

# ME-251: Thermodynamics and energetics I Second Law IV

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Photo Credit: Trougnouf

<b>Year</b>	2025-2026
<b>Course</b>	Thermodynamics and energetics I
<b>Questionnaire</b>	📄 Indicative feedback of teaching (since 2022-2023)
<b>Nb Registered</b>	345
<b>Nb Answered</b>	87

## The running of the course enables my learning and an appropriate class climate



- Structure and pacing
- Use of CoolProp for exercise problems
- Other improvement suggestions (freezing previous slides, class noise level, writing clarity)

- Entropy balance for closed systems

# Entropy Balance for Closed Systems

change in the amount of entropy contained within the system during some time interval

=

net amount of entropy transferred in across the system boundary during the time interval

+

amount of entropy produced within the system during the time interval

$$S_2 - S_1 = \int_1^2 \left( \frac{\delta Q}{T} \right)_b + \sigma \quad \sigma \geq 0$$

entropy change
entropy transfer
entropy production

**Entropy balance is a re-interpretation of Clausius inequality**

***Entropy production is non-negative for any system***

For an isolated system (closed system with no heat transfer), entropy can only go up or stay the same

Internally reversible system:  $\sigma = 0$

$$\frac{dS}{dt} = \sum_j \frac{\dot{Q}_j}{T_j} + \dot{\sigma}$$

$dS/dt$ : rate of change of system entropy

$\frac{\dot{Q}_j}{T_j}$ : entropy transfer rate through the part of boundary that has temperature  $T_j$

$\dot{\sigma}$ : entropy generation rate in the system



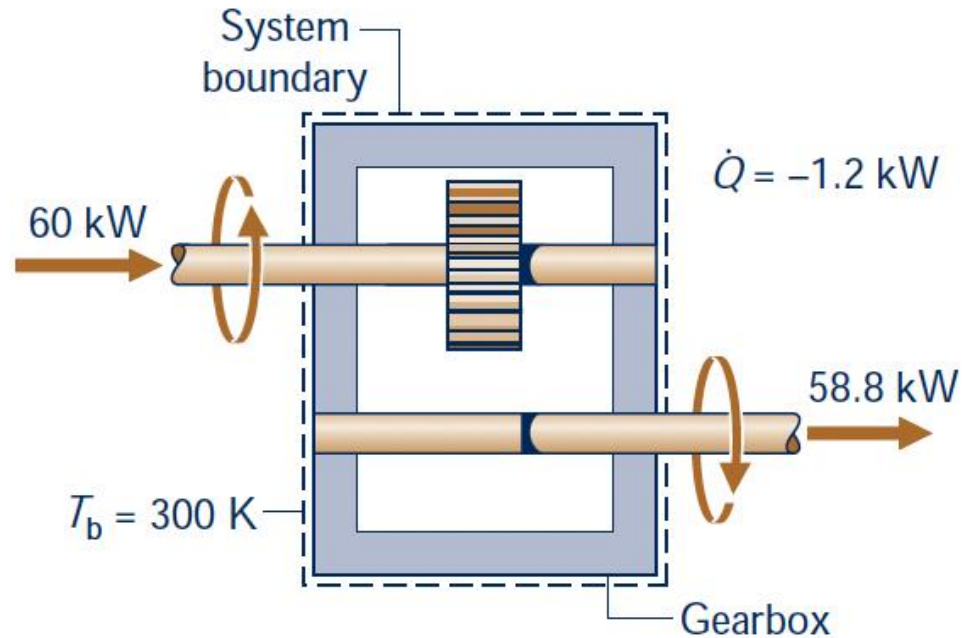
Typical tabulated entropy values: a certain reference state is chosen (for water, the entropy of saturated liquid at 0.01 °C is set to zero); **only entropy difference has physical meanings in the tabulated data**

