

# THERMODYNAMICS & ENERGETICS - II

What is thermodynamics?

Macroscopic framework of energy flows, and how they affect the properties of the system

— Macroscopic theory → no atomic scale information

— deals only with macroscopic variables

(eg) Volume ( $V$ )  
pressure ( $p$ )

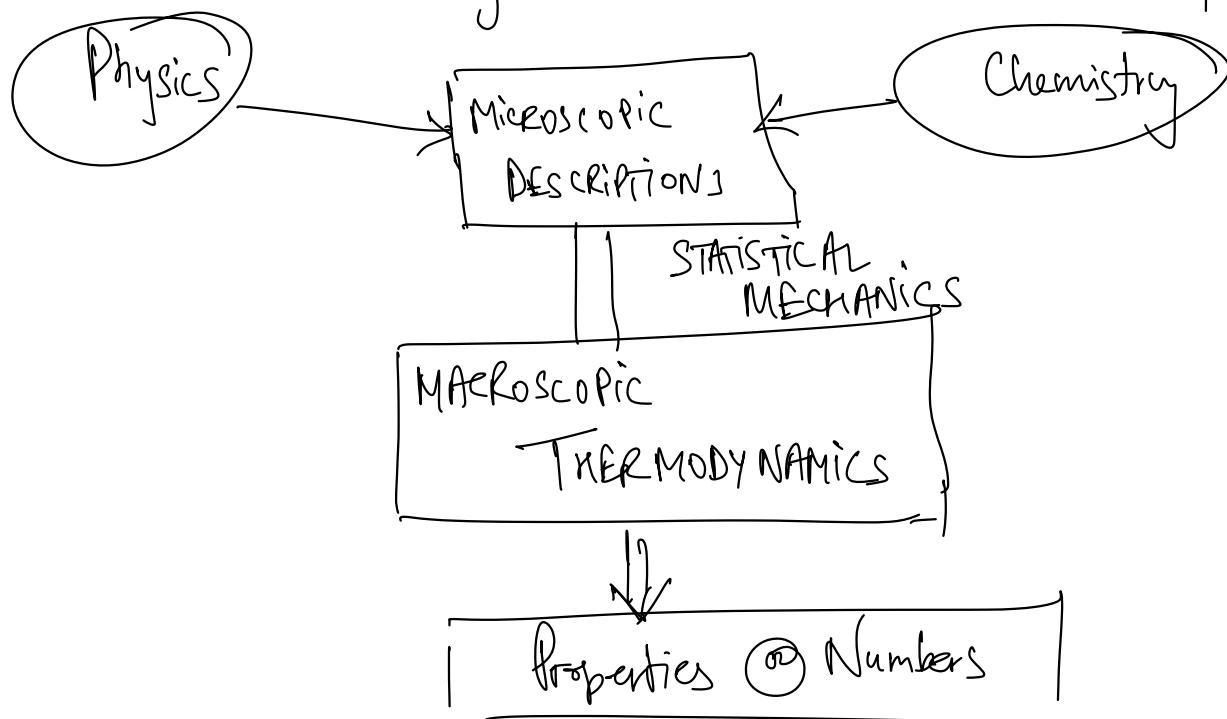
temperature ( $T$ ) ...

— Completely general framework that does not rely on atomic details.

— Thermodynamics is a "processor" @ "accountant"  
⇒ You have to put some information / physics into the framework to get other properties out

RELATION TO OTHER FIELDS:

Can derive thermodynamic relations from atomic / electronic



# Thermo & ITS RELATIONSHIP TO OTHER FIELDS

field	Key equation	Important variables
Mechanics	Newton (1643 - 1727)	$(m) \vec{v}, \vec{F}, \vec{x}$
Elasticity	Hooke's law (eqns. of elasticity) (1635-1703)	$\sigma, \epsilon, (E, G)$
Electricity & Magnetism	Maxwells equations (1831- 1879)	$\vec{E}, \vec{B}, \vec{M}, \vec{H}$ ( $\chi, \epsilon$ )
Thermal Sciences	Fourier Law (1768 - 1830)	$T, \vec{J}, (C_p, C_v)$

all different ways of storing energy in a system

Thermo COUPLES THESE Disciplines INTO ONE FRAMEWORK

- treating energy flows leads to the emergence of couplings between fields!

