

# Thermodynamics of Earth systems

Lecture 3:  
The First Law of  
Thermodynamics

# Material covered in Lecture 3

## Conclusion of Part 1: Introduction

- Hydrostatic equation: application to ocean and hypothetical constant density atmosphere; solid earth
- Hypsometric equation (atmosphere)

## *In-class worksheets*

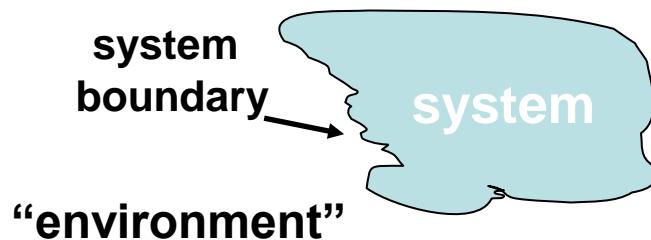
## Part 2: Framework

### *First Law of thermodynamics*

- Basic concepts & processes
- Work; expansion work
- Heat: heat capacity, basics of heat transfer mechanisms
- First law: internal energy, enthalpy, specific heats, heat capacity.
- Applications of first law to ideal gases

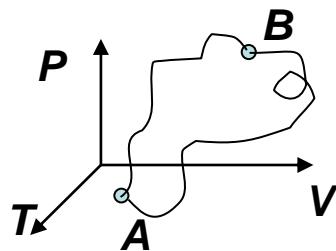
# Internal Energy

The **internal energy** of a system of particles,  $U$ , is the sum of the kinetic energy *in the reference frame in which the center of mass is at rest* and the potential energy arising from the forces of the particles on each other.



Difference between the total energy and the internal energy?

$$U = \text{kinetic} + \text{potential}$$



The internal energy is a **state function** – it depends only on the values of macroparameters (the state of a system), not on the method of preparation of this state (the “path” in the macroparameter space is irrelevant).

In equilibrium [  $f(P,V,T)=0$  ] :  $U = U(V, T)$

$U$  depends on the kinetic energy of particles in a system and an average inter-particle distance ( $\sim V^{1/3}$ ) – interactions.

For an ideal gas (no interactions) :  $U = U(T)$  - “pure” kinetic

# Internal Energy of an Ideal Gas

The internal energy of an ideal gas with  $f$  degrees of freedom:

$$U = \frac{f}{2} N k_B T$$

$f \Rightarrow 3$  (monatomic),  $5$  (diatomic),  $6$  (polyatomic)

(here we consider only trans.+rotat. degrees of freedom, and neglect the vibrational ones that can be excited at very high temperatures)

How does the internal energy of air in this (not-air-tight) room change with  $T$  if the external  $P = \text{const}$ ?

$$U = \frac{f}{2} N_{\text{in room}} k_B T = \left[ N_{\text{in room}} = \frac{PV}{k_B T} \right] = \frac{f}{2} PV$$

- does not change at all, an increase of the kinetic energy of individual molecules with  $T$  is compensated by a decrease of their number.

# Work and Heating ("Heat")

We are often interested in  $\Delta U$ , not  $U$ .  $\Delta U$  is due to:

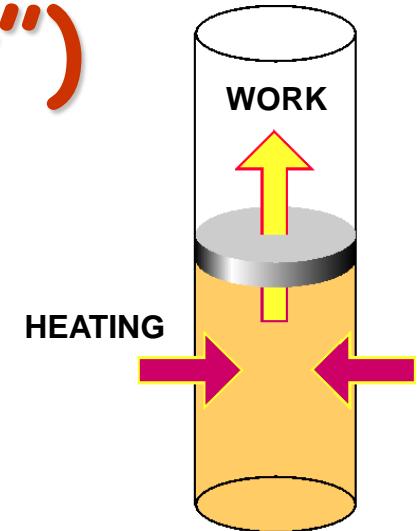
- $Q$  - energy flow between a system and its environment due to  $\Delta T$  across a boundary and a finite thermal conductivity of the boundary

– **heating ( $Q > 0$ ) /cooling ( $Q < 0$ )**

(there is no such physical quantity as “heat”; to emphasize this fact, it is better to use the term “heating” rather than “heat”)

- $W$  - any other kind of energy transfer across boundary

– **work**



*Work and Heating are both defined to describe energy transfer across a system boundary.*

***Heating/cooling processes:***

***conduction:*** the energy transfer by molecular contact – fast-moving molecules transfer energy to slow-moving molecules by collisions;

***convection:*** by macroscopic motion of gas or liquid

***radiation:*** by emission/absorption of electromagnetic radiation.

# The First Law

**The first law of thermodynamics:** the internal energy of a system can be changed by doing work on it or by heating/cooling it.