In two-phase region, we use the quality  $x$  in the same way as we retrieve  $u$ ,  $h$ ,  $v$ , *etc.*

$$s_x(T) = s_f(T)(1 - x) + s_g(T)x$$

saturation                  saturation  
liquid                          vapor

A gearbox operates at steady state

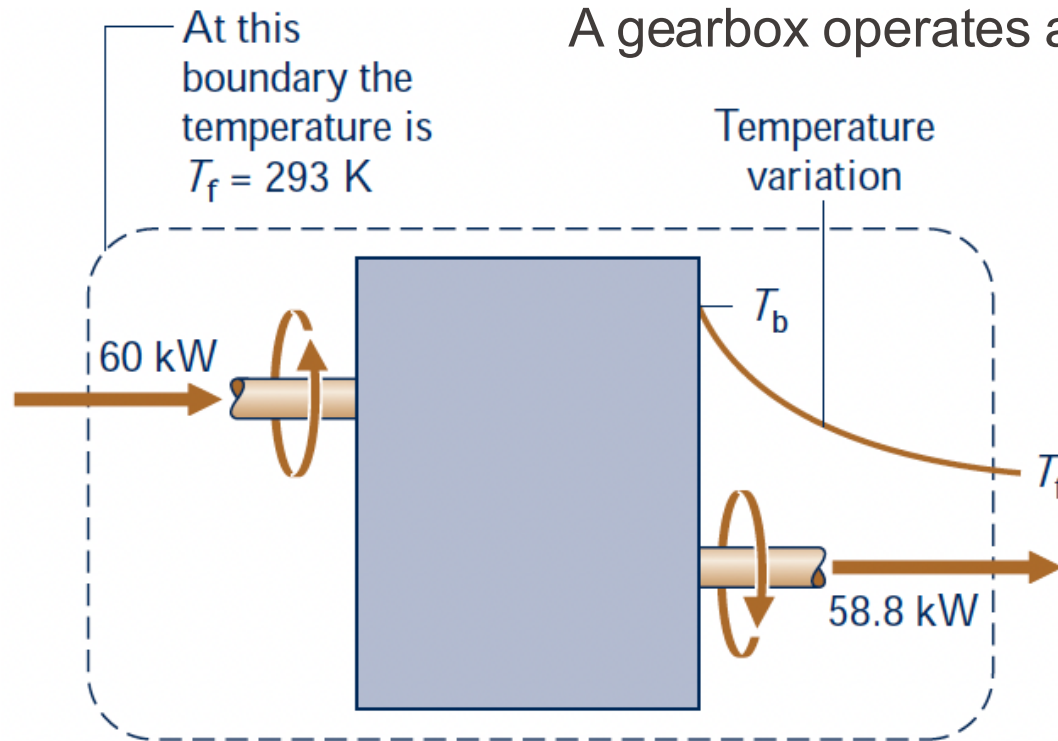


Evaluate the entropy production rate  $\dot{\sigma}_g$  inside the gearbox

$$\frac{dS}{dt} = \frac{\dot{Q}}{T_b} + \dot{\sigma}_g$$

$$\dot{\sigma}_g = -\frac{\dot{Q}}{T_b} = 4 \left[ \frac{\text{W}}{\text{K}} \right]$$

Irreversibility within the gearbox



Evaluate the entropy production rate  $\dot{\sigma}_s$  for an enlarged system consisting of the gearbox and enough of its surroundings such that the new boundary has temperature  $T_f = 293 \text{ K}$

$$\frac{dS}{dt} = \frac{\dot{Q}}{T_f} + \dot{\sigma}_s$$

$$\dot{\sigma}_s = -\frac{\dot{Q}}{T_f} = 4.1 \left[ \frac{\text{W}}{\text{K}} \right]$$

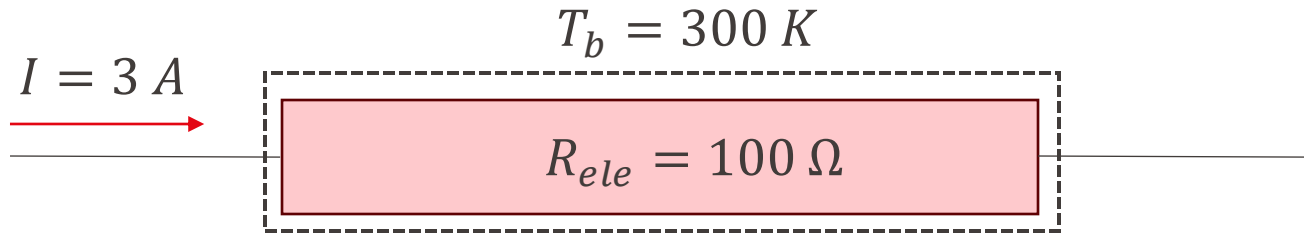
Irreversibility within the gearbox and associated with heat transfer to ambient

