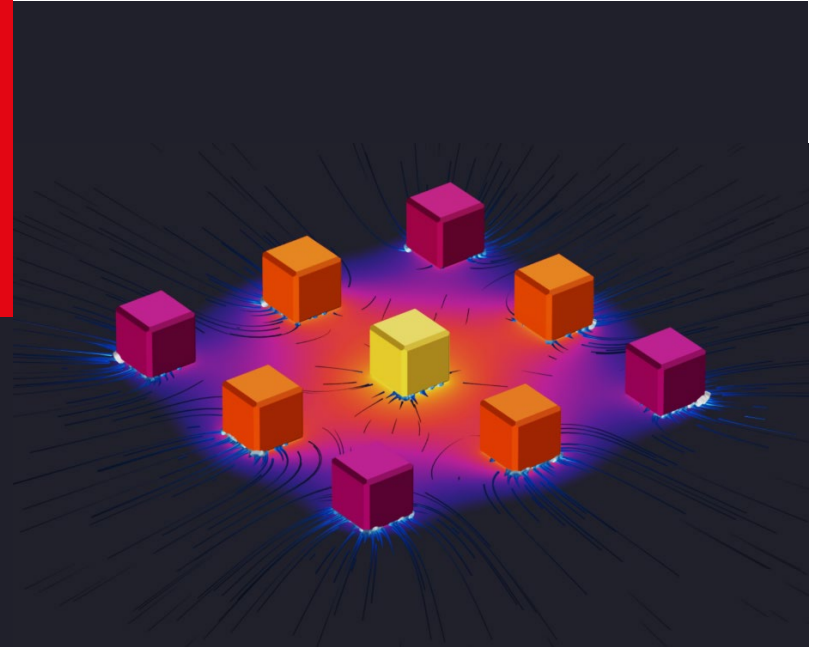


# Heat and Mass Transfer ME-341

*Instructor:* Giulia Tagliabue



Spring Semester

# Previously



Introduction to Heat Exchangers



Overall Heat Transfer Coefficient

## Learning Objectives:



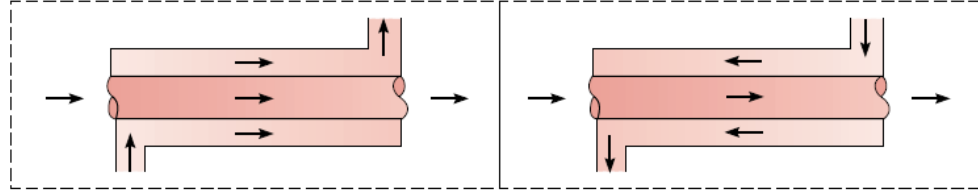
Understand the concept and possible design of heat exchangers



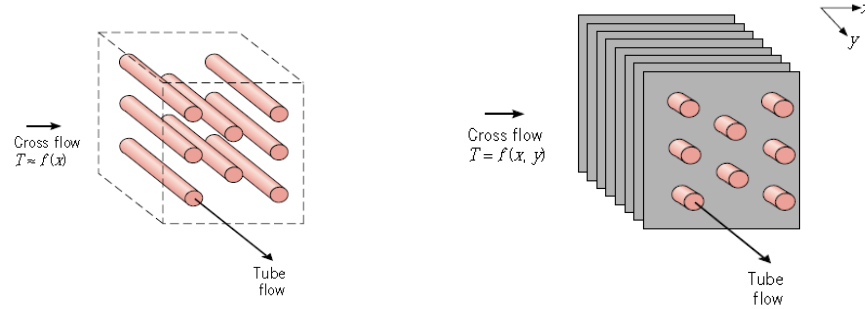
Calculate the overall heat transfer coefficient

# Introduction to Heat Exchangers

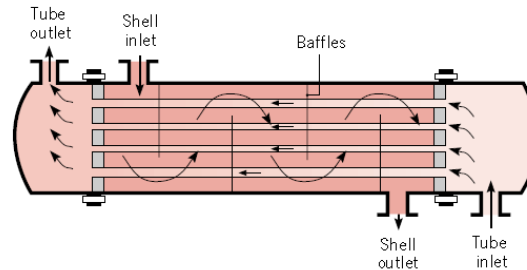
## A. Concentric Flow



## B. Cross-Flow



## C. Shell-and-Tube



# Overall Heat Transfer Coefficient

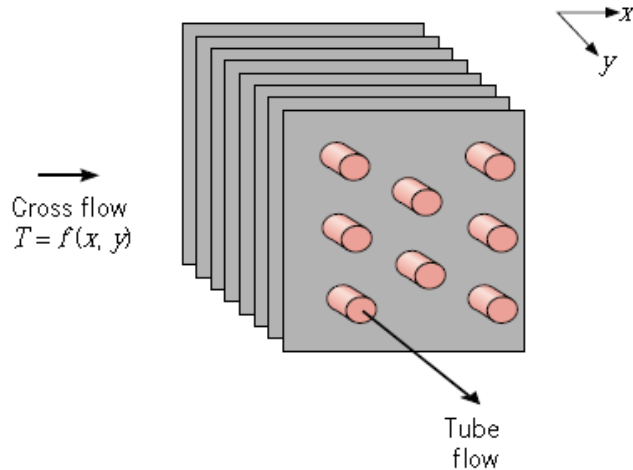
$$Q = \frac{\Delta T}{R_{tot}} = \mathbf{U} A \Delta T$$



$$U \equiv \frac{1}{R_{tot} A}$$



$$\frac{1}{UA} \equiv R_{tot}$$



- Conduction thermal resistances
  - Planar/radial conduction
- Convection thermal resistances
  - Internal and external
- Array of Fins
- Fouling

