



# ME-446: Liquid-gas interfacial heat and mass transfer

## Condensation

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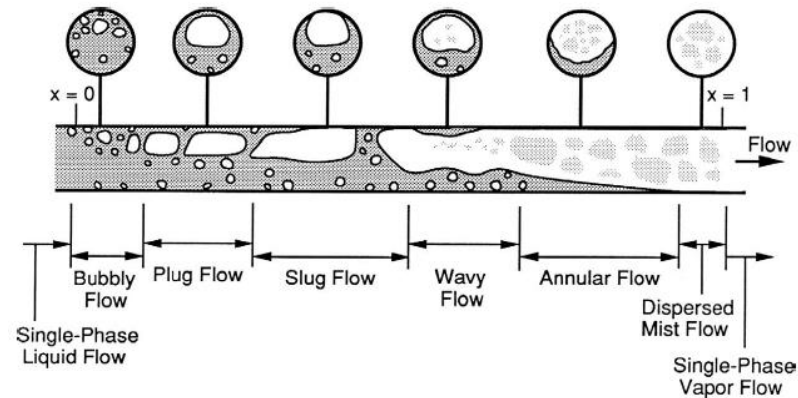
Energy Transport Advances  
Laboratory

EPFL Mechanical Engineering

2025 Fall Semester

Photo Credit: Trougnouf

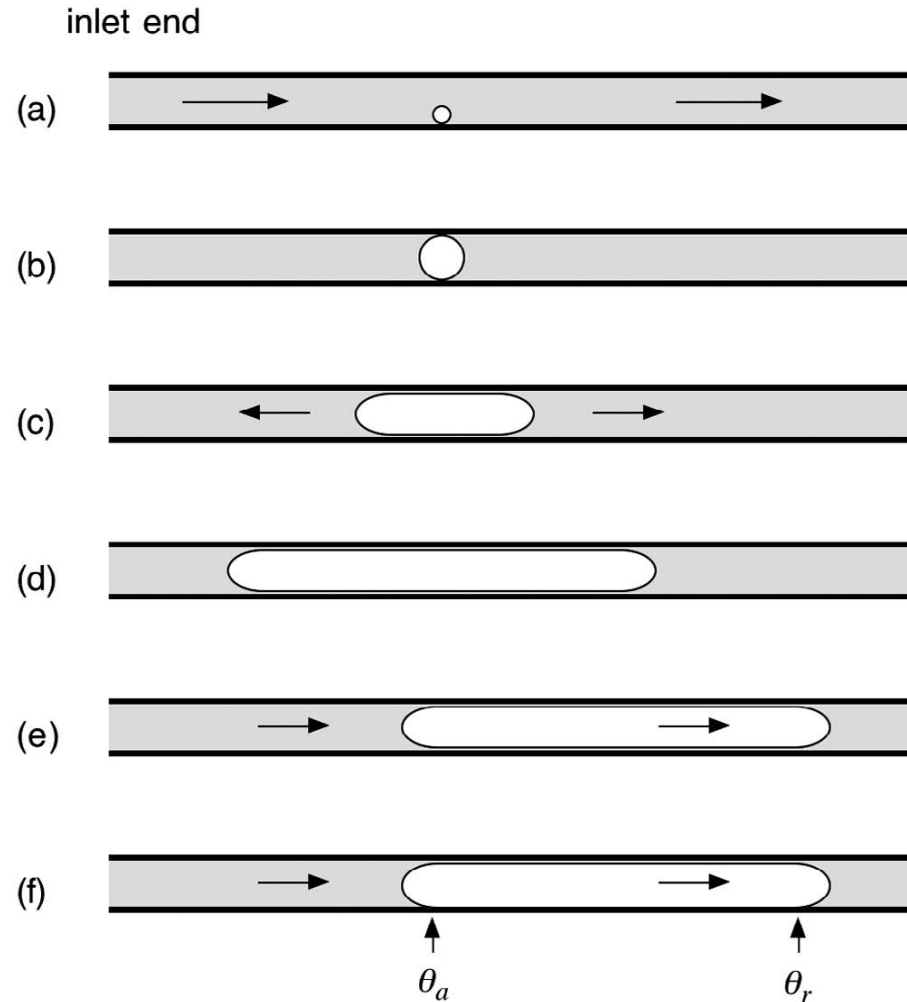
- How wicking surfaces enhance CHF
- Different flow boiling regimes



- Evaluate ONB, HTC, and CHF in flow boiling with correlations
  - ONB: Sato and Matsumura
  - HTC: Rohsenow's partition (subcooled), Gungor and Winterton (saturated), Kandlikar (saturated)
  - CHF: Celata (subcooled), Groeneveld lookup table (saturated)

- Miniaturization forces heat dissipation problem into microchannel regime
- Microchannel leads to higher surface-to-volume ratio and lower liquid layer thickness
- How confined the channel needs to be for us to consider it as micro?  
(Kew and Cornwell [https://doi.org/10.1016/S1359-4311\(96\)00071-3](https://doi.org/10.1016/S1359-4311(96)00071-3))

# Bubble Confined in Microchannels



Once confinement is reached:

- Bubble elongation dominates
- Thin-film evaporation becomes the key heat transfer mechanism
- Flow patterns collapse to slug–annular regimes
- Macroscale boiling correlations break down

Fig. 12.31 in Carey

- Liu and Garimella <https://doi.org/10.1115/1.2754944>

$$h = h_{sp}F + h_{nb}S$$

$$h_{sp} = 1.86 \left( \frac{k_l}{D_h} \right) \left[ \frac{\text{Re}_l \text{Pr}_l D}{L} \right]^{1/3} \left( \frac{\mu_l}{\mu_s} \right)^{0.14}$$

$$\text{Re}_l = \frac{G(1-x)D}{\mu_l} \quad \mu_s: \text{liquid viscosity at surface temperature}$$

- Liu and Garimella <https://doi.org/10.1115/1.2754944>

$$h = h_{sp}F + h_{nb}S$$

$$F = 2(\phi_l^2)^{0.25} \left(\frac{\mu_{tp}}{\mu_l}\right)^{0.105} \left(\frac{c_{p,tp}}{c_{p,l}}\right)^{0.25} \left(\frac{k_{tp}}{k_l}\right)^{0.75} \text{Pr}_l^{0.167}$$

$$\phi_l^2 = 1 + \frac{5}{X} + \frac{1}{X^2} \quad X^2 = \left(\frac{1-x}{x}\right) \left(\frac{\rho_v}{\rho_l}\right) \left(\frac{\mu_l}{\mu_v}\right) \quad \text{for laminar flow}$$

Subscript tp: two phase properties

- Liu and Garimella <https://doi.org/10.1115/1.2754944>

$$h = h_{sp}F + h_{nb}S$$

$$h_{nb} = h_0 \left( \frac{q''}{q_0''} \right) \left( \frac{R_p}{R_{p0}} \right)^{0.133} \left[ 1.73 p_r^{0.27} + \left( 6.1 + \frac{0.68}{1 - p_r} \right) p_r^2 \right]$$

