lim_{\tilde{x} \rightarrow +\infty} \text{DI} = \lim_{x \rightarrow 0} \text{Dext}$$

2D Flow

$$\frac{\partial u}{\partial x} + \frac{\partial v}{\partial y} = 0$$

$$\left(u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y} \right) u = -\frac{1}{\rho} \frac{\partial p}{\partial x} + \nu \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) u$$

$$\left(u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y} \right) v = -\frac{1}{\rho} \frac{\partial p}{\partial y} + \nu \left(\frac{\partial^2}{\partial x^2} + \frac{\partial^2}{\partial y^2} \right) v$$

Scales

$$\tilde{x} = \frac{x}{L} \qquad \tilde{y} = \frac{y}{\delta}$$

$$\tilde{u} = \frac{u}{U_{\infty}} \qquad \tilde{v} = \frac{v}{V} \qquad \tilde{p} = \frac{p}{\rho U_{\infty}^2}$$

$$\frac{\delta}{L} \ll 1 \qquad \frac{V}{U_{\infty}} \ll 1$$

Dominant balance

$$\frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \frac{\partial \tilde{v}}{\partial \tilde{y}} = 0$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = - \frac{\partial \tilde{p}}{\partial \tilde{x}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \left(\frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{u}}{\partial \tilde{x}^2} \right)$$

$$\frac{V}{U_\infty} \tilde{u} \frac{\partial \tilde{v}}{\partial \tilde{x}} + \left(\frac{V}{U_\infty} \right)^2 \frac{L}{\delta} \tilde{v} \frac{\partial \tilde{v}}{\partial \tilde{y}} = - \frac{L}{\delta} \frac{\partial \tilde{p}}{\partial \tilde{y}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \frac{V}{U_\infty} \left(\frac{\partial^2 \tilde{v}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{v}}{\partial \tilde{x}^2} \right)$$

Dominant balance

$$\frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \frac{\partial \tilde{v}}{\partial \tilde{y}} = 0$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = - \frac{\partial \tilde{p}}{\partial \tilde{x}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \left(\frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{u}}{\partial \tilde{x}^2} \right)$$

$$\frac{V}{U_\infty} \tilde{u} \frac{\partial \tilde{v}}{\partial \tilde{x}} + \left(\frac{V}{U_\infty} \right)^2 \frac{L}{\delta} \tilde{v} \frac{\partial \tilde{v}}{\partial \tilde{y}} = - \frac{L}{\delta} \frac{\partial \tilde{p}}{\partial \tilde{y}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \frac{V}{U_\infty} \left(\frac{\partial^2 \tilde{v}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{v}}{\partial \tilde{x}^2} \right)$$

Dominant balance

$$\frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \frac{\partial \tilde{v}}{\partial \tilde{y}} = 0$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = - \frac{\partial \tilde{p}}{\partial \tilde{x}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \left(\frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{u}}{\partial \tilde{x}^2} \right)$$

$$\frac{V}{U_\infty} \tilde{u} \frac{\partial \tilde{v}}{\partial \tilde{x}} + \left(\frac{V}{U_\infty} \right)^2 \frac{L}{\delta} \tilde{v} \frac{\partial \tilde{v}}{\partial \tilde{y}} = - \frac{L}{\delta} \frac{\partial \tilde{p}}{\partial \tilde{y}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \frac{V}{U_\infty} \left(\frac{\partial^2 \tilde{v}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{v}}{\partial \tilde{x}^2} \right)$$

Dominant balance

$$\frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \frac{\partial \tilde{v}}{\partial \tilde{y}} = 0$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = - \frac{\partial \tilde{p}}{\partial \tilde{x}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \left(\frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{u}}{\partial \tilde{x}^2} \right)$$

$$\frac{V}{U_\infty} \tilde{u} \frac{\partial \tilde{v}}{\partial \tilde{x}} + \left(\frac{V}{U_\infty} \right)^2 \frac{L}{\delta} \tilde{v} \frac{\partial \tilde{v}}{\partial \tilde{y}} = - \frac{L}{\delta} \frac{\partial \tilde{p}}{\partial \tilde{y}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \frac{V}{U_\infty} \left(\frac{\partial^2 \tilde{v}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{v}}{\partial \tilde{x}^2} \right)$$

Dominant balance

$$\frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \frac{\partial \tilde{v}}{\partial \tilde{y}} = 0$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = - \frac{\partial \tilde{p}}{\partial \tilde{x}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \left(\frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{u}}{\partial \tilde{x}^2} \right)$$

$$\frac{V}{U_\infty} \tilde{u} \frac{\partial \tilde{v}}{\partial \tilde{x}} + \left(\frac{V}{U_\infty} \right)^2 \frac{L}{\delta} \tilde{v} \frac{\partial \tilde{v}}{\partial \tilde{y}} = - \frac{L}{\delta} \frac{\partial \tilde{p}}{\partial \tilde{y}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \frac{V}{U_\infty} \left(\frac{\partial^2 \tilde{v}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{v}}{\partial \tilde{x}^2} \right)$$

Dominant balance

Leading order terms

$$\frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \frac{\partial \tilde{v}}{\partial \tilde{y}} = 0$$

$$\tilde{u} \frac{\partial \tilde{u}}{\partial \tilde{x}} + \frac{L}{\delta} \frac{V}{U_\infty} \tilde{v} \frac{\partial \tilde{u}}{\partial \tilde{y}} = - \frac{\partial \tilde{p}}{\partial \tilde{x}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \left(\frac{\partial^2 \tilde{u}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{u}}{\partial \tilde{x}^2} \right)$$

$$\frac{V}{U_\infty} \tilde{u} \frac{\partial \tilde{v}}{\partial \tilde{x}} + \left(\frac{V}{U_\infty} \right)^2 \frac{L}{\delta} \tilde{v} \frac{\partial \tilde{v}}{\partial \tilde{y}} = - \frac{L}{\delta} \frac{\partial \tilde{p}}{\partial \tilde{y}}$$

$$+ \frac{1}{\text{Re}} \frac{L^2}{\delta^2} \frac{V}{U_\infty} \left(\frac{\partial^2 \tilde{v}}{\partial \tilde{y}^2} + \frac{\delta^2}{L^2} \frac{\partial^2 \tilde{v}}{\partial \tilde{x}^2} \right)$$

Scales

$$\frac{L}{\delta} \frac{V}{U_{\infty}} = 1$$

$$\frac{L^2}{\delta^2} \frac{1}{\text{Re}} = 1$$

$$\delta = \frac{L}{\text{Re}^{1/2}} \quad V = \frac{U_{\infty}}{\text{Re}^{1/2}}$$