# Overall Heat Transfer Coefficient

## *Conduction & Convection Thermal Resistances*

Planar wall

$$R_{th,cond} = \frac{L}{kA} \quad [ \text{K/W} ]$$

$$R_{th,conv} = \frac{1}{hA} \quad [ \text{K/W} ]$$

Radial System

$$R_{th,cond-cyl} = \frac{\ln(r_2/r_1)}{2\pi Lk} \quad [ \text{K/W} ]$$

$$R_{th,conv} = \frac{1}{h2\pi rL} \quad [ \text{K/W} ]$$

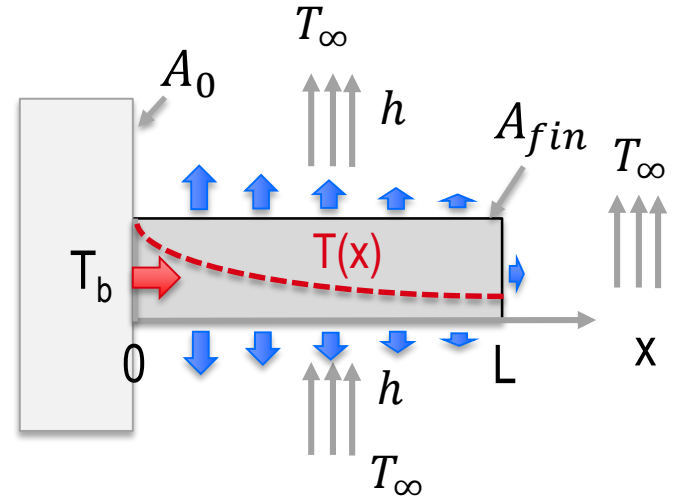
# Overall Heat Transfer Coefficient

## Fins – Efficiency and Resistance (1 Fin)

If  $T_b$  is the temperature of the fin base:

$$\Rightarrow \eta_f \equiv \frac{Q_f}{Q_{f,max}} = \frac{Q_f}{hA_f(T_b - T_\infty)} = \frac{Q_f}{hA_f\theta_b}$$

$$\Rightarrow R_f \equiv \frac{(T_b - T_\infty)}{Q_f} = \frac{1}{hA_f\eta_f}$$



e.g. for infinite fin we get:  $R_f \equiv \frac{\theta_b}{Q_f} = \frac{\theta_b}{M} = \frac{1}{\sqrt{hPkA_c}}$

# Overall Heat Transfer Coefficient

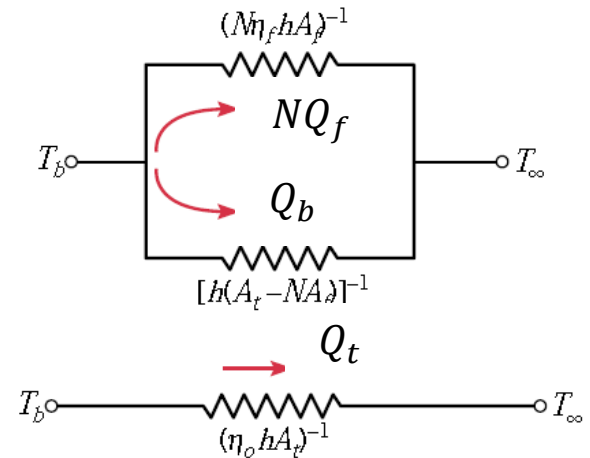
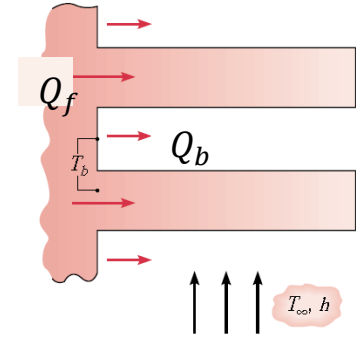
## Fins – Efficiency and Resistance (Array of Fins)

$$\eta_o \equiv \frac{Q_t}{Q_{t,max}} = \frac{Q_t}{hA_t(T_0 - T_\infty)} = \frac{NQ_f + Q_b}{h(NA_f + A_b)(T_0 - T_\infty)}$$

$$\Rightarrow \eta_o \equiv 1 - \frac{NA_f}{A_t}(1 - \eta_f) \quad \text{Overall efficiency}$$

Single fin efficiency

$$\Rightarrow R_o \equiv \frac{(T_0 - T_\infty)}{Q_t} = \frac{1}{\eta_o h A_t}$$



# Overall Heat Transfer Coefficient

## Fouling



Dramatic increase in thermal resistance due to poor conduction through the scaling layer

