

# Chapter 4: Propellers & Vortex

ME-342 Introduction to  
turbomachinery

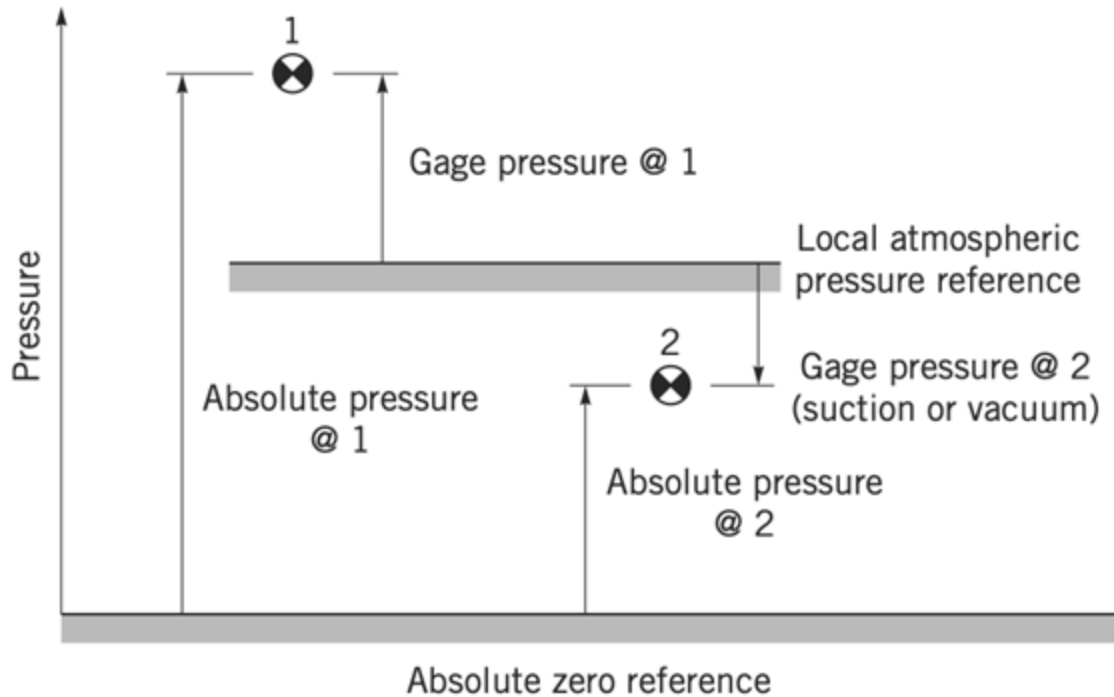
Prof. Eunok Yim, HEAD-lab.

# Questions from last exercise session

- Pressure coefficient,  $C_p$

Pressure coefficient	$C_p = \frac{p - p_\infty}{\frac{1}{2}\rho U^2}$	$\frac{\text{Static pressure}}{\text{Dynamic pressure}}$	Aerodynamics, hydrodynamics
Lift coefficient	$C_L = \frac{L}{\frac{1}{2}\rho U^2 A}$	$\frac{\text{Lift force}}{\text{Dynamic force}}$	Aerodynamics, hydrodynamics
Drag coefficient	$C_D = \frac{D}{\frac{1}{2}\rho U^2 A}$	$\frac{\text{Drag force}}{\text{Dynamic force}}$	Aerodynamics, hydrodynamics

The pressure coefficient is a dimensionless form of the pressure



- Ideal gas law → absolute pressure

$$\rho = \frac{p}{RT}$$

The pressure in the ideal gas law must be expressed as an **absolute pressure**, denoted (abs), which means that it is measured relative to absolute zero pressure (a pressure that would only occur in a perfect vacuum).

$$P_{abs} = P_{atm} + P_{gage}$$

The pressure at a point within a fluid mass is designated as either an **absolute pressure** or a **gage pressure**.

measured relative to a perfect vacuum (absolute zero pressure)

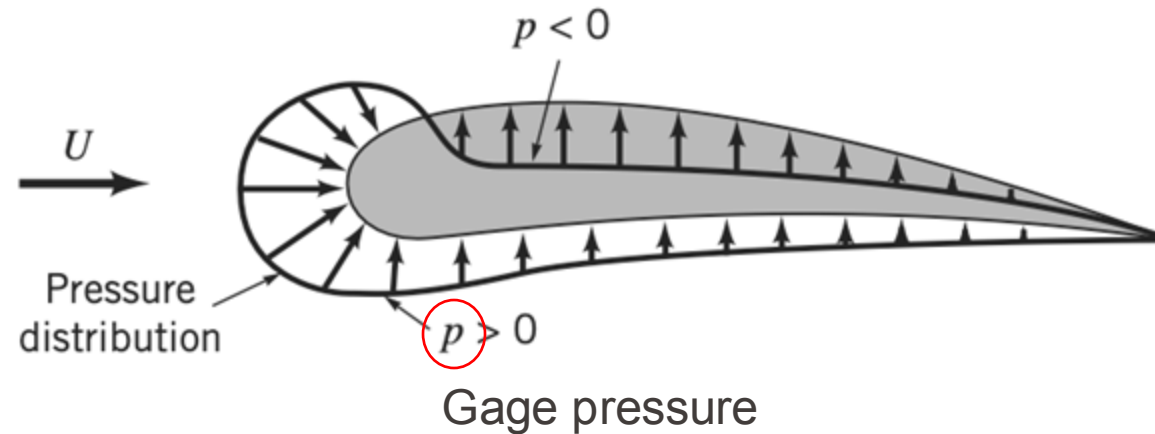
measured relative to the local atmospheric pressure

- Absolute pressures are always positive
- Gage pressure can be positive (above atm) or negative (below atm)
- Zero gage pressure: equal to the local atmospheric pressure
- Negative gage pressure is also referred to as a **suction** or **vacuum pressure**

# When flow is involved...

- **Static Pressure**

- Exerted by a fluid when it is at rest or when there is no directional motion effect at a given point
- Measured perpendicular to the fluid flow in motion
- Can be expressed as absolute pressure or gage pressure



- **Absolute static pressure = Static pressure + Atmospheric pressure**

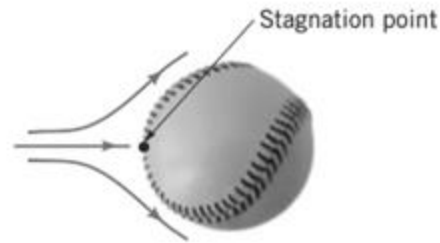
Total absolute pressure = Static (absolute) pressure + Dynamic pressure

- If static pressure is measured **relative to atmospheric pressure**, then it is the same as **gage** pressure.
- If static pressure is measured **relative to a vacuum**, then it is **absolute** pressure.

# Stagnation, total pressure

If elevation effects are neglected, the **stagnation pressure**,  $p + \rho V^2/2$  is the **largest pressure** obtainable along a given streamline.