Scan the QR code with  
your device  
Or go to [echo360poll.eu](https://echo360poll.eu)



Enter Code

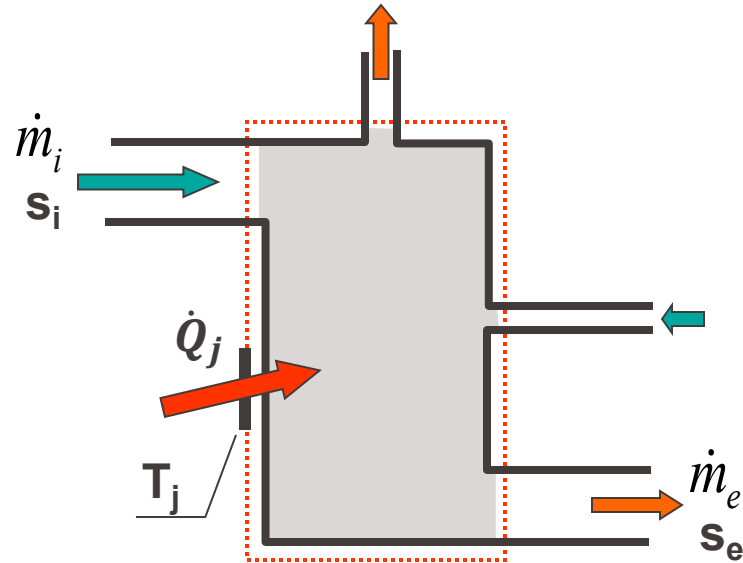
**luepfl**



Assuming steady-state, what is the entropy generation rate in the CV drawn around the resistor

- A. 0 [W/K]
- B. 1 [W/K]
- C. 3 [W/K]**
- D. I don't know

- Open system entropy rate balance
  - Isentropic process and isentropic efficiency
  - Internally reversible, steady-state flow processes
- Reading: 6.9-6.13



$$\frac{dm_{cv}}{dt} = \sum_i \dot{m}_i - \sum_e \dot{m}_e$$

$$\frac{dE_{cv}}{dt} = \sum_j \dot{Q}_j - \dot{W}_{cv} + \sum_i \dot{m}_i \left( h_i + \frac{\vec{V}_i^2}{2} + gz_i \right) - \sum_e \dot{m}_e \left( h_e + \frac{\vec{V}_e^2}{2} + gz_e \right)$$

$$\frac{dS_{cv}}{dt} = \sum_j \frac{\dot{Q}_j}{T_j} + \underbrace{\sum_i \dot{m}_i s_i - \sum_e \dot{m}_e s_e}_{\text{Entropy transfer}} + \underbrace{\dot{\sigma}_{cv}}_{\text{Entropy generation}} \quad [\text{W/K}]$$