➡ Introduce a *fouling* resistance per unit area (fouling factor)  $R_f''$

$$\begin{aligned} \text{➡} \quad R_f'' &\equiv A R_{foul} \quad \text{➡} \quad R_{foul} = \frac{R_f''}{A} \end{aligned}$$



# Overall Heat Transfer Coefficient

$$Q = \frac{\Delta T}{R_{tot}} = \mathbf{U} A \Delta T \quad \rightarrow \quad U \equiv \frac{1}{R_{tot} A} \quad \rightarrow \quad \frac{1}{UA} \equiv R_{tot}$$

In the most general case we could have fins present also on the inner side:

$$\frac{1}{UA} = \frac{1}{\eta_{o,out} h_{out} A_{out}} + \frac{R_{f,o}''}{\eta_{o,out} A_{out}} + \underbrace{R_{cond}} + \frac{R_{f,i}''}{\eta_{o,in} A_{in}} + \frac{1}{\eta_{o,in} h_{in} A_{in}}$$

Includes all the layers of conduction!!

$$U_{in} \equiv \frac{1}{R_{tot} A_{in}} \neq U_{out} \equiv \frac{1}{R_{tot} A_{out}}$$

# This Lecture

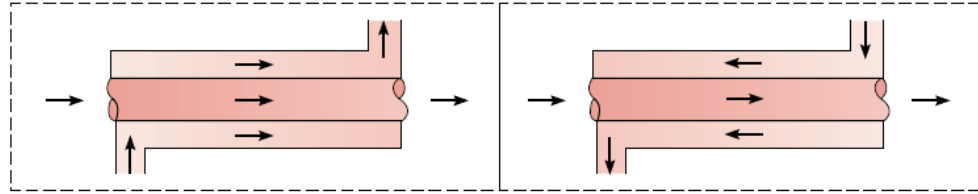
- ❑ Parallel Flow Design
  - ❑ Temperature Profile and Total Heat Transfer
- ❑ Counter-flow Design

## Learning Objectives:

- ❑ Calculate the total heat transfer for parallel flow HE
- ❑ Calculate the total heat transfer for counter flow HE

# Introduction to Heat Exchangers

## A. Concentric Flow



Parallel Flow

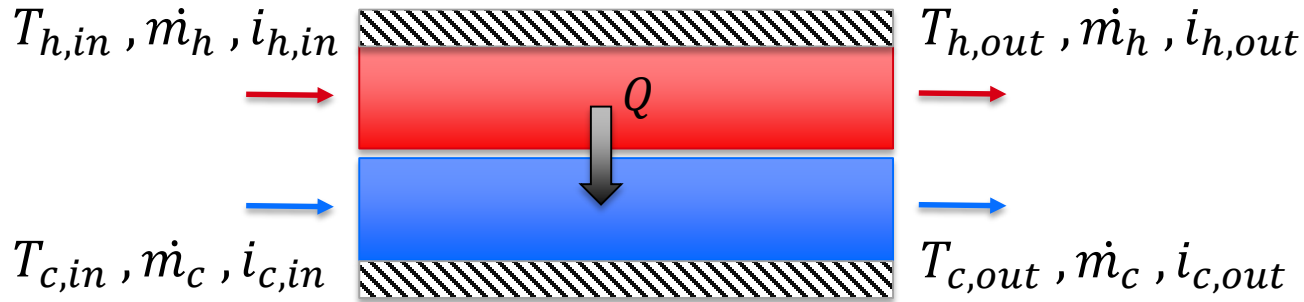
Counter Flow

# Parallel Flow Heat Exchanger

*Energy balance*

$i = \text{enthalpy}$

*We use this anomalous symbol to avoid confusion with convection coefficient*



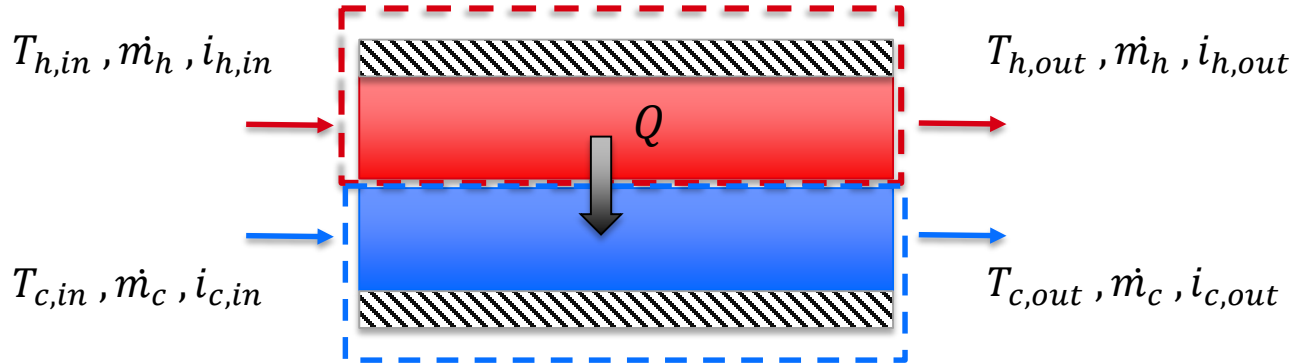
# Parallel Flow Heat Exchanger

Energy balance  $Q = Q_c = -Q_h$

$i$  = **enthalpy**

We use this anomalous symbol to avoid confusion with convection coefficient

$$Q_h = \dot{m}_h(i_{h,out} - i_{h,in}) = \dot{m}_h c_{p,h}(T_{h,out} - T_{h,in}) < 0$$



$$Q_c = \dot{m}_c(i_{c,out} - i_{c,in}) = \dot{m}_c c_{p,c}(T_{c,out} - T_{c,in}) > 0$$