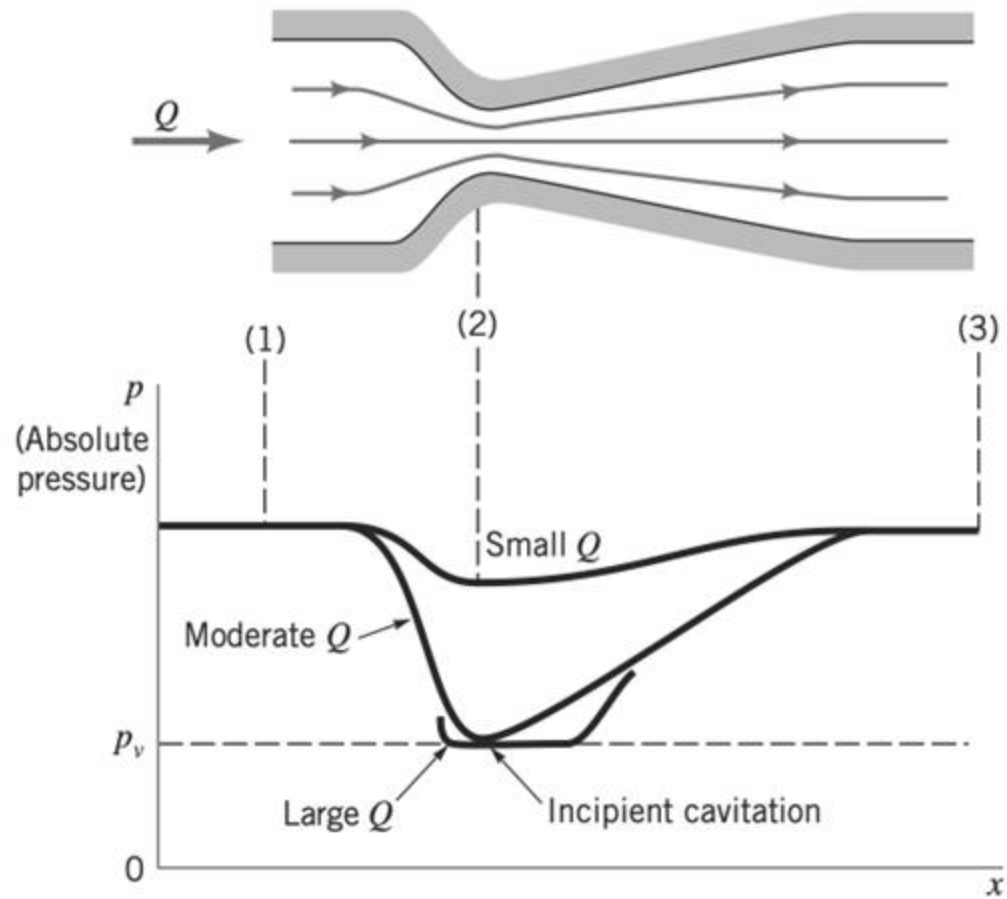
→ conversion of all of the kinetic energy into a pressure rise



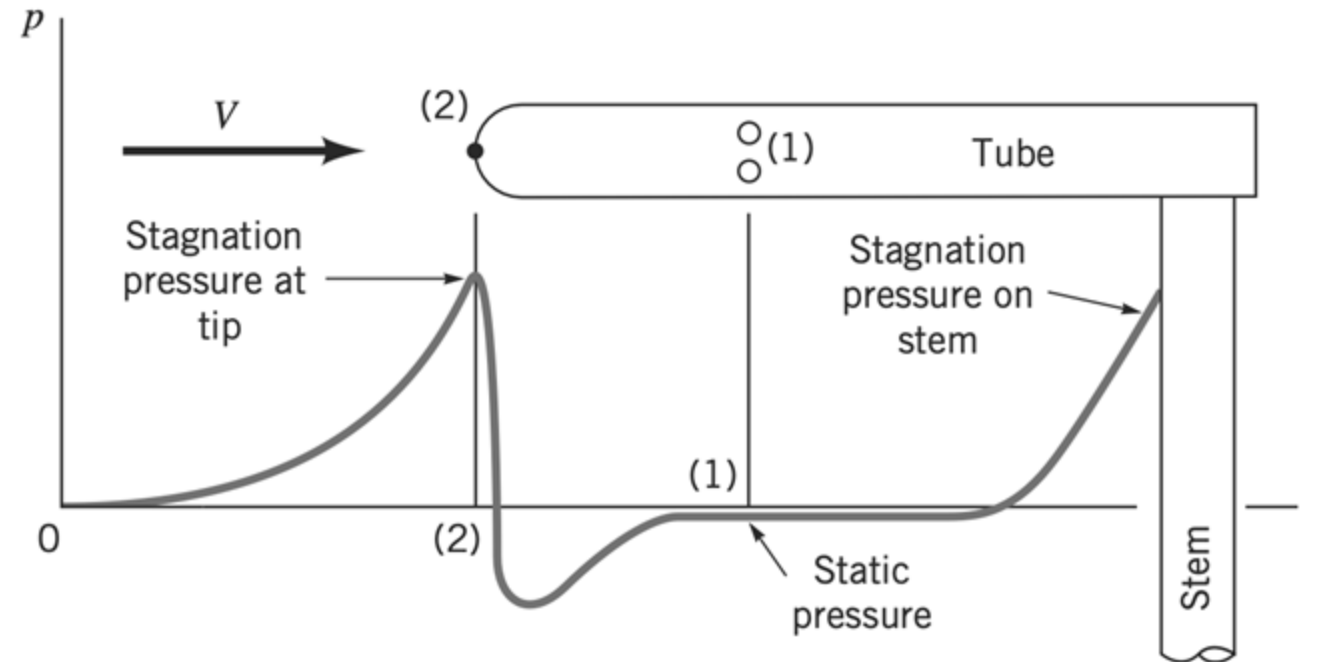
$$p + \frac{1}{2}\rho V^2 + \gamma z = p_T = \text{constant along a streamline}$$