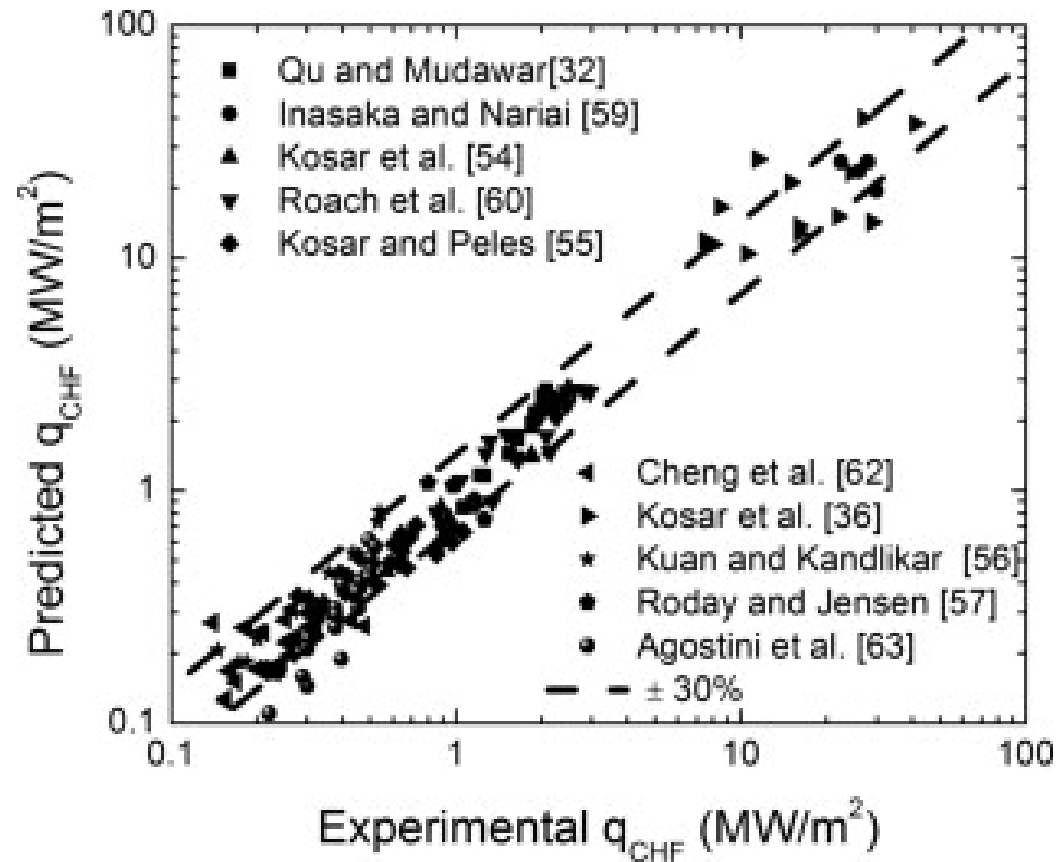
$$S = \exp \left\{ 36.57 - \frac{55746}{\text{Re}_l F^3} - 3.4 \ln(\text{Re}_l F^3) \right\}$$

$$h_0 = 5600 \text{ W/m}^2\text{K}, q_0'' = 20 \text{ kW/m}^2, R_{p0} = 0.4 \text{ }\mu\text{m}$$

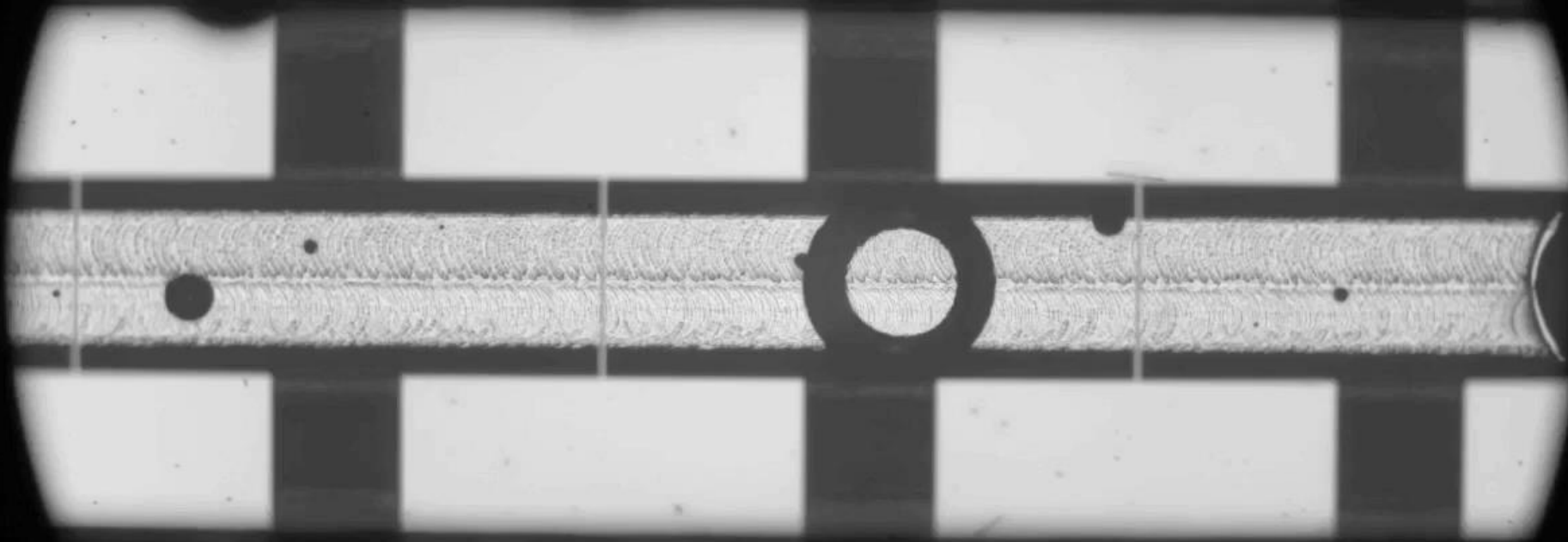
$R_p$ : surface roughness (set as 1  $\mu\text{m}$  by Liu and Garimella)

$p_r = P/P_c$  reduced pressure (normalized to critical pressure)

