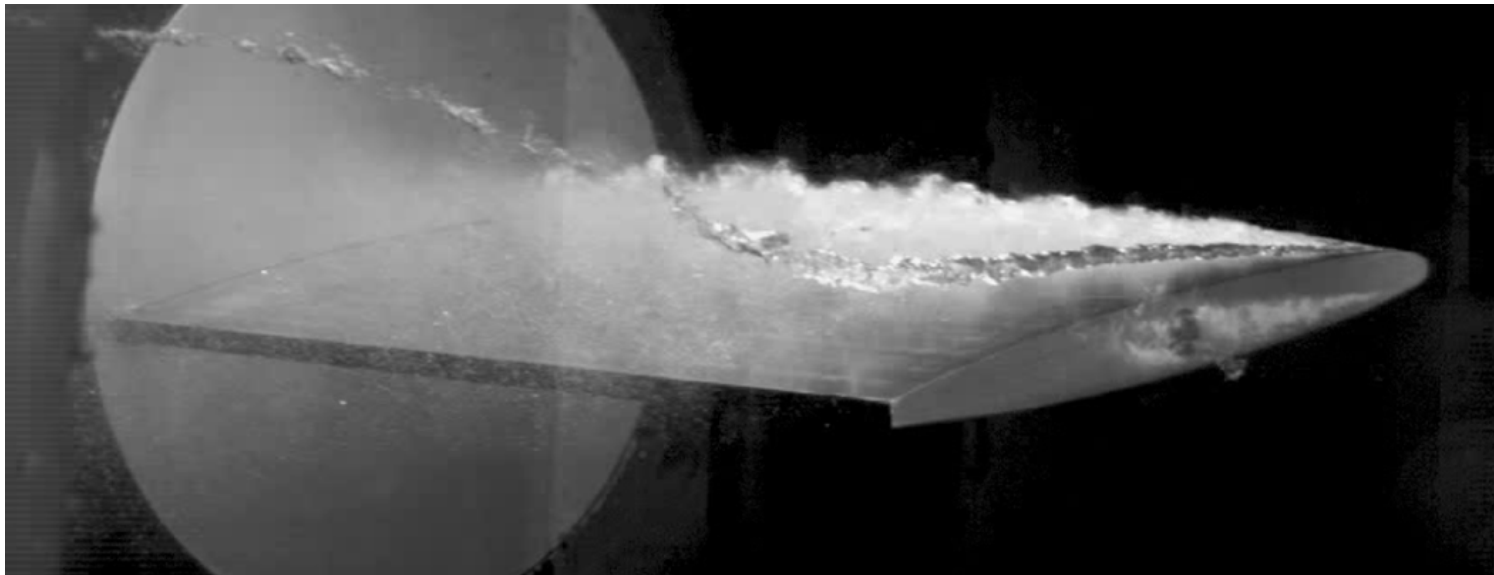


ECOLE POLYTECHNIQUE FEDERALE DE LAUSANNE
SGM - 6th & 8th Semester, Fall 2025
CAVITATION AND INTERFACE PHENOMENA

Chapter 5: Vortex Cavitation
5.4 : Tip Leakage Vortex Cavitation

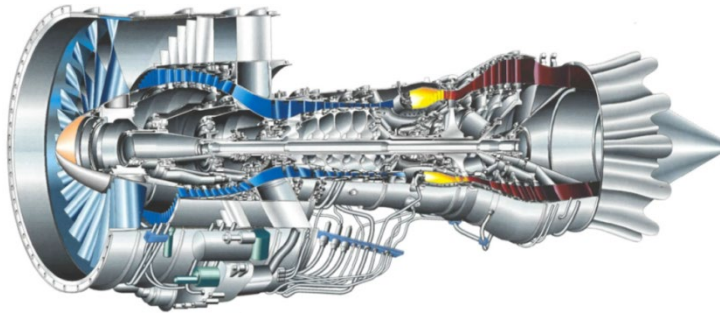


Dr Mohamed FARHAT Assistants: Th. Berger, R. Fuzzati
EPFL – Cavitation Research Group, Avenue de Cour 33 bis, 1007 Lausanne

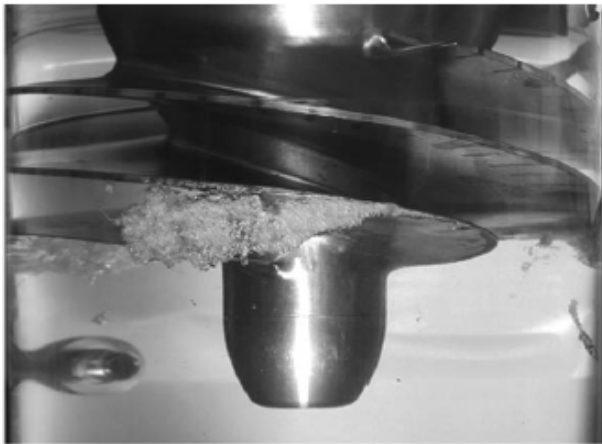
On The Tip Leakage Vortex (TLV)

- *TLV may develop at the tip of the blades of any ducted rotating impeller*
 - *The flow leaks and rolls up from pressure to suction sides through the gap*

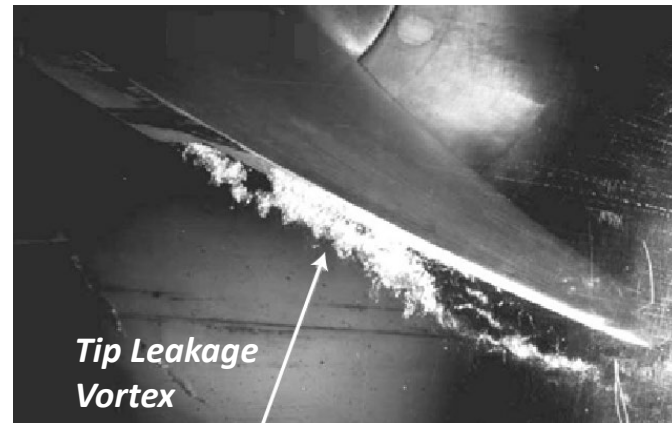
- *Turbo jet*



- *Space rocket inducers*

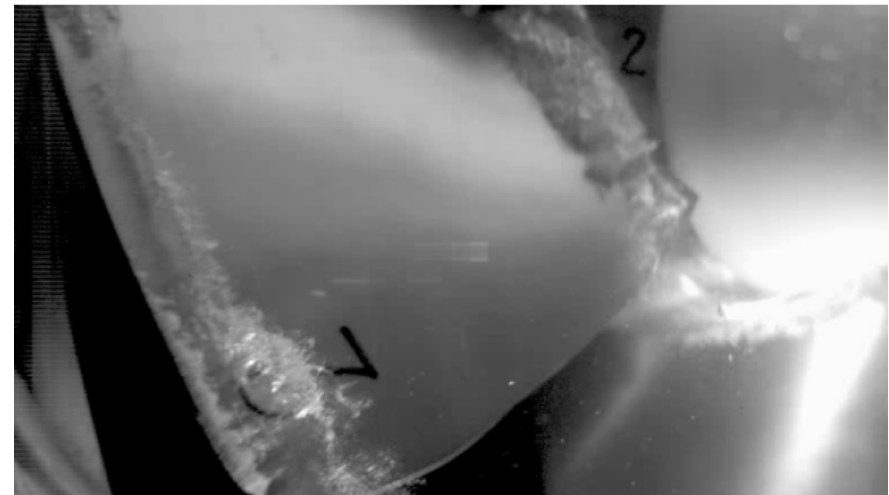
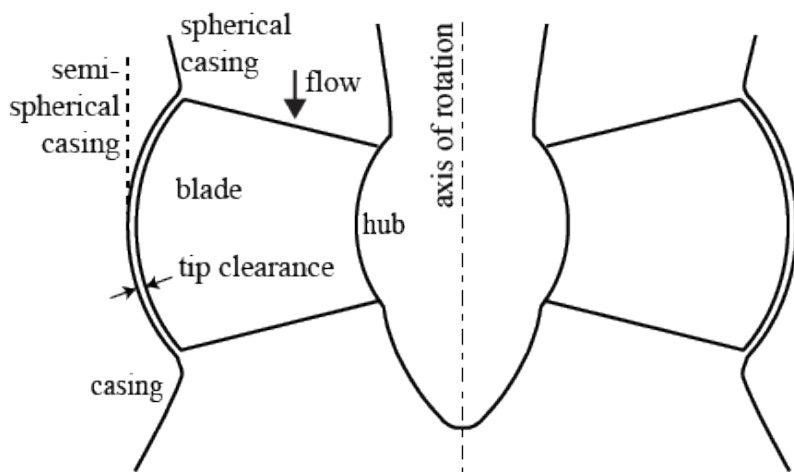


- *Axial turbines*



Tip leakage vortex in hydraulic turbines

- **Tip leakage vortex (TLV) cavitation** → **severe erosion in axial turbines**
 - **Cavitation highly dependent on gas content, Reynolds Nb and gap width**
- **TLV cavitation prediction?**
 - **Actual Numerical simulations are not reliable**
 - **Model tests:**
 - **Scale up rules are insufficient and may lead to wrong predictions.**
 - **Similarity of (i) gas content, (ii) Reynolds number and (iii) gap width impossible to achieve**