(eg) elasticity & EM  $\Rightarrow$  piezoelectricity.

Thermo needs subdisciplines to provide data & constitutive relations on materials behavior

(e.g) elasticity  $\epsilon = C \sigma$  Materials specific properties.

Magnetism  $M = \chi n$  tensor  
susceptibility.

## DEFINITIONS:

\* SYSTEM: Any collection of matter that can be uniquely specified (and over which macroscopic averages can be specified)

\* SURROUNDING / ENVIRONMENT: Universe - System

## VARIABLES:

- extensive: scale with the size of the system

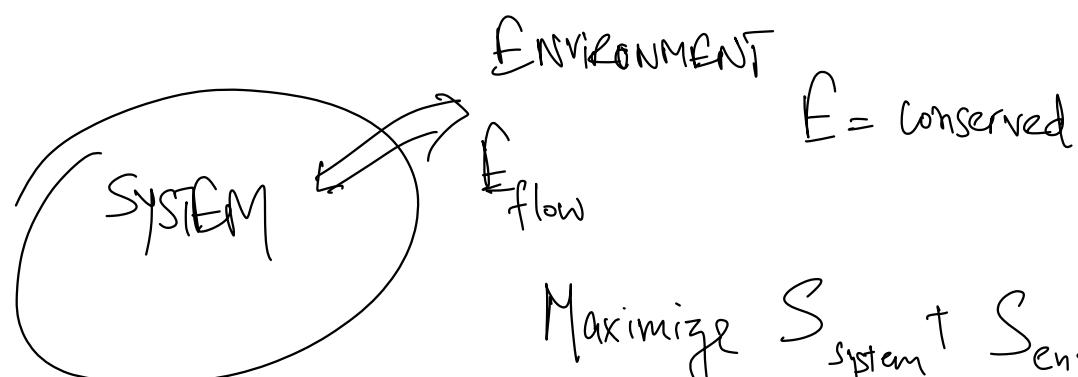
$$(V, N_i, U, \dots) \rightarrow \text{normalized extensive variables: } \frac{N_i}{V} = x_i$$

- intensive: do not scale with the size of the system

$$(T, p, \epsilon, \dots)$$

↓  
"densities"

## Thermo summarized



Maximize  $S_{\text{system}} + S_{\text{environment}}$

$$\underline{\text{Solve}} \quad dS_{\text{system}}(dE_{\text{system}}) + dS_{\text{environment}}(-dE_{\text{system}})$$

## STEPS TO SOLVING A THERMO PROBLEM

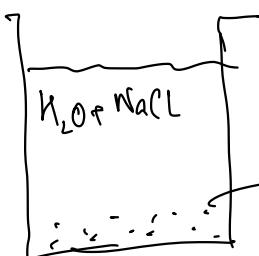
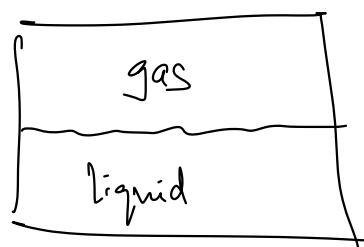
- ① Carefully define system
- ② Quantify the energy flows
- ③ Relate entropy to energy flows
- ④ Relate to properties

### \* BOUNDARIES

- permeable  $\rightarrow$  Open system  $N_i$  is allowed to change
- non-permeable  $\rightarrow$   $N_i$  is constant  $\rightarrow$  closed  $\xrightarrow{\quad}$  Open system
- Deformable  $\rightarrow$   $V$  can change
- Adiabatic  $\textcircled{a}$  Insulated  $\rightarrow$  no heat flow
- Diathermic  $\rightarrow$  perfect heat flow (System is at same temperature as environment)
- Rigid  $\rightarrow$   $V = \text{constant}$

# THERMODYNAMICS DEALS WITH EQUILIBRIUM STATES:

- \* **HOMOGENEOUS EQUILIBRIUM:** All bulk physical properties of the system are uniform throughout and do not change with time
- \* **HETEROGENEOUS EQUILIBRIUM:** Collection of equilibrium phases coexisting



undissolved  $NaCl$       The beaker contains "distinct portions" whose properties markedly differ from one another