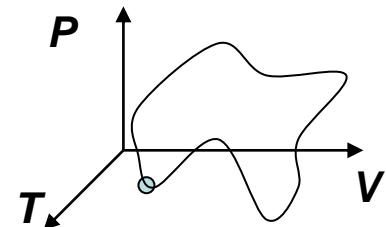
$$\Delta U = Q + W$$

conservation of energy.

**Sign convention:** we consider  $Q$  and  $W$  to be **positive** if energy flows *into* the system.

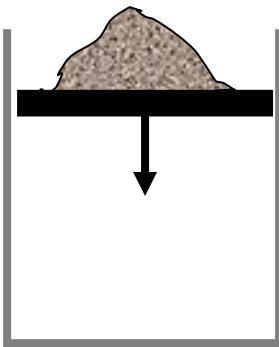
For a cyclic process ( $U_i = U_f$ )  $\Rightarrow Q = -W$ .

If, in addition,  $Q = 0$  then  $W = 0$



An equivalent formulation:

Perpetual motion machines of the first type do not exist.



# Quasi-Static Processes

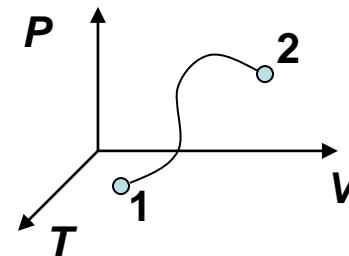
**Quasi-static (quasi-equilibrium) processes** – sufficiently slow processes, *any* intermediate state can be considered as an equilibrium state (the macroparameters are well-defined for all intermediate states).

**Advantage:** the state of a system that participates in a quasi-equilibrium process can be described with the **same (small) number of macro parameters** as for a system in equilibrium (e.g., for an ideal gas in quasi-equilibrium processes, this could be  $T$  and  $P$ ). By contrast, for **non-equilibrium processes** (e.g. turbulent flow of gas), we need a **huge number of macro parameters**.

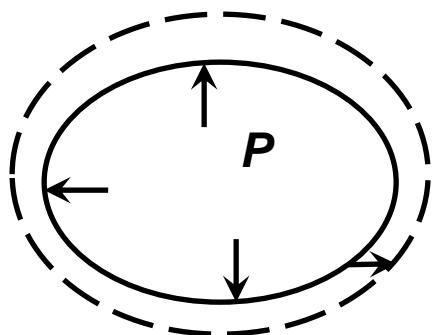
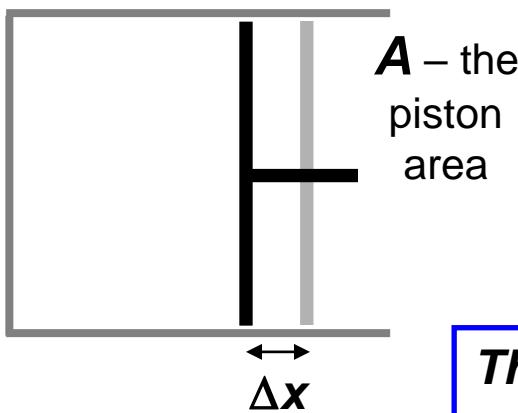
## Examples of quasi-equilibrium processes:

- isochoric:  $V = \text{const}$
- isobaric:  $P = \text{const}$
- isothermal:  $T = \text{const}$
- adiabatic:  $Q = 0$

For quasi-equilibrium processes,  $P$ ,  $V$ ,  $T$  are **well-defined** – the “path” between two states is a *continuous lines* in the  $P$ ,  $V$ ,  $T$  space.



# Work



**The work done by an external force** on a gas enclosed within a cylinder fitted with a piston:

$$W = (PA) dx = P(A dx) = \underbrace{-PdV}_{\text{force}}$$

**The sign:** if the volume is *decreased*,  $W$  is **positive** (by compressing gas, we increase its internal energy); if the volume is *increased*,  $W$  is **negative** (the gas decreases its internal energy by doing some work on the environment).

$$W_{1-2} = - \int_{V_1}^{V_2} P(T, V) dV$$

$$dU = Q - PdV$$

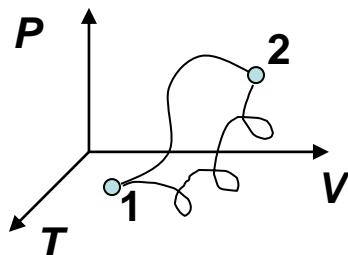
$W = -PdV$  - applies to any shape of system boundary

The work is not necessarily associated with the volume changes – e.g., in the Joule's experiments on determining the “mechanical equivalent of heat”, the system (water) was heated by stirring.