CHF model by Kandlikar <https://doi.org/10.1115/1.4001124>



Vapor Backflow Instability

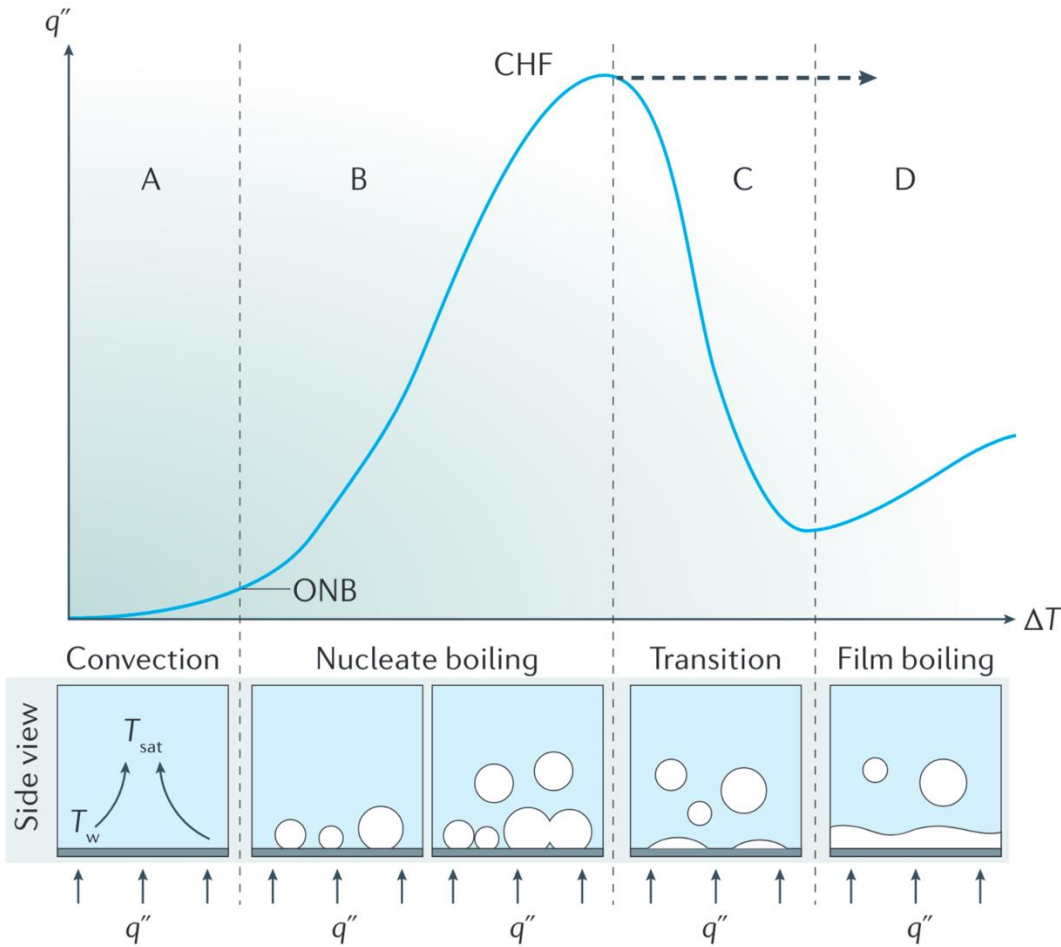


60 fps; real time duration: 210 ms

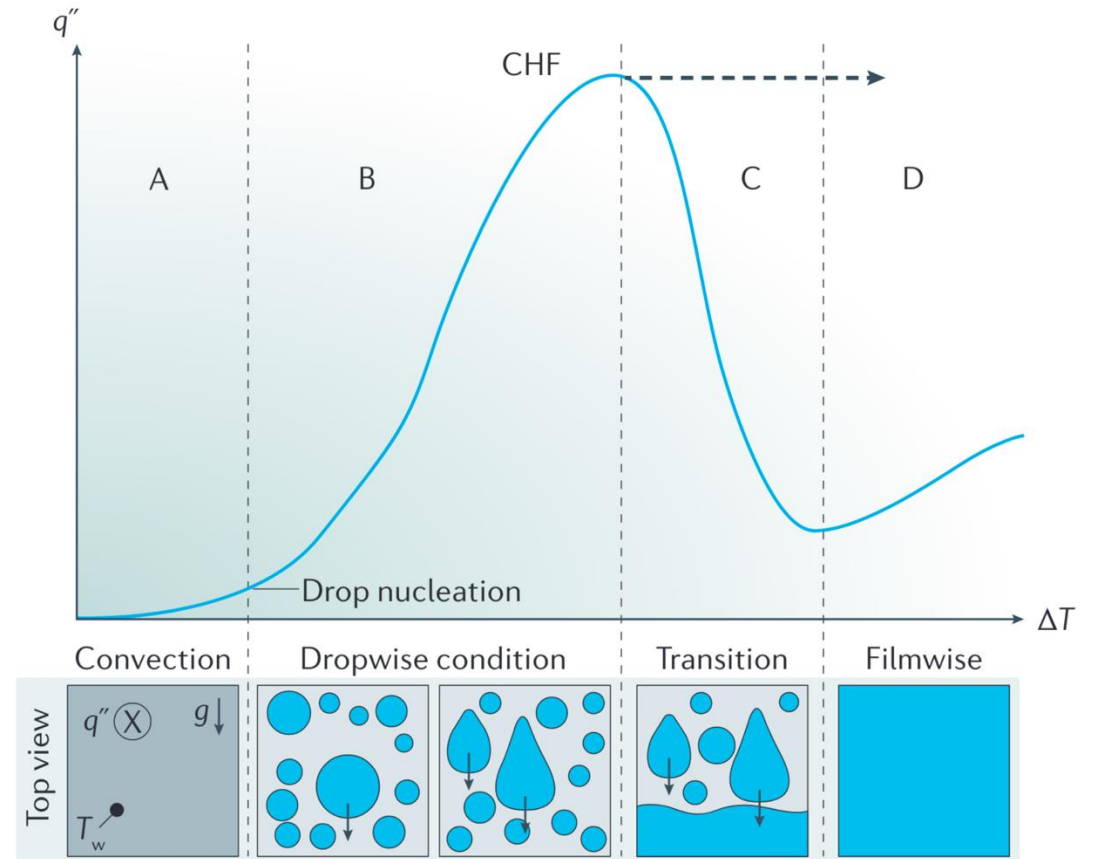
Credit: Mark  
Schepperle

# Boiling vs Condensation

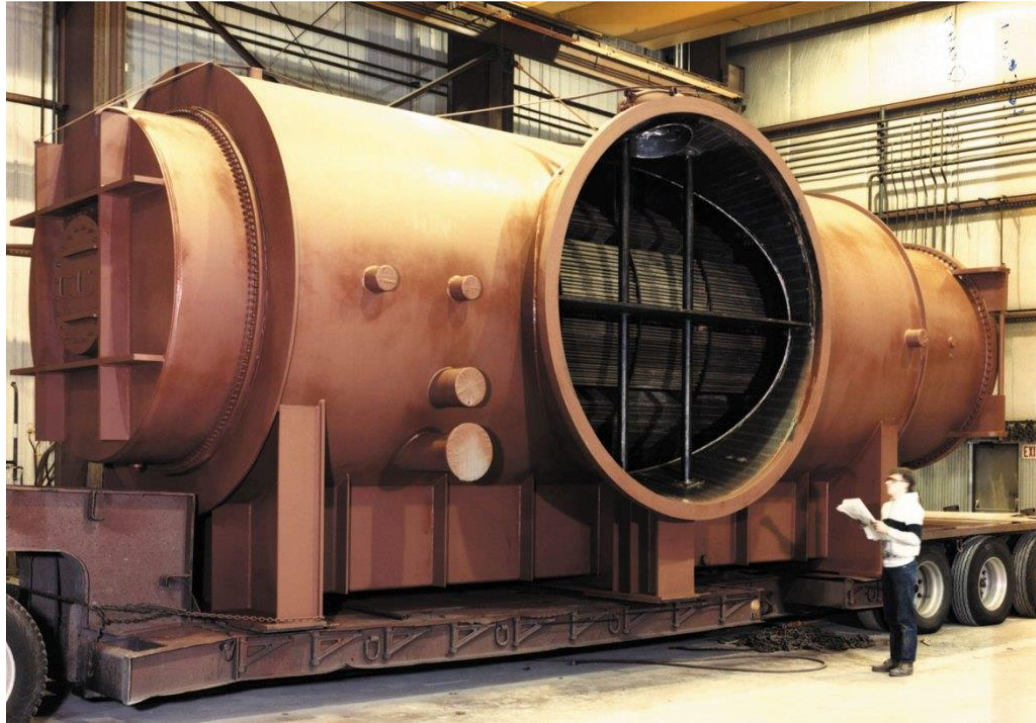