(eg)  $\chi_{NaCl}$  in the liquid v/s  $\chi_{NaCl}$  in the solid

$$\chi_{\text{liquid}} \neq \chi_{\text{solid}}$$

Each "distinct portion"  $\longrightarrow$  PHASE

IN EQUILIBRIUM: system has well-defined extensive & intensive variables eg ( $T, V, P, \mu, \dots$ ) that uniquely characterize the STATE of the system

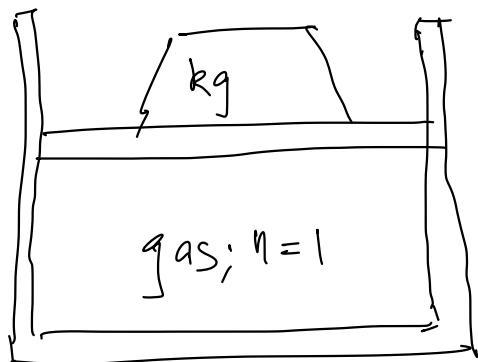
$\longrightarrow$  These variables are called STATE VARIABLES

State Variables are independent of how equilibrium is reached.

EXAMPLE: ideal gas: Equation of state  $\rightarrow$  thermo does not predict this

Must be derived from

- experiment
- physics  $\Rightarrow$  STAT. MECH



$$pV = nRT$$

fix  $p, T \rightarrow V$  is automatically determined in equilibrium

{

- does not depend on how the state was obtained
- fix 2-variables  $\rightarrow$  third is fixed by equilibrium

$V$  = state function ;  $V = f(T, p)$  [1 mole of gas]

$$\textcircled{w} \quad f(V, T, p) = 0$$

all equilibrium states for one mole of gas lie on this surface

CONSEQUENCE (Recall from calculus)

$$dV = \left( \frac{\partial V}{\partial p} \right)_T dp + \left( \frac{\partial V}{\partial T} \right)_p dT$$