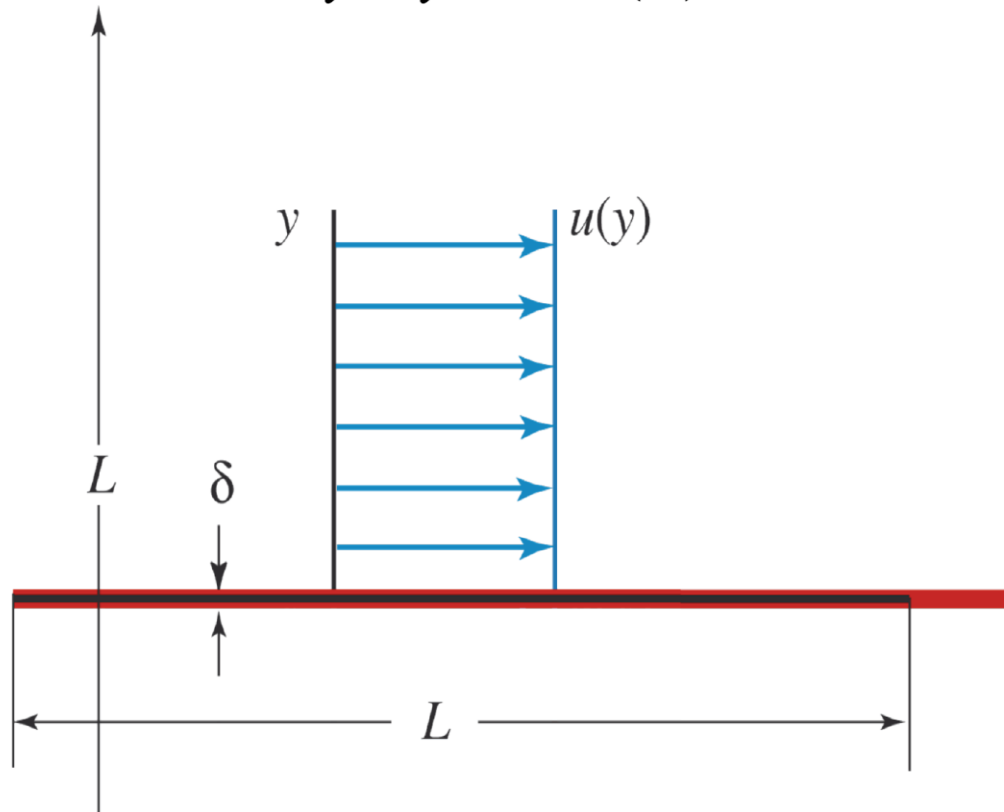
Small parameter

$$\left(\frac{1}{\text{Re}} \right)^{1/2} = \frac{\delta}{L} \ll 1$$

Outer approximation

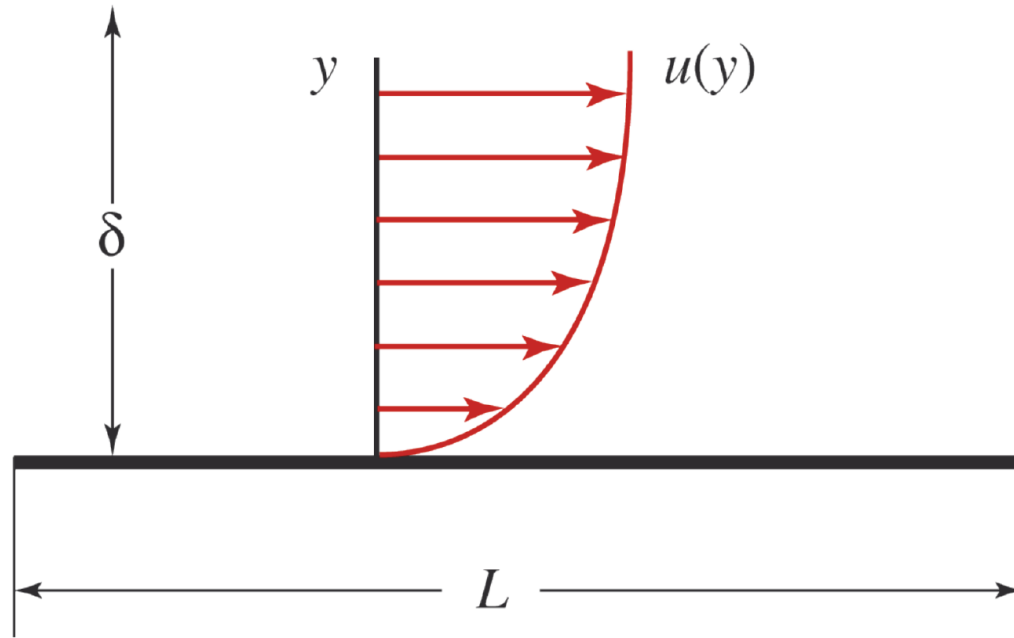
Inviscid fluid

$$y = \bar{y} L = \mathcal{O}(L)$$

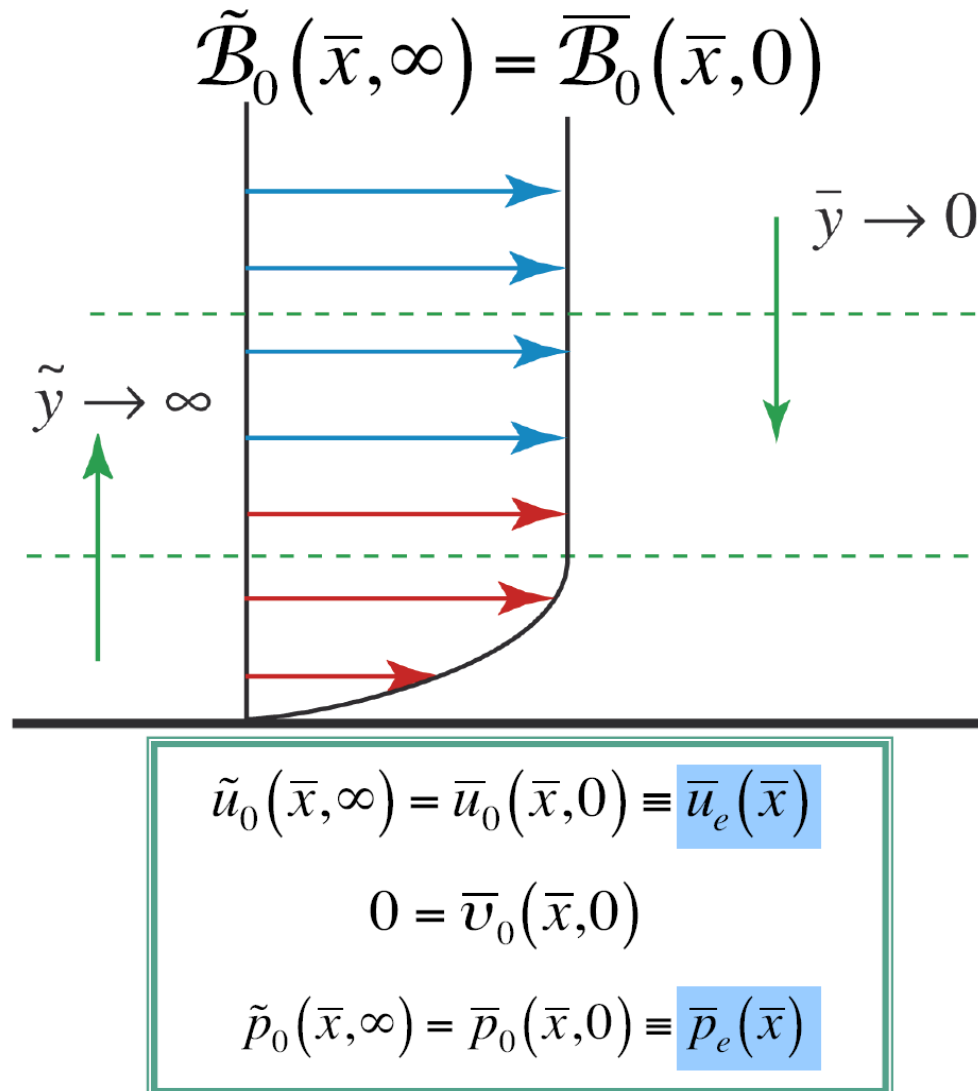


Inner approximation Boundary layer equations

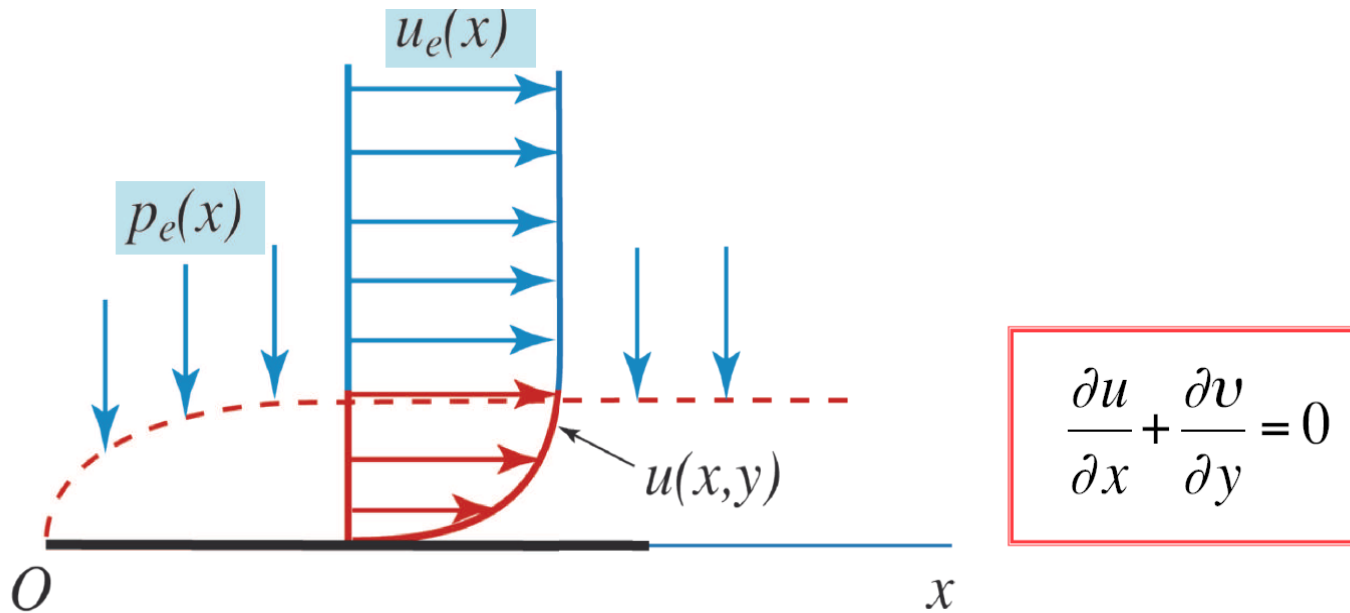
$$y = \tilde{y} \delta = \mathcal{O}(\delta)$$