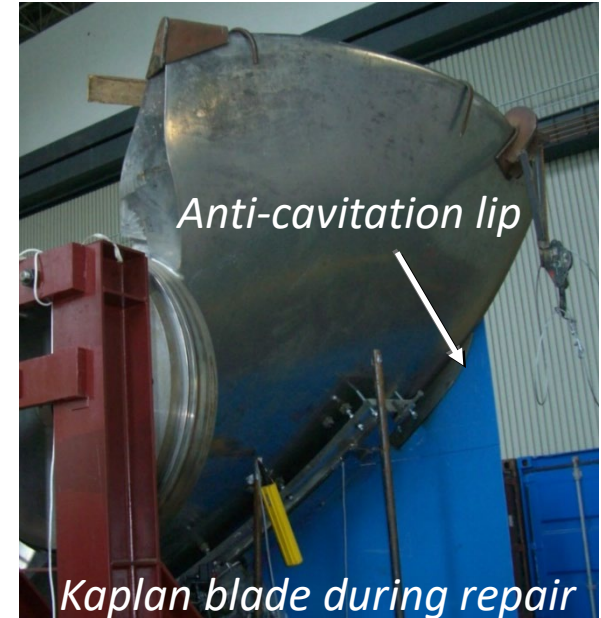
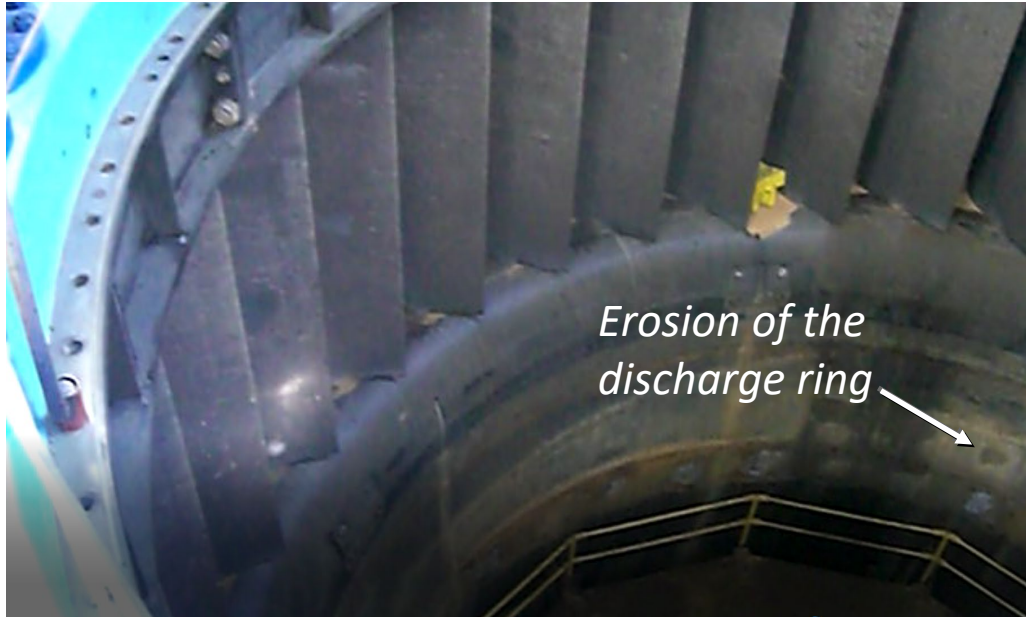
Tip leakage vortex in hydraulic turbines

- ***The case of “IRON GATES” Power Plant***
 - ***On the Danube river, on the Romanian-Serbian border***
 - ***6 x Kaplan units: 200 MW/unit, 35 m water head, 9.5 m runner diameter***
 - ***2nd world largest Kaplan turbines***
 - ***Severe cavitation erosion on the prototype***
 - ***No cavitation observed during model tests !***



Tip leakage vortex in hydraulic turbines

- *The case of “IRON GATES” Power Plant*



- *Severe erosion of the blades tip & the discharge ring (TLV cavitation)*
 - *Periodic repairs (increased operational cost)*
 - *Remedy:*
 - *Coating of exposed zones with more resistant material (e.g. Stellite)*
 - *Removable anti-cavitation lip*
 - *Search for more viable solution still underway*

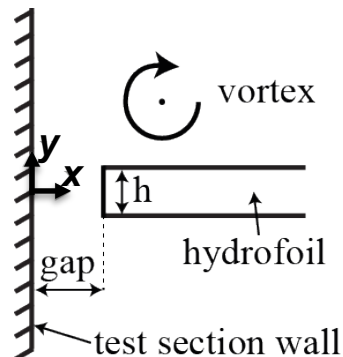
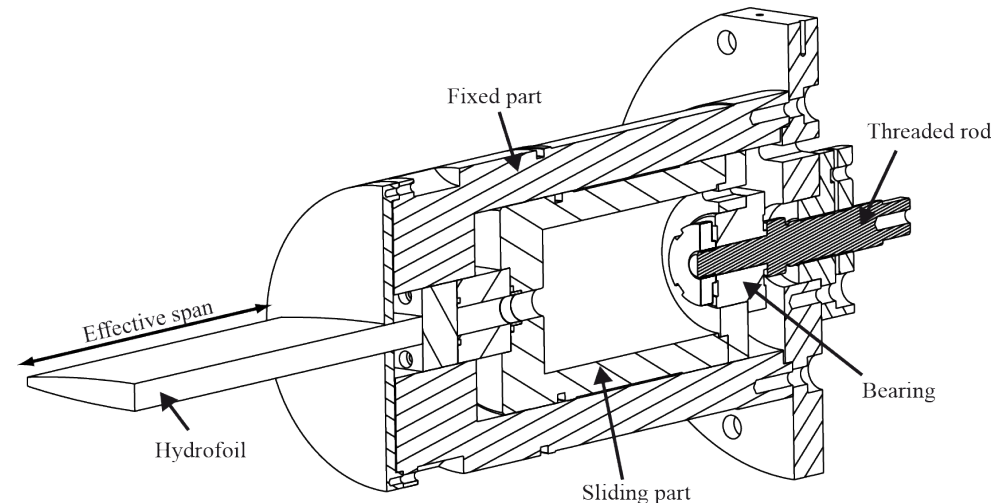
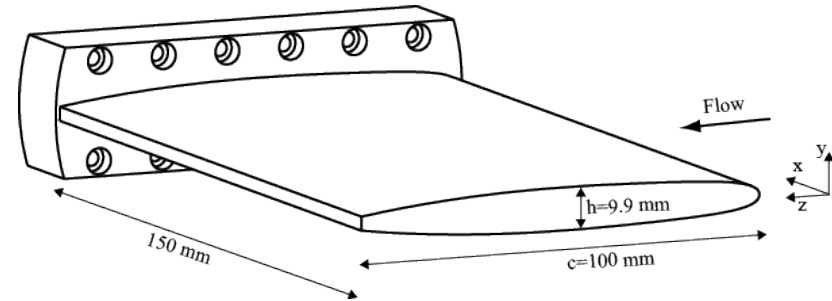
Tip Leakage Vortex Cavitation

- **Current procedure for reduced scale model testing of axial turbines**
 - **IEC 60193 standard**
 - **Geometrical similarity of the impeller**
 - **Full-scale clearance must not exceed the one of reduced scale model**
 - **Geometrical similarity is hard to satisfy in practice**
 - **The gap is measured in dewatered and still conditions**
 - **no direct measurement during turbine operation**
 - **Gap width variations due to pressure and thermal stresses**
- **Open Questions**
 - **Effect of gap width on the TLV ?**
 - **Control of the TLV in hydraulic turbines ?**