# W and Q are not State Functions

$$W_{1-2} = - \int_{V_1}^{V_2} P(T, V) dV$$

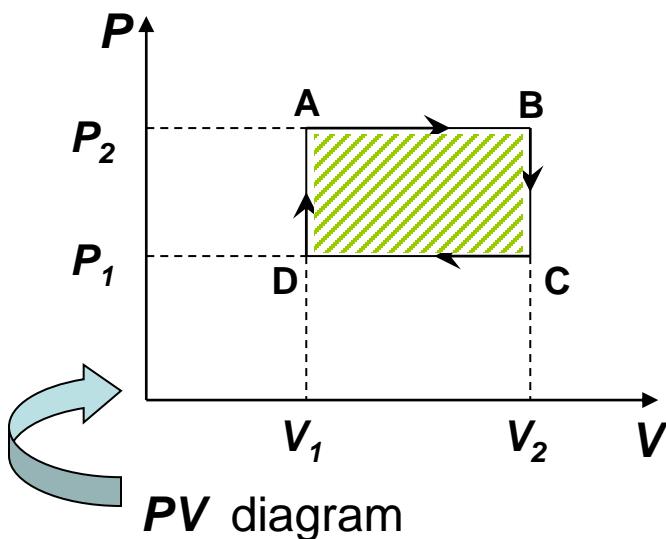
- we can bring the system from state 1 to state 2 along infinite # of paths, and for each path  $P(T, V)$  will be different.



Since **the work** done on a system depends not only on the **initial** and **final** states, but also on the **intermediate** states, it **is not a state function**.

$$\Delta U = Q + W$$

**U** is a state function, **W** - is not  $\Rightarrow$  thus, **Q is not a state function either**.



$$\begin{aligned} W_{net} &= W_{AB} + W_{CD} = -P_2(V_2 - V_1) - P_1(V_1 - V_2) \\ &= -(P_2 - P_1)(V_2 - V_1) < 0 \end{aligned}$$

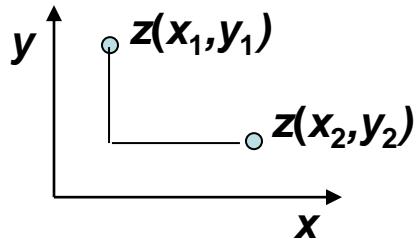
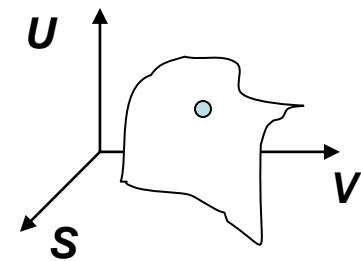
- the work is negative for the “clockwise” cycle; if the cyclic process were carried out in the reverse order (counterclockwise), the net work done on the gas would be positive.

# Comment on State Functions

$U$ ,  $P$ ,  $T$ , and  $V$  are the state functions,  $Q$  and  $W$  are not. Specifying an initial and final states of a system does not fix the values of  $Q$  and  $W$ , we need to know the whole process (the intermediate states). Analogy: in classical mechanics, if a force is not conservative (e.g., friction), the initial and final positions do not determine the work, the entire path must be specified.

In math terms,  $Q$  and  $W$  are not exact differentials of some functions of macroparameters. To emphasize that  $W$  and  $Q$  are NOT the state functions, we will use sometimes the curled symbols  $\delta$  (instead of  $d$ ) for their increments ( $\delta Q$  and  $\delta W$ ).

$$dU = T dS - P dV \quad \text{- an exact differential}$$



$dz = A_x(x, y)dx + A_y(x, y)dy$  - it is an exact differential if it is the difference between the values of some (state) function  $z(x, y)$  at these points:  $dz = z(x + dx, y + dy) - z(x, y)$

A necessary and sufficient condition for this:  $\frac{\partial A_x(x, y)}{\partial y} = \frac{\partial A_y(x, y)}{\partial x}$

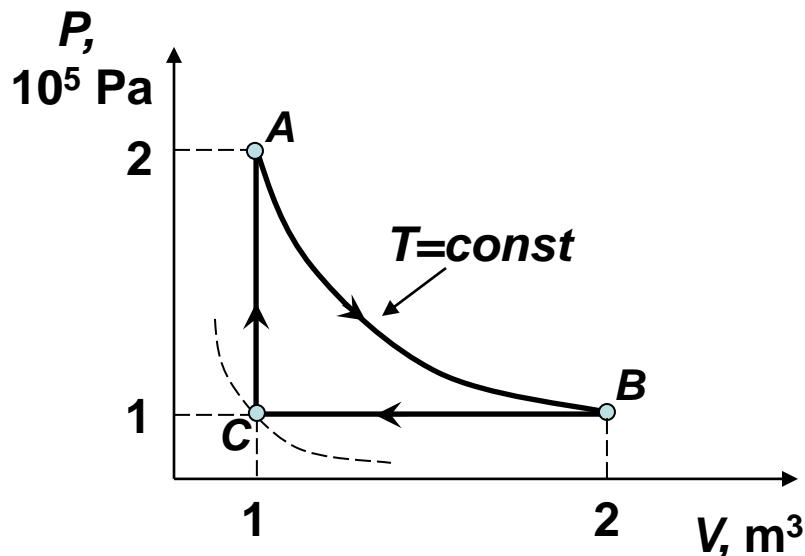
If this condition holds:

$$A_x(x, y) = \frac{\partial z(x, y)}{\partial x} \quad A_y(x, y) = \frac{\partial z(x, y)}{\partial y} \quad dz = \left( \frac{\partial z}{\partial x} \right)_y dx + \left( \frac{\partial z}{\partial y} \right)_x dy$$

e.g., for an ideal gas:  $\delta Q = dU + PdV = Nk_B \left( \frac{f}{2} dT + \frac{T}{V} dV \right)$  - cross derivatives are not equal