**Total pressure,  $p_T$ :** sum of the static pressure, hydrostatic pressure, and dynamic pressure

- Venturi tube



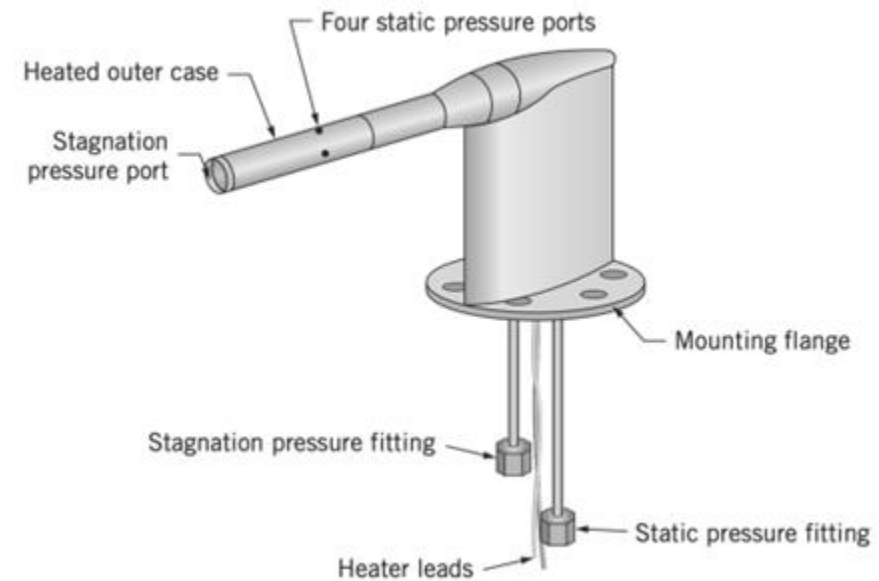
- Pitot tube



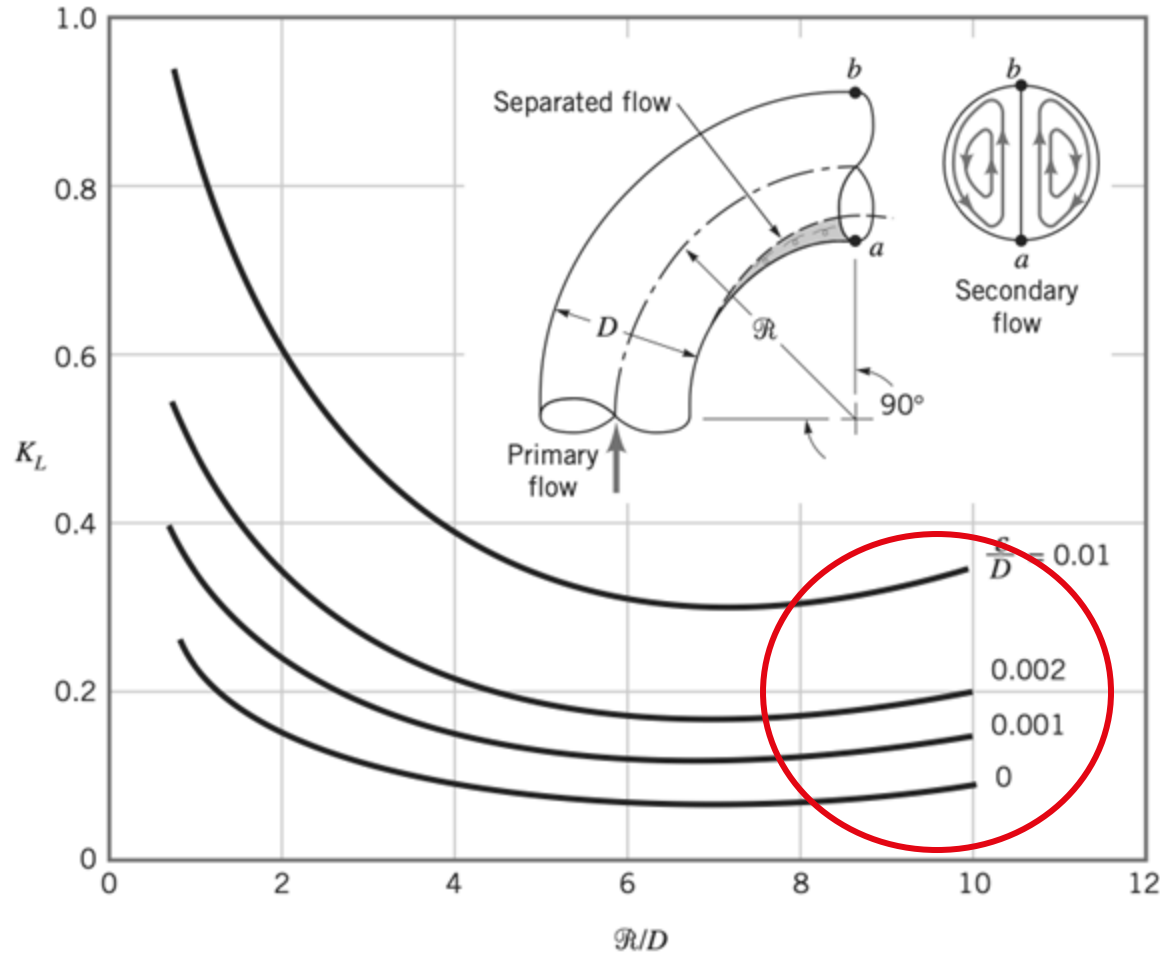
$$p_2 + \frac{1}{2}\rho V_2^2 = p_1$$



# Do you remember?



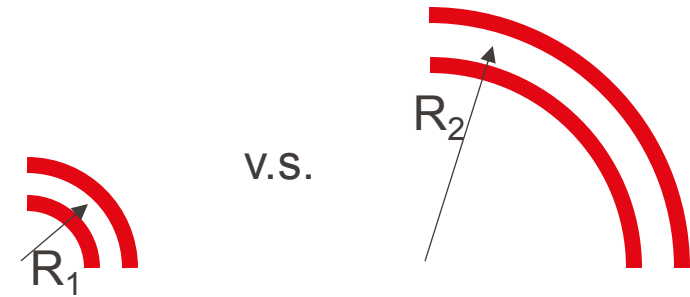
# Why does the loss coefficient increase slightly?



$$h_{L \text{ major}} = f \frac{\ell}{D} \frac{V^2}{2g}$$

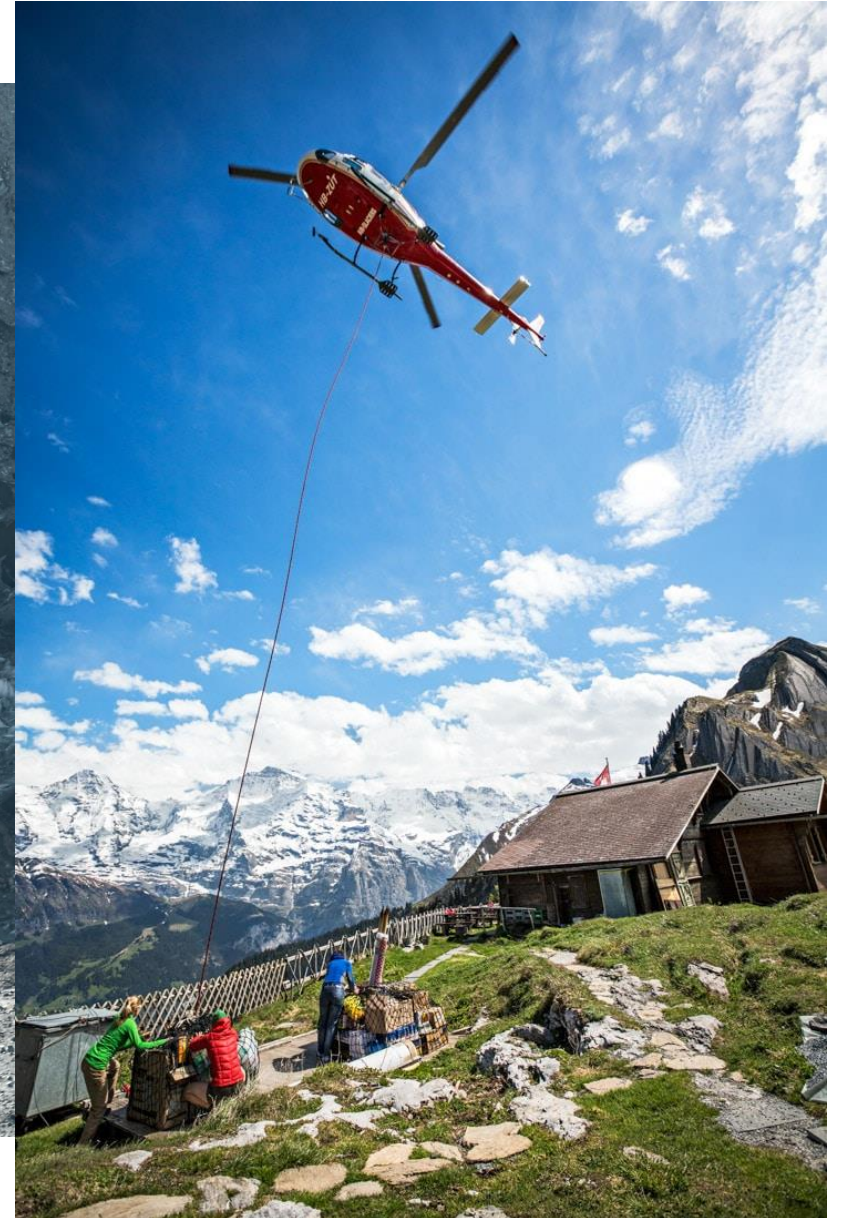
$$h_{L \text{ minor}} = K_L \frac{V^2}{2g}$$

- Increased flow path
  - for a given  $D$ , the flow path is increased as  $R$
  - Velocity profile difference between inner and outer bend is larger  $\rightarrow$  secondary flow separation





# One example turbomachinery, which is directly linked to the lift and drag of airfoils?



# Propellers (Helicopter)



# 1-Continuity equation (Mass conservation)

Time rate of change of the system mass = 0 :

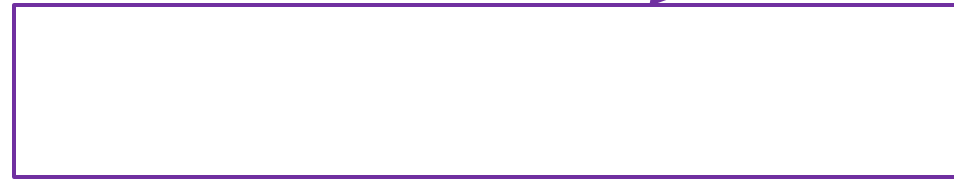
$$\frac{DM_{\text{sys}}}{Dt} = 0$$

The system mass,  $M_{\text{sys}}$  is

$$M_{\text{sys}} = \int_{\text{sys}} \rho dV$$

Reynolds  
transport  
theory

- Continuity equation



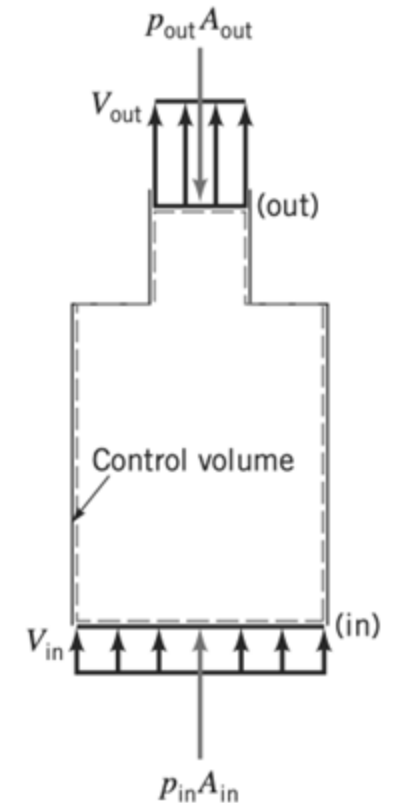
- Mass flow rate  $\dot{m} = \rho Q = \rho AV$



$$B = mb$$

$$\frac{DB_{\text{sys}}}{Dt} = \frac{\partial B_{\text{cv}}}{\partial t} + \dot{B}_{\text{out}} - \dot{B}_{\text{in}}$$

$$\frac{DB_{\text{sys}}}{Dt} = \frac{\partial B_{\text{cv}}}{\partial t} + \rho_2 A_2 V_2 b_2 - \rho_1 A_1 V_1 b_1$$