Incompressible fluid  
True if no phase change

Average temperature  
on that cross-section

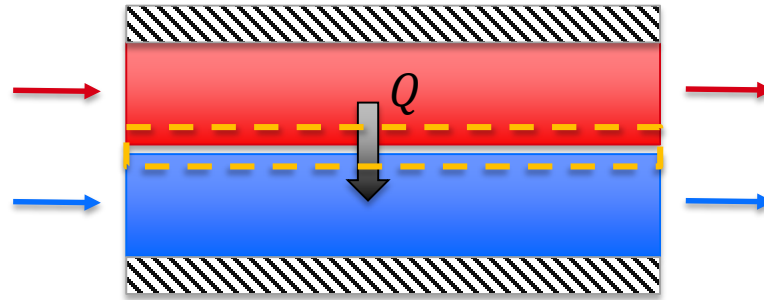
# Parallel Flow Heat Exchanger

Energy balance

$i = \text{enthalpy}$

*We use this anomalous symbol to avoid confusion with convection coefficient*

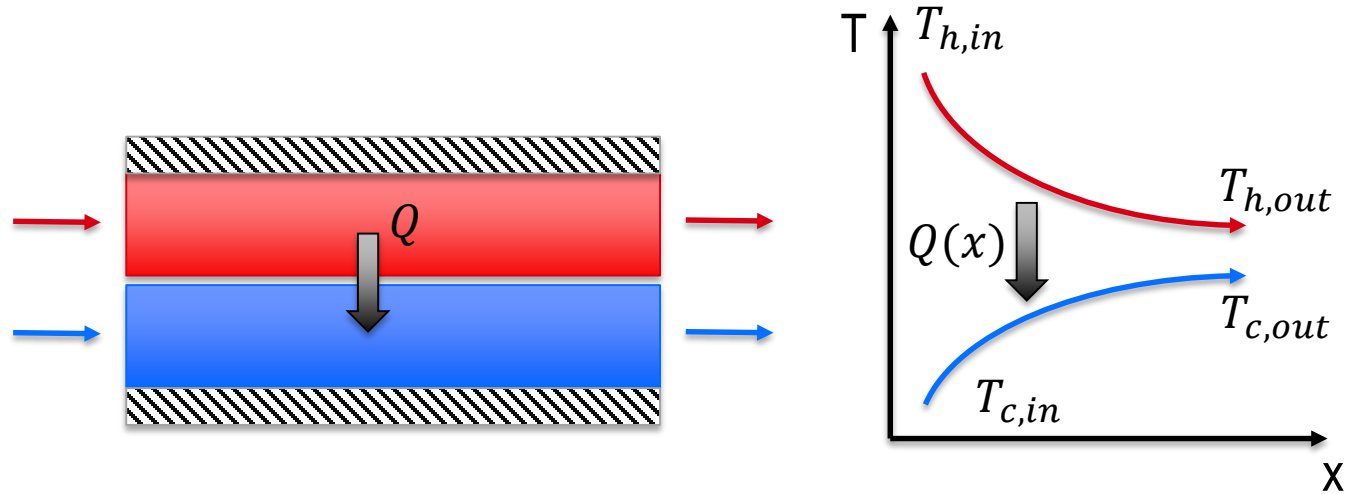
$$Q = UA\Delta T_m$$



- ➡ What is an appropriate  $\Delta T_m$ ?
- ➡ It is related to the temperature profile