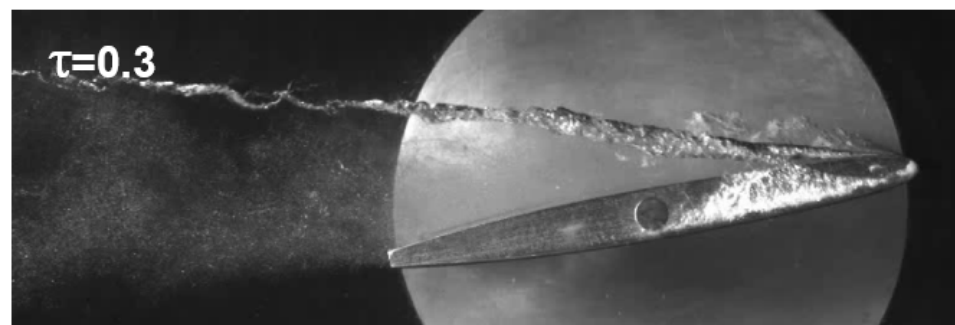
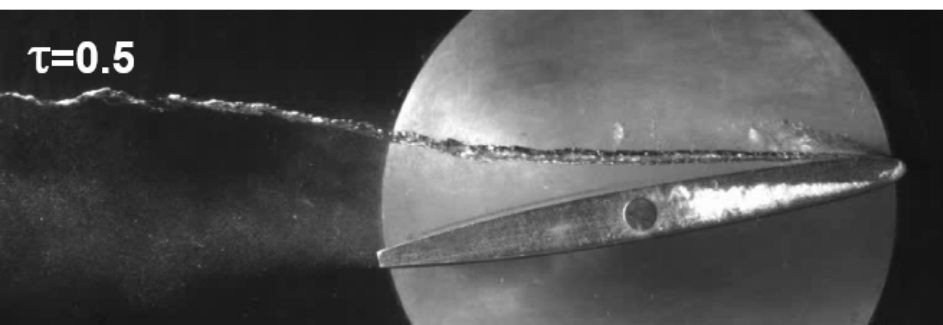
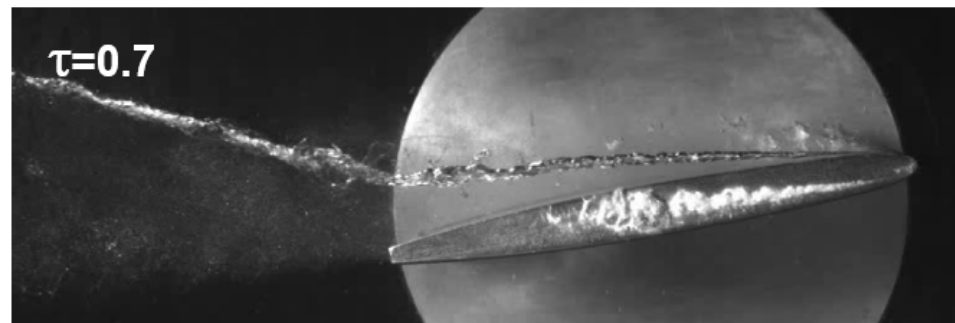
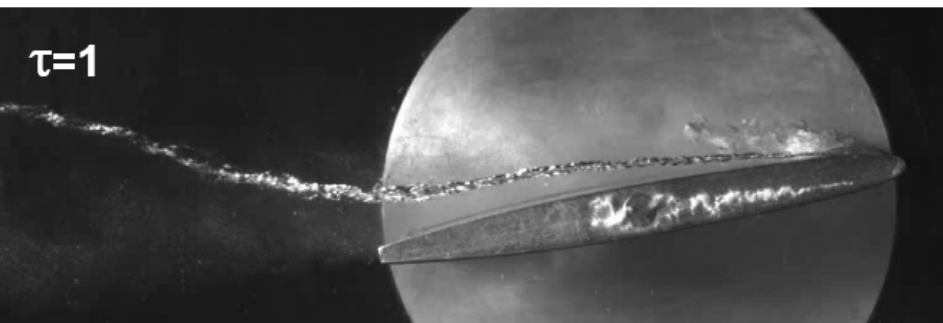
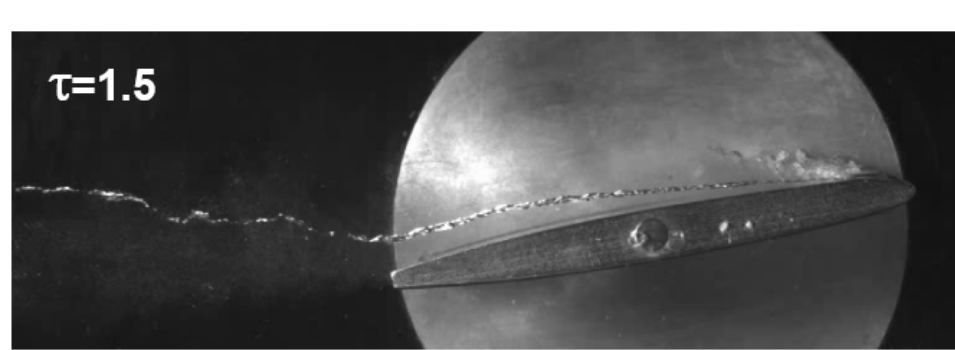
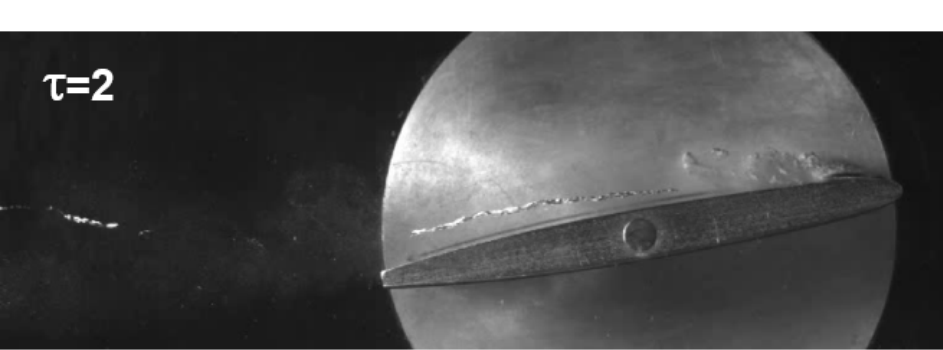
Effect of gap width on TLV: A Simplified Case Study

- High speed cavitation tunnel
- NACA0009 hydrofoil
 - *Smooth and roughened leading edge (natural and tripped transition of boundary layer to turbulence)*
- Sliding support of the hydrofoil
- Velocity measurements
- High-speed flow visualizations



normalized tip clearance:

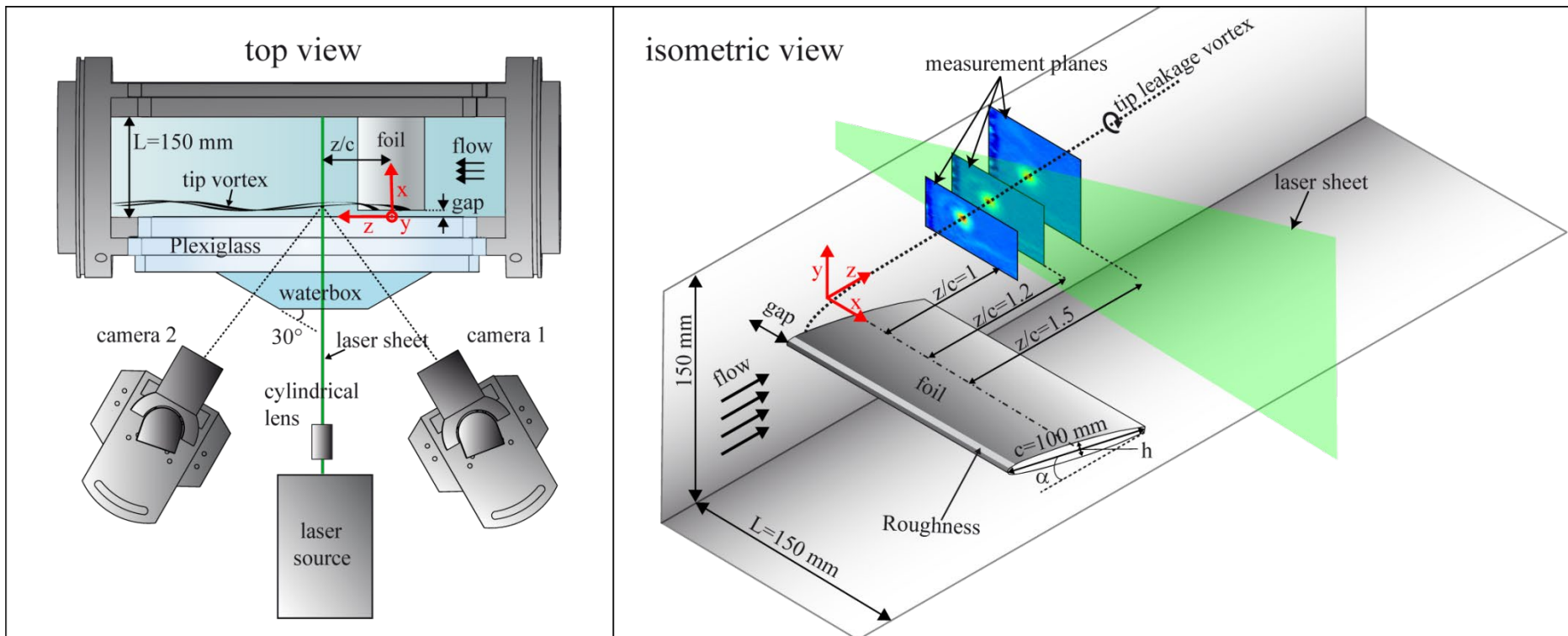
$$\tau = \text{gap} / h$$



Effect of gap width on the TLV

- Stereo PIV (SPIV)

- Non-intrusive measurement of 3 components of the velocity in 3 planes ($z/c = 1, 1.2$ and 1.5)
- To minimize optical distortions, a water-filled box is fitted to the test section to minimize optical distortions



- **PIV - Particle Image Velocimetry: working principle**
 - Divide images in interrogation windows (e.g. 32x32 pixels)