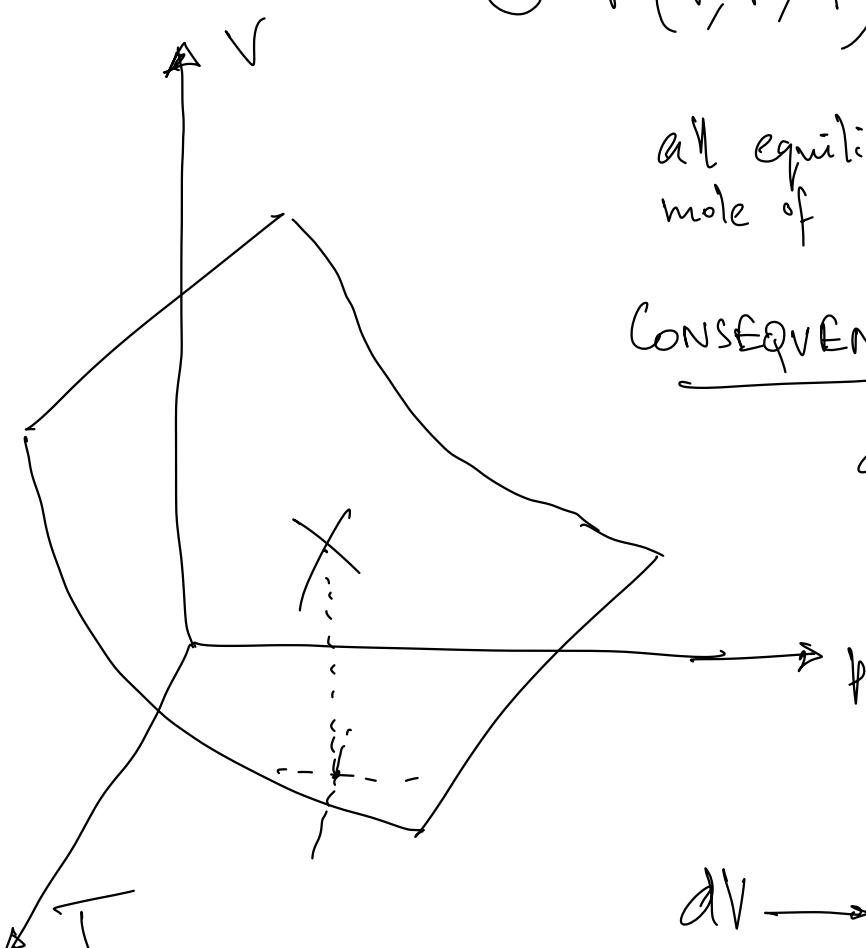
slope @

Constant  $T$

slope @

Const.  $p$

$dV \rightarrow$  total differential  
exact differential



## Measuring the equation of state

- ① Directly measure  $V(T, p)$  → Several experiments @  $(T_1, p_1), (T_2, p_2), \dots$
- ② Measure derived quantities:
  - \* Compressibility:  $\kappa = -\frac{1}{V} \left( \frac{\partial V}{\partial p} \right)_T$
  - \* thermal volumetric expansion:  $\beta = \frac{1}{V} \left( \frac{\partial V}{\partial T} \right)_p$
$$\Rightarrow \boxed{dV = -V \kappa dp + V \beta dT}$$

NOTE: Can do the same thing for  
 $p(V, T)$  or  $T(V, p)$

\* State Variables are determined by the allowed changes of the system

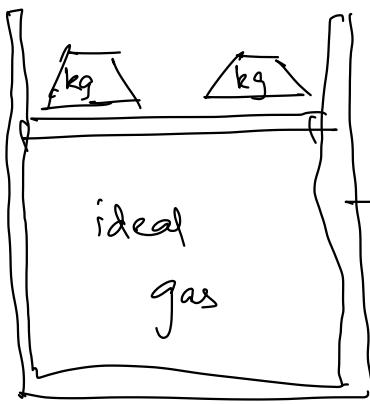
CHANGE OF STATE (process) - between stable and/or metastable states

- describe in terms of state variables
- $\left. \begin{array}{l} X \rightarrow \text{extensive } (V, N, \dots) \\ Y \rightarrow \text{intensive } (T, p, \mu, \dots) \end{array} \right\}$

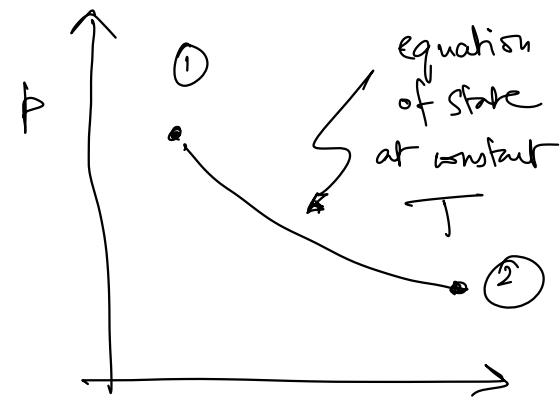
Two ways of changing the state of a system:

- \* perform work  $W$   $W > 0$  if work is performed on the system
- \* Exchange heat  $Q$   $Q > 0$  if heat is supplied to the system.