Matching condition



Boundary layer equations



$$\left(u \frac{\partial}{\partial x} + v \frac{\partial}{\partial y} \right) u = - \frac{1}{\rho} \frac{dp_e}{dx} + \nu \frac{\partial^2 u}{\partial y^2}$$

$$p(x, y) = p_e(x)$$

$$u(x, 0) = v(x, 0) = 0$$

$$u(x, \infty) = u_e(x)$$

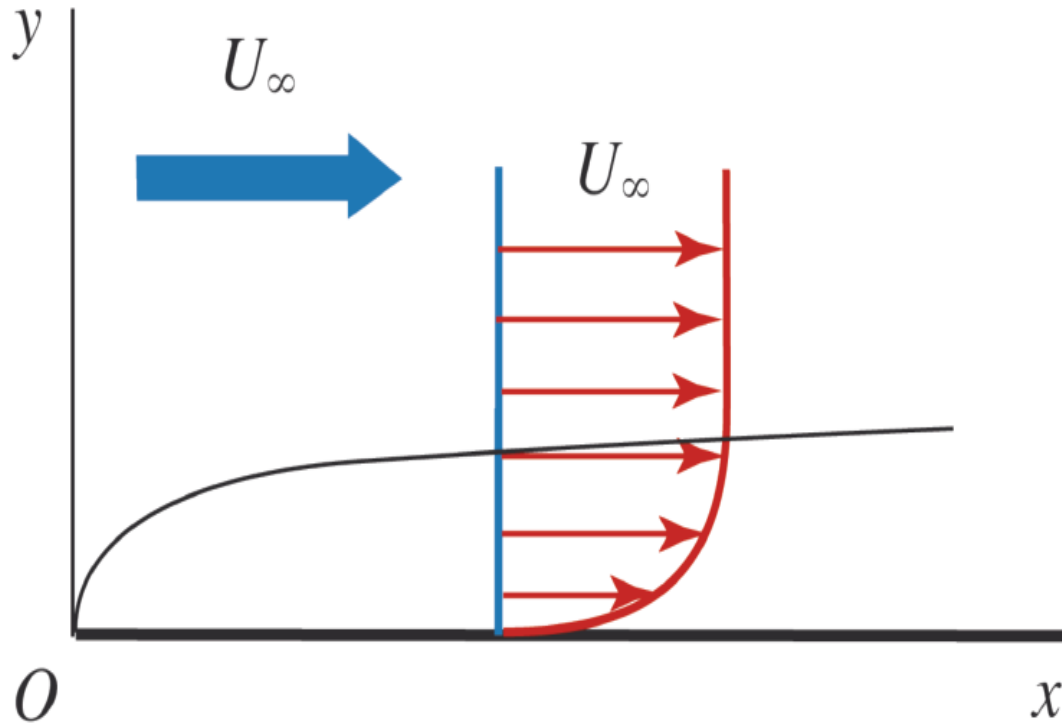
Boundary layer equations in streamfunction formulation

$$\left(\frac{\partial \psi}{\partial y} \frac{\partial}{\partial x} - \frac{\partial \psi}{\partial x} \frac{\partial}{\partial y} \right) \frac{\partial \psi}{\partial y} = - \frac{1}{\rho} \frac{dp_e}{dx} + \nu \frac{\partial^3 \psi}{\partial y^3}$$

$$\psi(x, 0) = \frac{\partial \psi}{\partial y}(x, 0) = 0$$

$$\psi(x, y) \sim u_e(x)y, \quad y \rightarrow \infty$$

Boundary layer on a semi-infinite plate



Self-similar solution?

$$\psi = A\psi' \quad x = Bx' \quad y = Cy'$$

$$v = Dv' \quad U_\infty = EU'_\infty$$

$$\frac{AC}{BD} \left(\frac{\partial \psi'}{\partial y'} \frac{\partial}{\partial x'} - \frac{\partial \psi'}{\partial x'} \frac{\partial}{\partial y'} \right) \frac{\partial \psi'}{\partial y'} = v' \frac{\partial^3 \psi'}{\partial y'^3}$$

$$\psi'(x', 0) = \frac{\partial \psi'}{\partial y'}(x', 0) = 0$$

$$\frac{A}{CE} \psi'(x', y') \sim U'_\infty y' \quad , \quad y' \rightarrow \infty$$

$$\frac{AC}{BD} = 1 \quad \frac{A}{CE} = 1$$

Self-similar solution

$$\frac{\psi y}{\nu x}$$

$$\frac{\psi}{U_{\infty} y}$$

$$\frac{\psi^2}{U_{\infty} \nu x}$$

$$\frac{y^2}{\nu x / U_{\infty}}$$

$$\underbrace{\frac{\psi}{(U_{\infty} \nu x)^{1/2}}}_{U_{\infty} \delta(x)}$$

$$\underbrace{\frac{y}{(\nu x / U_{\infty})^{1/2}}}_{\delta(x)}$$

Boundary layer on a semi-infinite plate

Blasius solution

$$\frac{\psi(x, y)}{U_{\infty} \delta(x)} = f(\eta)$$

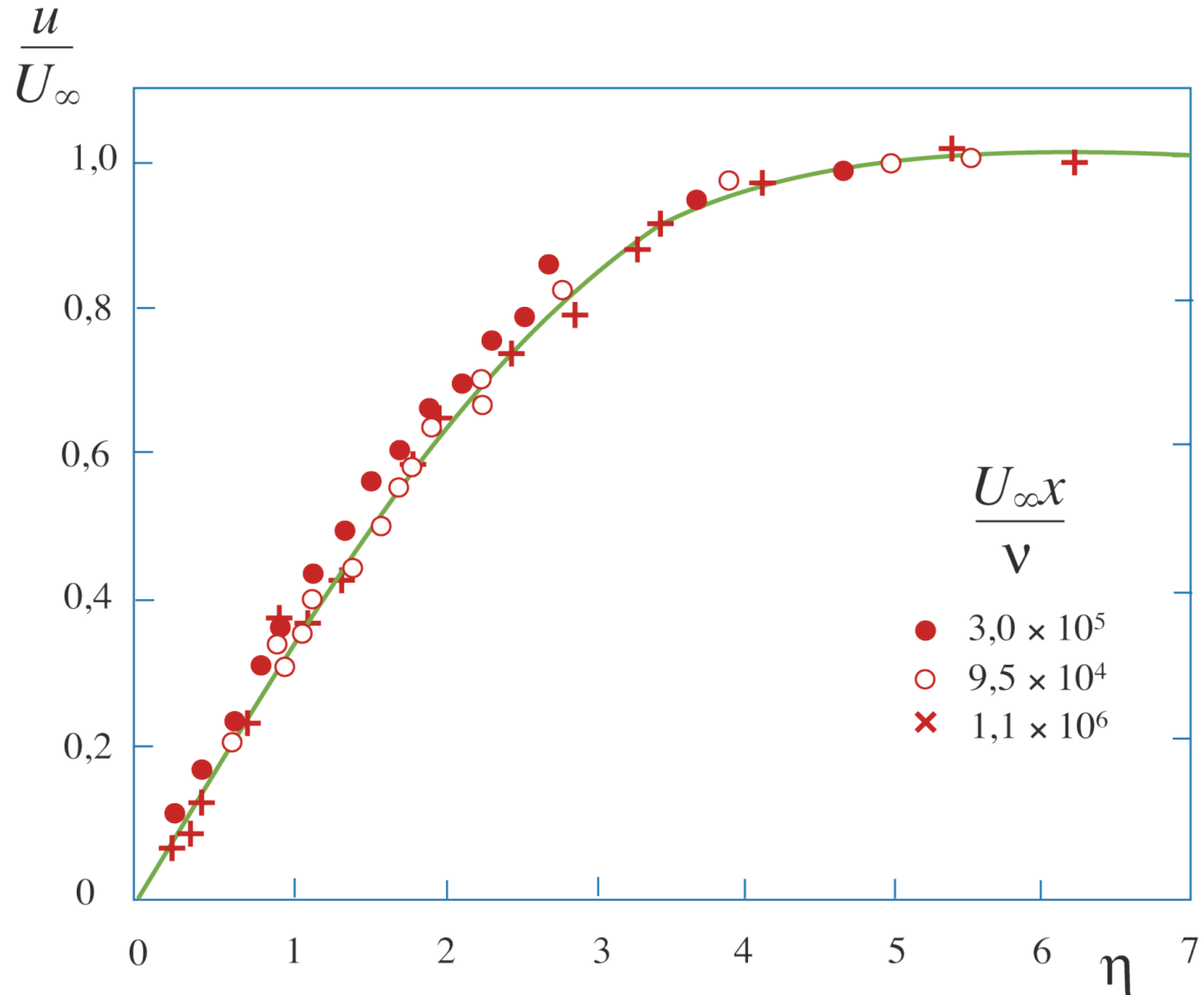
$$\eta = \frac{y}{\delta(x)} = \frac{y}{(\nu x / U_{\infty})^{1/2}}$$

