

## Problem Set 7: Navier-Stokes Equations

### 1 Uniform flow with a sink

#### 1.1 Mathematics

Consider a flow that consists of a sink of strength  $M$  located at the origin and a uniform flow, as shown in Figure 1. Infinitely far away from the origin, the flow is parallel to the  $x$ -axis with a uniform velocity,  $U$ , and pressure,  $p_\infty$ . The density of the fluid is  $\rho$ , which is constant.

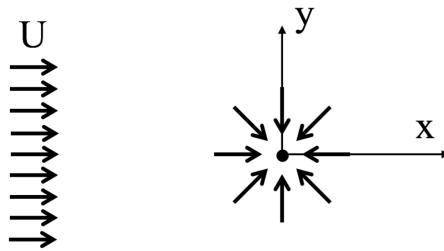


Figure 1: See problem 1.1.

- (a) Using superposition, determine the stream function,  $\psi(r, \theta)$ , and velocity potential,  $\phi(r, \theta)$ , for this system.
- (b) Calculate the velocity field for the flow.
- (c) Identify the location of any stagnation point in the flow.
- (d) Determine the equation for the stagnation streamline.
- (e) Determine the pressure along the line  $y = 0$  (i.e. calculate  $p(x)$  along  $y = 0$ ). Does anything unusual happen to the pressure at  $x = 0$ ? Explain.
- (f) Find the value of  $x$  at which the pressure along the line  $y = 0$  is maximized.

## 1.2 Flow into a slit

Water flows over a flat surface at  $1.2 \text{ m/s}$  as shown in Figure 2. A pump draws off water through a narrow slit at a volume rate of  $0.01 \text{ m}^3/\text{s}$  per meter length of the slit. Assume that the fluid is incompressible and inviscid and can be represented by the combination of a uniform flow and a sink. How far above the surface,  $H$ , must the fluid be so that it does not get sucked into the slit? Use the results from 1.1.

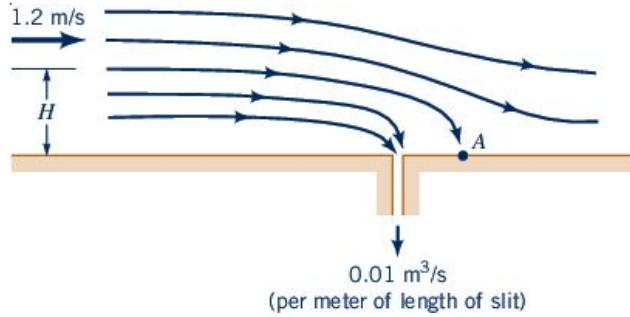


Figure 2: See problem 1.2.

## 2 Quonset hut

Wind at velocity  $U_\infty$  and pressure  $p_\infty$  flows past a Quonset hut which is a half-cylinder of radius  $a$  and length  $L$  (Figure 3). The internal pressure is  $p_i$ . Derive an expression for the upward force on the hut due to the difference between  $p_i$  and  $p_s$ .

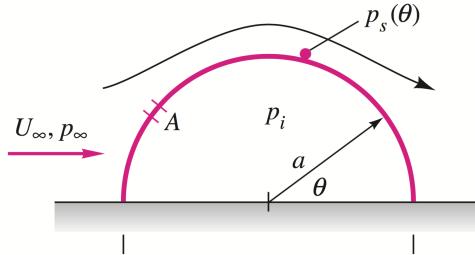


Figure 3: See problem 2.

### 3 Rotating shaft in a pipe

Consider the fully developed flow of an incompressible fluid with viscosity  $\mu$  and density  $\rho$  enclosed between two concentric cylinders of radii  $R_1$  and  $R_2$ , ( $R_1 < R_2$ ), as shown in Figure 4. The outer pipe is held stationary, while the inner pipe rotates slowly with a constant angular velocity  $\omega$ . Due to the symmetry of the system, the pressure gradient only varies spatially in the radial direction; that is,  $p = p(r)$ .