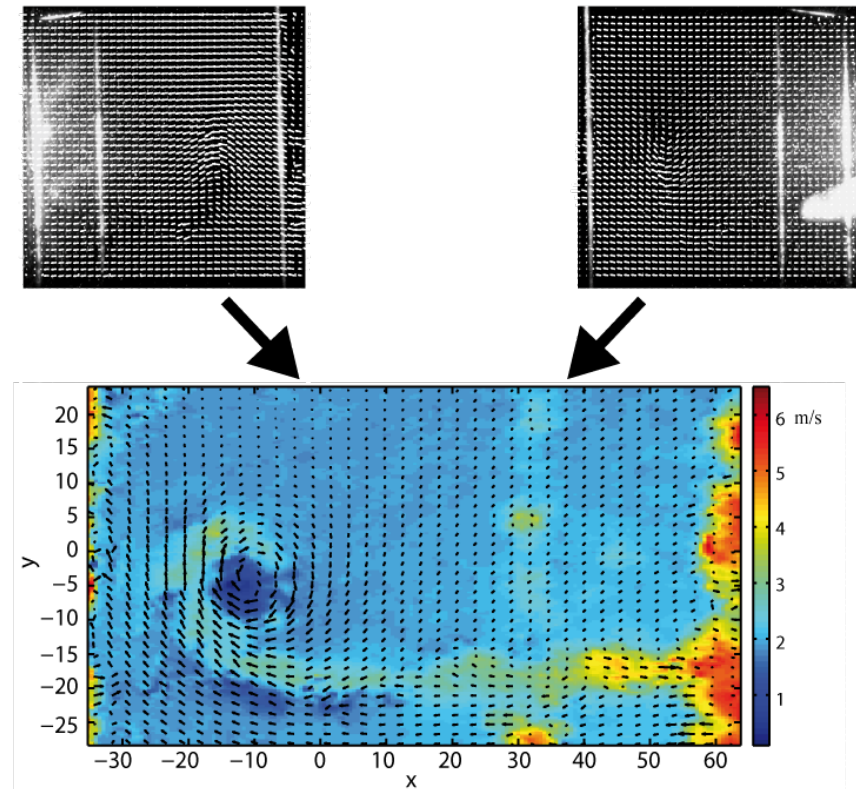
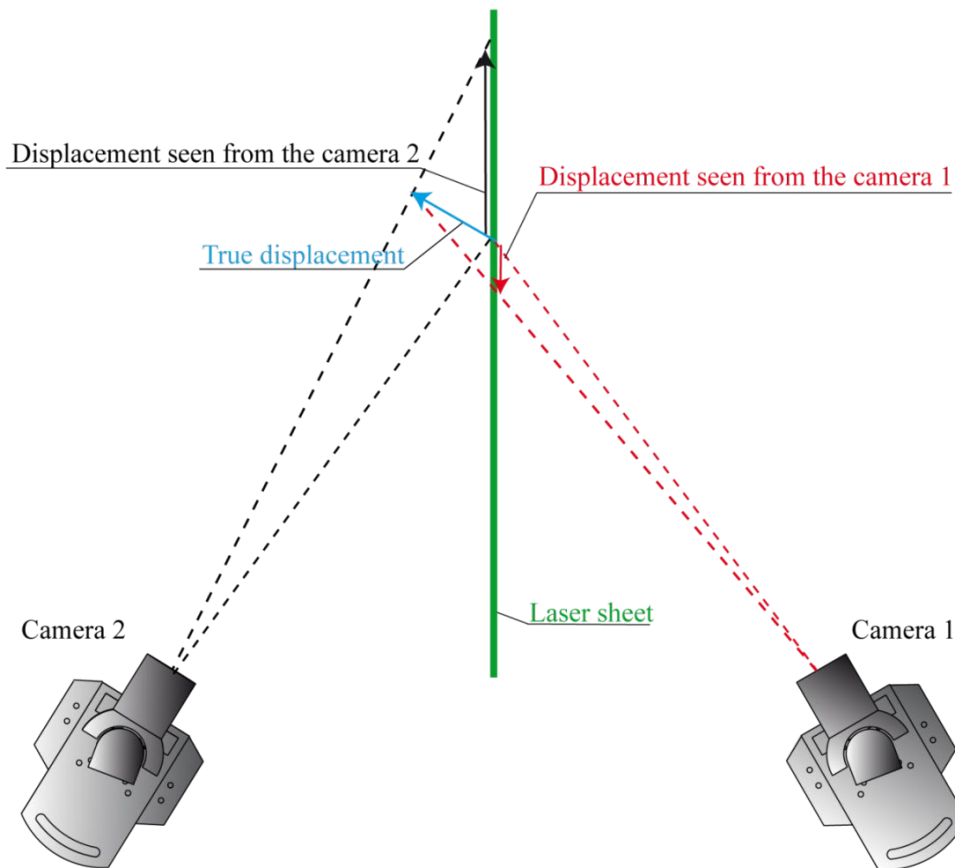
- **Hypothesis:**
 - Particles travel with the flow at the same speed
 - Uniform motion within every interrogation window
 - Cross correlation → Velocity estimate

- **Seeding particles:**
 - Density similar to water density
 - Typical size: $\sim 5 \mu\text{m}$ to $50 \mu\text{m}$

	PSP Polyamide seeding particles	HGS Hollow glass spheres	S-HGS Silver-coated hollow glass spheres	FPP Fluorescent polymer particles
Mean particle size (μm)	5, 20, 50	10	10	10, 30
Size distribution	1 - 10 μm 5 - 35 μm 30 - 70 μm	2 - 20 μm	2 - 20 μm	1 - 20 μm 20 - 50 μm
Particle shape	non-spherical but round	spherical	spherical	spherical
Density (g/cm ³)	1.03	1.1	1.4	1.19
Melting point (°C)	175	740	740	125
Refractive index	1.5	1.52	—	1.479
Material	Polyamide 12	Borosilicate glass	Borosilicate glass	Poly (Methyl methacrylate) (Labeled with Rhodium B)

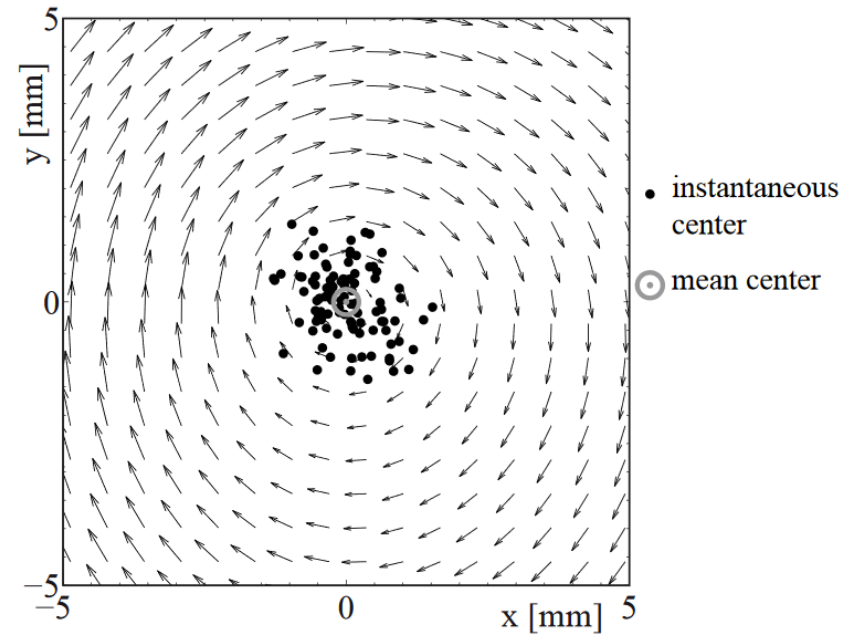
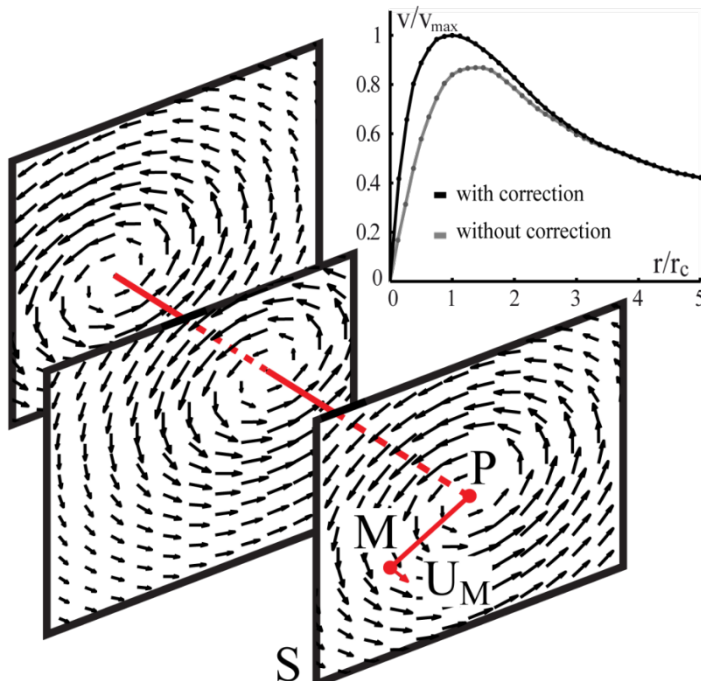
- **Stereo PIV (SPIV)**

- **Use of 2 cameras → Measurement of 3 components of the velocity**
 - *Requires a precise calibration*



- **SPIV - data processing**

- *Issue: Wandering (random displacement of the vortex axis due to turbulence)*
- *Correction of vortex wandering*
 - *Vortex center identification*
 - *Shifting and aligning velocity maps*
 - *Mean and statistic computation*



$$\Pi(P) = \frac{1}{S} \int_{M \in S} \frac{(PM \wedge U_M) \cdot z}{\|PM\| \cdot \|U_M\|} dS$$

Graftieaux criteria: $\text{Min}(\Pi) = \text{Vortex Axis}$

- **SPIV - data processing**

- Key vortex parameters
 - Vortex model best fit
 - Viscous core size estimation (r_c)
 - Circulation (Γ)

Lamb-Oseen model

$$v_{\theta}(r) = \frac{\Gamma}{2\pi r} \left(1 - e^{-\alpha r^2/r_c^2} \right)$$

Vatistas model

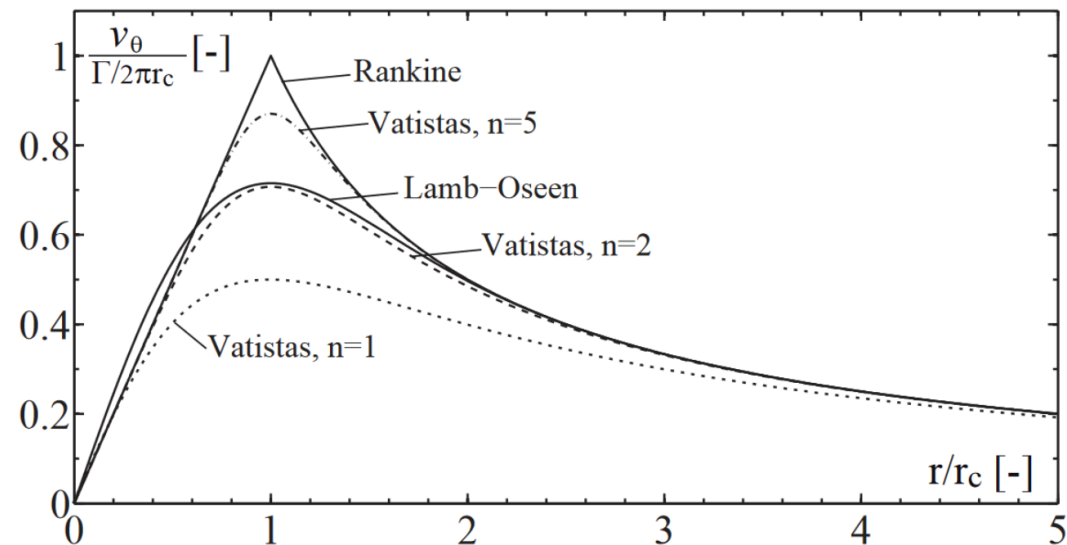
$$v_{\theta}(r) = \frac{\Gamma}{2\pi} \left(\frac{r}{(r_c^{2n} + r^{2n})^{1/n}} \right)$$