**a Boiling**



**b Condensation**



# Why Condensation Matters



Condenser in Power Plants, Holtec

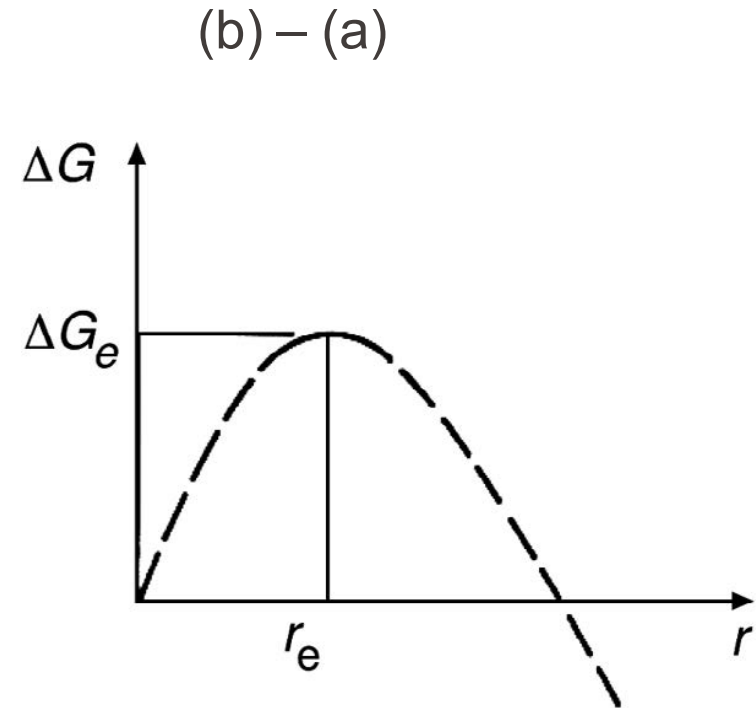
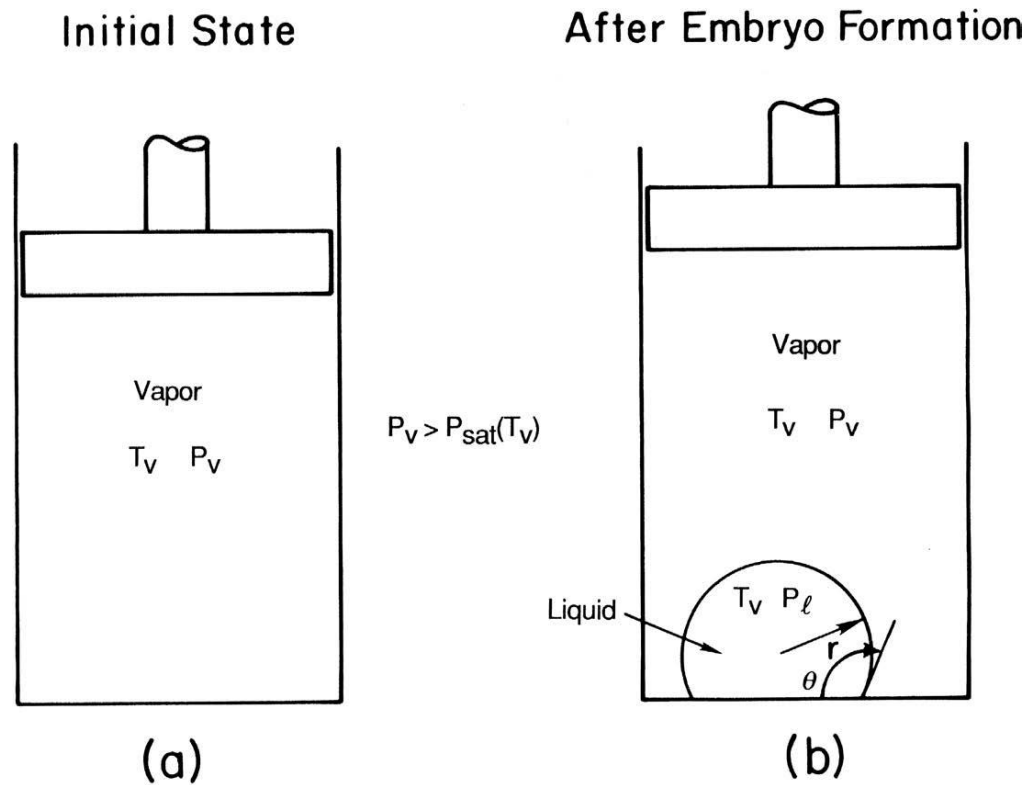


Condenser in Cooling Systems, STACK

# Intended Learning Objectives

- Nucleation in condensation
- Rose's analysis of dropwise condensation
- Nusselt's analysis of filmwise condensation