# Parallel Flow Heat Exchanger

## Temperature Profile

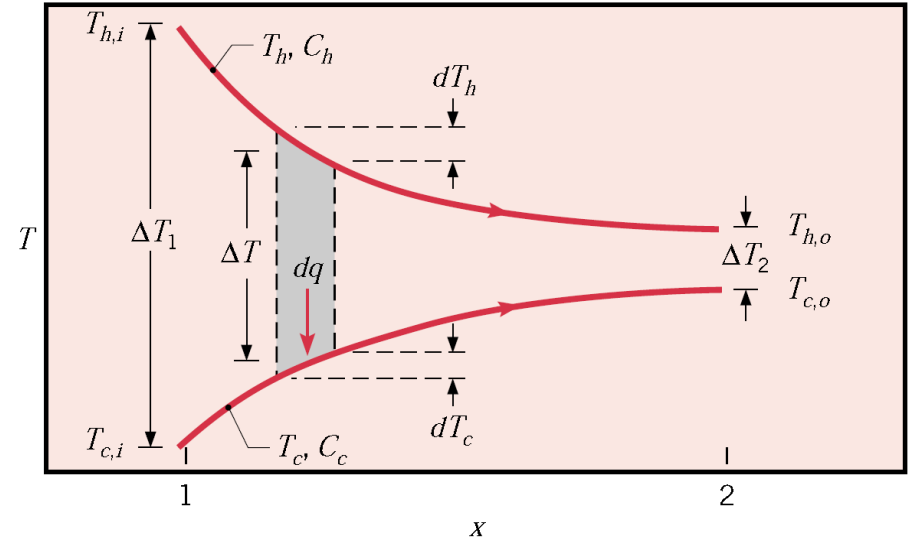
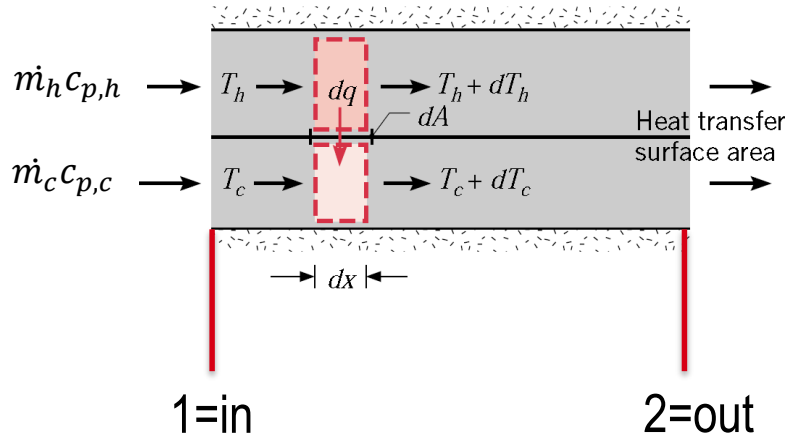


➡ What is an appropriate  $\Delta T_m$ ?

➡ Let's write a local energy balance, similar to what we did for internal forced convection

# Parallel Flow heat Exchanger

## Temperature Profile

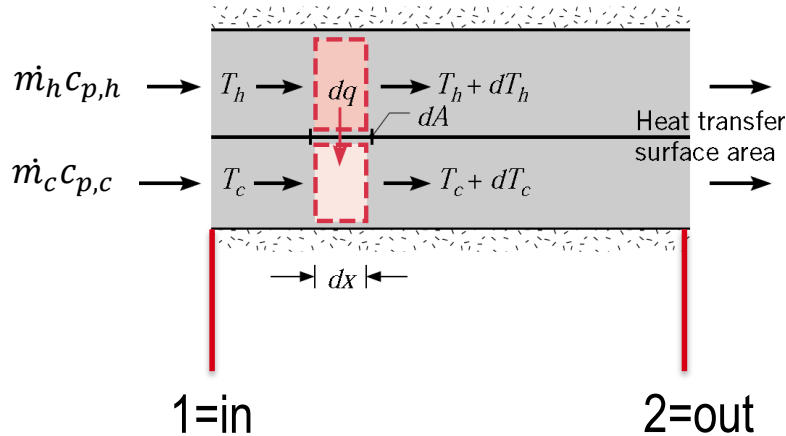


Need to write a differential equation for  $\Delta T$



# Parallel Flow heat Exchanger

## Temperature Profile



1. The heat exchanger is insulated from its surroundings, in which case the only heat exchange is between the hot and cold fluids.
2. Axial conduction along the tubes is negligible.
3. Potential and kinetic energy changes are negligible.
4. The fluid specific heats are constant.
5. The overall heat transfer coefficient is constant.

$$dq = \dot{m} (i_{x+dx} - i_x)$$

$$dq = \dot{m}_c c_{p,c} dT_c \equiv C_c dT_c \quad \text{Heat gain cold side}$$

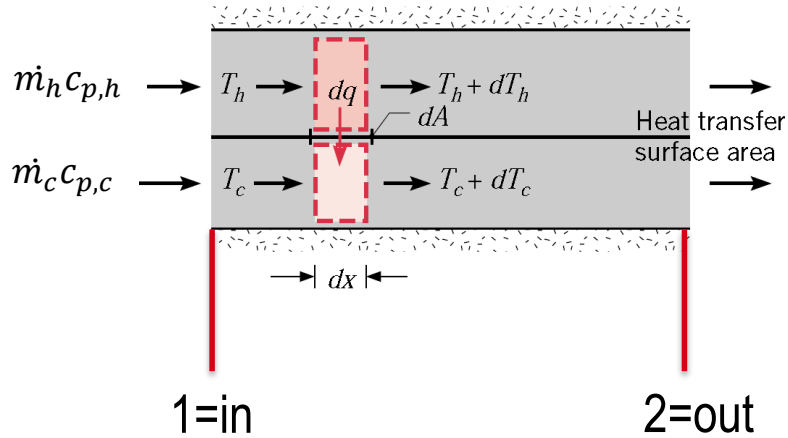
$$= -\dot{m}_h c_{p,h} dT_h \equiv -C_h dT_h \quad \text{Heat loss hot side}$$