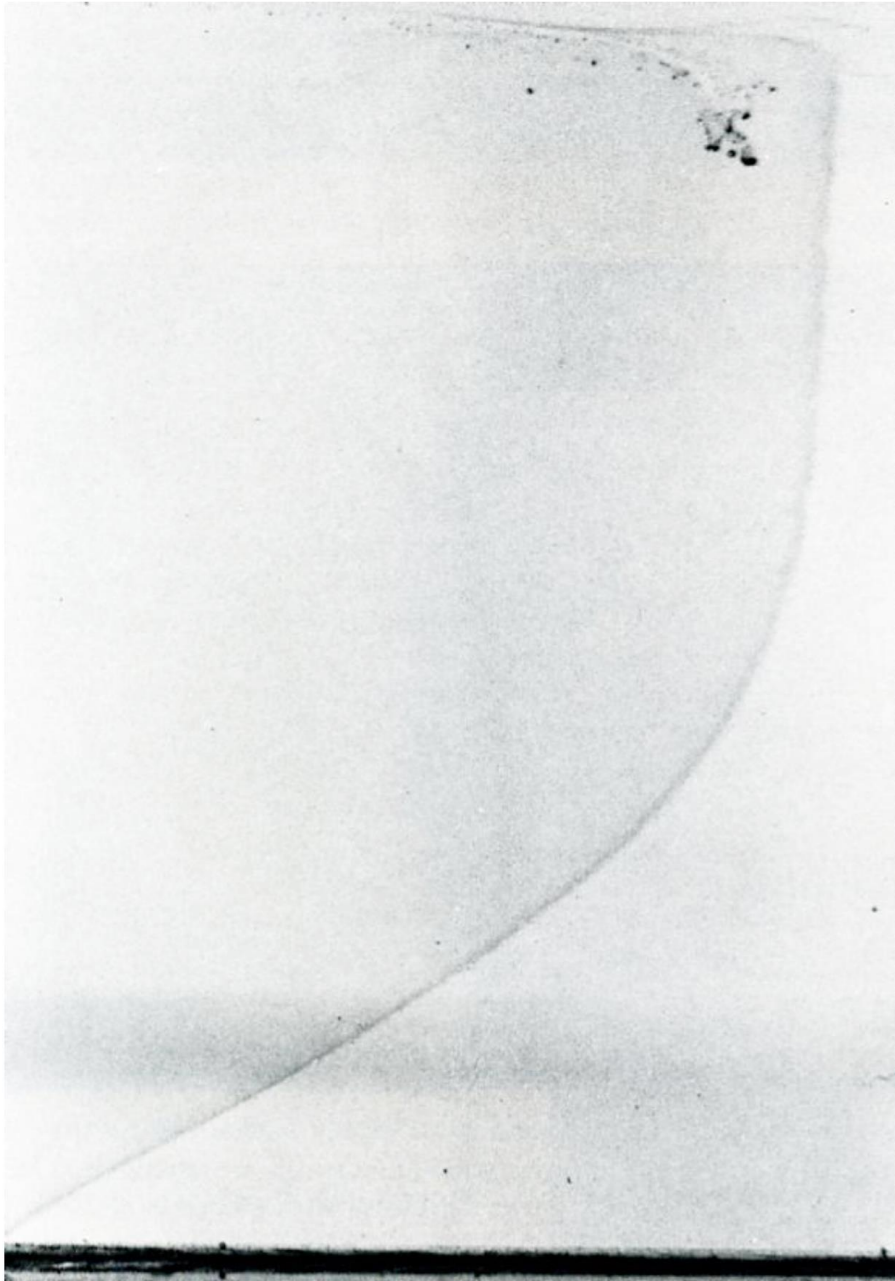
$$2f''' + ff'' = 0$$

$$f(0) = f'(0) = 0$$

$$f'(\infty) = 1$$

Velocity profiles in BL

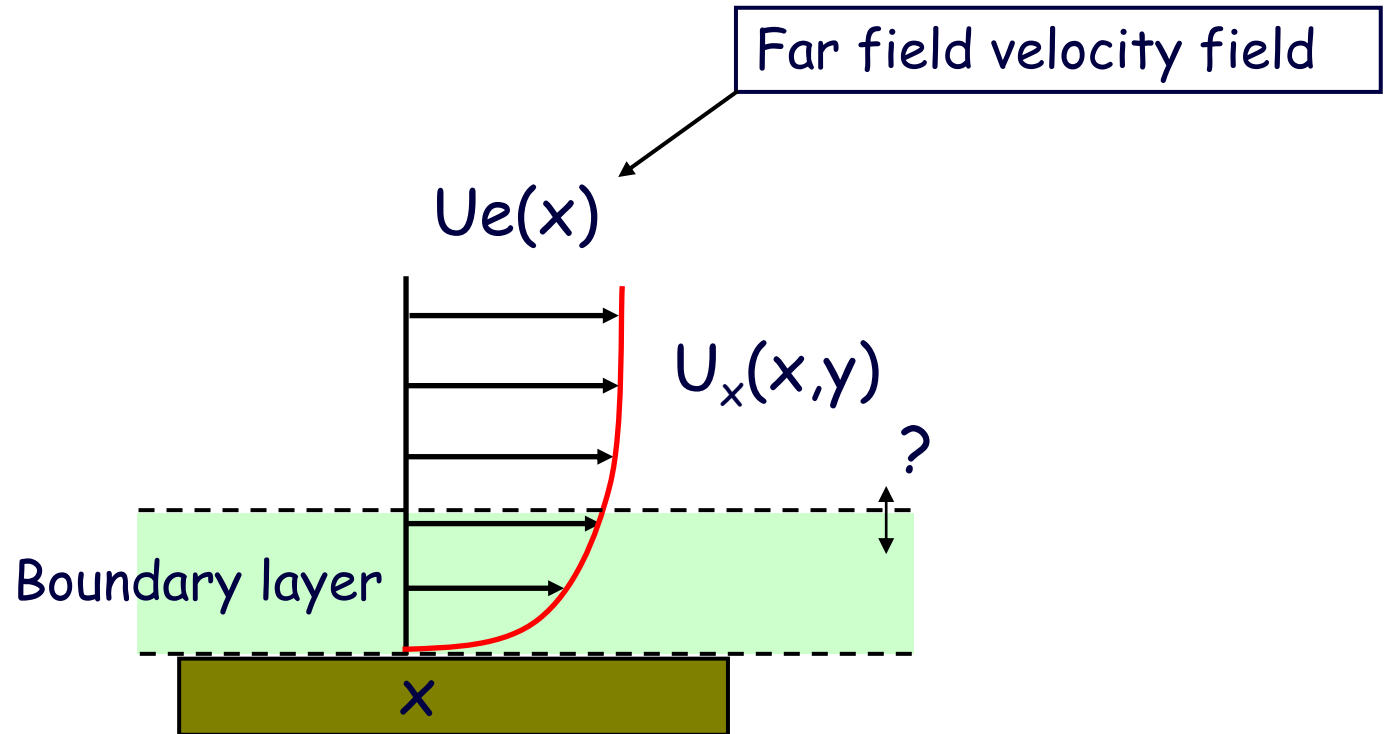




Profil de vitesse expérimental
dans la couche limite de
Blasius à $Re_x = 500$

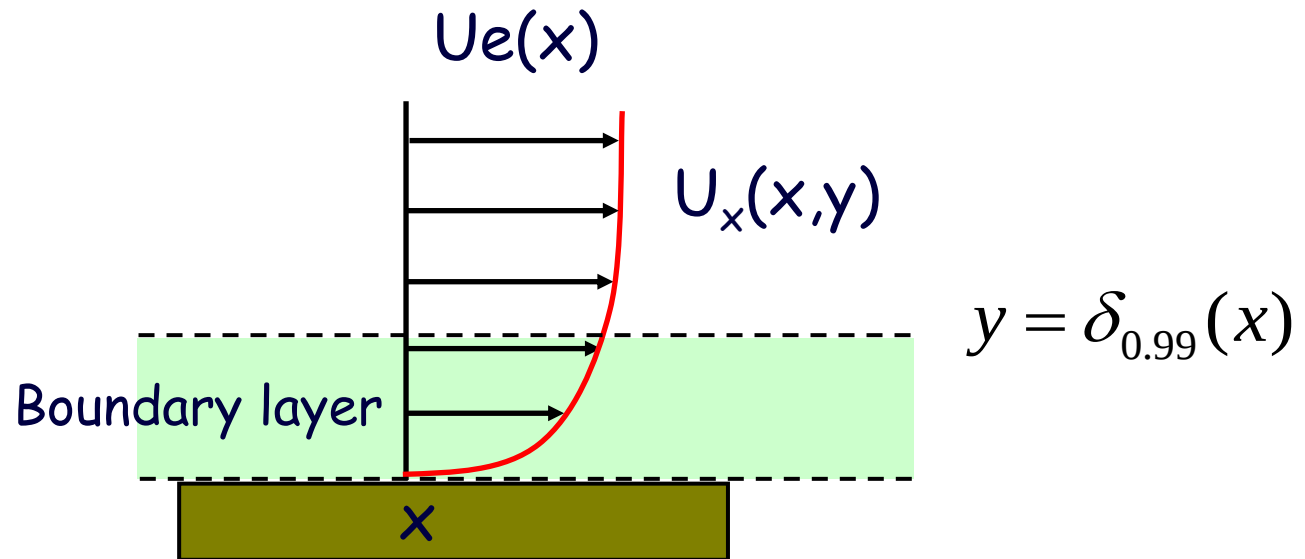
Wortmann (1982)

Boundary layer thickness