Carey, Chapter 9

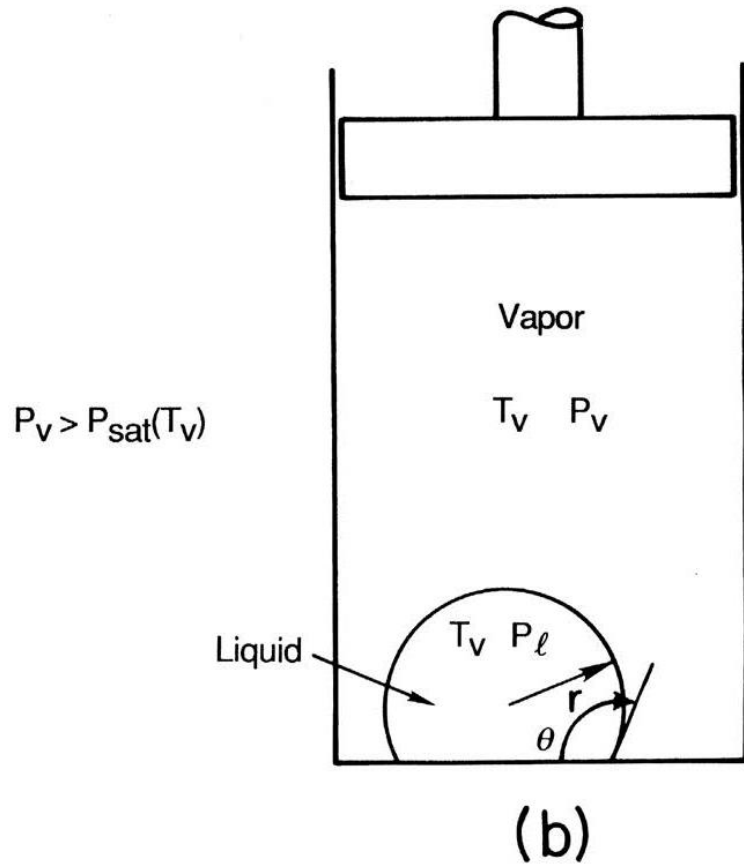


If  $r < r_e$ , the droplet collapses  
 If  $r > r_e$ , the droplet grows

Figure 9.2 in Carey

# Equilibrium Droplet Radius

After Embryo Formation

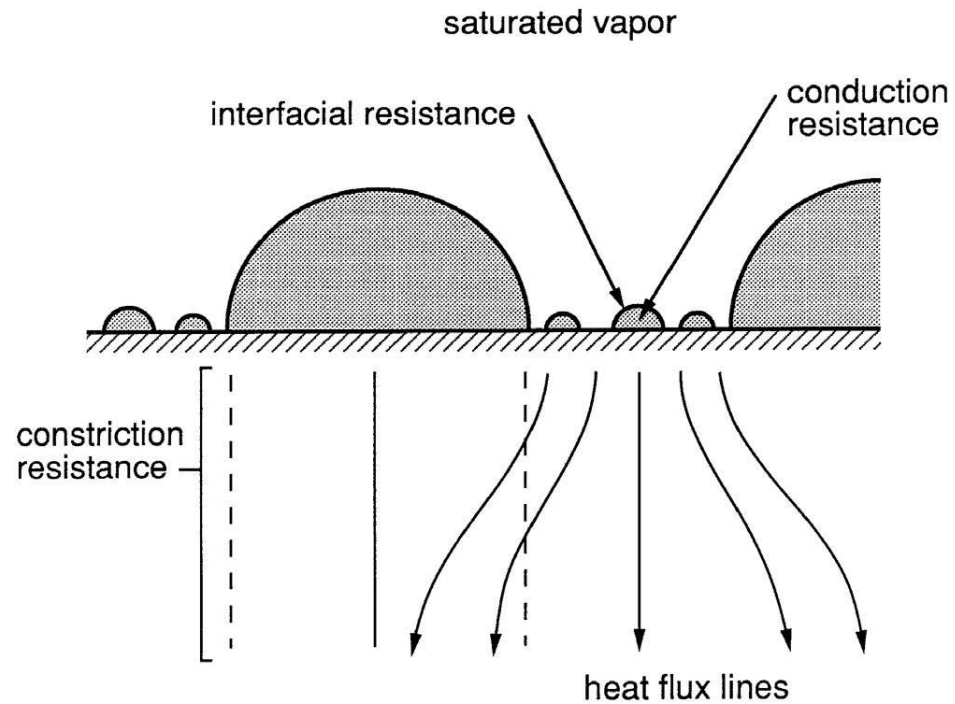


# Dropwise Condensation



<https://doi.org/10.1021/nl303835d>

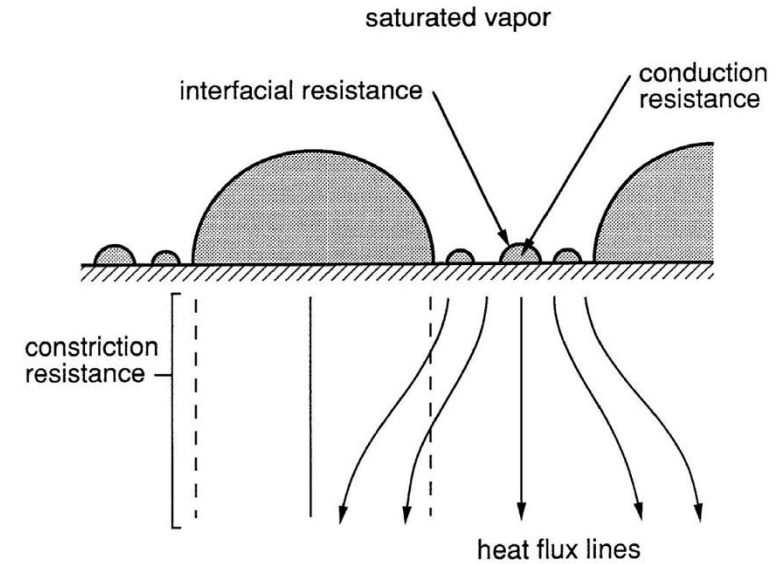
# Overall Droplet Resistance



- Interfacial resistance
- Curvature induced resistance
- Conduction through droplet
- Constriction resistance

- Schrage equation (condensation)

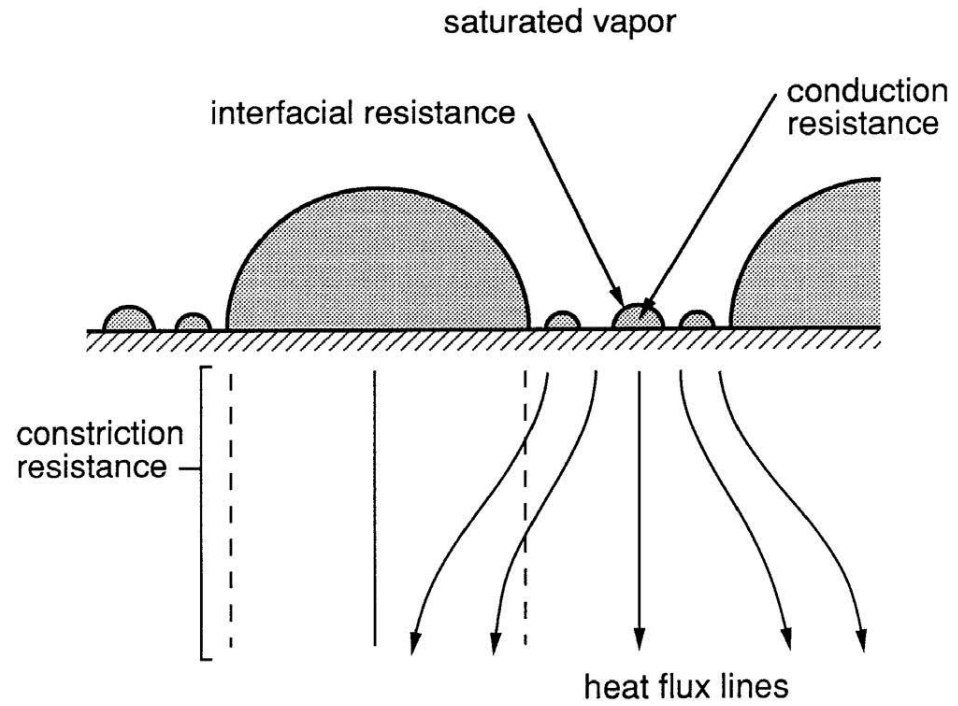
$$q_i'' = \left( \frac{2\hat{\sigma}}{2 - \hat{\sigma}} \right) h_{lv} \sqrt{\frac{1}{2\pi R} \left( \frac{P_v}{\sqrt{T_v}} - \frac{P_{\text{sat}}(T_l)}{\sqrt{T_l}} \right)}$$



