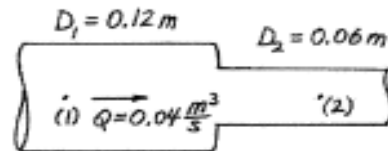


Series 12 - Solutions

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8.58

8.58 Water flows at a rate of $0.040 \text{ m}^3/\text{s}$ in a 0.12-m -diameter pipe that contains a sudden contraction to a 0.06-m -diameter pipe. Determine the pressure drop across the contraction section. How much of this pressure difference is due to losses and how much is due to kinetic energy changes?



$$\frac{p_1}{\rho} + \frac{V_1^2}{2g} + Z_1 = \frac{p_2}{\rho} + \frac{V_2^2}{2g} + Z_2 + K_L \frac{V_2^2}{2g}, \text{ where } Z_1 = Z_2 \quad (1)$$

and

$$V_1 = \frac{Q}{A_1} = \frac{0.04 \frac{\text{m}^3}{\text{s}}}{\frac{\pi}{4}(0.12\text{m})^2} = 3.54 \frac{\text{m}}{\text{s}}, \quad V_2 = \frac{Q}{A_2} = \frac{0.04 \frac{\text{m}^3}{\text{s}}}{\frac{\pi}{4}(0.06\text{m})^2} = 14.1 \frac{\text{m}}{\text{s}}$$

Thus, with $\frac{A_2}{A_1} = \left(\frac{D_2}{D_1}\right)^2 = \left(\frac{0.06\text{m}}{0.12\text{m}}\right)^2 = 0.25$ we obtain from Fig. 8.30

$$K_L = 0.40$$

Hence, from Eq. (1)

$$p_1 - p_2 = \frac{1}{2} \rho \left[K_L V_2^2 + V_2^2 - V_1^2 \right] = \frac{1}{2} (999 \frac{\text{kg}}{\text{m}^3}) \left[0.40 (14.1 \frac{\text{m}}{\text{s}})^2 + (14.1 \frac{\text{m}}{\text{s}})^2 - (3.54 \frac{\text{m}}{\text{s}})^2 \right]$$

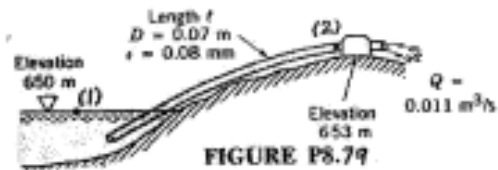
or

$$p_1 - p_2 = 39.7 \times 10^3 \frac{\text{N}}{\text{m}^2} + 93.0 \times 10^3 \frac{\text{N}}{\text{m}^2} = \underline{\underline{133 \text{ kPa}}}$$

This represents a 39.7 kPa drop from losses and a 93.0 kPa drop due to an increase in kinetic energy.

8.79

8.79 Water at 10 °C is pumped from a lake as shown in Fig. P8.79. If the flowrate is 0.011 m³/s, what is the maximum length inlet pipe, ℓ , that can be used without cavitation occurring?



$$\frac{p_1}{\rho} + \frac{V_1^2}{2g} + z_1 = \frac{p_2}{\rho} + \frac{V_2^2}{2g} + z_2 + \left(f \frac{\ell}{D} + \sum K_L\right) \frac{V^2}{2g}, \text{ where } p_1 = 101 \text{ kPa}, z_1 = 650 \text{ m} \quad (1)$$

$V_1 = 0$, $V_2 = V$, $z_2 = 653 \text{ m}$, and from Table B.2 $p_2 = p_v = 1.228 \text{ kPa}$

$$\text{Also, } V = \frac{Q}{A} = \frac{0.011 \frac{\text{m}^3}{\text{s}}}{\frac{\pi}{4} (0.07 \text{ m})^2} = 2.86 \frac{\text{m}}{\text{s}} \text{ so that}$$

$$Re = \frac{VD}{\nu} = \frac{(2.86 \frac{\text{m}}{\text{s}})(0.07 \text{ m})}{1.307 \times 10^{-6} \frac{\text{m}^2}{\text{s}}} = 1.53 \times 10^5. \text{ With this } Re \text{ and from Table 8.1 with}$$

$$\frac{\epsilon}{D} = \frac{0.08 \text{ mm}}{70 \text{ mm}} = 0.00114 \text{ we obtain } f = 0.0216 \text{ (see Fig. 8.20)}$$