Entropy change

Entropy transfer

Entropy generation

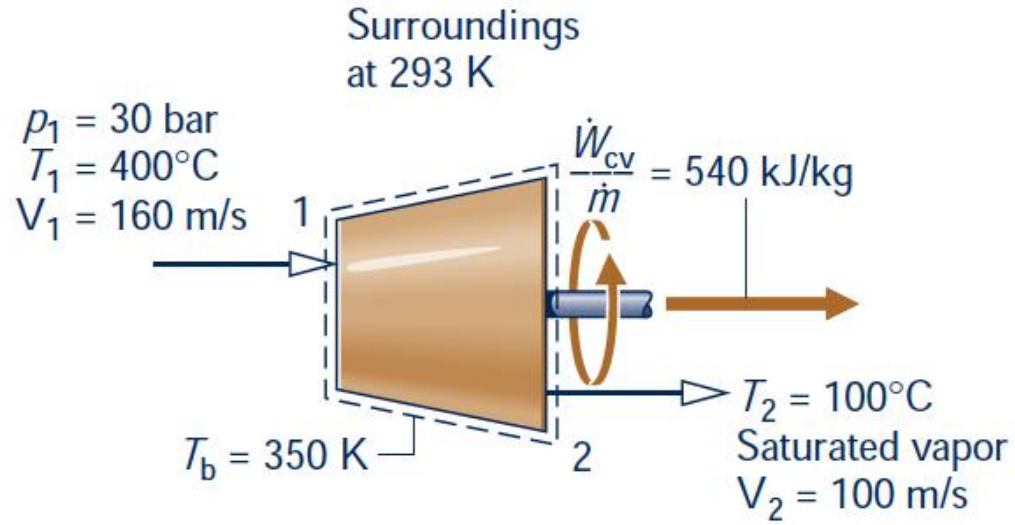
$$\cancel{\frac{dS_{cv}}{dt}} = \sum_j \frac{\dot{Q}_j}{T_j} + \sum_i \dot{m}_i s_i - \sum_e \dot{m}_e s_e + \dot{\sigma}_{cv}$$

One inlet (1), one exit (2) control volume

$$0 = \sum_j \frac{\dot{Q}_j}{T_j} + \dot{m}(s_1 - s_2) + \dot{\sigma}_{cv}$$

$$0 = \frac{1}{\dot{m}} \sum_j \frac{\dot{Q}_j}{T_j} + s_1 - s_2 + \frac{\dot{\sigma}_{cv}}{\dot{m}}$$

1 bar =  $10^5$  Pa

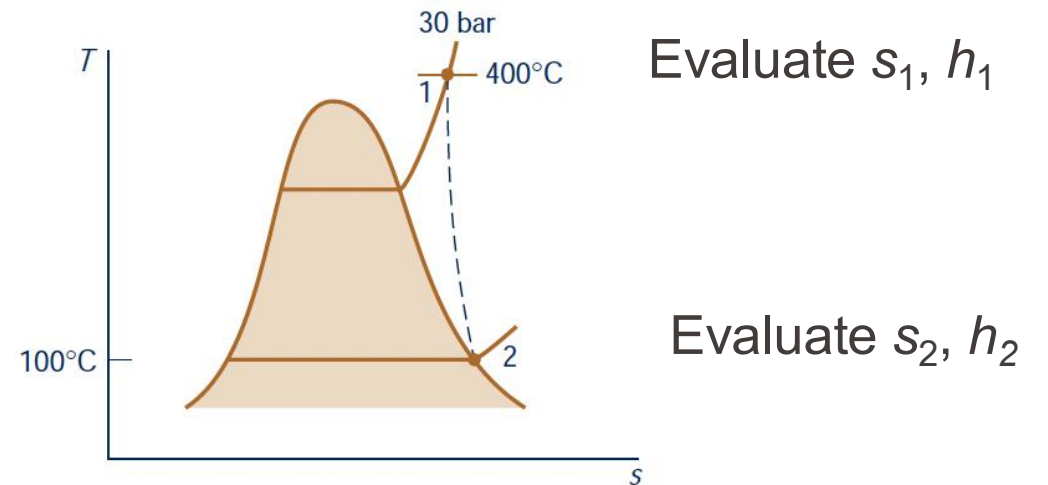


$$0 = \frac{\dot{Q}_{cv}}{\dot{m}T_b} + (s_1 - s_2) + \frac{\dot{\sigma}_{cv}}{\dot{m}}$$

$$0 = \frac{\dot{Q}_{cv}}{\dot{m}} - \frac{\dot{W}_{cv}}{\dot{m}} + \left( h_1 + \frac{\vec{V}_1^2}{2} - h_2 - \frac{\vec{V}_2^2}{2} \right)$$