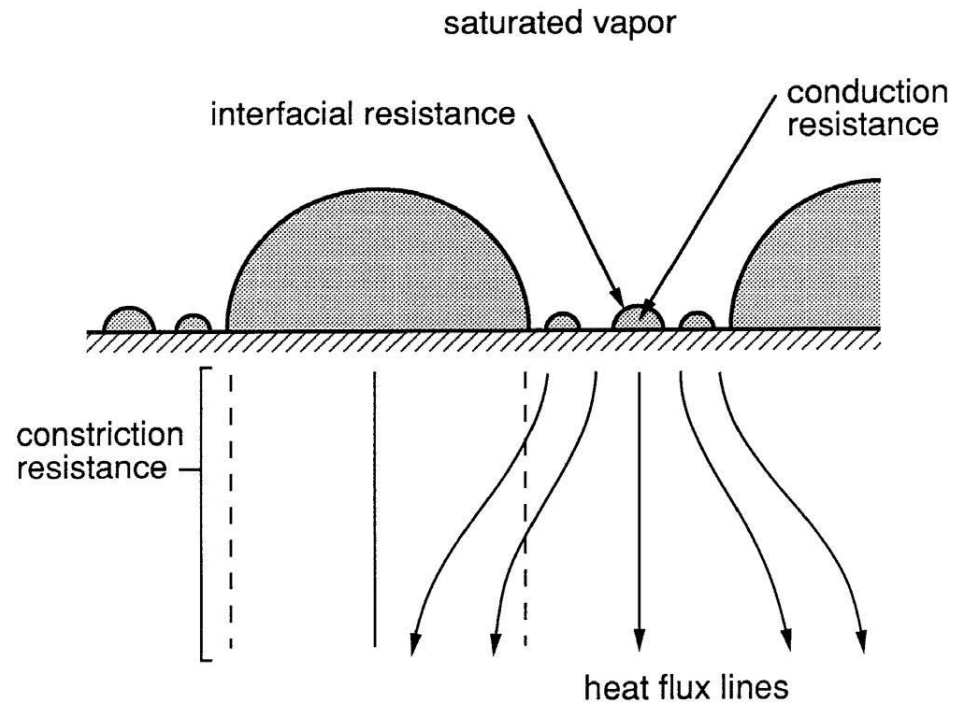
Given a droplet radius  $r$ , the equilibrium droplet temperature at the interface should follow

$$r \approx \frac{2\sigma}{\frac{RT_{eq}}{v_l} \ln\left(\frac{P_v}{P_{sat}(T_{eq})}\right)}$$

Clausius-Clapeyron  $\frac{dP_{sat}}{dT} = \frac{\rho_v h_{lv}}{T}$



# Overall Droplet Resistance



- Interfacial resistance
- Curvature induced resistance
- Conduction through droplet
- Constriction resistance

- Cumulative distribution function (postulated form)

$$F(r) = \left(\frac{r}{r_{max}}\right)^{1/3} \text{ for } r_e < r \leq r_{max};$$

For dropwise condensation of steam at pressures below 1 atm, Rose et al. recommended the following empirical correlation for the heat transfer coefficient

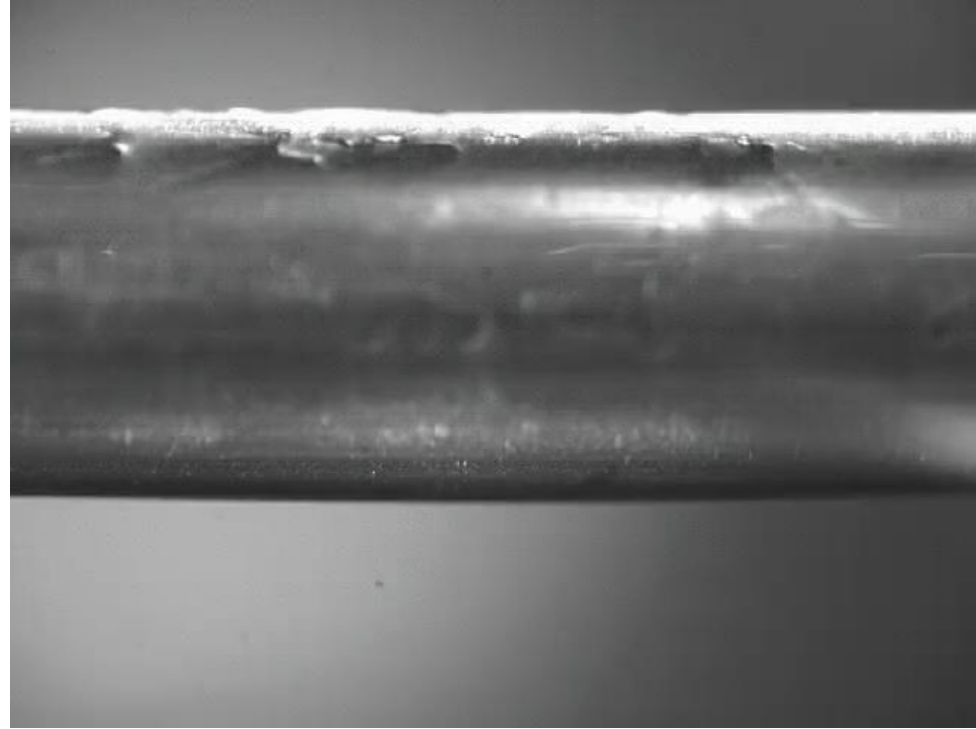
$$h_{dc} = T_v^{0.8} [5 + 0.3(T_{sat} - T_w)] \quad \text{Eq. 9.42 in Carey}$$

Temperature in Celsius and HTC in kW/m<sup>2</sup>K

# Dropwise vs Filmwise Condensation



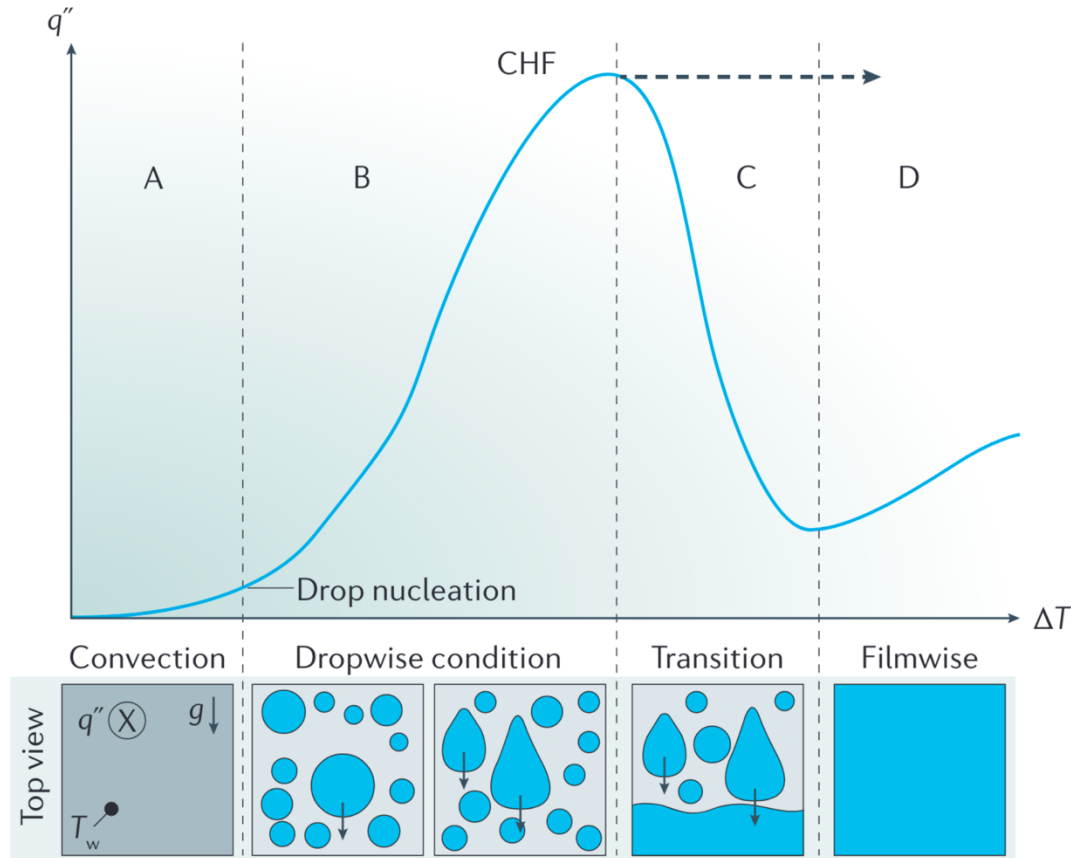
Dropwise condensation



Filmwise condensation

# Dropwise vs Filmwise Condensation

## b Condensation



Data obtained by Takeyama and Shimizu [9.40] for condensation of steam on a short vertical copper surface.