Hence, with $\sum K_L = 0.8$ for the entrance, Eq. (1) becomes

$$\frac{(101 - 1.228) \times 10^3 \frac{\text{N}}{\text{m}^2}}{9.80 \times 10^3 \frac{\text{N}}{\text{m}^3}} + 650 \text{ m} = 653 \text{ m} + \left(1 + (0.0216) \left(\frac{\ell}{0.07 \text{ m}}\right) + 0.8\right) \frac{(2.86 \frac{\text{m}}{\text{s}})^2}{2(9.81 \frac{\text{m}}{\text{s}^2})}$$

$$\text{or } \ell = \underline{50.0 \text{ m}}$$

8.88

8.88 A fan is to produce a constant air speed of 40 m/s throughout the pipe loop shown in Fig. P8.88. The 3-m-diameter pipes are smooth, and each of the four 90-degree elbows has a loss coefficient of 0.30. Determine the power that the fan adds to the air.

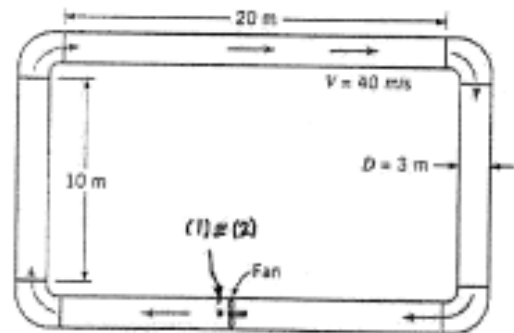


FIGURE P8.88

$$\frac{p_1}{\rho} + Z_1 + \frac{V_1^2}{2g} - h_L + h_s = \frac{p_2}{\rho} + Z_2 + \frac{V_2^2}{2g}$$

If we locate (1) and (2) at the same place it follows that

$$p_1 = p_2, V_1 = V_2, \text{ and } Z_1 = Z_2.$$

Thus,

$$h_s = h_L = \left(f \frac{L}{D} + \sum K_{L_i} \right) \frac{V^2}{2g} \quad \text{where } \sum K_{L_i} = 4(0.30) = 1.2$$

$$\text{Also, } Re = \frac{\rho V D}{\mu} = \frac{1.23 \frac{\text{kg}}{\text{m}^3} (40 \frac{\text{m}}{\text{s}}) (3 \text{ m})}{1.79 \times 10^{-5} \frac{\text{N} \cdot \text{s}}{\text{m}^2}} = 8.25 \times 10^6$$

$$\text{and } \frac{\epsilon}{D} = 0 \text{ so that from Fig. 8.20, } f = 0.0083$$

Hence,

$$h_s = \left(0.0083 \frac{(20+20+10+10) \text{ m}}{3 \text{ m}} + 1.2 \right) \frac{(40 \frac{\text{m}}{\text{s}})^2}{2(9.81 \frac{\text{m}}{\text{s}^2})} = 111 \text{ m}$$

so that

$$\begin{aligned} \dot{W}_s &= \gamma Q h_s = \rho g Q h_s = (1.23 \frac{\text{kg}}{\text{m}^3}) (9.81 \frac{\text{m}}{\text{s}^2}) \left[\frac{\pi}{4} (3 \text{ m})^2 (40 \frac{\text{m}}{\text{s}}) \right] 111 \text{ m} \\ &= 3.79 \times 10^5 \frac{\text{N} \cdot \text{m}}{\text{s}} = \underline{\underline{379 \text{ kW}}} \end{aligned}$$

8.90

8.90 Water flows from the nozzle attached to the spray tank shown in Fig. P8.90. Determine the flowrate if the loss coefficient for the nozzle (based on upstream conditions) is 0.75 and the friction factor for the rough hose is 0.11.

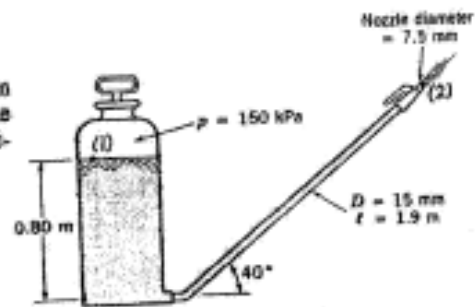


FIGURE P8.90

$$\frac{p_1}{\rho} + \frac{V_1^2}{2g} + z_1 = \frac{p_2}{\rho} + \frac{V_2^2}{2g} + z_2 + \left(f \frac{l}{D} + K_L\right) \frac{V^2}{2g}, \text{ where } p_1 = 150 \text{ kPa}, p_2 = 0, \quad (1)$$

$$z_1 = 0.80 \text{ m}, z_2 = l \sin 40^\circ = (1.9 \text{ m}) \sin 40^\circ = 1.22 \text{ m}, V_1 = 0,$$

$$V = \frac{Q}{A}, \text{ and } V_2 = \frac{Q}{A_2} = \left(\frac{A}{A_2}\right)V = \left(\frac{D}{D_2}\right)^2 V = \left(\frac{15 \text{ mm}}{7.5 \text{ mm}}\right)^2 V = 4V$$