$$\frac{\partial p}{\partial r} = \rho v_{\theta}^2 / r$$



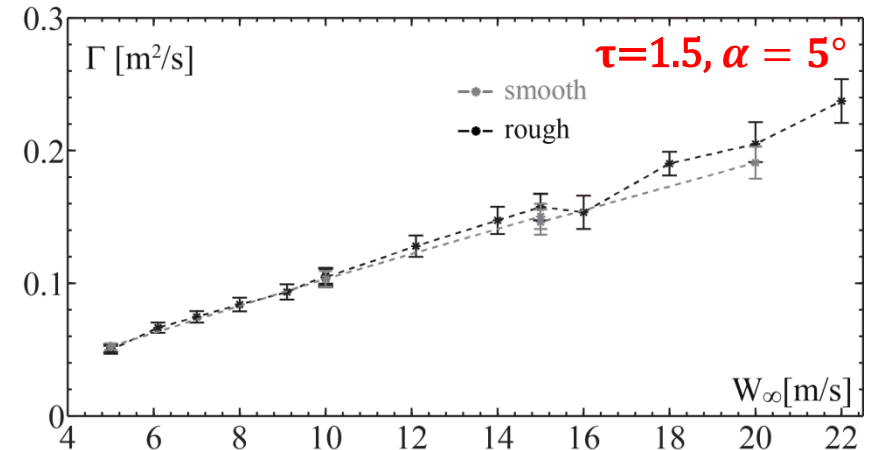
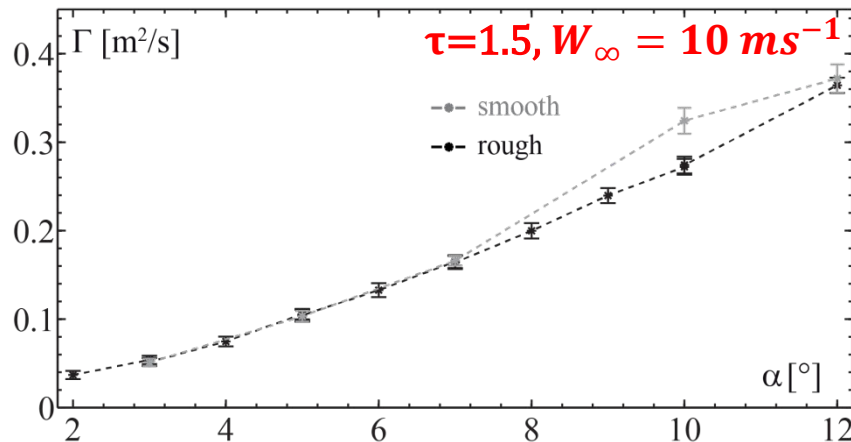
$$C_{pmin} = -\beta \left(\frac{\Gamma}{W_{\infty} r_c} \right)^2$$

$$\Gamma = \int_A \vec{\omega} \cdot \vec{n} \cdot dS$$



TLV Characteristics

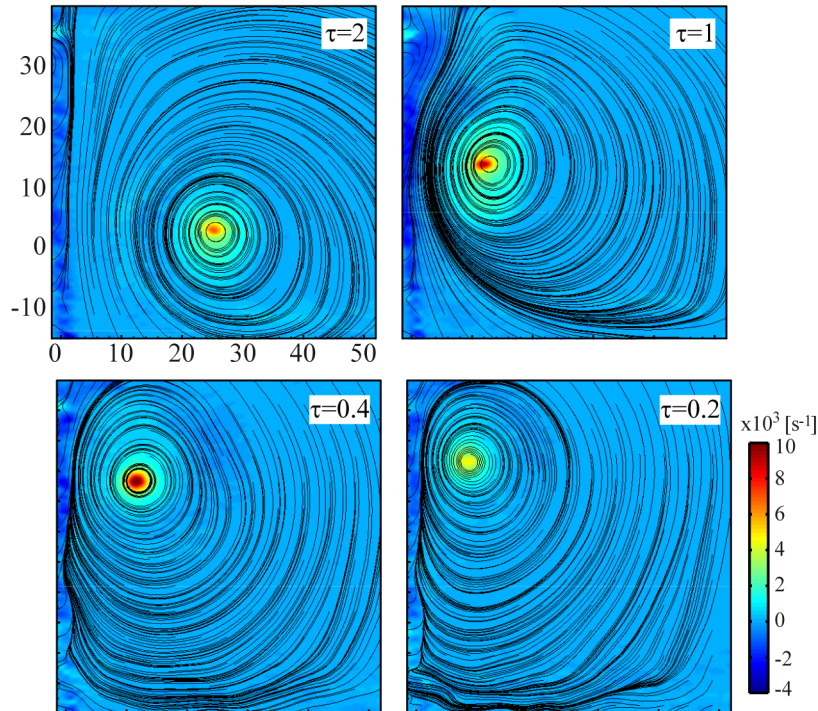
- **Circulation (Γ):**
 - *Estimated from velocity fields by fitting the Lamb-Oseen model*
 - *Almost linear relationship with incidence angle and upstream velocity*
 - *No significant influence of leading edge roughness*



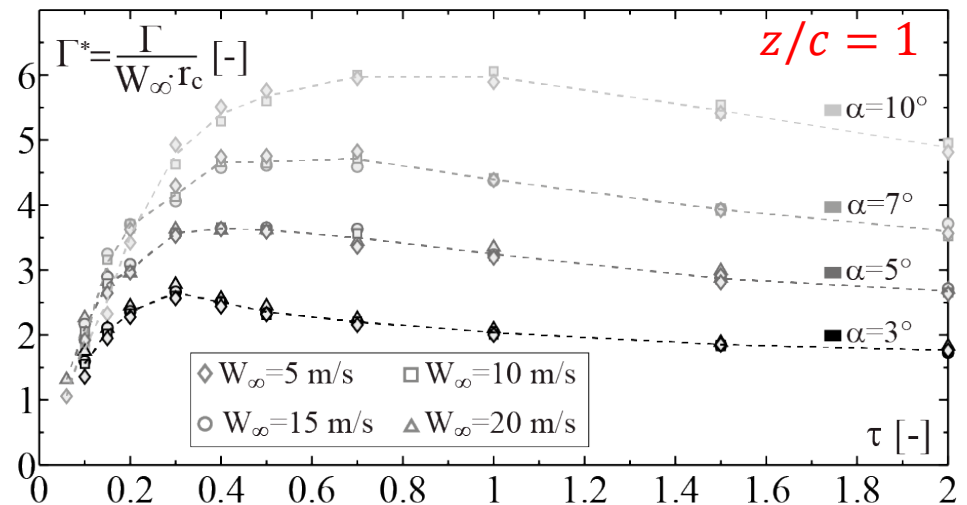
TLV Characteristics

- Influence of gap width (τ) on vortex intensity
 - Non-dimensional circulation Γ^* at $z/c = 1$ (roughened leading edge)
 - $\Gamma^*(\tau)$ highly dependent on incidence angle

Streamwise vorticity ($W_\infty = 10 \text{ ms}^{-1}$, $\alpha = 10^\circ$)



$$\Gamma^* = \frac{\Gamma}{W_\infty r_c} \quad r_c \text{ is the viscous core radius}$$

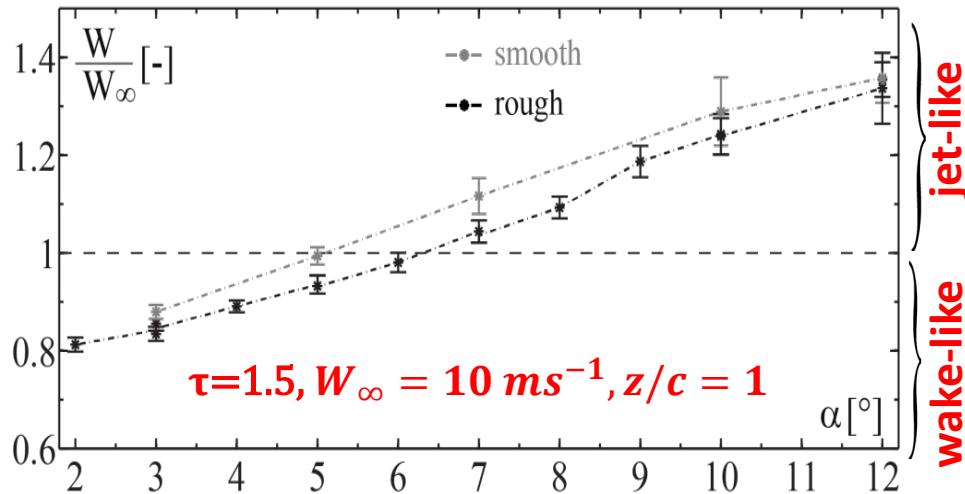


TLV Characteristics

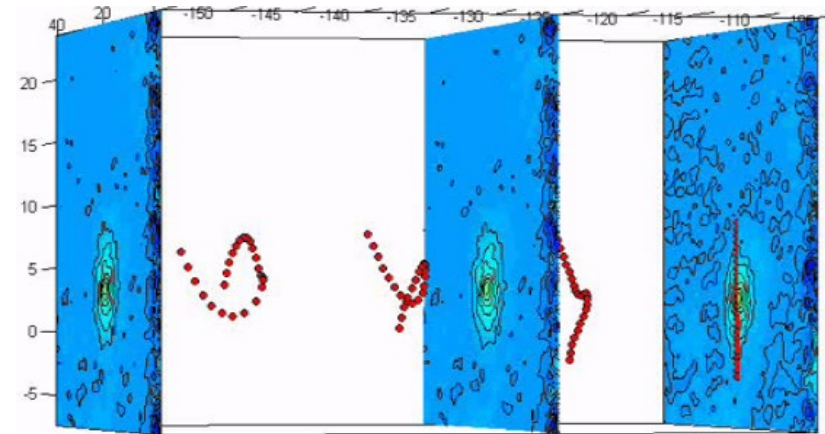
- Influence of incidence angle (α) on axial velocity