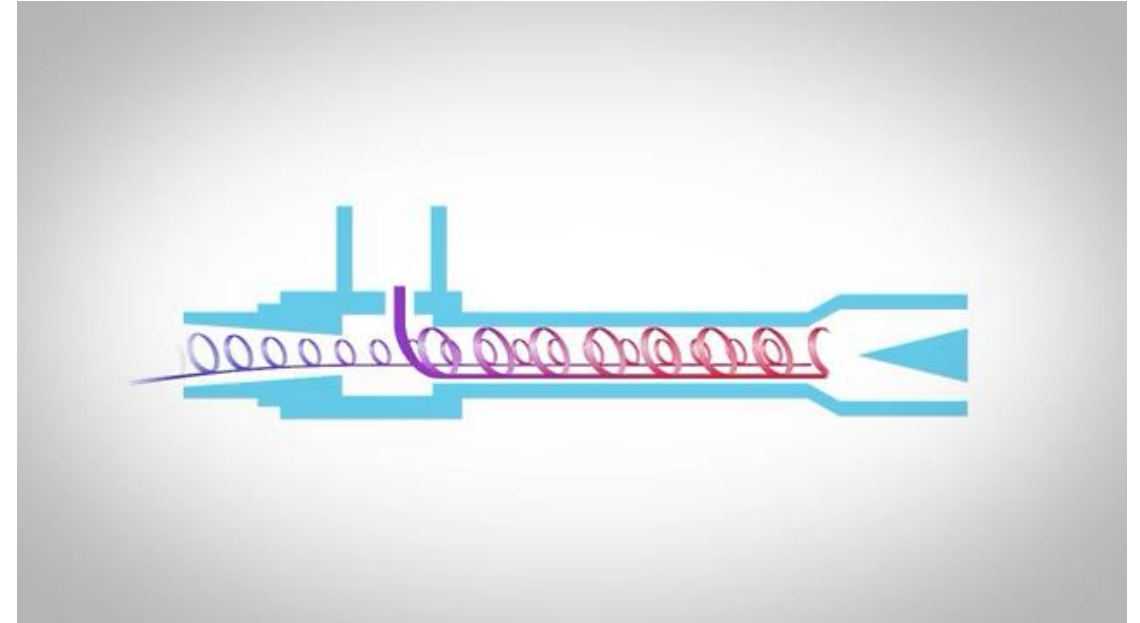
Steam expands through a turbine at steady state (change of potential energy ignored)

Determine the entropy generation per kg of steam flowing through turbine  $\frac{\dot{\sigma}_{cv}}{\dot{m}}$