Thus, with $f = 0.11$ and $K_L = 0.75$ Eq. (1) gives

$$\frac{150 \times 10^3 \frac{\text{N}}{\text{m}^2}}{9800 \frac{\text{N}}{\text{m}^3}} + 0.80 \text{ m} = 1.22 \text{ m} + \left(4^2 + 0.11 \left(\frac{1.9 \text{ m}}{0.015 \text{ m}}\right) + 0.75\right) \frac{V^2}{2(9.81 \frac{\text{m}}{\text{s}^2})}$$

or

$$V = 3.09 \frac{\text{m}}{\text{s}}$$

$$\text{Thus, } Q = AV = \frac{\pi}{4} (0.015 \text{ m})^2 (3.09 \frac{\text{m}}{\text{s}}) = \underline{\underline{5.46 \times 10^{-4} \frac{\text{m}^3}{\text{s}}}}$$

8-99

8.94

8.94 The pump shown in Fig. P8.94 adds 25 kW to the water and causes a flowrate of $0.04 \text{ m}^3/\text{s}$. Determine the flowrate expected if the pump is removed from the system. Assume $f = 0.016$ for either case and neglect minor losses.

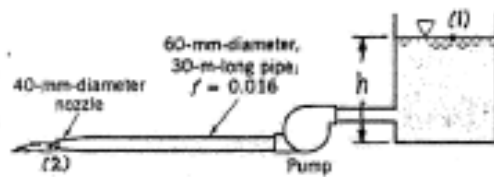


FIGURE P8.94

$$\frac{p_1}{\rho} + \frac{V_1^2}{2g} + z_1 + h_p = \frac{p_2}{\rho} + \frac{V_2^2}{2g} + z_2 + f \frac{L}{D} \frac{V^2}{2g}, \text{ where } p_1 = p_2 = 0, z_1 = h, z_2 = 0,$$

$$V_1 = 0, V_2 = \frac{Q}{A_2} = \frac{0.04 \frac{\text{m}^3}{\text{s}}}{\frac{\pi}{4}(0.04 \text{ m})^2} = 31.8 \frac{\text{m}}{\text{s}}, V = \frac{Q}{A} = \frac{0.04 \frac{\text{m}^3}{\text{s}}}{\frac{\pi}{4}(0.06 \text{ m})^2} = 14.15 \frac{\text{m}}{\text{s}}$$

Thus,

$$h + h_p = \frac{(31.8 \frac{\text{m}}{\text{s}})^2}{2(9.81 \frac{\text{m}}{\text{s}^2})} + 0.016 \left(\frac{30 \text{ m}}{0.06 \text{ m}} \right) \frac{(14.15 \frac{\text{m}}{\text{s}})^2}{2(9.81 \frac{\text{m}}{\text{s}^2})} = 133.2 \text{ m}$$

but,

$$h_p = \frac{P}{\rho Q} = \frac{25 \times 10^3 \frac{\text{N} \cdot \text{m}}{\text{s}}}{(980 \times 10^3 \frac{\text{N}}{\text{m}^3})(0.04 \frac{\text{m}^3}{\text{s}})} = 63.8 \text{ m}$$

Hence,

$$h = 133.2 \text{ m} - 63.8 \text{ m} = 69.5 \text{ m}$$

Without the pump $h_p = 0$ and $z_1 = \frac{V_2^2}{2g} + f \frac{L}{D} \frac{V^2}{2g}$ where $h = 69.5 \text{ m} = z_1$,

and

$$V_2 = \frac{AV}{A_2} = \left(\frac{D}{D_2} \right)^2 V \text{ or } V_2 = \left(\frac{60 \text{ mm}}{40 \text{ mm}} \right)^2 V = 2.25 V$$