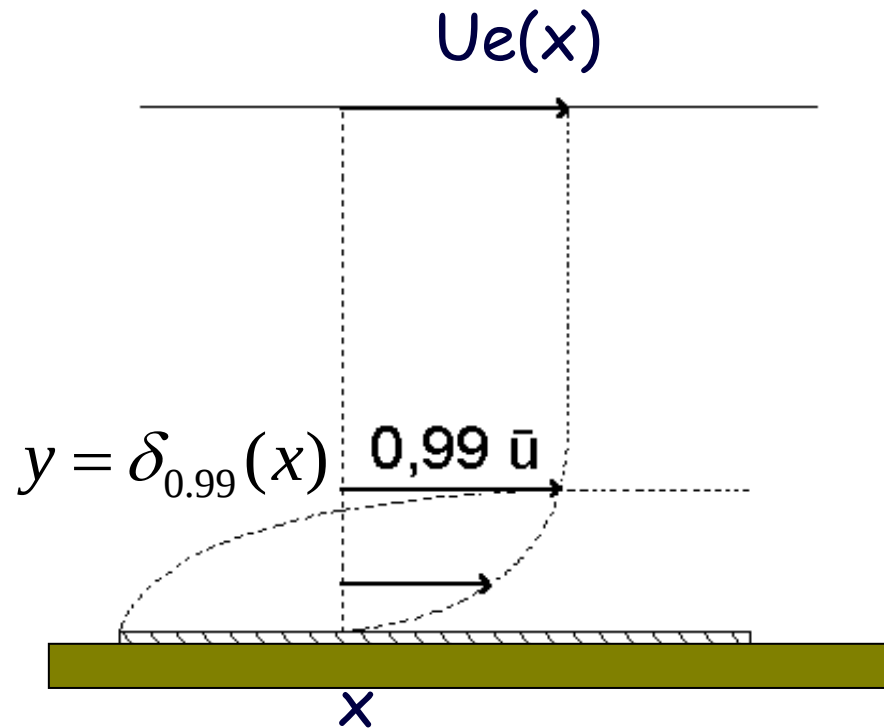
How to characterize or define the boundary layer thickness?

$\delta_{0.99}$ thickness



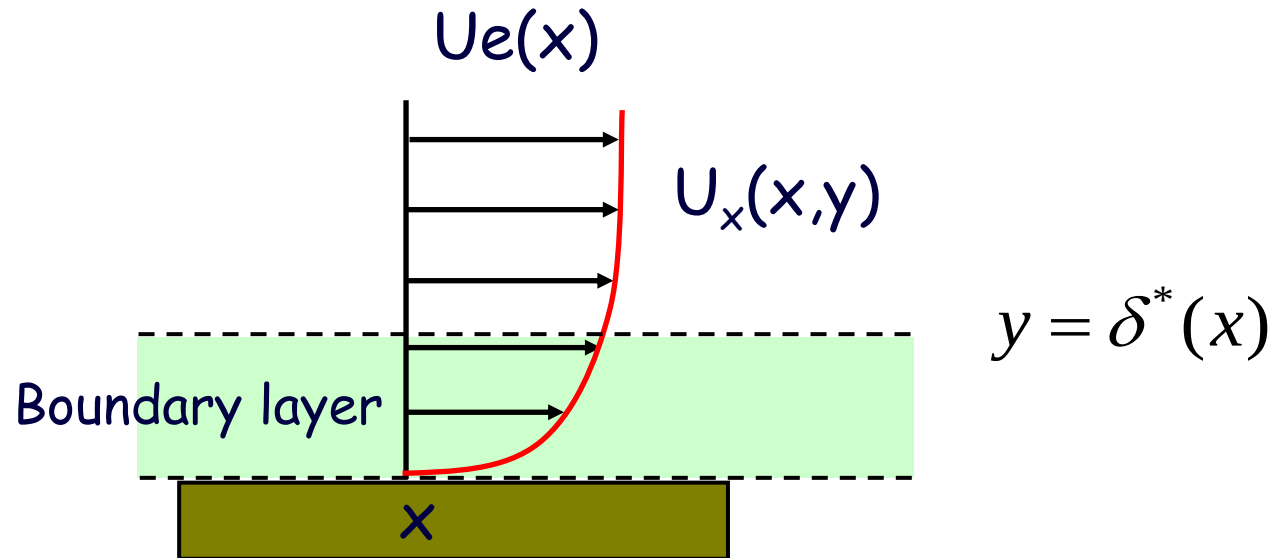
Thickness such that $U_x(x, \delta_{0.99}) = 0.99U_e(x)$

Thickness $\delta_{0.99}$



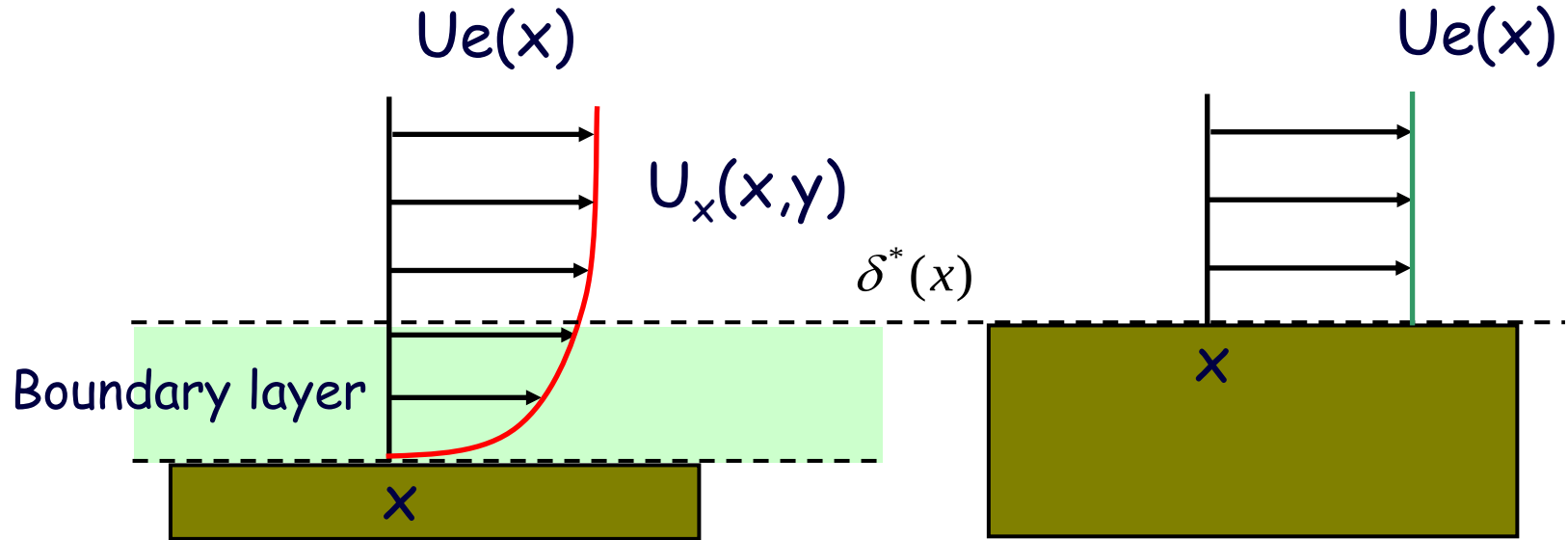
→ The boundary layer thickness is local!