# Problem

Imagine that an ideal monatomic gas is taken from its initial state **A** to state **B** by an *isothermal* process, from **B** to **C** by an *isobaric* process, and from **C** back to its initial state **A** by an *isochoric* process. Fill in the signs of **Q**, **W**, and  $\Delta U$  for each step.

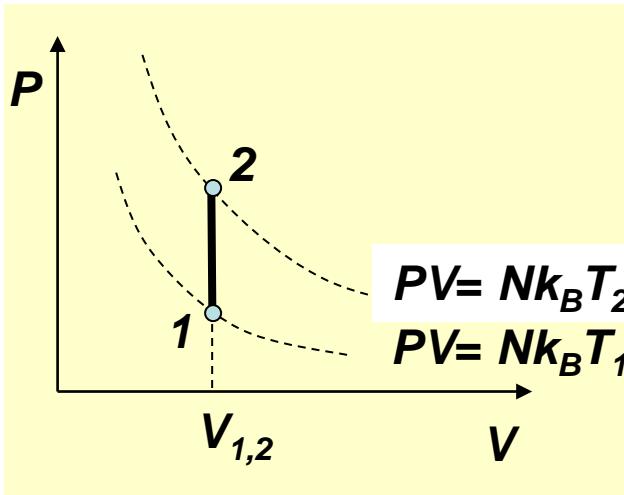


Step	<b>Q</b>	<b>W</b>	$\Delta U$
<b>A</b> $\rightarrow$ <b>B</b>	+	--	0
<b>B</b> $\rightarrow$ <b>C</b>	--	+	--
<b>C</b> $\rightarrow$ <b>A</b>	+	0	+

$$U = \frac{f}{2} N k_B T$$

$$PV = N k_B T$$

# Quasistatic Processes in an Ideal Gas



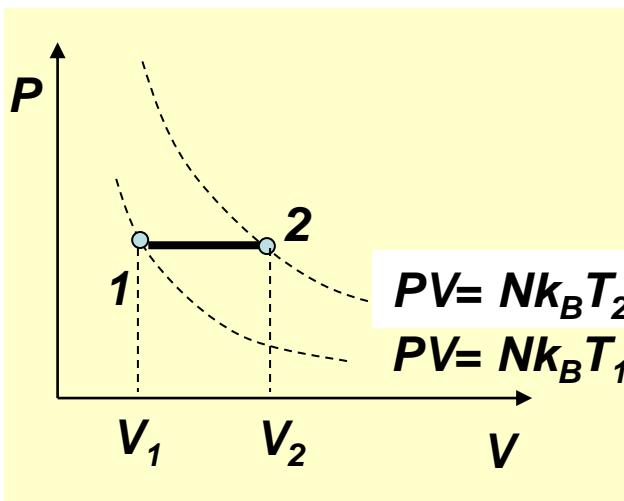
- isochoric ( $V = \text{const}$ )

$$W_{1 \rightarrow 2} = 0$$

$$Q_{1 \rightarrow 2} = \frac{3}{2} Nk_B (T_2 - T_1) > 0 \quad (= C_V \Delta T)$$

(see the last slide)

$$dU = Q_{1 \rightarrow 2}$$



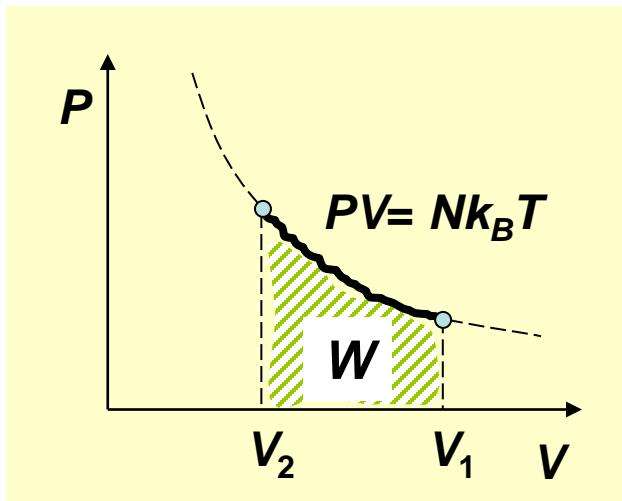
- isobaric ( $P = \text{const}$ )