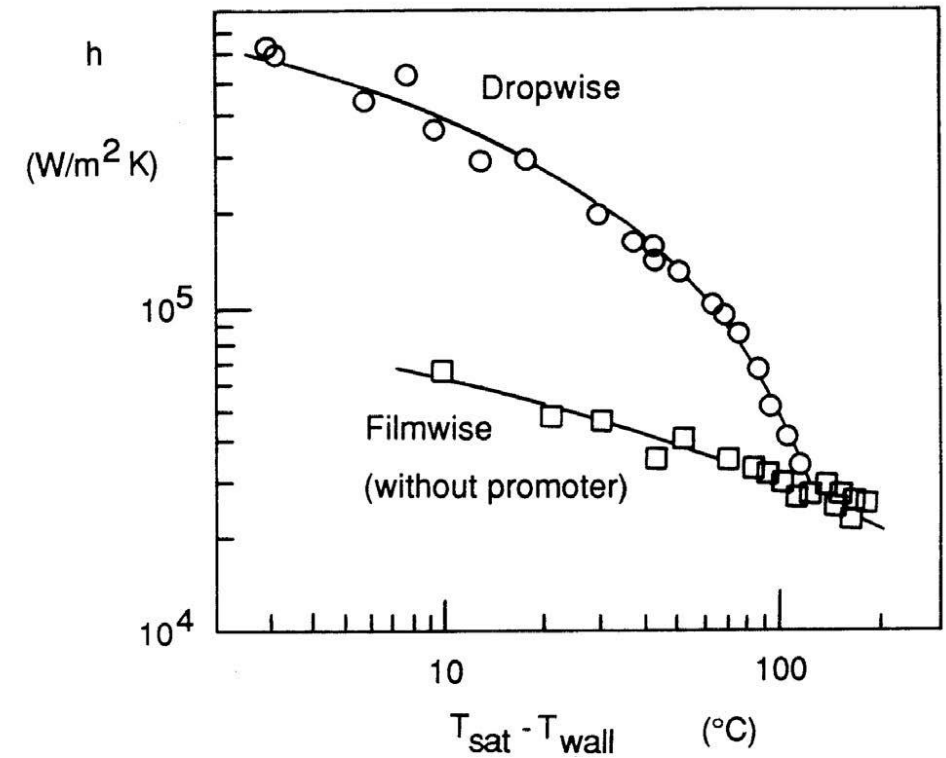
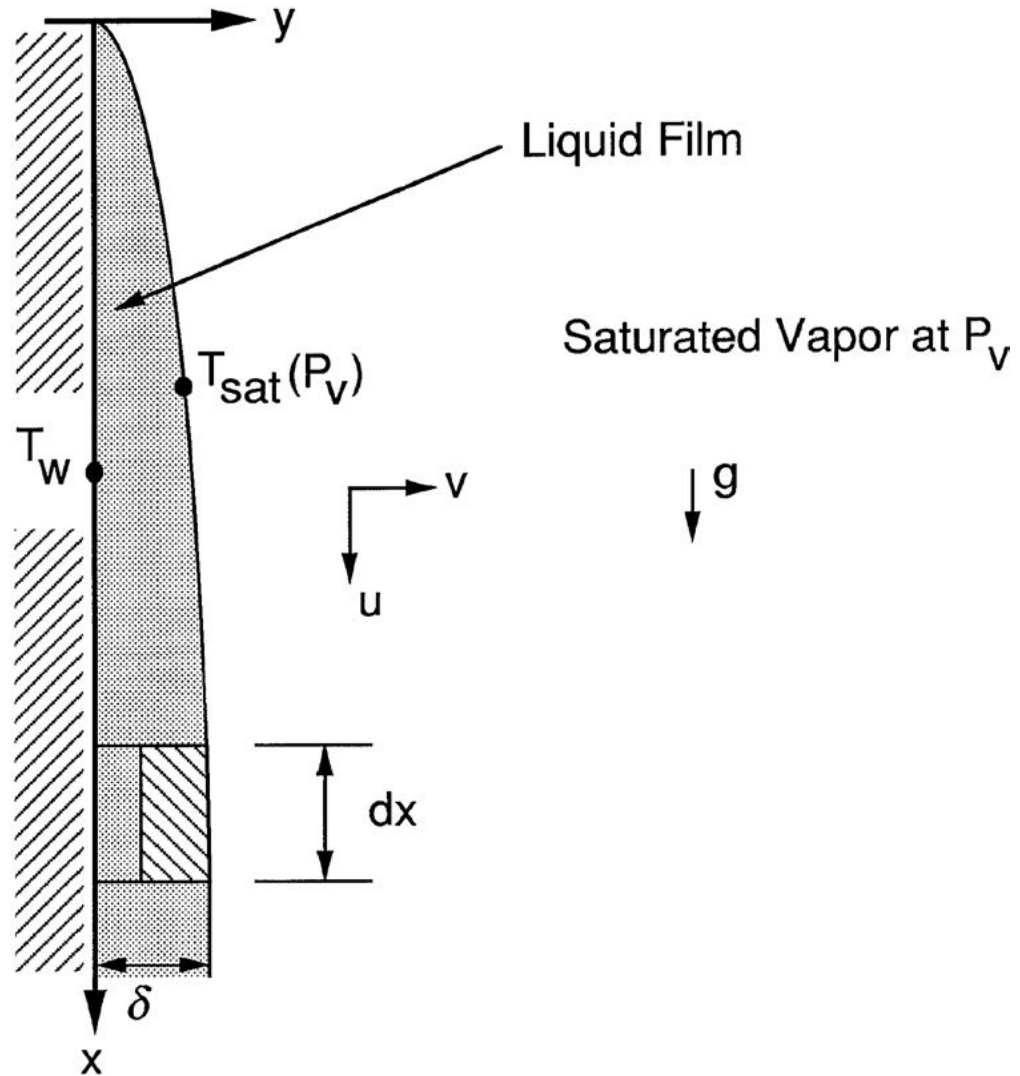
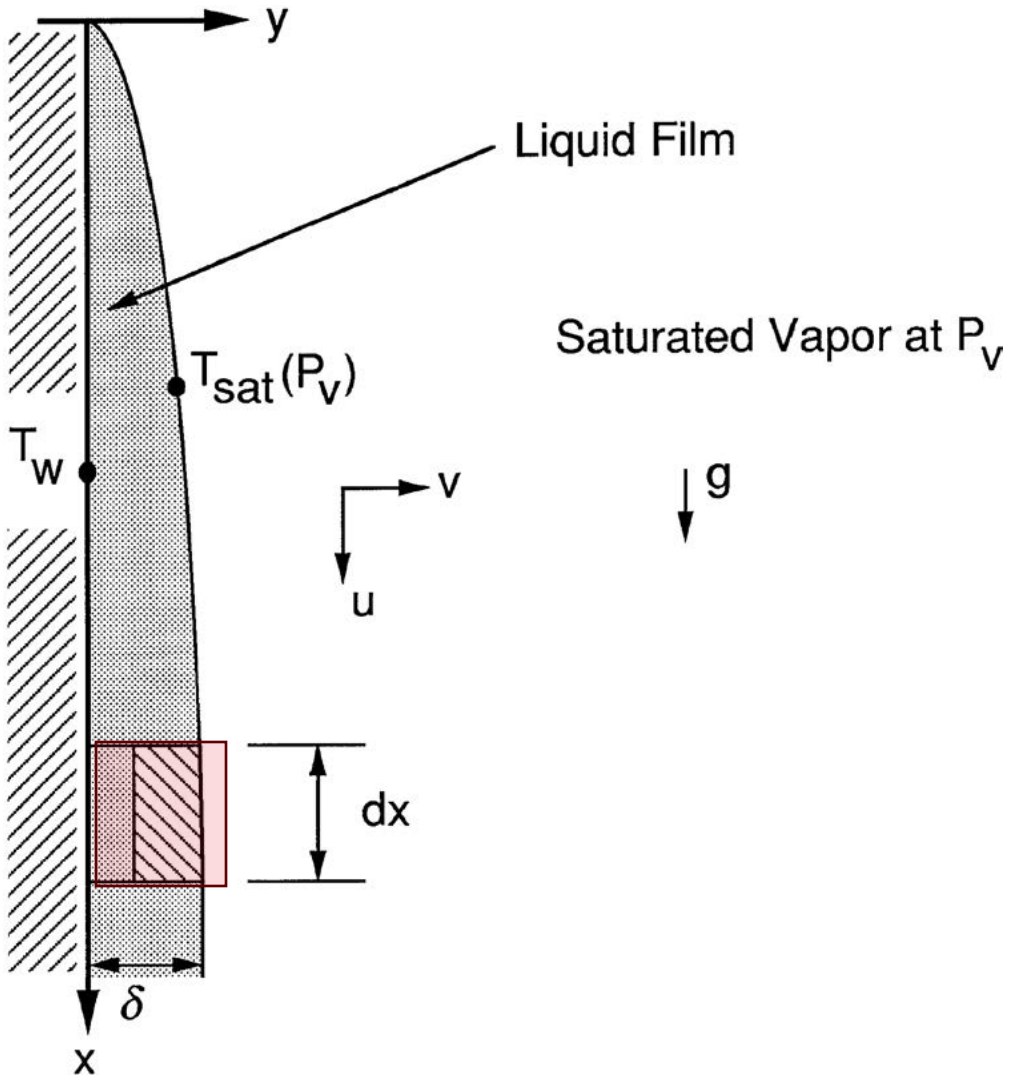


