

# Inverted pendulum

## System model

Consider the system depicted in Figure 1 composed by a pendulum fixed on a cart and controlled through a DC motor.

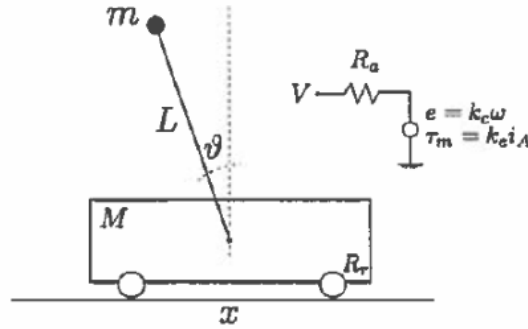


Figure 1: Cart-pendulum system.

The model parameters are given as  $M = 10 \text{ kg}$ ,  $m = 2 \text{ kg}$ ,  $L = 1 \text{ m}$ ,  $R_r = 0.1 \text{ m}$ ,  $R_a = 10 \Omega$ ,  $K_c = 2 \text{ V s/rad}$ ,  $K_e = 2 \text{ Nm/A}$ . The goal of this work is to stabilize the pendulum in its vertical position (around  $\vartheta = 0$ ) acting on the motor voltage  $v$ .

To derive the dynamical model the system can be decomposed as in Figure 2.

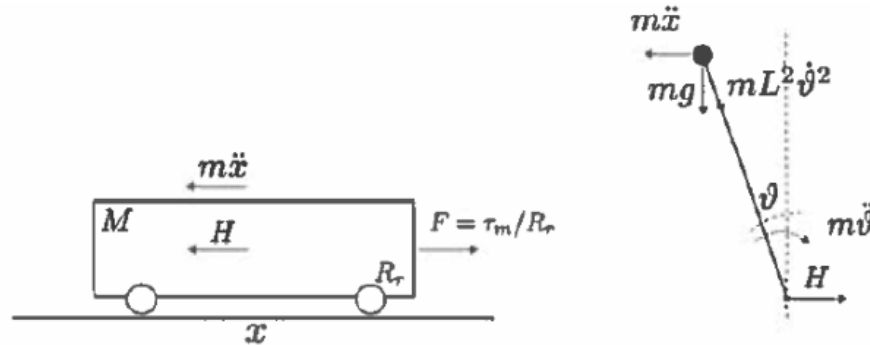


Figure 2: Cart-pendulum system.

- Cart:

$$M \ddot{x} + H = F$$

$$\tau_m = k_e i_a = k_e \frac{v - k_c \omega}{R_a} \quad , \quad \omega = \frac{\dot{x}}{R_r}$$

- Pendulum:

$$\begin{aligned} H &= m \ddot{x} + mL\ddot{\theta} \cos(\theta) - mL^2\dot{\theta}^2 \sin(\theta) \\ mL^2\ddot{\theta} &= +mgL\sin(\theta) - mL\ddot{x} \cos(\theta) \end{aligned}$$

Rearranging the terms to remove the force H we obtain

$$(M + m)\ddot{x} + mL\ddot{\theta} \cos(\theta) - mL^2\dot{\theta}^2 \sin(\theta) = \frac{\tau_m}{R_r}$$

$$m L^2\ddot{\theta} - mgL\sin(\theta) + mL\ddot{x} \cos(\theta) = 0$$

and substituting the terms  $\ddot{\theta}$  and  $\tau_m$  in the first equation we have the following model

$$\begin{cases} (M + m - m \cos^2(\theta))\ddot{x} = -mgL\sin(\theta) \cos(\theta) + mL^2\dot{x}^2 \sin(\theta) + \frac{k_e}{R_r R_a} v - \frac{k_e k_c}{R_r^2 R_a} x_2 \\ \ddot{\theta} = \frac{g}{L} \sin(\theta) - \frac{\ddot{x}}{L} \cos(\theta) \end{cases}$$

To use a state-space representation define the variables  $x_1 = x$ ,  $x_2 = \dot{x}$ ,  $x_3 = \theta$ ,  $x_4 = \dot{\theta}$

$$\begin{cases} \dot{x}_1 = x_2 \\ \dot{x}_2 = \frac{1}{M + m - m \cos^2(x_3)} \left( -mg \sin(x_3) \cos(x_3) + m L^2 x_2^2 \sin(x_3) + \frac{k_e}{R_r R_a} v - \frac{k_e k_c}{R_r^2 R_a} x_2 \right) \\ \dot{x}_3 = x_4 \\ \dot{x}_4 = \frac{g}{L} \sin(x_3) - \frac{\cos(x_3)}{L(M + m - m \cos^2(x_3))} \left( -mg \sin(x_3) \cos(x_3) + m L^2 x_2^2 \sin(x_3) + \frac{k_e}{R_r R_a} v - \frac{k_e k_c}{R_r^2 R_a} x_2 \right) \end{cases}$$

## Equilibrium and linearization

We are now interested in linearizing the system model around the vertical pendulum position with zero input, i.e.  $\bar{x} = (0,0,0,0)$  and  $\bar{v} = 0$ ; the resulting linearized model is the following

$$\begin{cases} \delta \dot{x}_1 = \delta x_2 \\ \delta \dot{x}_2 = -\frac{k_e k_c}{M R_r^2 R_a} \delta x_2 - \frac{gm}{M} \delta x_3 + \frac{k_e}{M R_a R_r} \delta v \\ \delta \dot{x}_3 = \delta x_4 \\ \delta \dot{x}_4 = -\frac{k_e k_c}{M R_r^2 R_a L} \delta x_2 + \left( \frac{g}{L} + \frac{gm}{M L} \right) \delta x_3 - \frac{k_e}{M R_a R_r L} \delta v \end{cases}$$

Substituting the parameters values the linearized system model can be written as

$$\delta \dot{x} = \begin{bmatrix} 0 & 1 & 0 & 0 \\ 0 & -4 & -1.9620 & 0 \\ 0 & 0 & 0 & 1 \\ 0 & 4 & 11.7720 & 0 \end{bmatrix} \delta x + \begin{bmatrix} 0 \\ 0.2 \\ 0 \\ -0.2 \end{bmatrix} \delta v$$

In matlab there are several ways of obtaining the linearized model without making all the computations by hand. **One possible solution is to construct a Simulink model of the system, put it at the equilibrium and use the ‘time-based linearization’ block.** You will do this in Exercise Session 6.