Also we can write:

$$\Delta T = T_h - T_c \quad \Rightarrow \quad dq = U \Delta T dA$$

# Parallel Flow heat Exchanger

## Temperature Profile



$$\Delta T = T_h - T_c \quad \Rightarrow \quad d(\Delta T) = dT_h - dT_c$$

Where:  $dT_h = -\frac{dq}{C_h}$   $dT_c = \frac{dq}{C_c}$

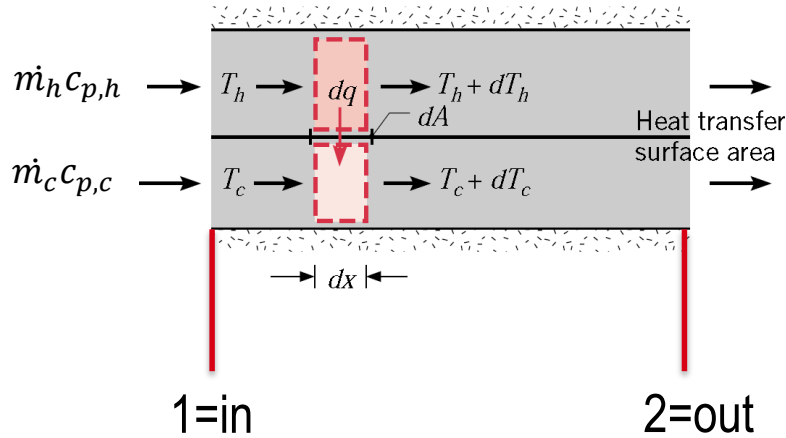
$$\Rightarrow d(\Delta T) = -dq \left( \frac{1}{C_h} + \frac{1}{C_c} \right)$$

$$\Rightarrow d\Delta T = -U\Delta T dA \left( \frac{1}{C_h} + \frac{1}{C_c} \right)$$

$$\Rightarrow \int_1^2 \frac{d(\Delta T)}{\Delta T} = -U \left( \frac{1}{C_h} + \frac{1}{C_c} \right) \int_1^2 dA$$

# Parallel Flow heat Exchanger

## Temperature Profile



$$\int_1^2 \frac{d(\Delta T)}{\Delta T} = -U \left( \frac{1}{C_h} + \frac{1}{C_c} \right) \int_1^2 dA$$

$$\Rightarrow \ln \left( \frac{\Delta T_2}{\Delta T_1} \right) = -UA \left( \frac{1}{C_h} + \frac{1}{C_c} \right)$$

$$Q = -Q_h = -\dot{m}_h c_{p,h} (T_{h,1} - T_{h,2}) = C_h (T_{h,1} - T_{h,2})$$

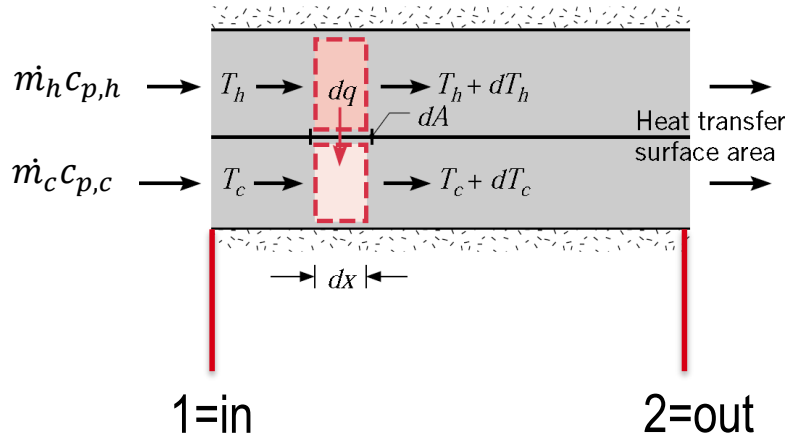
$$Q = Q_c = \dot{m}_c c_{p,c} (T_{c,2} - T_{c,1}) = C_c (T_{c,2} - T_{c,1})$$

$$C_h = Q / (T_{h,1} - T_{h,2}) \quad C_c = Q / (T_{c,2} - T_{c,1})$$

$$\Rightarrow \ln \left( \frac{\Delta T_2}{\Delta T_1} \right) = -UA \left( \frac{(T_{h,1} - T_{h,2})}{Q} + \frac{(T_{c,2} - T_{c,1})}{Q} \right)$$

# Parallel Flow heat Exchanger

*Temperature Profile and Total Heat Transfer*



$$\ln\left(\frac{\Delta T_2}{\Delta T_1}\right) = -UA \left( \frac{(T_{h,1} - T_{c,1})}{Q} + \frac{(T_{c,2} - T_{h,2})}{Q} \right)$$