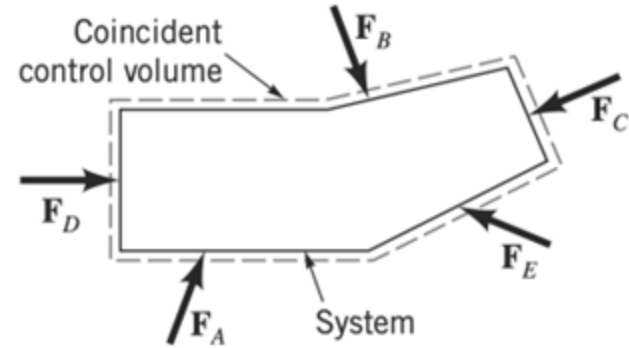


- cs: control **surface**
- cv: control **volume**

# 2-Linear momentum equation

Newton's second law of motion

The change of **motion** of an object is **proportional** to the **force** impressed; and is made in the direction of the straight line in which the force is impressed



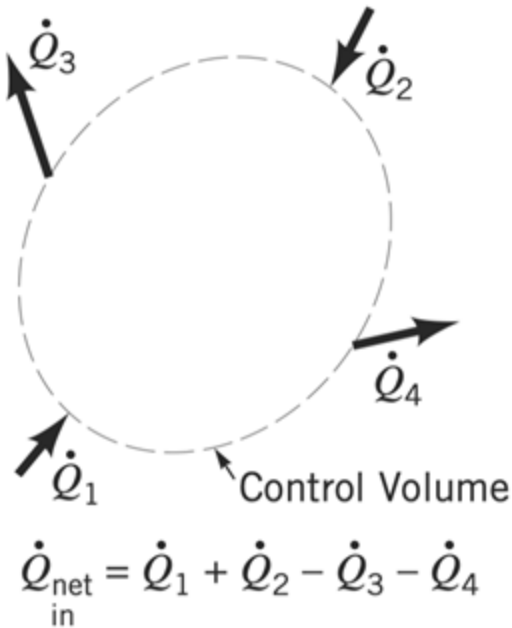
- Linear momentum equation

$$\frac{\partial}{\partial t} \int_{cv} \mathbf{V} \rho dV + \int_{cs} \mathbf{V} \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA = \sum \mathbf{F}_{\text{contents of the control volume}}$$



# 3-Energy equation

- cs: control **surface**
- cv: control **volume**



The first law of thermodynamics

$$\frac{D}{Dt} \int_{\text{sys}} e \rho dV = \frac{\partial}{\partial t} \int_{\text{cv}} e \rho dV + \int_{\text{cs}} e \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA$$

Time rate  
of increase  
of the total  
stored energy  
of the system

=  
time rate of increase  
of the total stored  
energy of the contents  
of the control volume

+  
net rate of flow  
of the total stored energy  
out of the control  
volume through the  
control surface

where  $e = \underbrace{\check{u}}_{\text{internal energy}} + \frac{V^2}{2} + gz$  total stored energy per unit mass

$$\dot{Q}_{\text{net in}} + \dot{W}_{\text{shaft net in}} = \frac{\partial}{\partial t} \int_{\text{cv}} e \rho dV + \int_{\text{cs}} \left( \check{u} + \frac{p}{\rho} + \frac{V^2}{2} + gz \right) \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA$$

Heat  
transfer ratio

Work transfer  
rate, power

# Three governing equations

- Continuity equation

$$\frac{\partial}{\partial t} \int_{\text{cv}} \rho dV + \int_{\text{cs}} \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA = 0$$

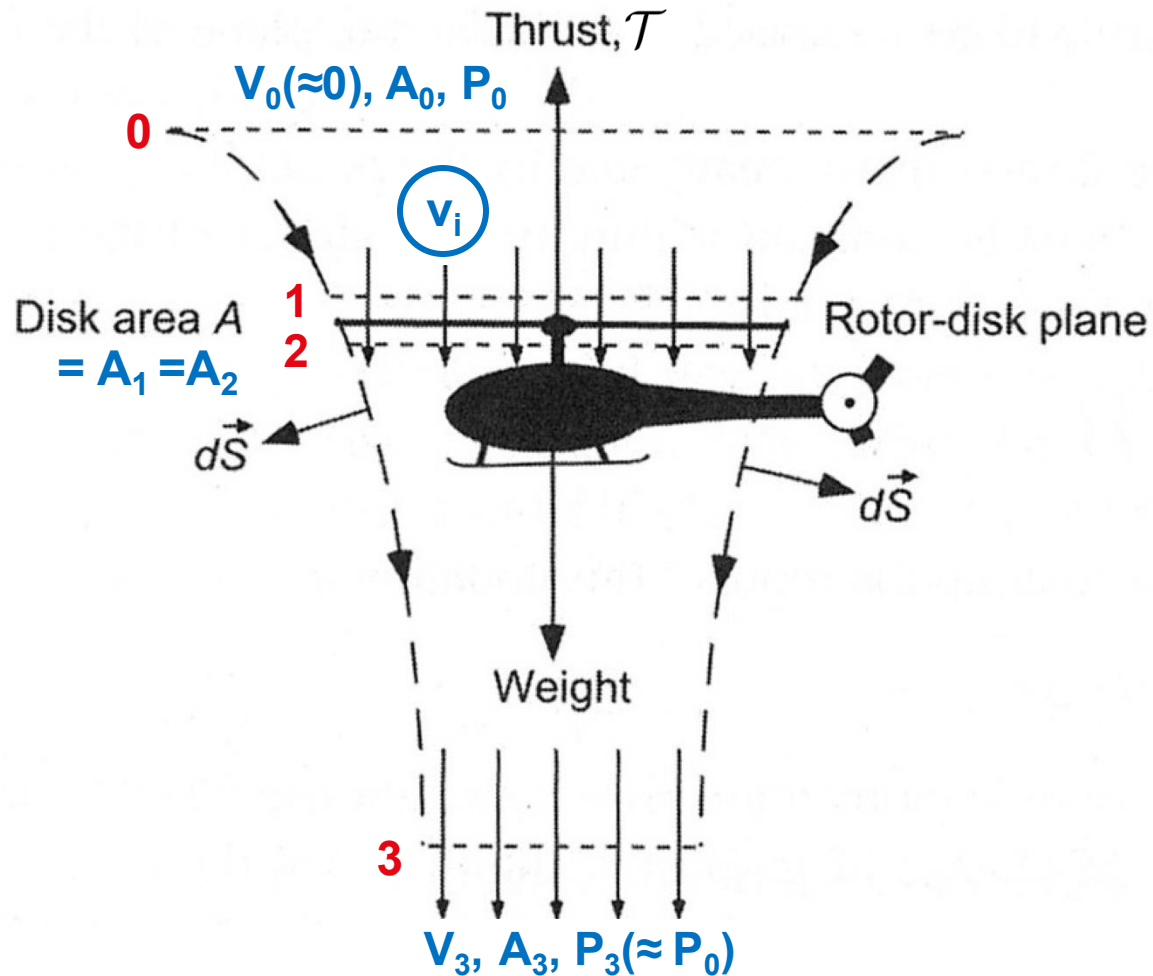
- Linear momentum equation

$$\frac{\partial}{\partial t} \int_{\text{cv}} \mathbf{V} \rho dV + \int_{\text{cs}} \mathbf{V} \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA = \sum \mathbf{F}_{\text{contents of the control volume}}$$