- 2 regimes:

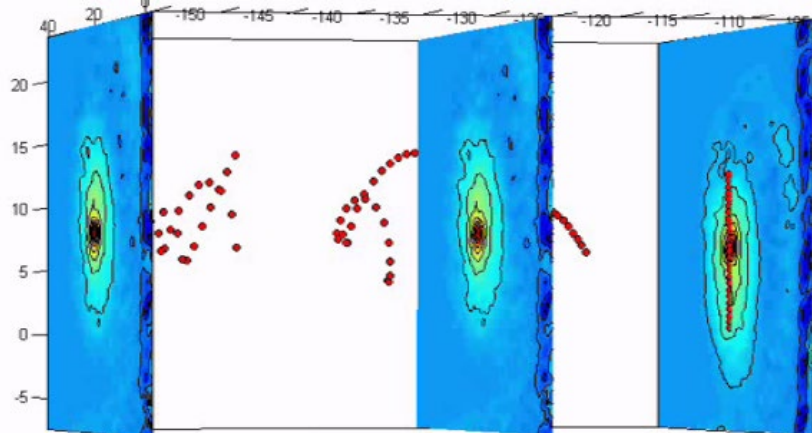
- $\alpha \lesssim 6^\circ$: wake-like profile ($W < W_\infty$)
- $\alpha \gtrsim 6^\circ$: Jet-like profile ($W < W_\infty$)
fluid particles, close to the vortex axis, travel faster in the stream-wise direction than the mean flow



$\alpha=3^\circ$: wake-like profile



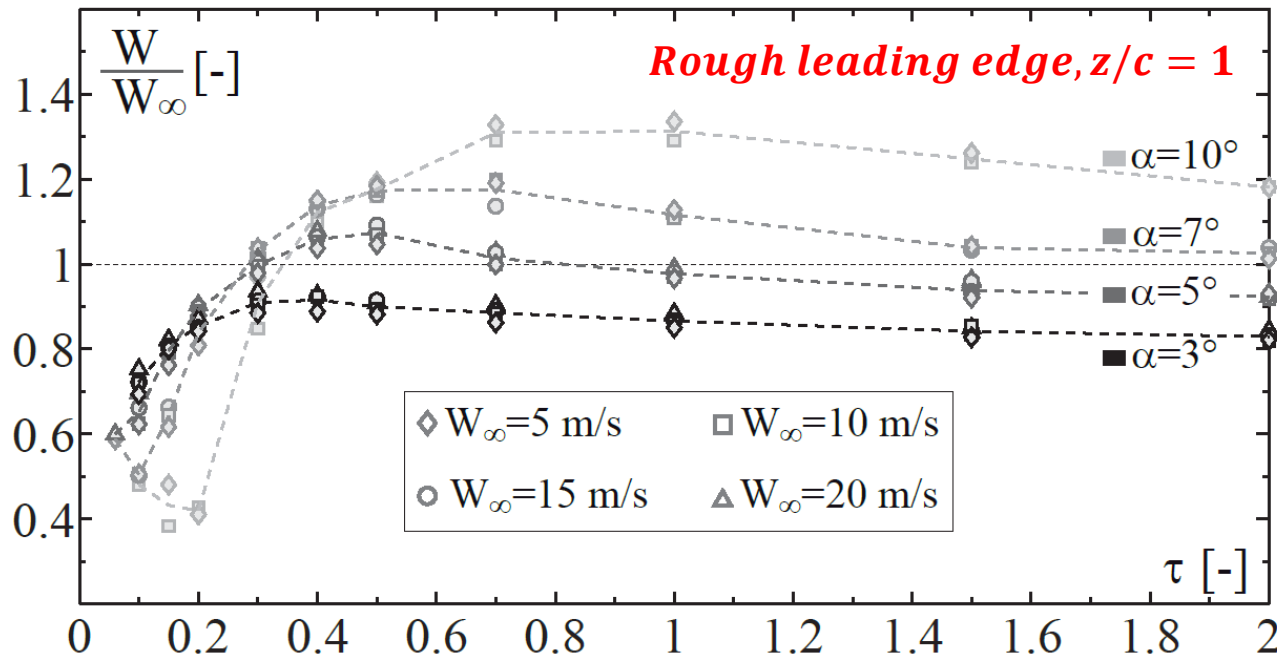
$\alpha=10^\circ$: jet-like profile



TLV Characteristics

- **Influence of gap width (τ) on axial velocity**

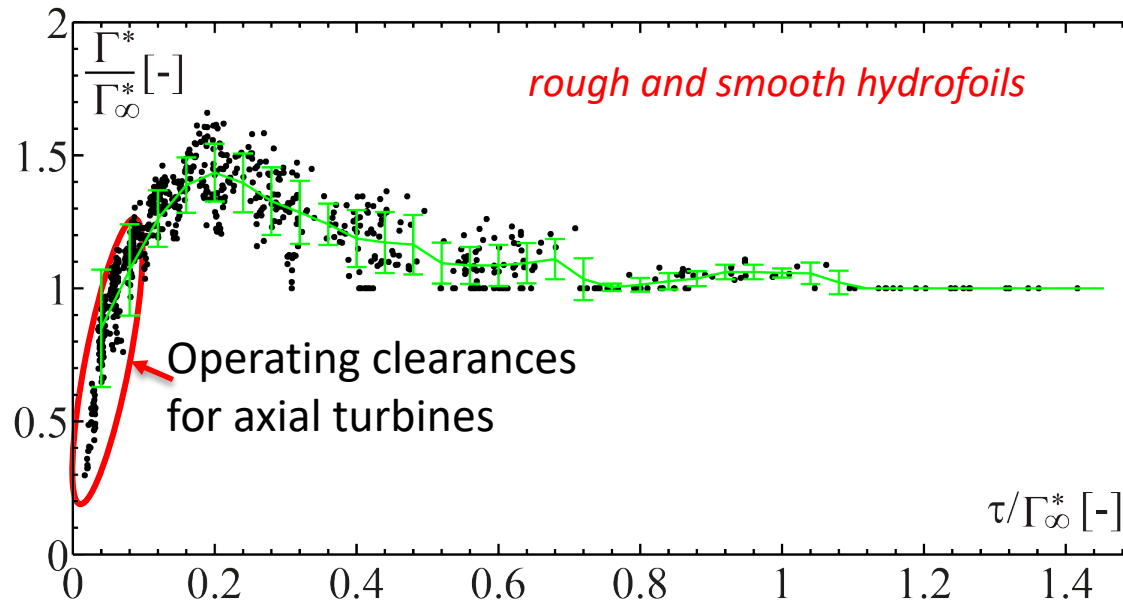
- The flow around the vortex axis tends to switch from wake-like to jet-like flow as the gap increases
- $W^*(\tau)$ highly dependent on incidence angle



Scaling law for TLV intensity

- Dimensionless coefficients

- Scaling of the gap width: τ/Γ_∞^* , with $\Gamma_\infty^* = \lim_{\tau \rightarrow \infty} (\Gamma^*) \approx \Gamma^*(\tau = 2)$
- Scaling of the TLV intensity: Γ^*/Γ_∞^*



- $C_{p_{min}}$: We can easily show in the case of Lamb-Oseen model, that:

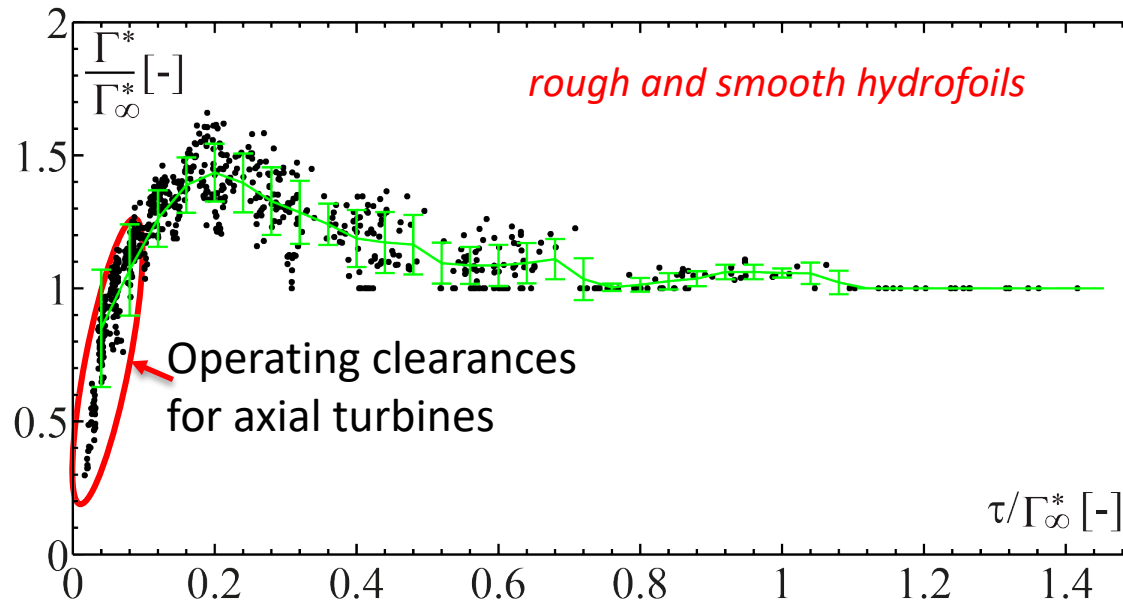
$$C_{p_{min}} \approx -0.044 \left(\frac{\Gamma}{W_\infty r_c} \right)^2$$

→ As the gap reduces, Γ increases and so does the cavitation risk

New scaling law for TLV intensity

- Dimensionless coefficients

- Scaling of the gap width: τ/Γ_∞^* , with $\Gamma_\infty^* = \lim_{\tau \rightarrow \infty} (\Gamma^*) \approx \Gamma^*(\tau = 2)$
- Scaling of the TLV intensity: Γ^*/Γ_∞^*



- For $\tau/\Gamma_\infty^* \lesssim 0.2$, Γ^*/Γ_∞^* is highly dependent on the gap width
- This corresponds to typical clearances of axial turbines and may explain the discrepancy between model tests and full scale results.