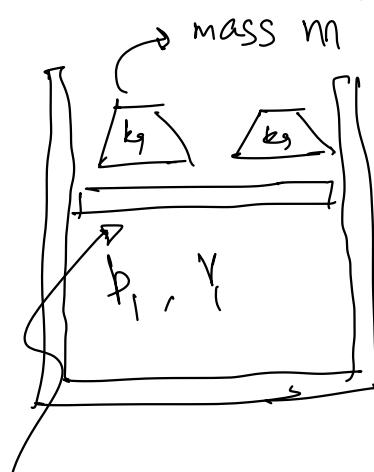
WORK



diathermal walls  
⇒ constant  $T$



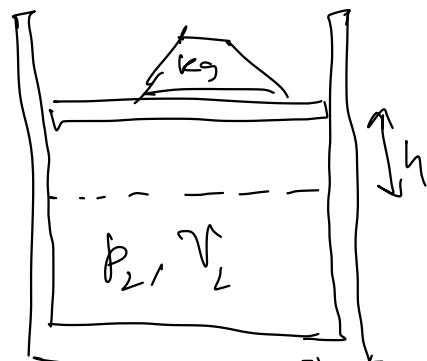
Change state from ① → ②



$$P_1 = \frac{mg}{A}$$

Cross-Section area

$A$

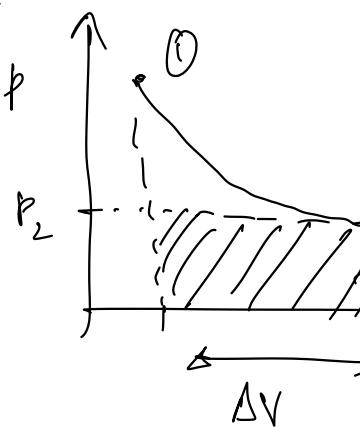


Work performed by the system to the environment

→ System lifted on weight of mass m by a height h

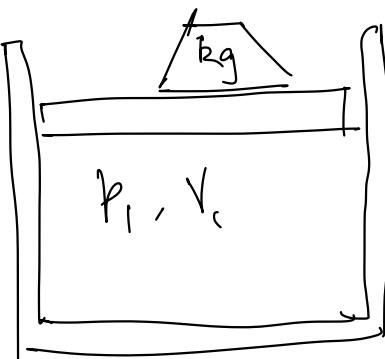
$$W = -mg \cdot h = -\frac{mg}{A} \cdot hA$$

$$W = -P_2 \cdot \Delta V$$

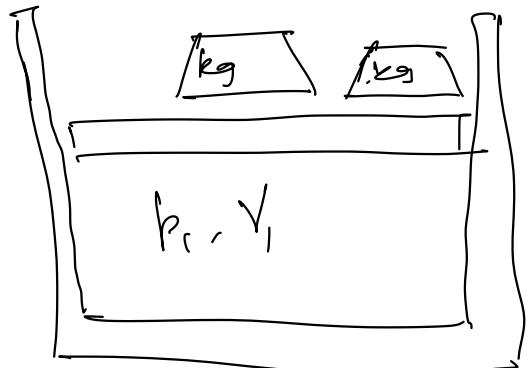


$$W_{1 \rightarrow 2} = -P_2 \Delta V$$

Go from ②  $\rightarrow$  ① :



②

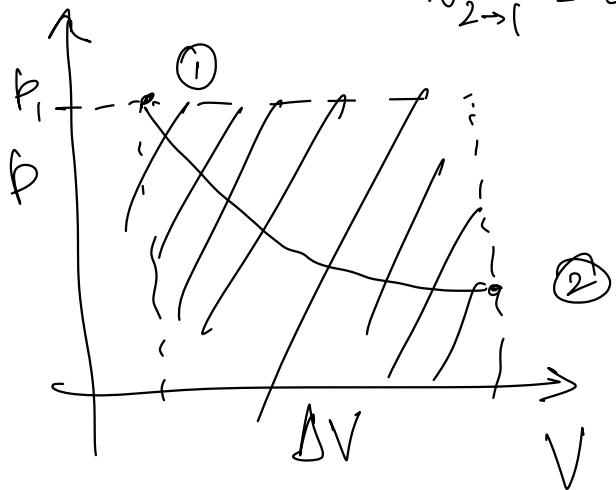


①

Work performed on the system

$\rightarrow$  two weights of mass  $m$  were dropped a height  $h$

$$W_{2 \rightarrow 1} = 2mg \cdot h = 2mg \cdot ht = p_1 \Delta V$$



$\rightarrow$  To achieve a change of state in reality we need either an overpressure or an underpressure  $\rightarrow$  for a real change of state

$$p_{\text{system}} \neq p_{\text{environment}}$$

$\rightarrow$  Work is different when reversing the process

REAL PROCESSES ARE IRREVERSIBLE

When is  $|W_{1 \rightarrow 2}| = |W_{2 \rightarrow 1}|$  ?

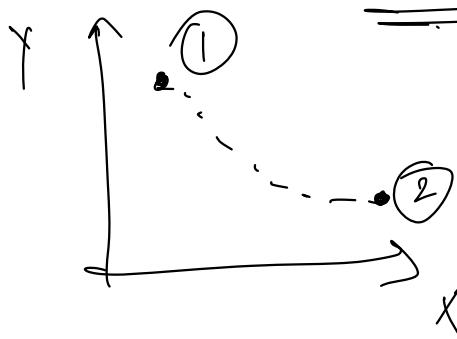
when  $p_{\text{system}} = p_{\text{env}}$  at all stages