- Energy equation

$$\dot{Q}_{\text{net in}} + \dot{W}_{\text{shaft net in}} = \frac{\partial}{\partial t} \int_{\text{cv}} e \rho dV + \int_{\text{cs}} \left( \check{u} + \frac{p}{\rho} + \frac{V^2}{2} + gz \right) \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA$$

# Helicopters- conservation law for hovering rotor



$v_i$  : induced air velocity

$V_3$  : far wake velocity

- Continuity & linear momentum equations:

$$\frac{\partial}{\partial t} \int_{cv} \rho dV + \int_{cs} \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA = 0$$

$d\vec{S}$

$$\frac{\partial}{\partial t} \int_{cv} \mathbf{V} \rho dV + \int_{cs} \mathbf{V} \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA = \sum \mathbf{F}_{\text{contents of the control volume}}$$

Steady-state approximation

$$\dot{m} = \int_A \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA$$

Thrust,  $T$  (reaction force)

$$\mathcal{T} = \dot{m} V_3$$

- Rotor power** (work done by rotor per unit time)

$$P_{\text{rotor}} = \mathcal{T} v_i$$

# Helicopters- conservation law for hovering rotor

- Energy equation (the gain of energy of the fluids per unit time)

adiabatic

steady

No internal energy change

$$\cancel{\dot{Q}_{\text{net in}}} + \dot{W}_{\text{shaft net in}} = \cancel{\frac{\partial}{\partial t} \int_{\text{cv}} e \rho dV} + \int_{\text{cs}} \left( \cancel{\check{u}} + \frac{p}{\rho} + \frac{V^2}{2} + gz \right) \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA$$

$d\vec{S}$

Mass flow rate

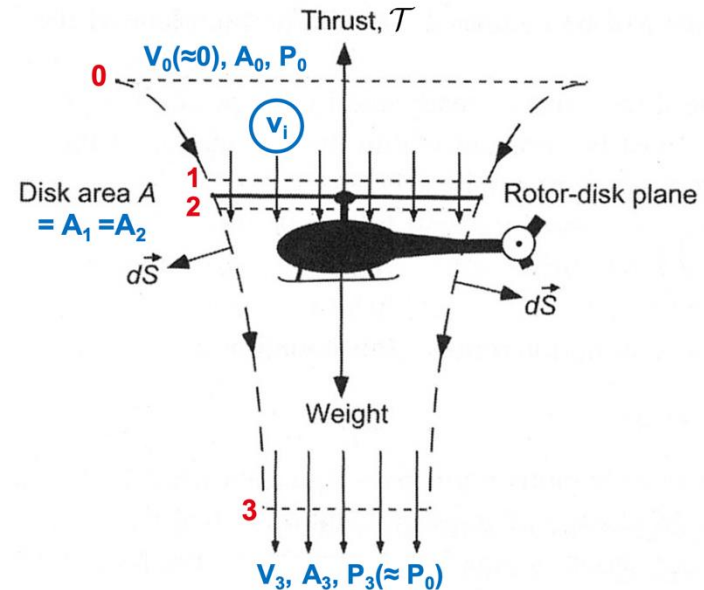
$$\dot{m} = \int_A \rho \mathbf{V} \cdot \hat{\mathbf{n}} dA$$

$$\dot{W}_{\text{shaft net in}} = P_{\text{rotor}} =$$

=

Potential energy change  $\ll$  kinetic energy

$$= \frac{V_3^2}{2} \dot{m}$$





- Continuity equation:

$$\mathcal{T} = \dot{m}V_3, \quad P_{\text{rotor}} = \mathcal{T}v_i$$

- Energy equation:  $P_{\text{rotor}} = \frac{V_3^2}{2}\dot{m}$



Induced velocity:

Far field velocity is twice of the induced velocity

- Mass flowrate:  $\dot{m} = \rho A_3 V_3 = \rho A_2 v_i = \rho A v_i$

$$\mathcal{T} = \dot{m}V_3 = \dot{m}(2v_i) = (\rho A v_i)(2v_i) = 2\rho A v_i^2$$

The induced velocity:

$$v_h \equiv v_i = \sqrt{\frac{\mathcal{T}}{2\rho A}} = \sqrt{\left(\frac{\mathcal{T}}{A}\right) \left(\frac{1}{2\rho}\right)}$$

Disk loading

# Helicopters- Figure of merit (FM)

$$P_{\text{rotor}} = \frac{V_3^2}{2} \dot{m} = \dot{m} V_3 v_i = 2 \dot{m} v_i^2$$

$$\dot{m} = \rho A v_i$$

$$v_i = \sqrt{\frac{\mathcal{T}}{2\rho A}} = \sqrt{\left(\frac{\mathcal{T}}{A}\right) \left(\frac{1}{2\rho}\right)}$$

$$P_{\text{rotor}} = \frac{\mathcal{T}^{3/2}}{\sqrt{2\rho A}} \quad \text{Ideal rotor power}$$