$$W_{1 \rightarrow 2} = - \int_1^2 P(V, T) dV = -P(V_2 - V_1) < 0$$

$$Q_{1 \rightarrow 2} = \frac{5}{2} Nk_B (T_2 - T_1) > 0 \quad (= C_P \Delta T)$$

$$dU = W_{1 \rightarrow 2} + Q_{1 \rightarrow 2}$$

# Isothermal Process in an Ideal Gas



• isothermal (  $T = \text{const}$  ) :

$$dU = 0$$

$$W_{1 \rightarrow 2} = - \int_{V_1}^{V_2} P(V, T) dV = -Nk_B T \int_{V_1}^{V_2} \frac{dV}{V} = -Nk_B T \ln \frac{V_2}{V_1}$$

$$W_{i-f} = Nk_B T \ln \frac{V_i}{V_f}$$

$$Q_{1 \rightarrow 2} = -W_{1 \rightarrow 2}$$

$W_{i-f} > 0$  if  $V_i > V_f$  (compression)

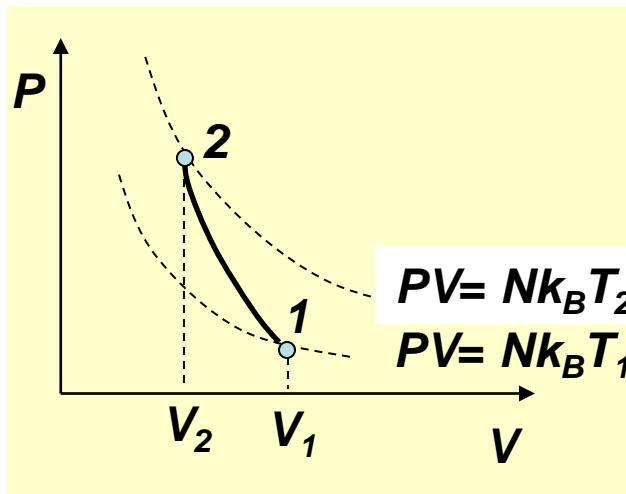
$W_{i-f} < 0$  if  $V_i < V_f$  (expansion)

# Adiabatic Process in an Ideal Gas

- adiabatic (*thermally isolated system*)

$$Q_{1 \rightarrow 2} = 0 \quad dU = W_{1 \rightarrow 2}$$

The amount of work needed to change the state of a thermally isolated system depends **only** on the *initial* and *final* states and not on the *intermediate* states.



$$W_{1 \rightarrow 2} = - \int_{V_1}^{V_2} P(V, T) dV$$

to calculate  $W_{1 \rightarrow 2}$ , we need to know  $P(V, T)$  for an adiabatic process

$$U = \frac{f}{2} Nk_B T \quad \Rightarrow \quad dU = \frac{f}{2} Nk_B dT = -PdV$$

(  $f$  – the # of “unfrozen” degrees of freedom )

$$PV = Nk_B T \quad \Rightarrow \quad PdV + VdP = Nk_B dT$$

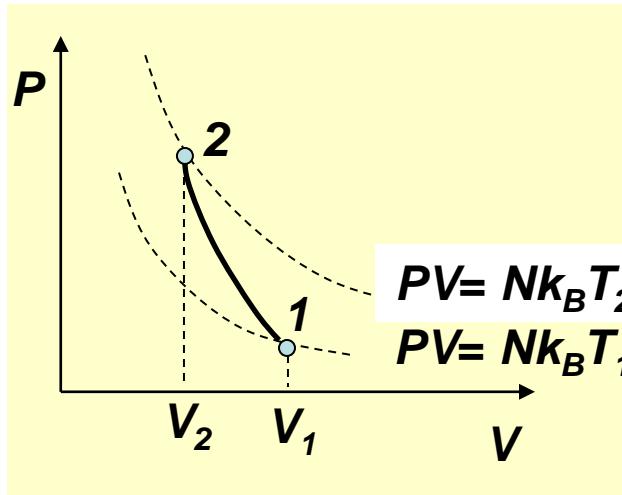
$$PdV + VdP = -\frac{2}{f} PdV \quad | \div PV$$

$$\frac{dV}{V} \left(1 + \frac{2}{f}\right) + \frac{dP}{P} = 0 \quad | \quad \int, \quad \gamma = 1 + \frac{2}{f} \quad \text{Adiabatic exponent}$$

$$\gamma \int_{V_1}^V \frac{dV}{V} + \int_{P_1}^P \frac{dP}{P} = 0$$

$$\ln \left( \frac{V}{V_1} \right)^\gamma = \ln \left( \frac{P_1}{P} \right) \Rightarrow \boxed{PV^\gamma = P_1 V_1^\gamma = \text{const}}$$

# Adiabatic Process in an Ideal Gas (cont.)



$$PV^\gamma = P_1 V_1^\gamma = \text{const}$$

An adiabata is “steeper” than an isotherma: in an adiabatic process, the work flowing out of the gas comes at the expense of its thermal energy  $\Rightarrow$  its temperature will decrease.