Displacement thickness δ^*



Thickness defined such that
$$\delta^*(x) = \int_{y=0}^{\infty} \left(1 - \frac{U_x(x, y)}{U_e(x)} \right) dy$$

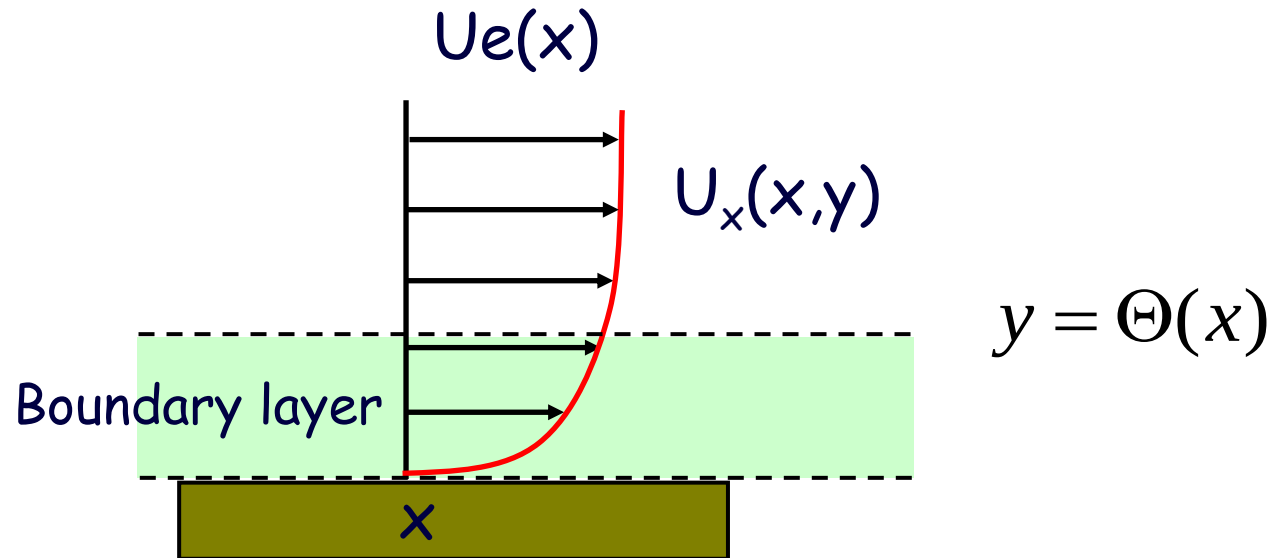
Physical meaning δ^*



Flow rate $Q_1 = \int_{y=0}^{\infty} U_x(x, y) dy$

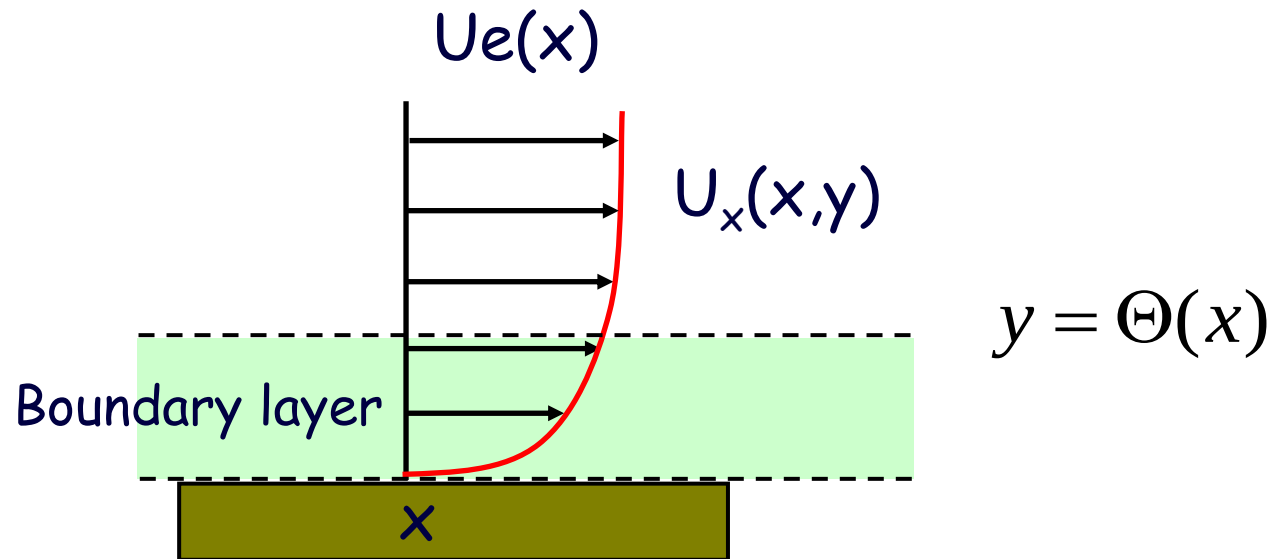
$$Q_2 = \int_{y=\delta^*(x)}^{\infty} U_e(x) dy = Q_1$$

Momentum thickness Θ



Defined as
$$\Theta(x) = \int_{y=0}^{\infty} \frac{U_x(x, y)}{U_e(x)} \left(1 - \frac{U_x(x, y)}{U_e(x)} \right) dy$$

Physical meaning of Θ

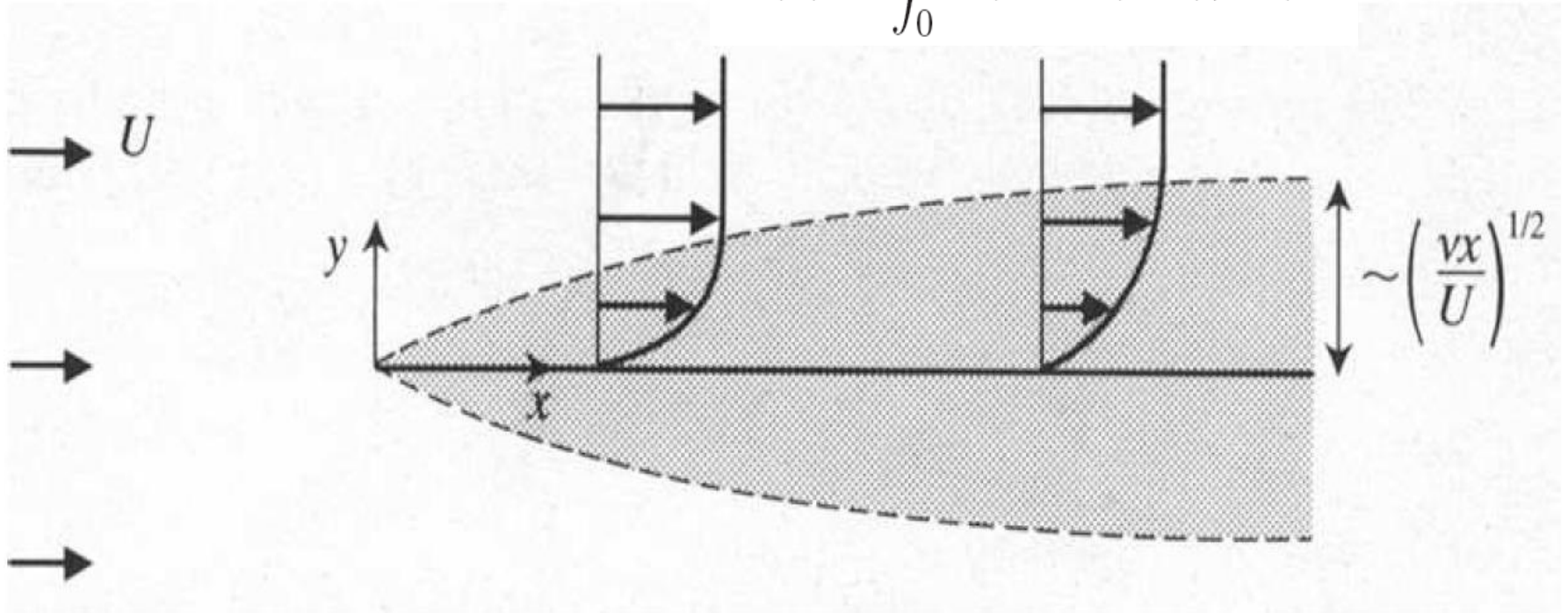


It is the thickness by which the wall should be increased to yield the same **momentum** for a inviscid flow.