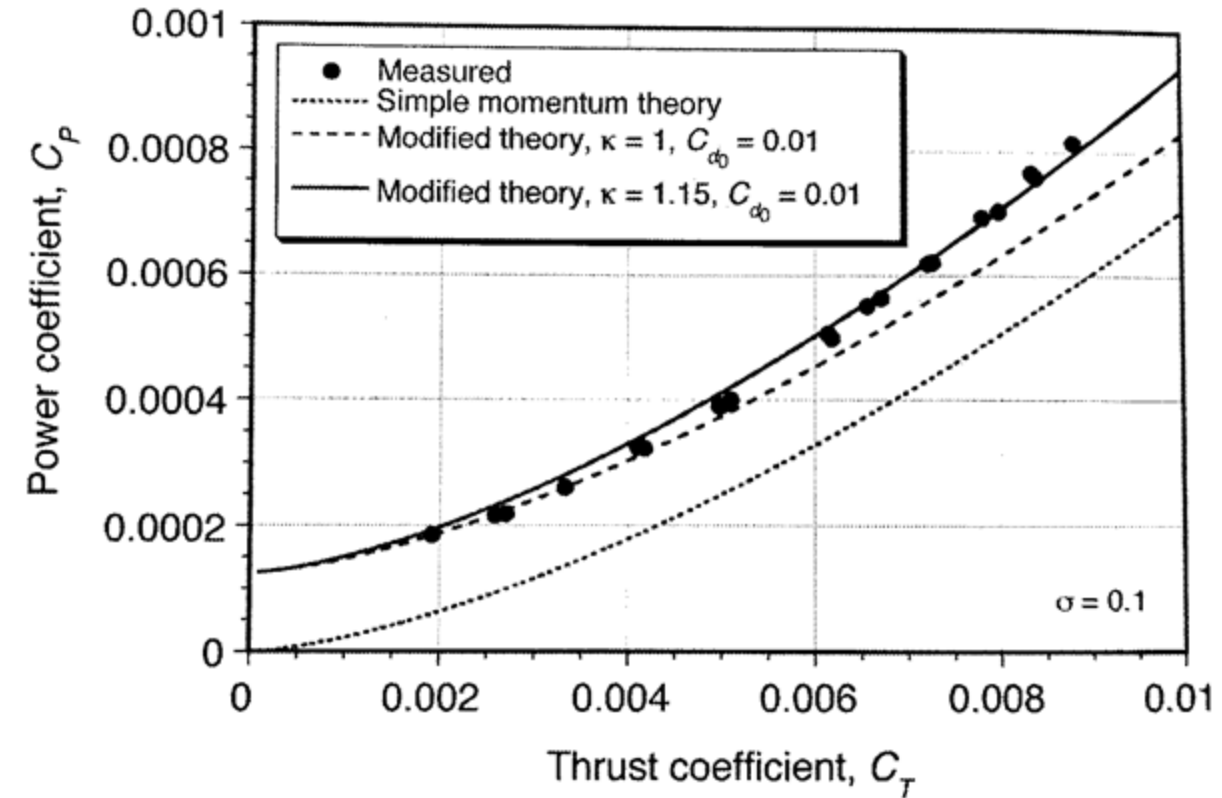
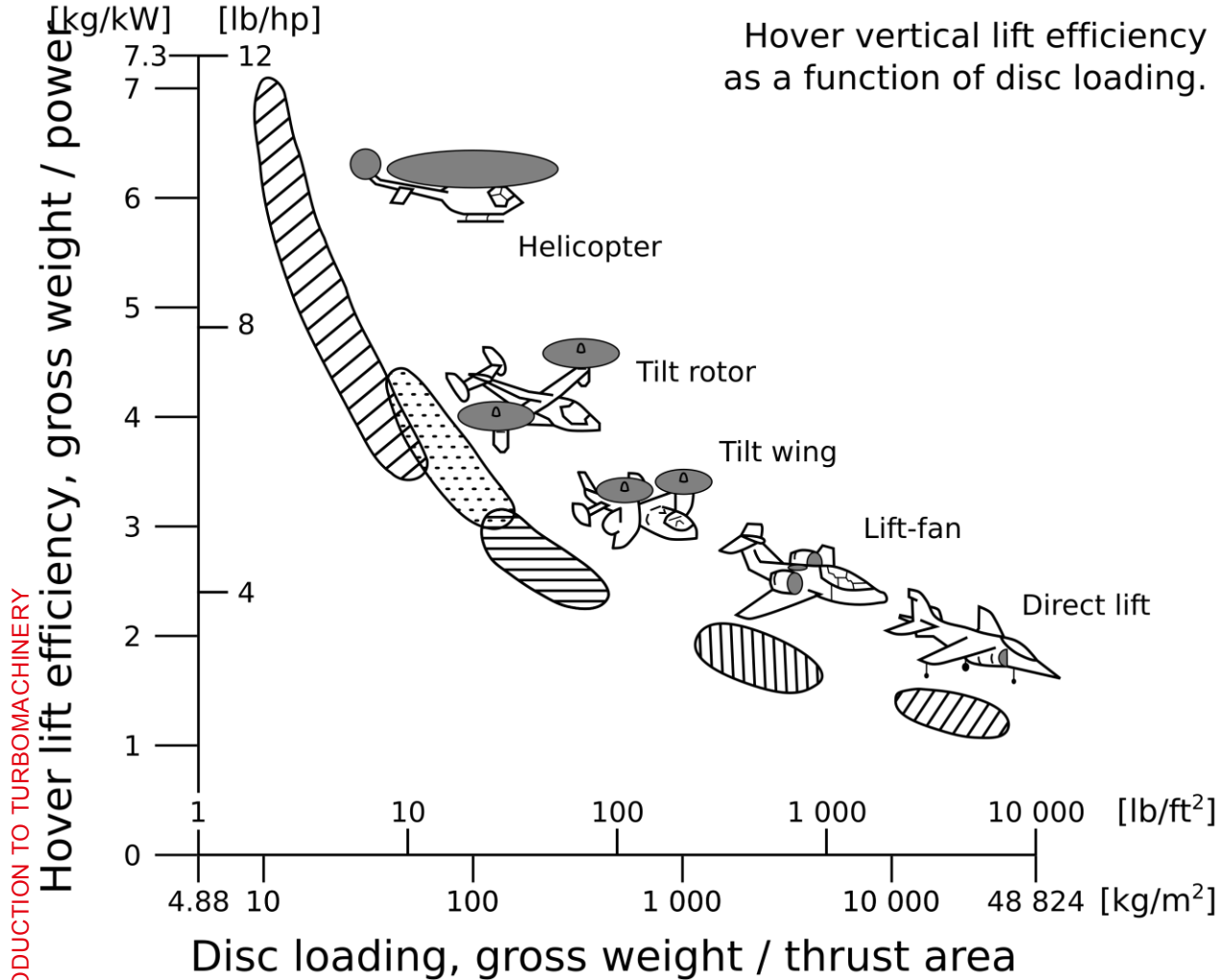
$$\text{Figure of merit, FM} = \frac{\text{Ideal rotor power}}{\text{Actual rotor power}}$$

If FM is 0.7 and the computed ideal power is 2000 kW, what is the actual power required?

# Hovering efficiency vs. disk loading

Ideal power:

$$P_{\text{rotor}} = \frac{\mathcal{T}^{3/2}}{\sqrt{2\rho A}}$$

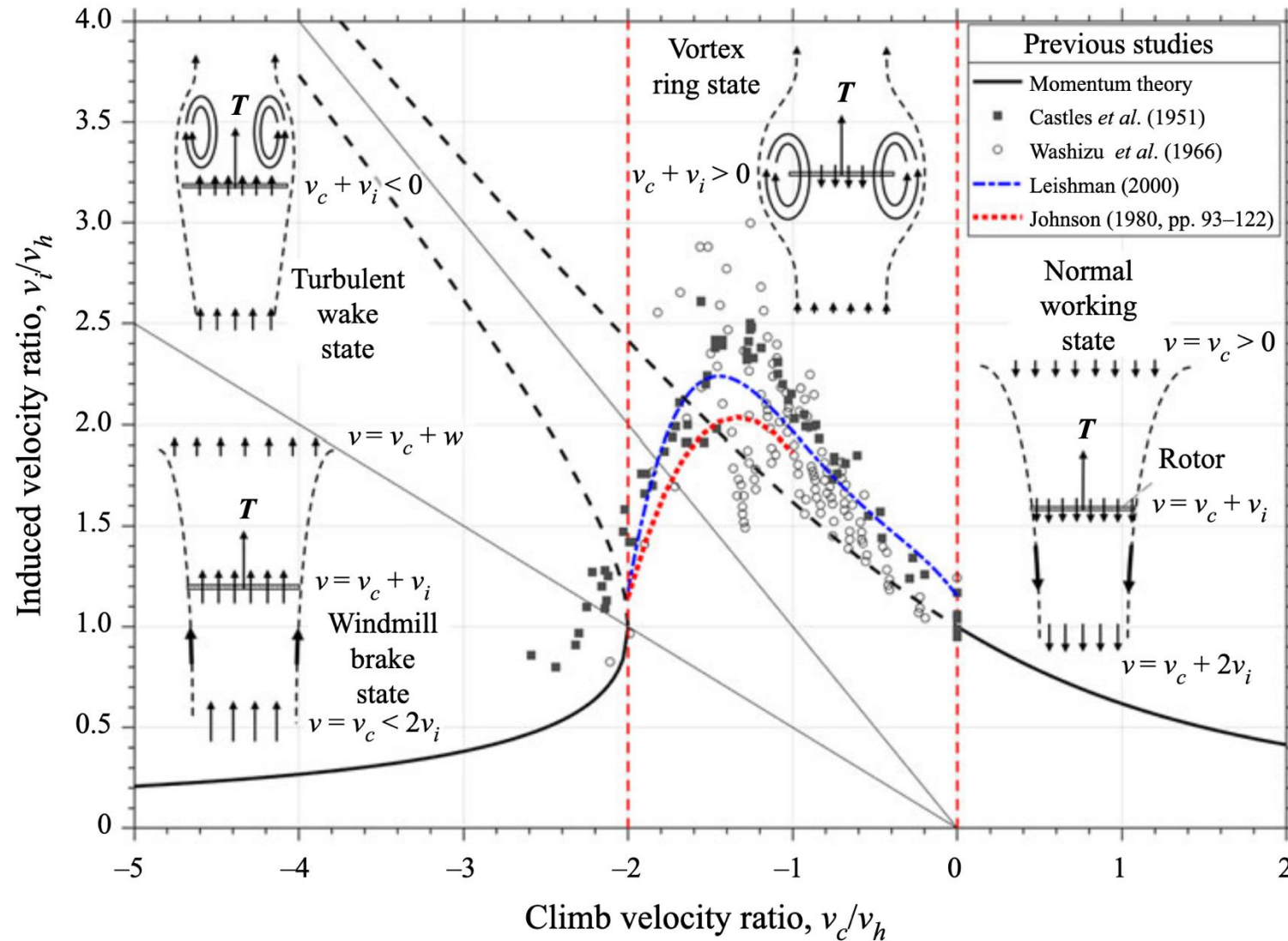


\*Note, here  $C_p$  is not the pressure coefficient ( $C_p$ )

$$\frac{\mathcal{T}}{A}$$

# Real helicopter dynamics are much more complex..

Vortex ring state





A helicopter is helping to the construction of a mountain cabane.