$$\begin{aligned}
 W_{1 \rightarrow 2} &= - \int_{V_1}^{V_2} P(V, T) dV = - \int_{V_1}^{V_2} \frac{P_1 V_1^\gamma}{V^\gamma} dV = - P_1 V_1^\gamma \frac{1}{-\gamma + 1} V^{-\gamma+1} \Big|_{V_1}^{V_2} \\
 &= P_1 V_1^\gamma \frac{1}{\gamma - 1} \left( \frac{1}{V_2^{\gamma-1}} - \frac{1}{V_1^{\gamma-1}} \right)
 \end{aligned}$$

$\gamma \Rightarrow 1+2/3 \approx 1.67$  (monatomic),  $1+2/5 = 1.4$  (diatomic),  $1+2/6 \approx 1.33$  (polyatomic)  
(again, neglecting the vibrational degrees of freedom)

Prove  $W_{1 \rightarrow 2} = \frac{f}{2} \Delta(PV) = \frac{f}{2} Nk_B \Delta T = \Delta U$

# Summary of quasi-static processes of ideal gases

$$\Delta U \equiv U_f - U_i$$

Quasi-Static process	$\Delta U$	$Q$	$W$	Ideal gas law
● isobaric ( $\Delta P=0$ )	$\Delta U = \frac{f}{2} N k_B \Delta T = \frac{f}{2} P \Delta V$	$\frac{f+2}{2} P \Delta V$	$-P \Delta V$	$\frac{V_i}{T_i} = \frac{V_f}{T_f}$
● isochoric ( $\Delta V=0$ )	$\Delta U = \frac{f}{2} N k_B \Delta T = \frac{f}{2} (\Delta P) V$	$\frac{f}{2} (\Delta P) V$	0	$\frac{P_i}{T_i} = \frac{P_f}{T_f}$
● isothermal ( $\Delta T=0$ )	0	$-W$	$-N k_B T \ln \frac{V_f}{V_i}$	$P_i V_i = P_f V_f$
● adiabatic ( $Q=0$ )	$\Delta U = \frac{f}{2} N k_B \Delta T = \frac{f}{2} \Delta(PV)$	0	$\Delta U$	$P_i V_i^\gamma = P_f V_f^\gamma$

# Problem

Imagine that we rapidly compress a sample of air whose initial pressure is  $10^5$  Pa and temperature is  $22^\circ\text{C}$  ( $= 295$  K) to a volume that is a quarter of its original volume (e.g., pumping bike's tire). What is its final temperature?

Rapid compression – approx. **adiabatic**, no time for the energy exchange with the environment due to thermal conductivity

$$\left. \begin{array}{l} P_1 V_1 = N k_B T_1 \\ P_2 V_2 = N k_B T_2 \\ P_1 V_1^\gamma = P_2 V_2^\gamma \end{array} \right\} \quad P_2 = \frac{P_1 V_1^\gamma}{V_2^{\gamma-1}} \quad \frac{P_1 V_1^\gamma}{V_2^{\gamma-1}} = N k_B T_2 = \frac{P_1 V_1}{T_1} T_2 \quad \Rightarrow \quad \left( \frac{V_1}{V_2} \right)^{\gamma-1} = \frac{T_2}{T_1}$$

For adiabatic processes:

$$T_1 V_1^{\gamma-1} = T_2 V_2^{\gamma-1} = \text{const}$$

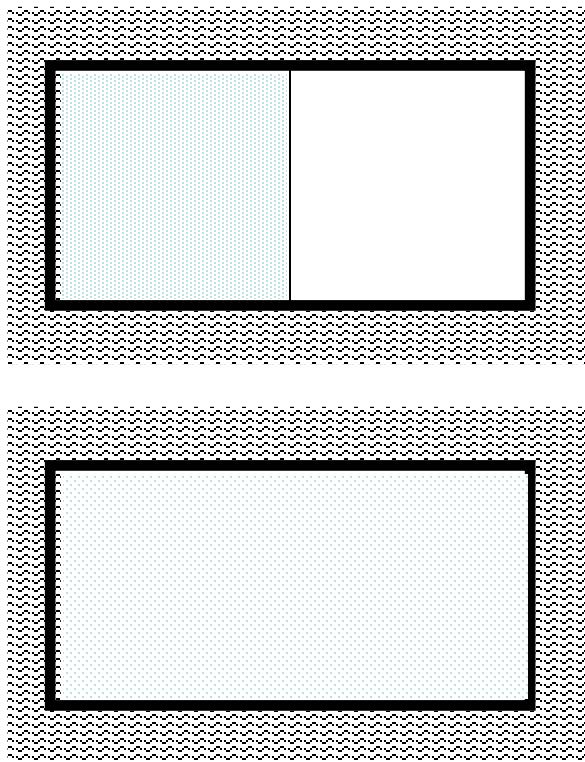
also  $P^{\gamma-1} / T^\gamma = \text{const}$

$$T_2 = T_1 \left( \frac{V_1}{V_2} \right)^{\gamma-1} = 295 \text{ K} \times 4^{0.4} \approx 295 \text{ K} \times 1.74 \approx 514 \text{ K}$$