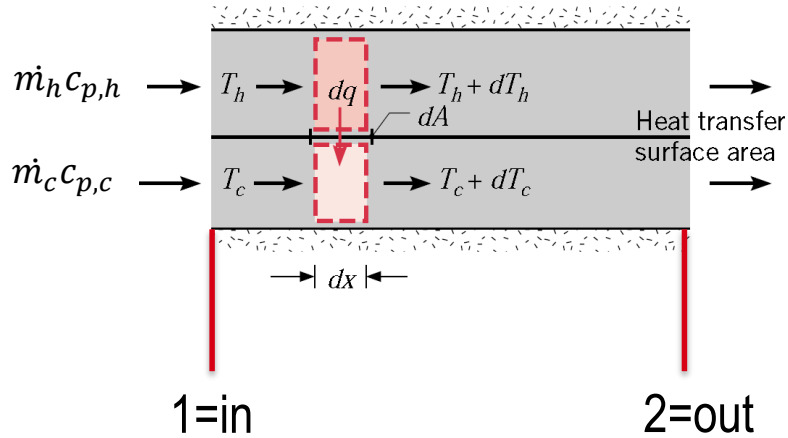
$$\Rightarrow \ln\left(\frac{\Delta T_2}{\Delta T_1}\right) = -UA \left( \frac{\Delta T_1}{Q} - \frac{\Delta T_2}{Q} \right)$$

$$\Rightarrow Q = UA \frac{\Delta T_2 - \Delta T_1}{\ln(\Delta T_2 / \Delta T_1)} = UA \Delta T_m$$

$$\Rightarrow \Delta T_m = \frac{\Delta T_2 - \Delta T_1}{\ln(\Delta T_2 / \Delta T_1)} \quad \begin{aligned} \Delta T_1 &= (T_{h,1} - T_{c,1}) \\ \Delta T_2 &= (T_{h,2} - T_{c,2}) \end{aligned}$$

# Parallel Flow heat Exchanger

## Temperature Profile and Total Heat Transfer



$$Q = UA \frac{\Delta T_2 - \Delta T_1}{\ln(\Delta T_2 / \Delta T_1)} = UA \Delta T_m$$

$$\Delta T_m = \frac{\Delta T_2 - \Delta T_1}{\ln(\Delta T_2 / \Delta T_1)}$$

$$\Delta T_1 = (T_{h,1} - T_{c,1}) = (T_{h,in} - T_{c,in})$$

$$\Delta T_2 = (T_{h,2} - T_{c,2}) = (T_{h,out} - T_{c,out})$$

# This Lecture



Parallel Flow Design



Temperature Profile and Total Heat Transfer



Counter-flow Design

Learning Objectives:



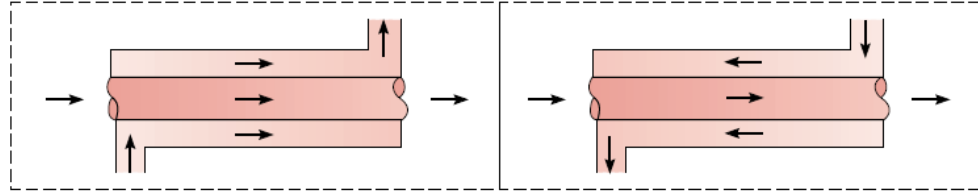
Calculate the total heat transfer for parallel flow HE



Calculate the total heat transfer for counter flow HE

# Introduction to Heat Exchangers

## A. Concentric Flow

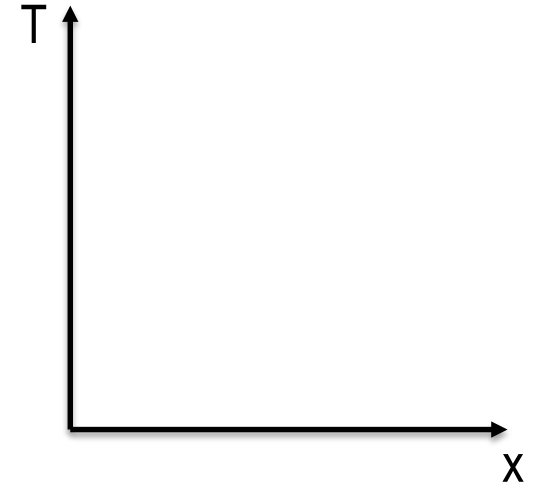
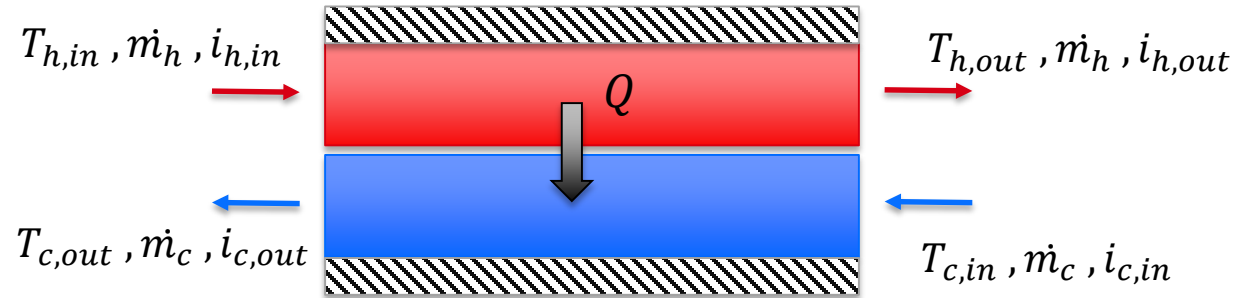