- (1) Estimate the power required for the helicopter to hover.
  - (2) What is the amount of fuel required for 10 min hover? (ignore mass change)
- Gross mass of helicopter 20 tonnes
  - Rotor diameter 12 m
  - Density of air 1.2 kg/m<sup>3</sup>
  - Figure of merit of the rotor 0.75
  - Fuel for helicopter (BP Avgas 80), 30 MJ/litre (= 8.3 kWh/litre)

$$P_{\text{rotor}} = \mathcal{T} v_i$$

$$v_i = \sqrt{\frac{\mathcal{T}}{2\rho A}}$$



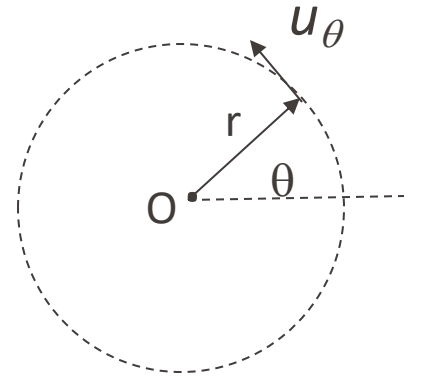
# Vortex

Vortex flow: A vortex may be seen as region of a flow in which the fluid rotates around a straight or curved line

- Free vortex (potential flow):
  - Velocity field:

$$\vec{u} = (0, u_\theta, 0), \quad u_\theta(r) = \frac{\Gamma}{2\pi r}$$

Vortex intensity  $\Gamma$



$$\Gamma = \oint_C \vec{u} \cdot d\vec{s} = \text{cst.}$$

$d\vec{s}$  = infinitesimal path element along the closed curve C

- Irrotational flow

$$\vec{\zeta} = \nabla \times \vec{u} = \left( 0, 0, \frac{1}{r} \frac{\partial (ru_\theta)}{\partial r} \right) = (0, 0, 0) \quad \forall r > 0$$

- Free vortex (potential flow):

- Pressure field :

- Navier-Stokes (radial equilibrium):  $\frac{\partial p}{\partial r} = \rho \frac{u_\theta^2}{r} = \rho \frac{\Gamma^2}{4\pi^2 r^3}$

$$\text{Integration} \rightarrow p(r) = p_\infty - \frac{\rho \Gamma^2}{8\pi^2 r^2} \quad u_\theta(r) = \frac{\Gamma}{2\pi r}$$

- Alternate method - Bernoulli equation:

Since the flow is irrotational and steady, Bernoulli equation reads:

$$\forall r > 0, \quad p(r) + \rho \frac{u_\theta^2}{2} = \text{constant}$$

$$\Rightarrow \forall r > 0, \quad p(r) = p_\infty - \rho \frac{u_\theta^2}{2} = p_\infty - \frac{\rho \Gamma^2}{8\pi^2 r^2}$$

- Limitations:

$$\lim_{r \rightarrow 0} u_\theta(r) = +\infty, \quad \lim_{r \rightarrow 0} p(r) = -\infty$$



- Rankine model:
  - Close to the axis, the viscous forces limits the velocity
  - Velocity field:

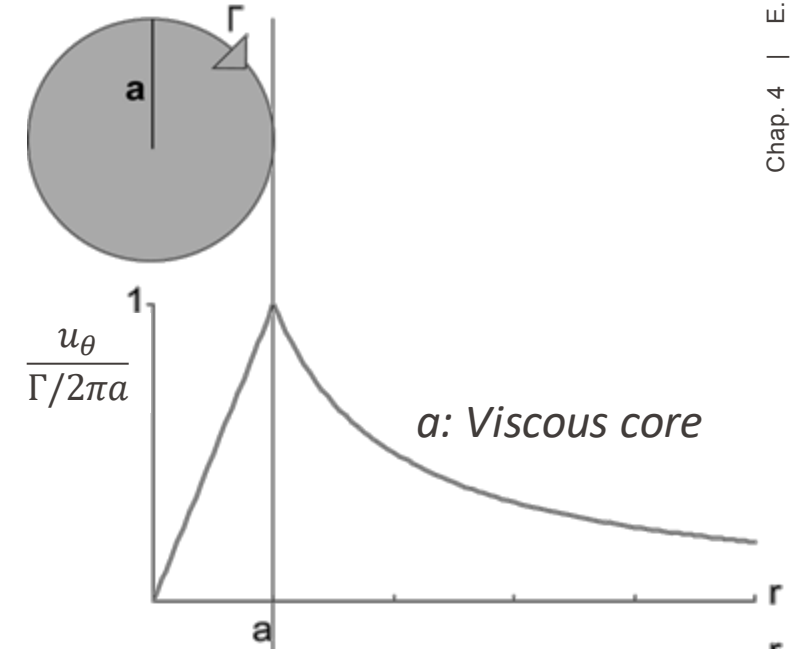
$$\begin{cases} r \geq a, & u_\theta = \frac{\Gamma}{2\pi r} \\ r \leq a, & u_\theta = \omega r \end{cases}$$

where  $\Gamma$  is the circulation:

$$\Gamma = \int_0^{2\pi} u_\theta r d\theta$$

- Continuity of the velocity at  $r = a$

$$\omega a = \frac{\Gamma}{2\pi a} \rightarrow \omega = \frac{\Gamma}{2\pi a^2} \quad \text{or} \quad \Gamma = 2\pi\omega a^2$$



- Rankine model:
  - Pressure field (Navier Stokes)

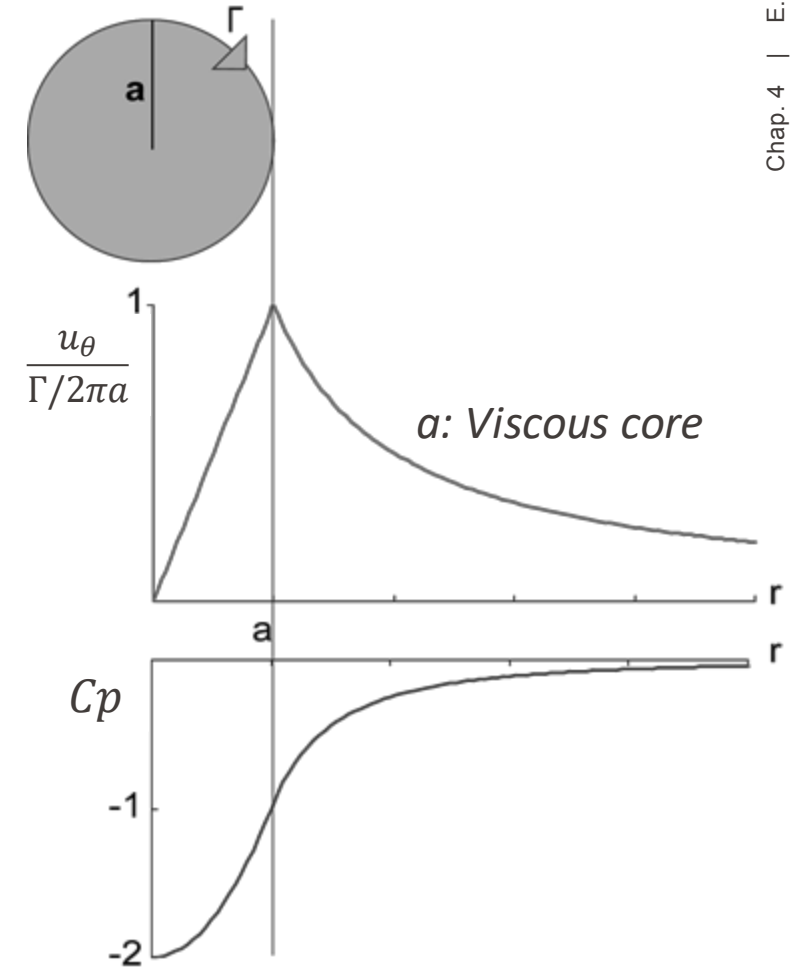
$$\frac{\partial p}{\partial r} = \rho \frac{u_\theta^2}{r}$$

- Integration:

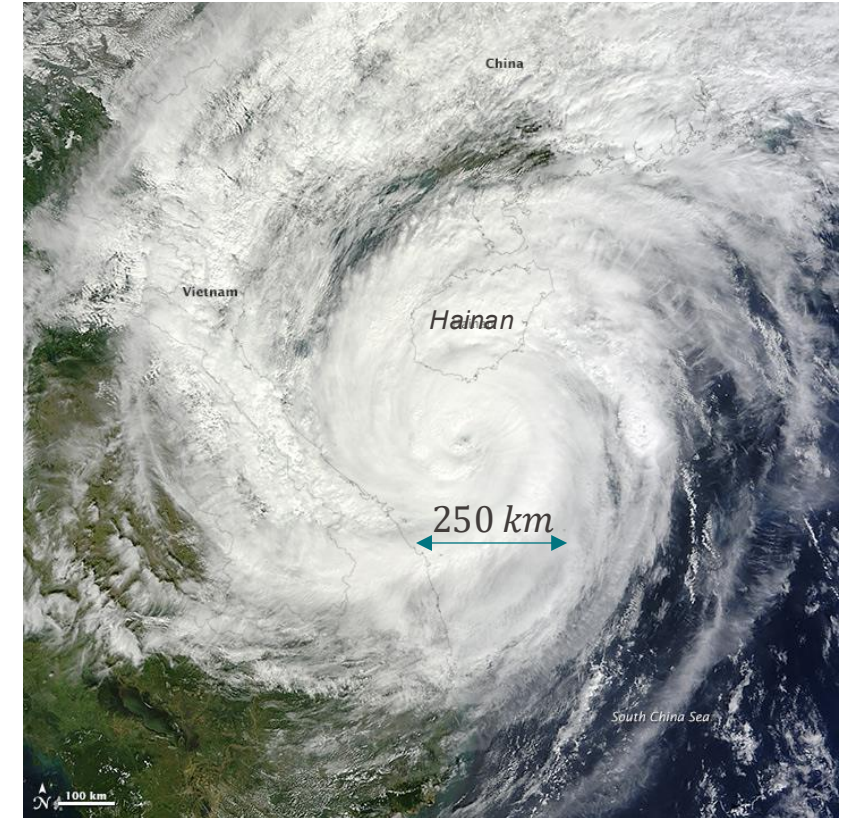
$$\begin{cases} r \geq a, & p(r) = p_\infty - \frac{\rho \Gamma^2}{8\pi^2 r^2} \\ r \leq a, & p_\infty - \frac{\rho \Gamma^2}{8\pi^2 a^2} \left( 2 - \frac{r^2}{a^2} \right) \end{cases}$$

$$C_p(r) = \frac{p(r) - p_\infty}{\frac{1}{2} \rho \left( \frac{\Gamma}{2\pi a} \right)^2} \quad C_{p,\min} = -2$$

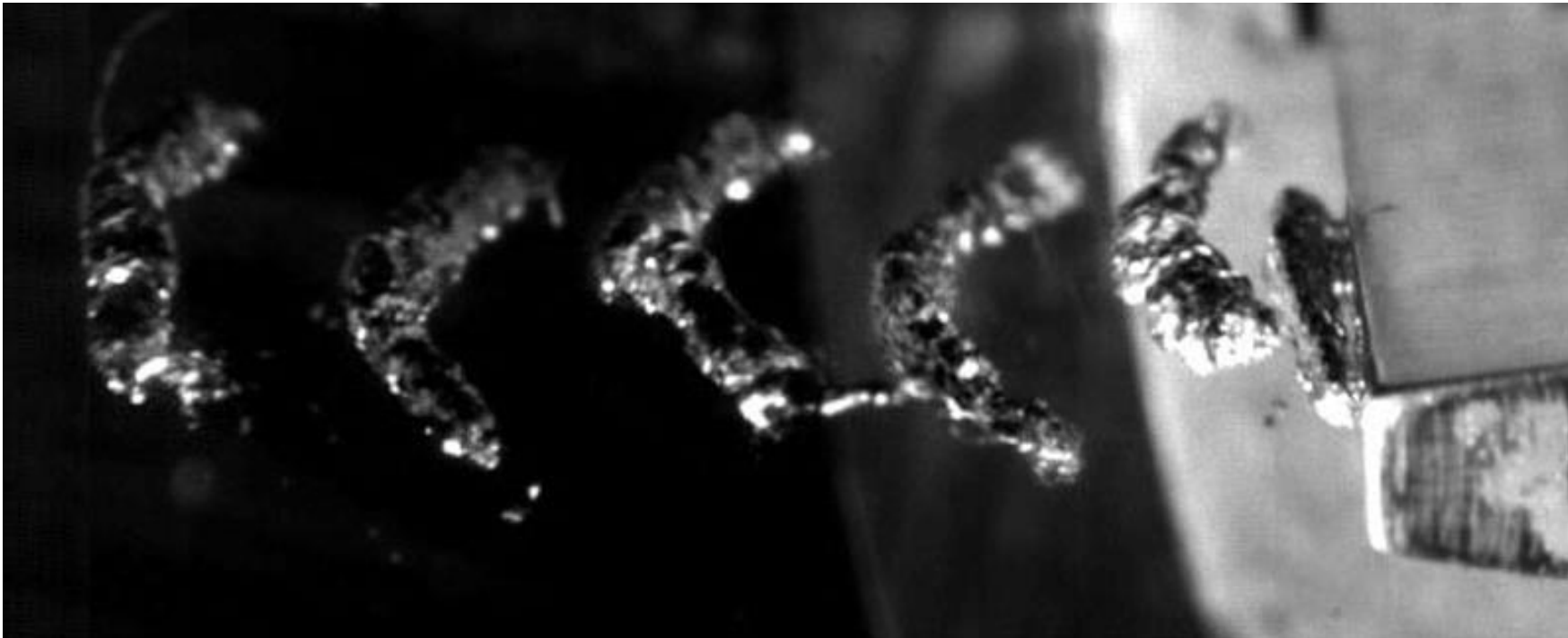
- Discontinuity of the velocity derivative at  $r = a$ 
  - More sophisticated models are available (e.g. Oseen model, Vestas model, ...)



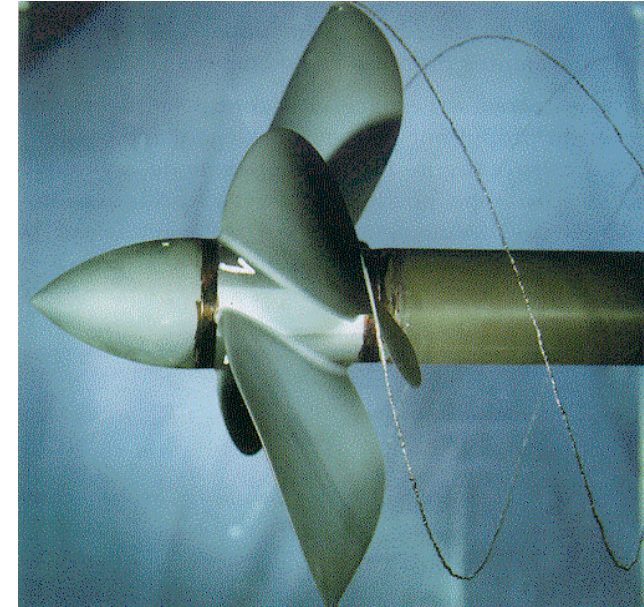
- Example of vortex flow: Super Typhoon Haiyan, Philippines, November 2013
  - Maximum wind speed  $\sim 360$  km/h
    - Pressure in the vortex axis ?
  - Assumptions:
    - Air density  $\rho = 1$  kg/m<sup>3</sup>
    - Atm. pressure = 1000 hPa
    - 2D flow



- Example of vortex flow: Karman vortices in the wake of a blunt hydrofoil
  - Flow instability in the wake → Alternate shedding of discrete vortices
  - Low pressure in the core of the vortices → Cavitation
  - Hydro-elastic coupling (Strong induced vibration):  
Vortex shedding frequency  $\sim$  resonance frequency of the hydrofoil (torsional mode)



- Example of vortex flow: Tip vortices in marine propellers
- Lifting blade with finite span (ducted or non-ducted impeller):
  - A swirl develop at the tip as the flow escapes from high to low pressure sides
  - Risk of cavitation → Source of noise and vibration





- Example of vortex flow: Rope in Francis turbines
  - The rope is a cavitating vortex that develops at the outlet of a Francis turbine
  - Due to the residual kinetic moment at part load condition
  - Source of strong hydraulic instabilities (noise and vibration)