Épaisseurs de couche limite

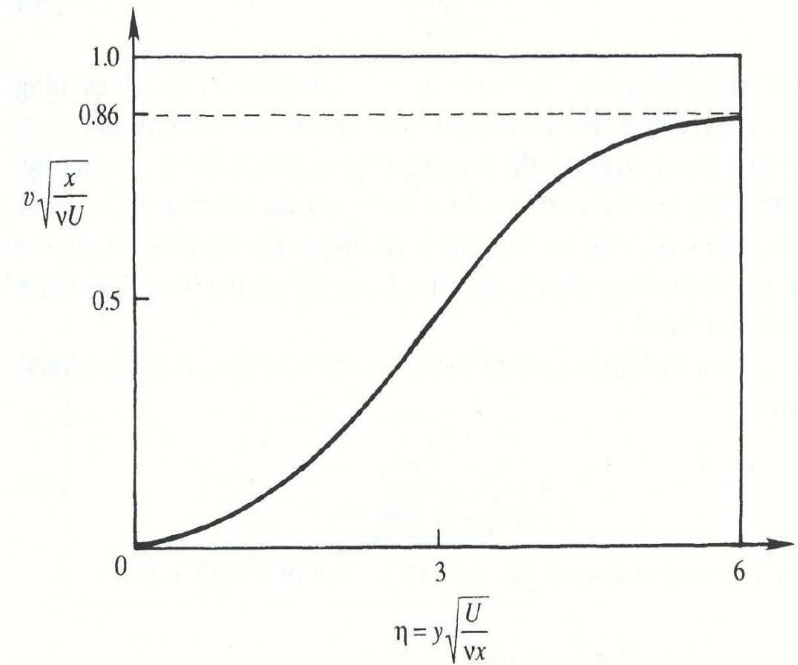
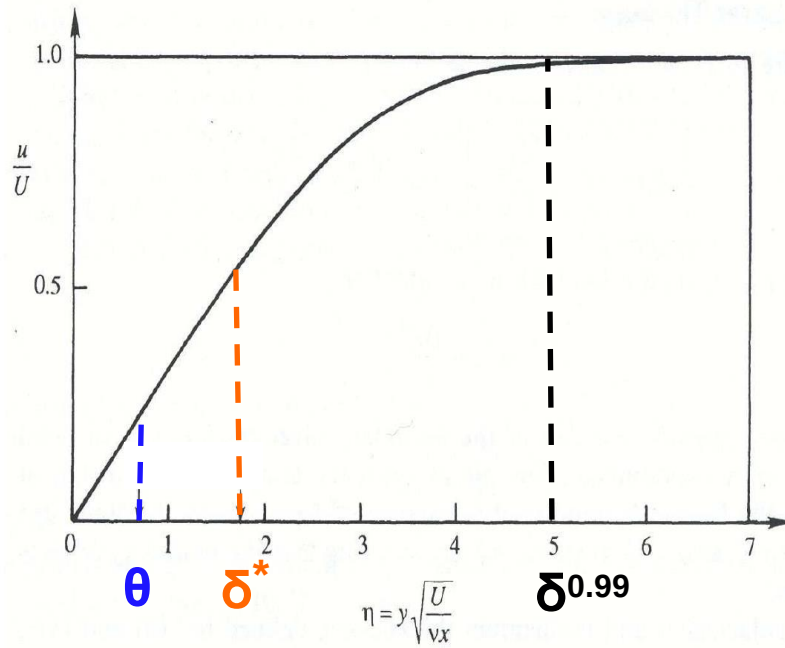
épaisseur à 99% $u[x, \delta_{0,99}(x)] = 0,99 U_e \sim 5 \delta$

épaisseur de déplacement $\delta^*(x) = \int_0^\infty (1 - u(x, y)/U_e) dy \sim 1.73 \delta$

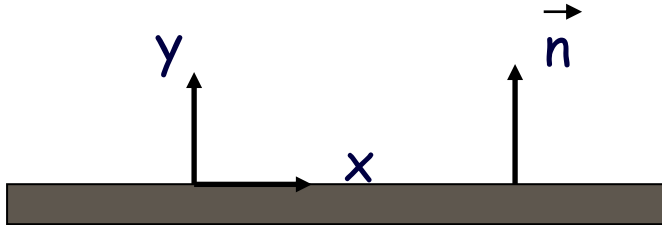


épaisseur de quantité de mouvement $\theta = \int_0^\infty \frac{u}{U_e} (1 - u/U_e) dy \sim .66 \delta$

Profils de couche limite



Viscous friction

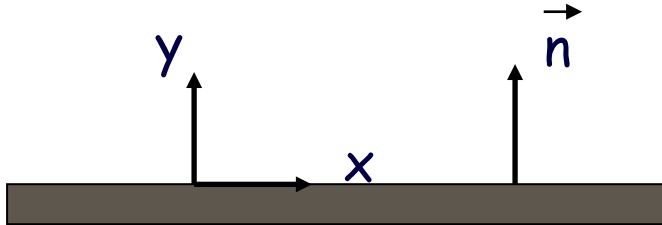


$$\begin{pmatrix} 2\mu \frac{\partial U_x}{\partial x} & \mu \left(\frac{\partial U_x}{\partial y} + \frac{\partial U_y}{\partial x} \right) \\ \mu \left(\frac{\partial U_x}{\partial y} + \frac{\partial U_y}{\partial x} \right) & 2\mu \frac{\partial U_y}{\partial y} \end{pmatrix} \cdot \begin{pmatrix} 0 \\ 1 \end{pmatrix} = \begin{pmatrix} \mu \frac{\partial U_x}{\partial y} + \mu \frac{\partial U_y}{\partial x} \\ 2\mu \frac{\partial U_y}{\partial y} \end{pmatrix}$$

$\downarrow$ $\vec{\tau}$ $\downarrow$ $\vec{n}$

A priori both drag and lift can be caused by viscous friction

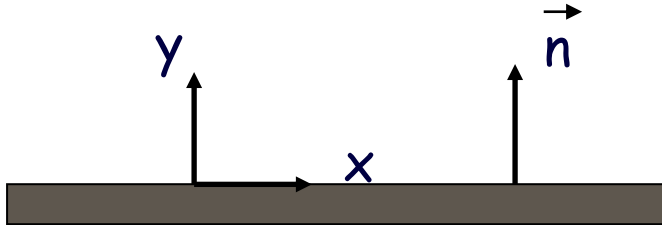
Viscous friction



$$\begin{pmatrix} 2\mu \frac{\partial U_x}{\partial x} & \mu \left(\frac{\partial U_x}{\partial y} + \frac{\partial U_y}{\partial x} \right) \\ \mu \left(\frac{\partial U_x}{\partial y} + \frac{\partial U_y}{\partial x} \right) & 2\mu \frac{\partial U_y}{\partial y} \end{pmatrix} \cdot \begin{pmatrix} 0 \\ 1 \end{pmatrix} = \begin{pmatrix} \mu \frac{\partial U_x}{\partial y} + \mu \frac{\partial U_y}{\partial x} \\ 2\mu \frac{\partial U_y}{\partial y} \end{pmatrix}$$

Note however $U_x \gg U_y$ in the boundary layer

Viscous friction

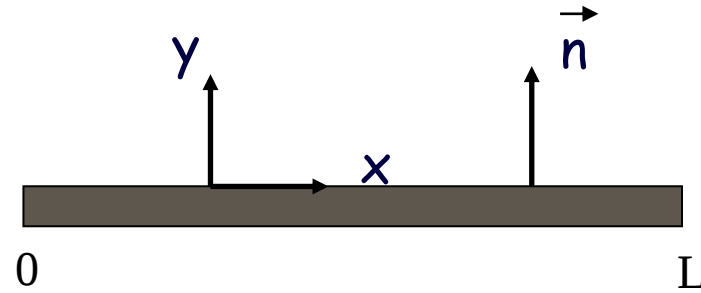


$$\begin{pmatrix} 2\mu \frac{\partial U_x}{\partial x} & \mu \left(\frac{\partial U_x}{\partial y} + \frac{\partial U_y}{\partial x} \right) \\ \mu \left(\frac{\partial U_x}{\partial y} + \frac{\partial U_y}{\partial x} \right) & 2\mu \frac{\partial U_y}{\partial y} \end{pmatrix} \cdot \begin{pmatrix} 0 \\ 1 \end{pmatrix} = \begin{pmatrix} \mu \frac{\partial U_x}{\partial y} + \cancel{\mu \frac{\partial U_y}{\partial x}} \\ \cancel{2\mu \frac{\partial U_y}{\partial y}} \end{pmatrix} \approx \begin{pmatrix} \mu \frac{\partial U_x}{\partial y} \\ 0 \end{pmatrix}$$