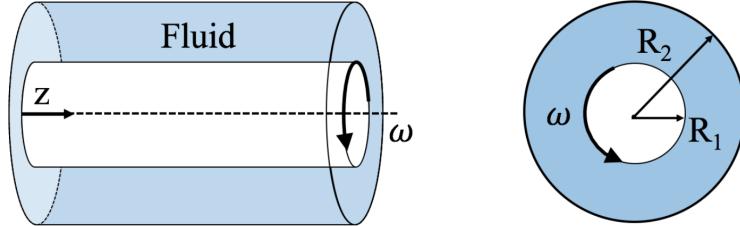


Figure 4: Rotating shaft in a pipe. See problem 3.

- (a) Simplify the Navier-Stokes equations to derive the flow's equation of motion. State your assumptions.
- (b) What are the boundary conditions for the flow?
- (c) Calculate the velocity profile of the flow.
- (d) If the pressure at  $r = R_1$  is  $p_0$ , what is the pressure distribution in the fluid,  $p(r)$ ?

*Hint:* The general form of the incompressible Navier-Stokes equations in cylindrical coordinates was given on the last exercise sheet.

### 4 Pulled shaft in a pipe

An incompressible Newtonian fluid flows steadily between two infinitely long, concentric cylinders as shown in Figure 5. The outer cylinder is fixed, but the inner cylinder moves with a longitudinal velocity  $V_0$  as shown. The pressure gradient in the axial directions is  $-\Delta p/l$  where  $\Delta p > 0$  is the magnitude of the pressure difference between two sections of distance  $l$ . For what value of  $V_0$  will the drag on the inner cylinder be zero? Assume that the flow is laminar, axisymmetric and fully developed.

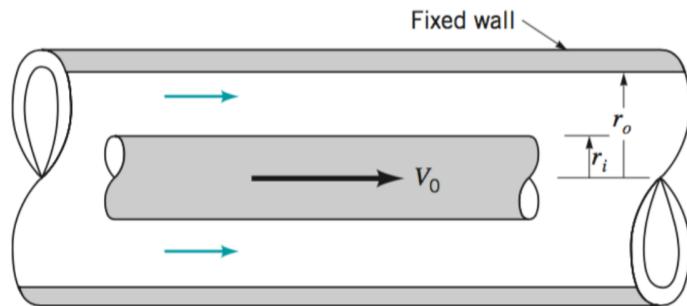


Figure 5: See problem 4.

## 5 Time-dependent channel flow

A section of a water channel is shown in Figure 6. It is  $l = 0.5 \text{ m}$  long and  $d = 1 \text{ cm}$  wide. The water is incompressible and has a dynamic viscosity of  $\mu = 10^{-3} \text{ Ns/m}^2$ . The fluid is initially at rest but at time  $t = 0$ , a pump is switched on and the fluid becomes subject to a pressure difference of  $\Delta p = p_{in} - p_{out} = 0.01 \text{ kPa}$  along the channel in  $x$ -direction. Due to *no slip* boundary conditions the velocity at the channel walls is zero:  $\mathbf{U}(y = 0, t) = \mathbf{U}(y = d, t) = 0$ .

(a) Simplify the Navier-Stokes Equations for the planar velocity  $\mathbf{U}(\mathbf{x}, t) = U(y, t) \hat{\mathbf{i}}$ .  
 Then, solve the time-dependent problem of the vertical velocity profile  $U(y, t)$  by calculating  
 (b) the steady state solution,  
 (c) the time-dependent solution for the deviations from the steady state.

*Hint:* Follow the same solution strategy as in the diffusion exercises in problem set 3 and 4.

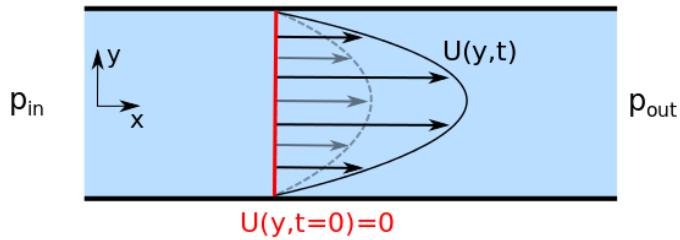


Figure 6: Pressure driven channel flow. See problem 5.