Figure 9.6 in Carey

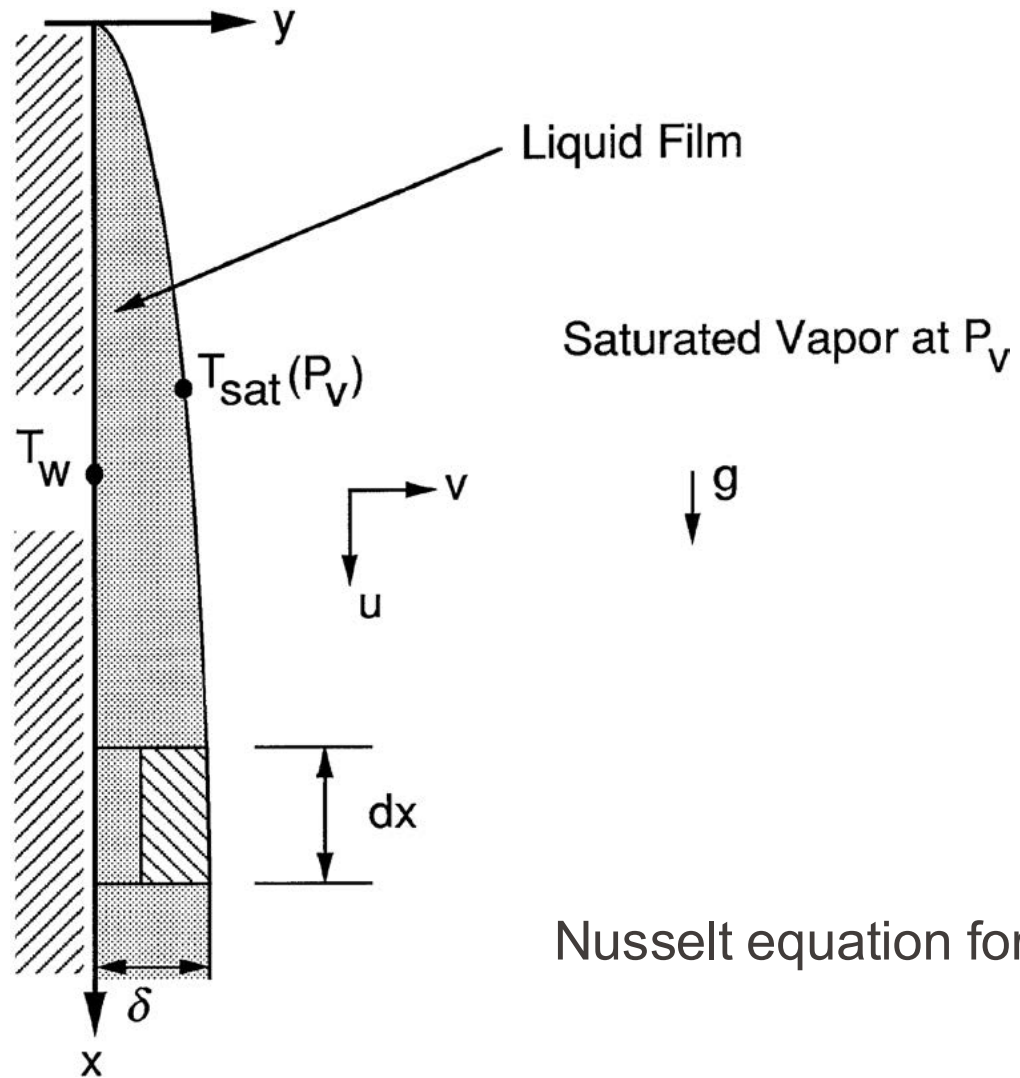
# Filmwise Condensation on a Flat Vertical Surface



# Energy Balance



# Filmwise Condensation HTC



Nusselt equation for filmwise condensation

- Filmwise condensation still the most common operation mode in industry
  - Not as bad as film boiling as liquid is much more conductive than vapor
- Dropwise condensation requires strong hydrophobicity at high supersaturation (droplets must roll off the surface quickly before flooding occurs). However, maintaining strong hydrophobicity over extended period is challenging.