Effect of gap width - Summary



- Vortex trajectory & intensity significantly influenced by the proximity of a neighboring wall.
- The axial flow may be wake-like or jet-like, depending on incidence angle and gap
- Existence of a critical gap width for which the TLV intensity is maximum, with an increase risk of cavitation.
- New dimensionless parameters (Γ^*/Γ_∞^* and τ/Γ_∞^*) allows to merge the data in almost a single curve.
- Sensitivity of TLV intensity to gap width explains the discrepancies of cavitation occurrence between models and prototypes.

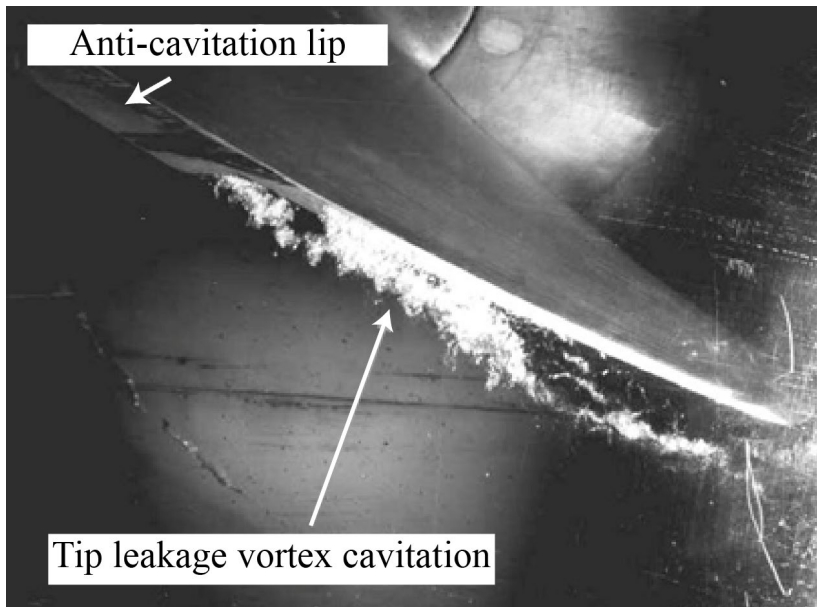
Related publications:

- M. Dreyer, J. Decaix, C. Münch-Alligné, and M. Farhat (2014) “Mind the Gap: A New Insight into the Tip Leakage Vortex Using Stereo-PIV.” *Experiments in Fluids* 55 (11): 1–13.
- J. Decaix, G. Balarac, M. Dreyer, M. Farhat, and C Münch (2015) “RANS and LES Computations of the Tip-Leakage Vortex for Different Gap Widths.” *Journal of Turbulence* 16(4):309-341

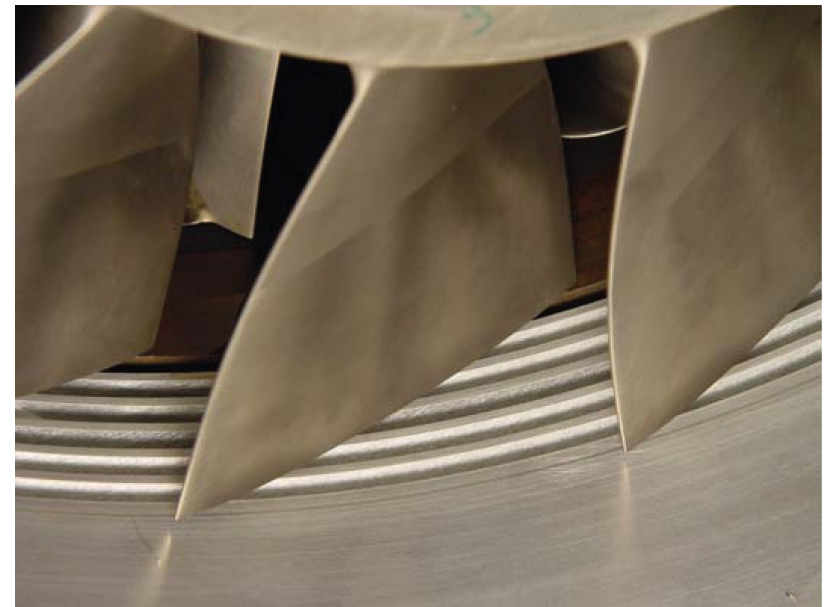
Mitigation of tip leakage vortex cavitation by grooves

- MOTIVATION AND CONTEXT

in hydraulic turbines



Grooves are used in gas compressors

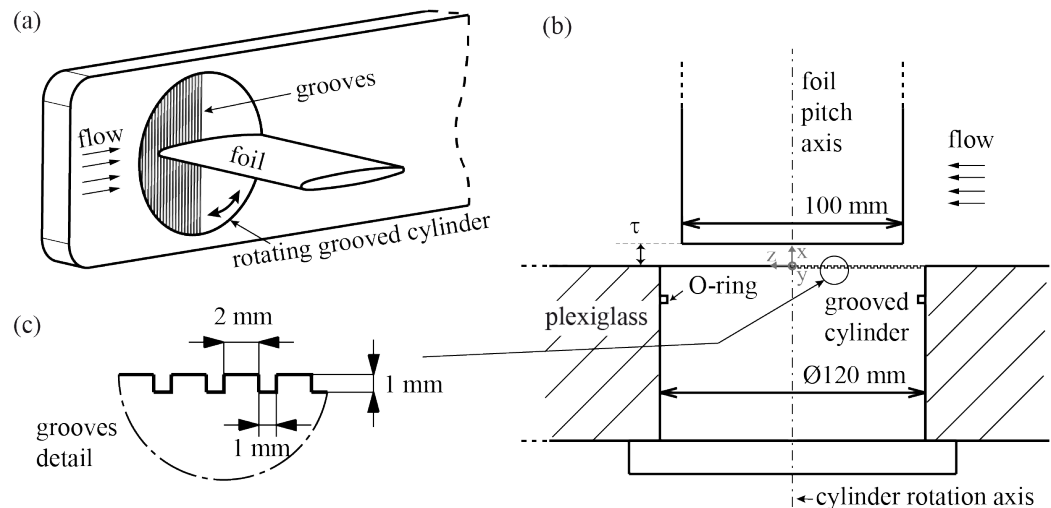


Muller et al. 2007. In ASME Turbo Expo 2007: Power for Land, Sea, and Air, 115–24.

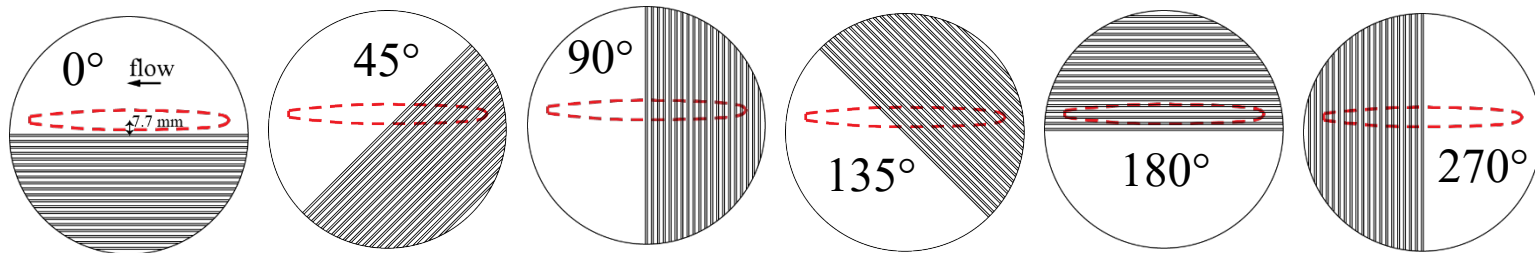
Mitigation of tip leakage vortex cavitation by grooves

- Case study – Experimental Setup

- Naca0009 hydrofoil
- Grooves implementation: on the lateral window, with adjustable orientation
- Grooves size of the order of the wall boundary layer thickness (~ 1 mm)