- poor approx. for a bike pump, works better for diesel engines

# Non-equilibrium Adiabatic Processes

Free expansion



1.  $TV^{\gamma-1} = \text{const}$   $V$  – increases  
 $\Rightarrow T$  – decreases (cooling)
2. On the other hand,  $\Delta U = Q + W = 0$   
 $U \sim T \Rightarrow T$  – unchanged  
(agrees with experimental finding)

Contradiction – because approach #1 cannot be justified – violent expansion of gas *is not* a quasi-static process.  $T$  must remain the same.

$TV^{\gamma-1} = \text{const}$  - applies only to ***quasi-equilibrium*** processes !!!

# The Enthalpy

Isobaric processes ( $P = \text{const}$ ):

$$dU = Q - P\Delta V = Q - \Delta(PV) \Rightarrow Q = \Delta U + \Delta(PV)$$

$$\Rightarrow H \equiv U + PV - \text{the enthalpy}$$

The enthalpy is a **state function**, because  $U$ ,  $P$ , and  $V$  are state functions. In isobaric processes, the energy received by a system by heating equals to the change in enthalpy.

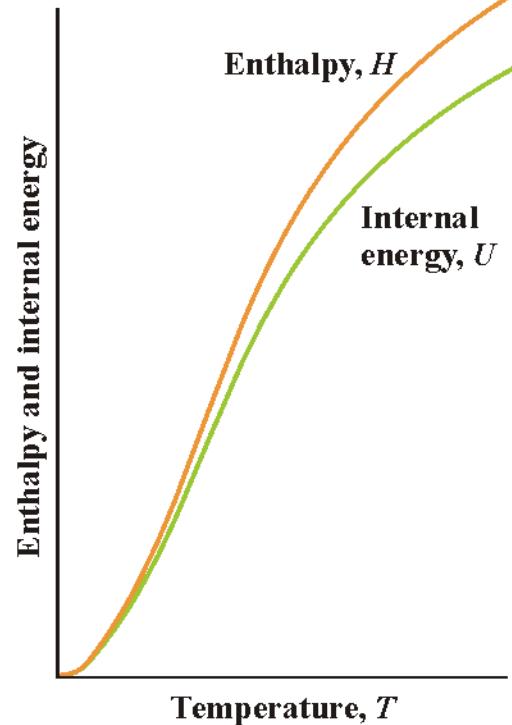
isochoric:

$$Q = \Delta U$$

isobaric:

$$Q = \Delta H$$

in both cases,  $Q$  does not depend on the path from 1 to 2.



**Consequence:** the energy released (absorbed) in chemical reactions at constant volume (pressure) depends only on the initial and final states of a system.

**The enthalpy of an ideal gas:**  $H = U + PV = \frac{f}{2} Nk_B T + Nk_B T = \left( \frac{f}{2} + 1 \right) Nk_B T$   
(depends on  $T$  only)

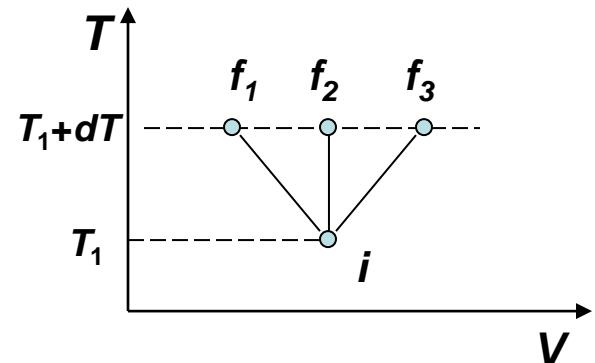
# Heat Capacity

**The heat capacity of a system** - the amount of energy transfer due to heating required to produce a unit temperature rise in that system

$$C \equiv \frac{\delta Q}{\Delta T}$$

**C** is NOT a state function (since **Q** is not a state function) – it depends on the path between two states of a system  $\Rightarrow$

( isothermic – **C** =  $\infty$ , adiabatic – **C** = 0 )