At leading order, only shear stress

Wall friction shear coefficient

Flat plate of length L



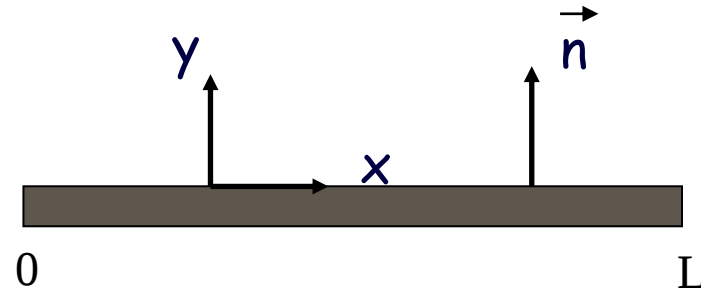
Linear density of friction coefficient

dimensional :
$$F_x = \int_0^L \tau_p(x) dx = \int_0^L \mu \frac{\partial U_x}{\partial y}(x) dx \rightarrow \text{N/m ! (2D)}$$

nondimensional :
$$C_x = \frac{F_x}{\frac{1}{2} \rho_\infty U_\infty^2 L}$$

Wall friction shear coefficient

Flat plate of length L



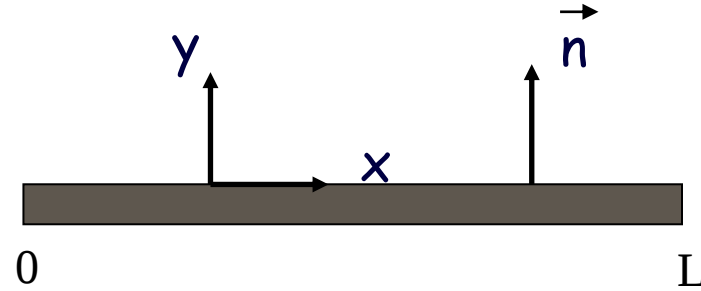
Linear density of friction coefficient

dimensional : $\tau_p(x) = \mu \frac{\partial U_x}{\partial y}(x) \rightarrow \text{N/m}^2$

nondimensional : $C_f(x) = \frac{\tau_p(x)}{\frac{1}{2} \rho_\infty U_\infty^2} = \frac{0.664}{\text{Re}_x^{1/2}}$

Wall friction shear coefficient C_f

Plaque de longueur L



Densité de contrainte pariétale (i.e. qui s'exerce à la paroi)

Dimensionné : $\tau_p(x) = \mu \frac{\partial U_x}{\partial y}(x) \rightarrow \text{N/m}^2$

$$C_f(x) = \frac{\tau_p(x)}{\frac{1}{2} \rho_\infty U_\infty^2} \approx \frac{\mu U_\infty}{\delta} \frac{1}{\frac{1}{2} \rho_\infty U_\infty^2}$$

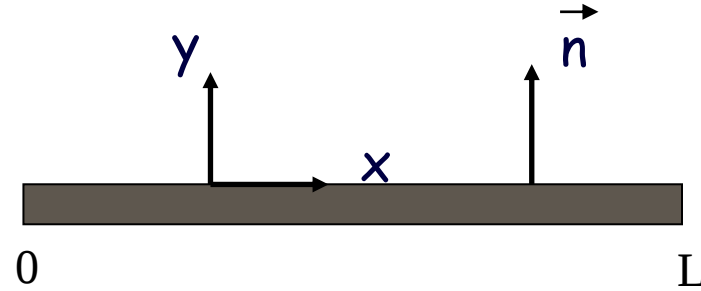
Adimensionné :

$$\approx \frac{\mu U_\infty}{\sqrt{\frac{\mu x}{\rho_\infty U_\infty}}} \frac{1}{\frac{1}{2} \rho_\infty U_\infty^2}$$

$$\approx \sqrt{\frac{\rho_\infty \mu U_\infty^3}{x}} \frac{1}{\frac{1}{2} \rho_\infty U_\infty^2}$$

Drag coefficient C_x

Plaque de longueur L

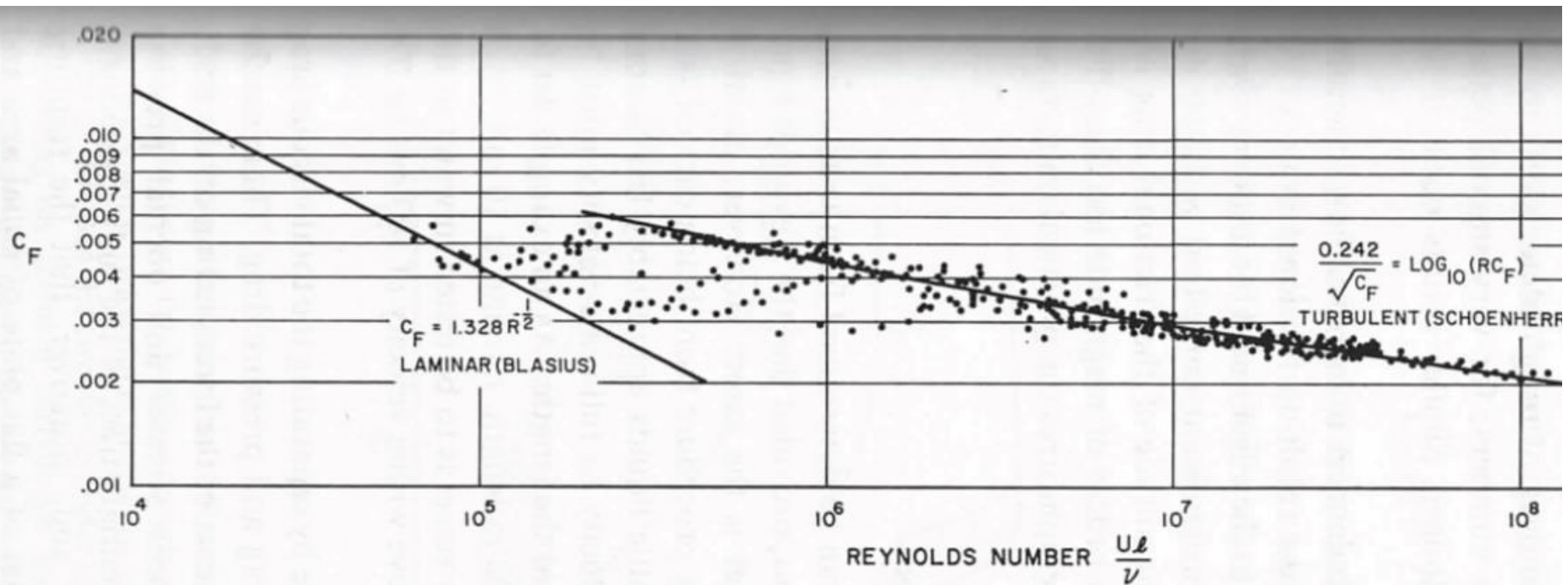


Force de trainée s'exerçant sur la paroi

Dimensionné : $F_x = \int_0^L \tau_p(x) dx = \int_0^L \mu \frac{\partial U_x}{\partial y}(x) dx \rightarrow \text{N/m ! (2D)}$

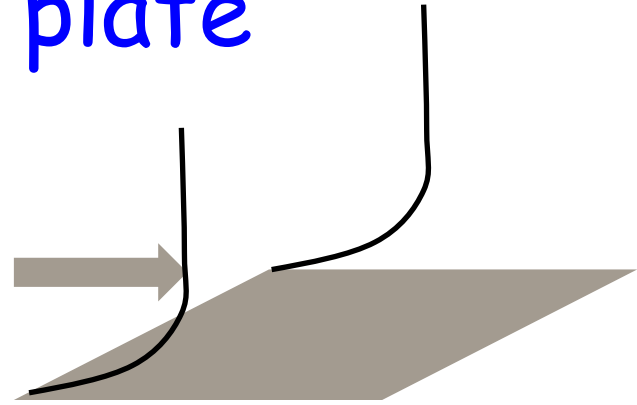
Adimensionné : $C_x = \frac{F_x}{\frac{1}{2} \rho_\infty U_\infty^2 L} \approx \frac{\sqrt{\rho_\infty \mu L U_\infty^3}}{\frac{1}{2} \rho_\infty U_\infty^2 L} \approx \sqrt{\frac{\mu}{\rho_\infty U_\infty L}} \approx \text{Re}^{-1/2}$

Drag of a plate



Drag of a plate

Plate of length L , width W



$$F_D \approx W \sqrt{\rho \mu L U_\infty^3}$$