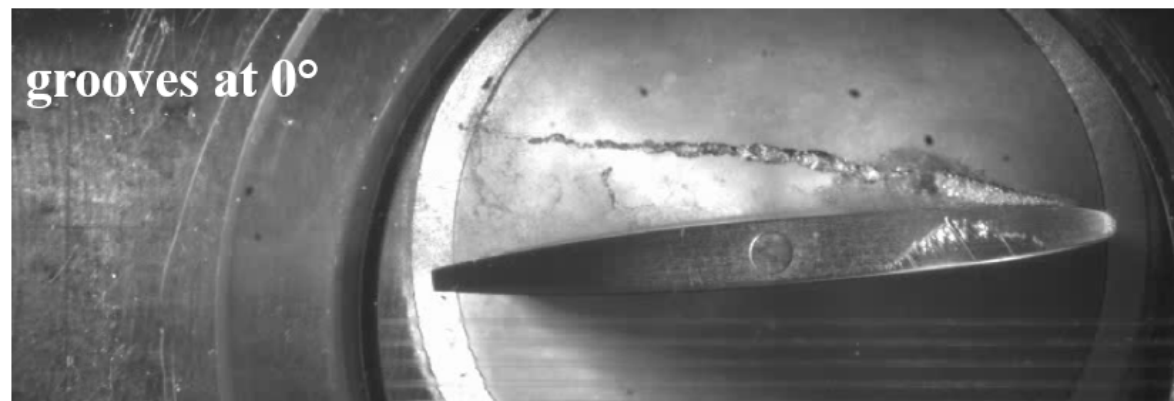
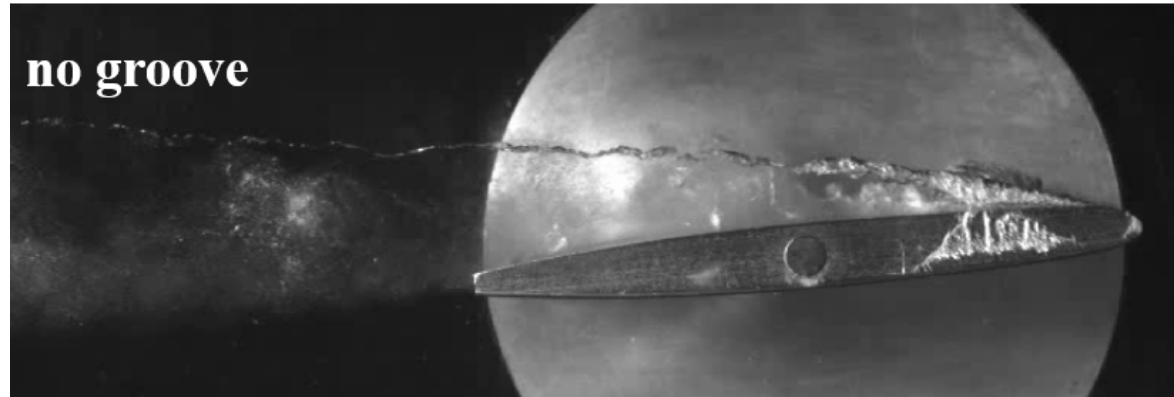


- Grooves orientations:



Mitigation of tip leakage vortex cavitation by grooves

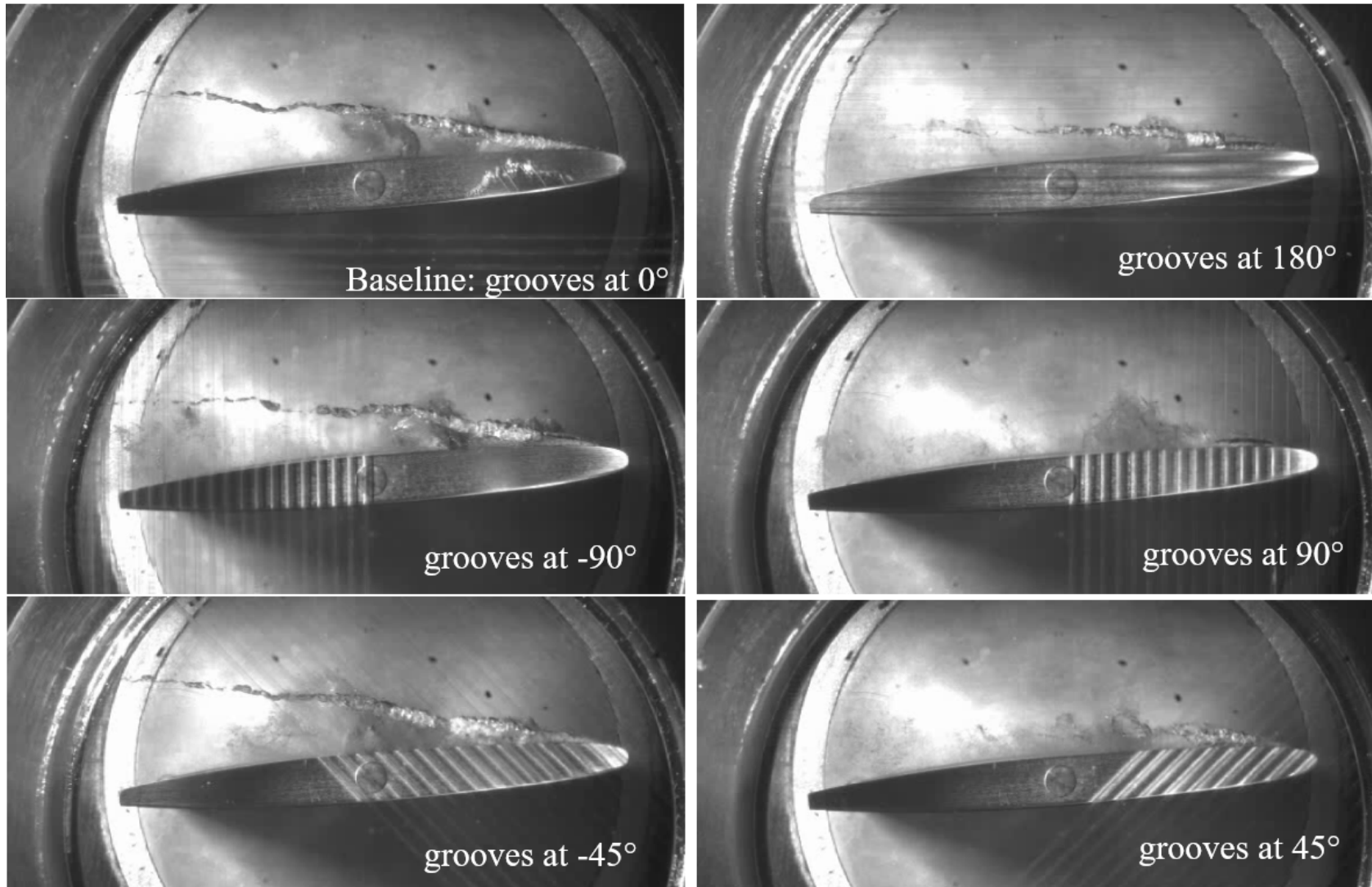
- No groove vs grooves below the foil
 - $\tau=0.1, W_{\infty} = 10 \text{ ms}^{-1}, \alpha = 5^{\circ}$
 - *No significant effect \rightarrow this case may be used as baseline for comparison*



Control of TLV in hydraulic Machines

Effect of grooves orientation on TLV cavitation development

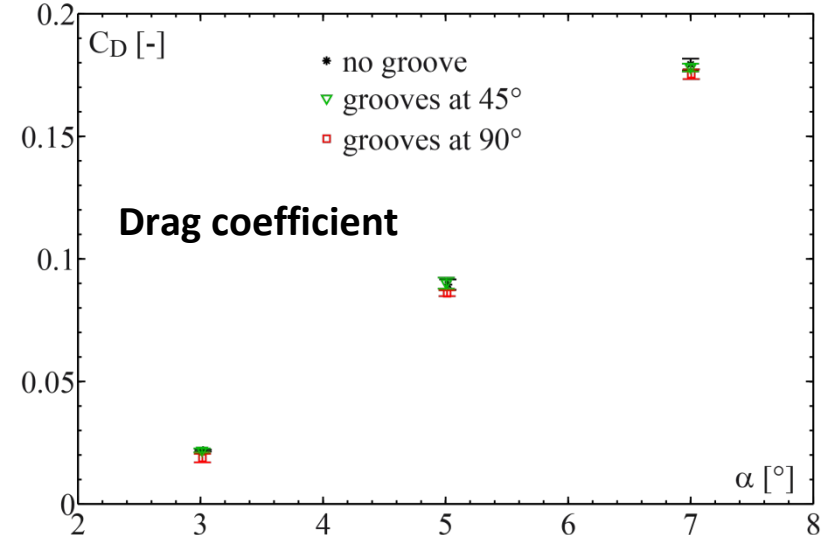
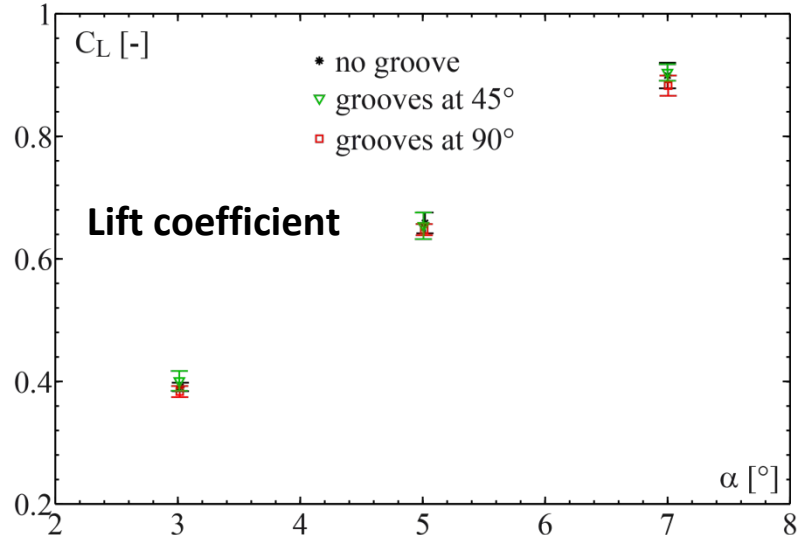
$$\tau=0.1, W_\infty = 10 \text{ ms}^{-1}, \sigma = 1.2, \alpha = 5^\circ$$



Optimum is obtained with grooves at $\sim 45^\circ$ to 90°

Effects on hydrodynamic performances

- Lift and drag measurements
 - Cavitation-free conditions, $\tau=0.1$, $W_\infty = 10 \text{ ms}^{-1}$



- TLV control with grooves - summary
 - Significant reduction of cavitation with grooves near the leading edge, oriented at 45° or 90°, without alteration of hydrodynamic performances (lift and drag)