Idealized  
REVERSIBLE  
change of state

# THREE TYPES OF PROCESSES

(I)

Discontinuous IRREVERSIBLE



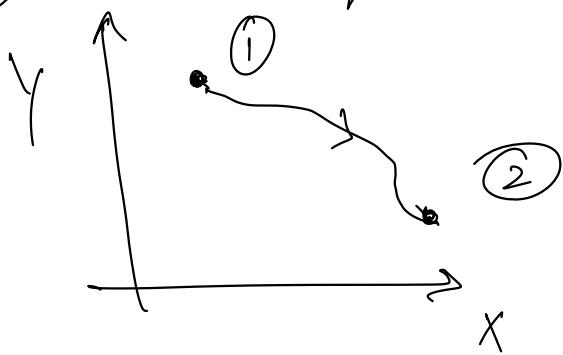
- $X, Y$  cannot be measured along path

- change is fast & turbulent

(eg)  $H_2, O_2$  combustion

(II)

Continuous quasi-static IRREVERSIBLE



- process is slow enough that  $X$  &  $Y$  can be specified along path

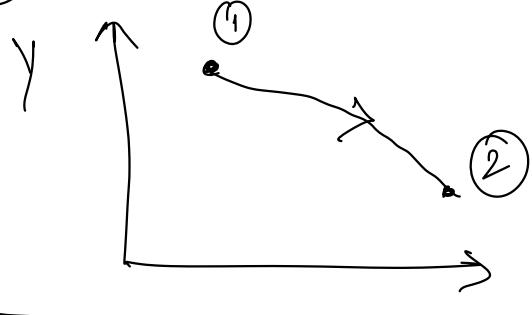
- System & environment are not in equilibrium

- make small changes to  $X$  &  $Y$  and allow to equilibrate

(eg) expansion of a gas  $p_{env} < p_{sys}$

(III)

Continuous quasi-static REVERSIBLE



- same as II but the system is always in equilibrium with the surroundings

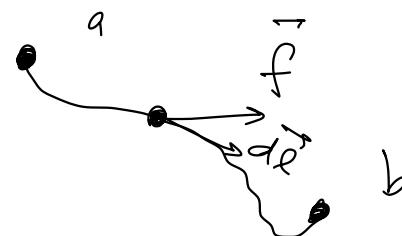
- can retrace steps restoring both system & environment to its original state

PURELY  
MATHEMATICAL

Change of state (process) through exchange of work and heat between system & environment

WORK: energy transfer from "displacement" under a "force"

\* classical mechanics



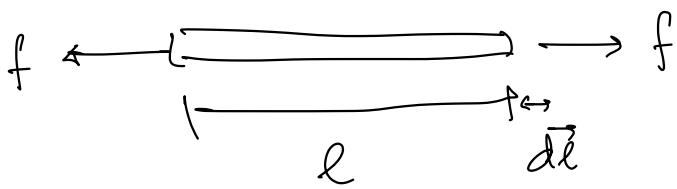
$$\delta W = \vec{f} \cdot \vec{dl}$$

$$\Delta W = \int_a^b \vec{f} \cdot \vec{dl}$$

$\delta W > 0$  if work is done on the system

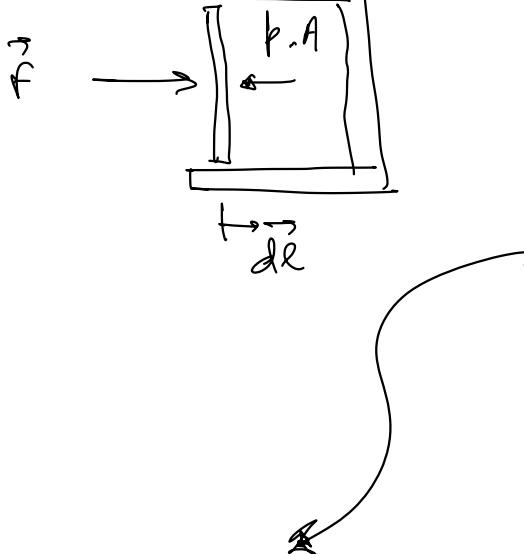
[path dependent]