**The specific heat capacity**  $c \equiv \frac{C}{m}$

# $C_V$ and $C_P$

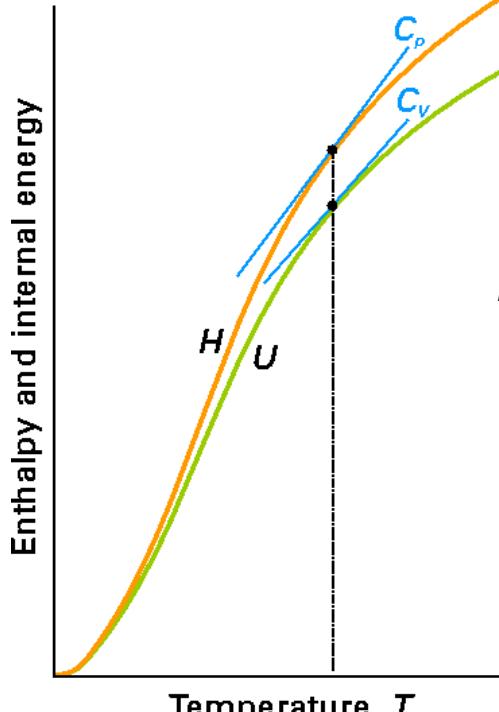
$$C = \frac{\delta Q}{dT} = \frac{dU + PdV}{dT}$$

$$V = \text{const} \rightarrow C_V = \left( \frac{\partial U}{\partial T} \right)_V$$

$$P = \text{const} \rightarrow C_P = \left( \frac{\partial H}{\partial T} \right)_P$$

**the heat capacity at constant volume**

**the heat capacity at constant pressure**



To find  $C_P$  and  $C_V$ , we need  $f(P, V, T) = 0$  and  $U = U(V, T)$

**For an ideal gas**  $U = \frac{f}{2} Nk_B T$   $H = \left( \frac{f}{2} + 1 \right) Nk_B T$

$$C_V = \frac{f}{2} Nk_B = \frac{f}{2} nR$$

↑  
# of moles

$$C_P = \left( \frac{f}{2} + 1 \right) nR$$

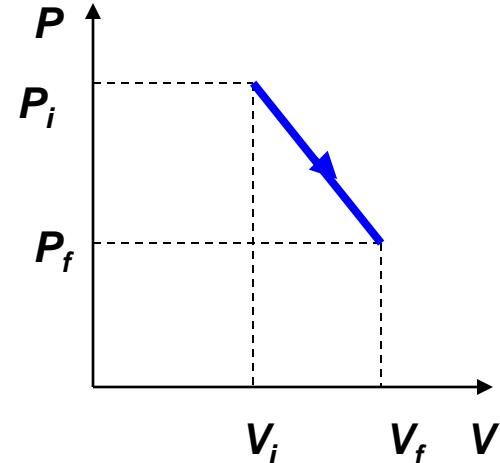
For one mole of a monatomic ideal gas:

$$C_V = \frac{3}{2} R \quad C_P = \frac{5}{2} R$$

# Another Problem

During the ascent of a meteorological helium-gas filled balloon, its volume increases from  $V_i = 1 \text{ m}^3$  to  $V_f = 1.8 \text{ m}^3$ , and the pressure inside the balloon decreases from 1 bar ( $=10^5 \text{ N/m}^2$ ) to 0.5 bar. Assume that the pressure changes linearly with volume between  $V_i$  and  $V_f$ .

- (a) If the initial  $T$  is 300K, what is the final  $T$ ?
- (b) How much work is done **by** the gas in the balloon?
- (c) How much “heat” does the gas absorb, if any?



$$P(V) = -0.625 \text{ bar/m}^3 \times V + 1.625 \text{ bar}$$

$$(a) \quad PV = Nk_B T \quad T = \frac{PV}{Nk_B} \quad T_f = T_i \frac{P_f V_f}{P_i V_i} = 300 \text{ K} \frac{0.5 \text{ bar} \times 1.8 \text{ m}^3}{1 \text{ bar} \times 1 \text{ m}^3} = 270 \text{ K}$$

$$(b) \quad \delta W_{ON} = - \int_{V_i}^{V_f} P(V) dV \quad \text{- work done on a system} \quad \delta W_{BY} = \int_{V_i}^{V_f} P(V) dV \quad \text{- work done by a system}$$

$$\boxed{\delta W_{ON} = -\delta W_{BY}} \quad \delta W_{BY} = \int_{V_i}^{V_f} P(V) dV = (0.5 \times 0.8 \text{ bar} \cdot \text{m}^3 + 0.5 \times 0.4 \text{ bar} \cdot \text{m}^3) = 0.6 \text{ bar} \cdot \text{m}^3 = 6 \cdot 10^4 \text{ J}$$

$$(c) \quad \Delta U = \delta Q + \delta W_{ON}$$

$$\delta Q = \Delta U - \delta W_{ON} = \frac{3}{2} Nk_B (T_f - T_i) - W_{ON} = \frac{3}{2} P_i V_i \left( \frac{T_f}{T_i} - 1 \right) + \delta W_{BY} = 1.5 \cdot 10^5 \text{ J} \times (-0.1) + 6 \cdot 10^4 \text{ J} = 4.5 \cdot 10^4 \text{ J}$$

# Worksheet time!

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Download Worksheet 3 from the Moodle page and work on it.