Thus,

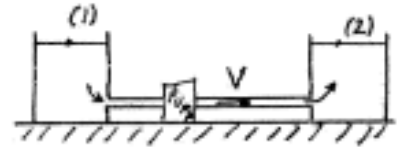
$$69.5 \text{ m} = \frac{(2.25 V)^2 + 0.016 \left(\frac{30 \text{ m}}{0.06 \text{ m}} \right) V^2}{2(9.81 \frac{\text{m}}{\text{s}^2})} \text{ or } V = 10.22 \frac{\text{m}}{\text{s}}$$

so that

$$Q = AV = \frac{\pi}{4} (0.06 \text{ m})^2 (10.22 \frac{\text{m}}{\text{s}}) = \underline{\underline{0.0289 \frac{\text{m}^3}{\text{s}}}}$$

8.101

8.101 Water is pumped between two large open reservoirs through 1.5 km of smooth pipe. The water surfaces in the two reservoirs are at the same elevation. When the pump adds 20 kW to the water the flowrate is 1 m³/s. If minor losses are negligible, determine the pipe diameter.



$$\frac{P_1}{\rho} + z_1 + \frac{V_1^2}{2g} + h_s - h_L = \frac{P_2}{\rho} + z_2 + \frac{V_2^2}{2g}, \text{ where } P_1 = P_2 = 0, V_1 = V_2 = 0, z_1 = z_2$$

Thus,

$$(1) \quad h_s = h_L \quad \text{where} \quad h_s = \frac{\dot{W}_s}{\rho Q} = \frac{20 \times 10^3 \text{ N}\cdot\text{m/s}}{(9.80 \times 10^3 \frac{\text{N}}{\text{m}^3})(1 \frac{\text{m}^3}{\text{s}})} = 2.04 \text{ m}$$

and

$$h_L = f \frac{L}{D} \frac{V^2}{2g} \quad \text{with} \quad V = \frac{Q}{A} = \frac{1 \text{ m}^3/\text{s}}{\frac{\pi}{4} D^2} = \frac{1.273}{D^2} \text{ m/s with } D \text{ in m}$$

Hence,

$$(2) \quad h_L = f \frac{1.5 \times 10^3 \text{ m}}{D} \frac{(1.273/D^2)^2 \text{ m}^2/\text{s}^2}{2(9.81 \text{ m/s}^2)} = 123.9 f / D^5 \text{ m}$$

$$\text{From Eqs (1) and (2), } 2.04 = 123.9 f / D^5 \text{ or } f = 0.0165 D^5$$

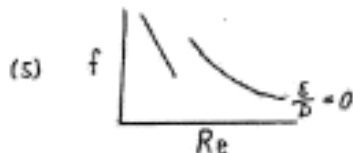
$$(3) \quad D = 2.27 f^{1/5}$$

Also,

$$Re = \frac{\rho V D}{\mu} = \frac{999 \frac{\text{kg}}{\text{m}^3} (1.273/D^2) \text{ m} D}{1.12 \times 10^{-3} \frac{\text{N}\cdot\text{s}}{\text{m}^2}} \quad \text{or}$$

$$(4) \quad Re = 1.14 \times 10^6 / D$$

Finally, with $\epsilon/D = 0$ the Moody chart (Fig. 8.20) is the final equation.



Trial and error solution of Eqs. (3), (4), and (5) for f , Re , and D :

Assume $f = 0.02$ so Eq (3) gives $D = 2.27 (0.02)^{1/5} = 1.04 \text{ m}$ and Eq (4) gives $Re = 1.14 \times 10^6 / 1.04 = 1.10 \times 10^6$. Thus, from Eq (5), $f = 0.0115$ which is not equal to the assumed $f = 0.02$. Try again with $f = 0.0115$ which gives $D = 0.931 \text{ m}$, $Re = 1.22 \times 10^6$, and $f = 0.0113 \neq 0.0115$. One final try with $f = 0.0113$ gives $D = 0.927 \text{ m}$, $Re = 1.23 \times 10^6$, and $f = 0.0113$ as assumed. Thus, $D = 0.927 \text{ m}$.

An alternate method is to use the Colebrook formula (Eq (8.35)) rather than the Moody chart (Eq (5)). Thus, with $\epsilon/D = 0$,

(con't)

8.101 (con't)

Eq(8.35) is

$$\frac{1}{\sqrt{f}} = -2.0 \log \left(\frac{2.51}{Re \sqrt{f}} \right)$$
 which, when combined with Eqs. (3) and (4), gives

$$(6) \quad \frac{1}{(0.0165 D^5)^{1/2}} = -2.0 \log \left[\frac{2.51 D}{1.14 \times 10^6 (0.0165 D^5)^{1/2}} \right]$$

Using a computer root-finding program to solve Eq (6) gives
 $D = 0.926$, which is consistent with the trial and error solution
 given above.