elongation of a rod



$$\delta W = \vec{f} \cdot \vec{dl}$$

piston work



$$\delta W = \vec{F} \cdot \vec{dl}$$

Assume mechanical equilibrium

$$\Rightarrow F = pA$$

$$\delta W = pA \underbrace{dl}_{-dV} = -pdV$$

ONLY TRUE IF IN MECHANICAL EQUILIBRIUM

$\Rightarrow$  ONLY FOR REVERSIBLE

PROCESSES

Work is expressed in terms of state variables of the system

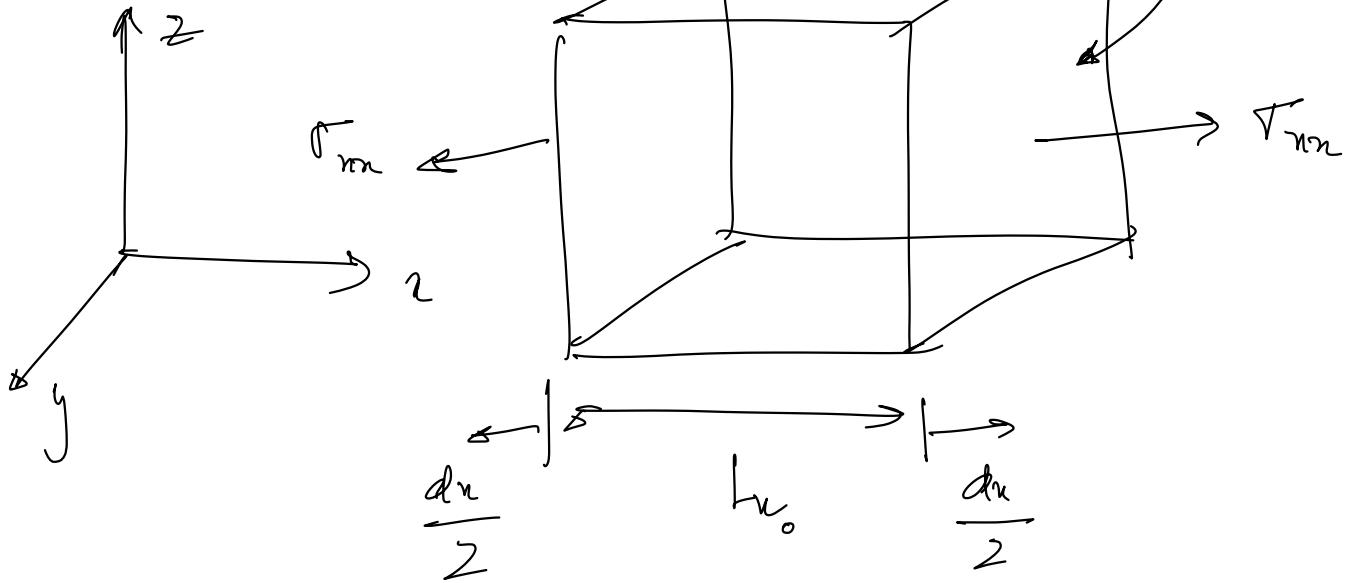
NOTE: ① quasi-static irreversible compression

$f > p \cdot A \rightarrow$  perform more work than the reversible case

② quasi-static irreversible expansion

$f < p \cdot A \rightarrow$  get less work out of the system

Strain Work



$$\delta W = \int F_{xx} A_0 \frac{dx}{2} = F_{xx} A_0 L_{x_0} \frac{dx}{L_{x_0}} = V_0 F_{xx} \frac{d\epsilon_{xx}}{V_0}$$

$$= V_0 F_{xx} d\epsilon_{xx}$$

In general : 
$$\boxed{\delta W = \sum_i \sum_j V_0 F_{ij} d\epsilon_{ij}}$$

Special case :

hydrostatic pressure

$$\left. \begin{aligned} F_{xx} &= F_{yy} = F_{zz} = -p \\ F_{xy} &= F_{yz} = F_{zx} = 0 \end{aligned} \right]$$

$$\boxed{\delta W = -pdV}$$

$V = V_0 (1 + \epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz})$   
Assuming infinitesimal strains

$$\delta W = -V_0 p d(\epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz})$$

$$\left. \begin{aligned} dV &= V_0 d(\epsilon_{xx} + \epsilon_{yy} + \epsilon_{zz}) \end{aligned} \right]$$