Parallel Flow

Counter Flow

# Counter Flow Heat Exchanger

*Temperature Profile*

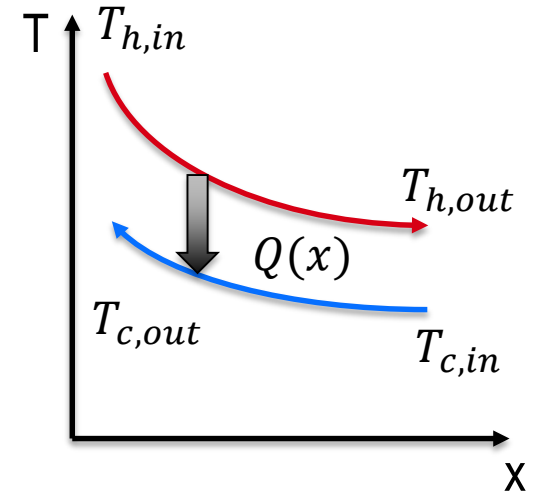
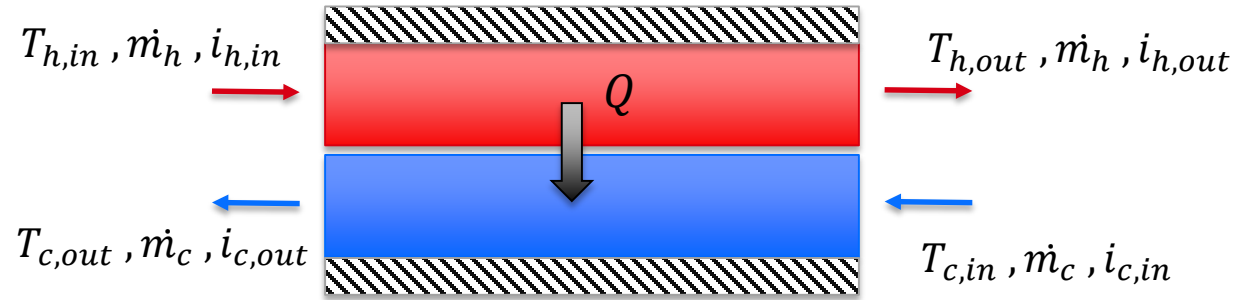


➡ What is an appropriate  $\Delta T_m$ ?



# Counter Flow Heat Exchanger

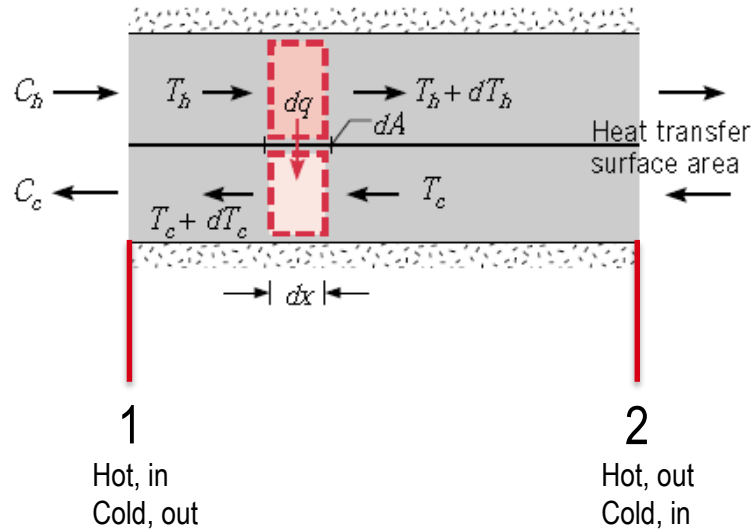
## Temperature Profile



➡ What is an appropriate  $\Delta T_m$ ?

# Counter Flow heat Exchanger

## Temperature Profile and Total Heat Transfer



$$Q = UA \frac{\Delta T_2 - \Delta T_1}{\ln(\Delta T_2 / \Delta T_1)} = UA \Delta T_m$$

$$\Delta T_m = \frac{\Delta T_2 - \Delta T_1}{\ln(\Delta T_2 / \Delta T_1)}$$

$$\Delta T_1 = (T_{h,1} - T_{c,1}) = (T_{h,in} - T_{c,out})$$

$$\Delta T_2 = (T_{h,2} - T_{c,2}) = (T_{h,out} - T_{c,in})$$

# This Lecture

- ✓ ☒ Parallel Flow Design
- ✓ ☒ Temperature Profile and Total Heat Transfer
- ✓ ☒ Counter-flow Design

## Learning Objectives:

- ✓ ☒ Calculate the total heat transfer for parallel flow HE
- ✓ ☒ Calculate the total heat transfer for counter flow HE

# Next Lecture

- ☐ Special Operating Conditions
- ☐ Exercises

Learning Objectives:

- ☐ Recognize specific conditions