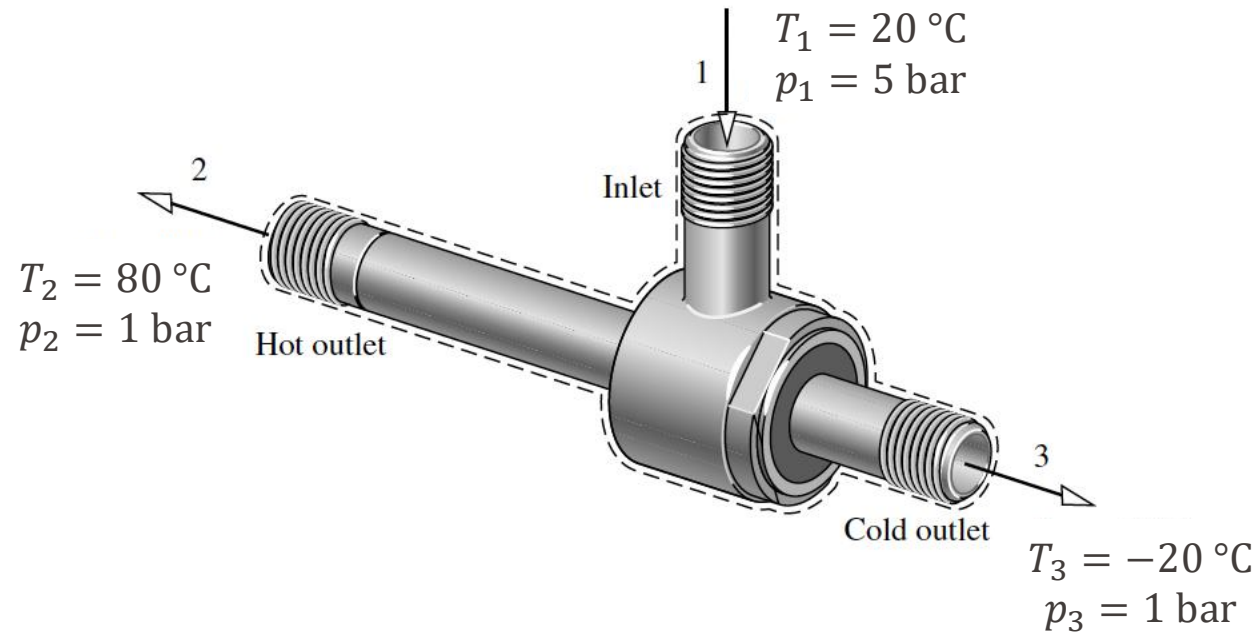


<https://www.youtube.com/watch?v=ZpMe193KoHM>



[https://www.youtube.com/watch?v=Q\\_y2FvH2DHE](https://www.youtube.com/watch?v=Q_y2FvH2DHE)

# Example 6.7 (number adjusted)



*Evaluate this invention claim*

A device producing hot and cold stream of air flows without additional external heat or work transfer at a steady state?

Model air as perfect gas with  $c_p = 3.5R$

**Does this invention violate 2<sup>nd</sup> law?**

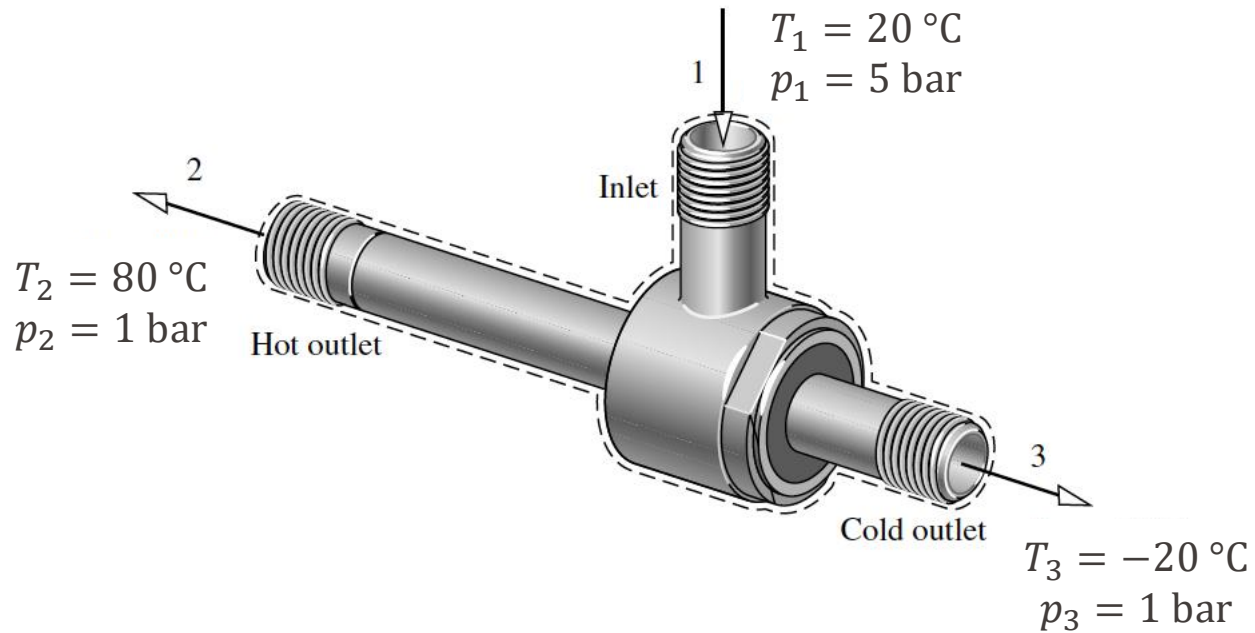
$$\dot{m}_1 = \dot{m}_2 + \dot{m}_3$$

$$0 = \dot{m}_1 h_1 - (\dot{m}_2 h_2 + \dot{m}_3 h_3)$$

$$0 = \dot{m}_2 (h_2 - h_1) + \dot{m}_3 (h_3 - h_1)$$

$$0 = \dot{m}_2 c_p (T_2 - T_1) + \dot{m}_3 c_p (T_3 - T_1)$$

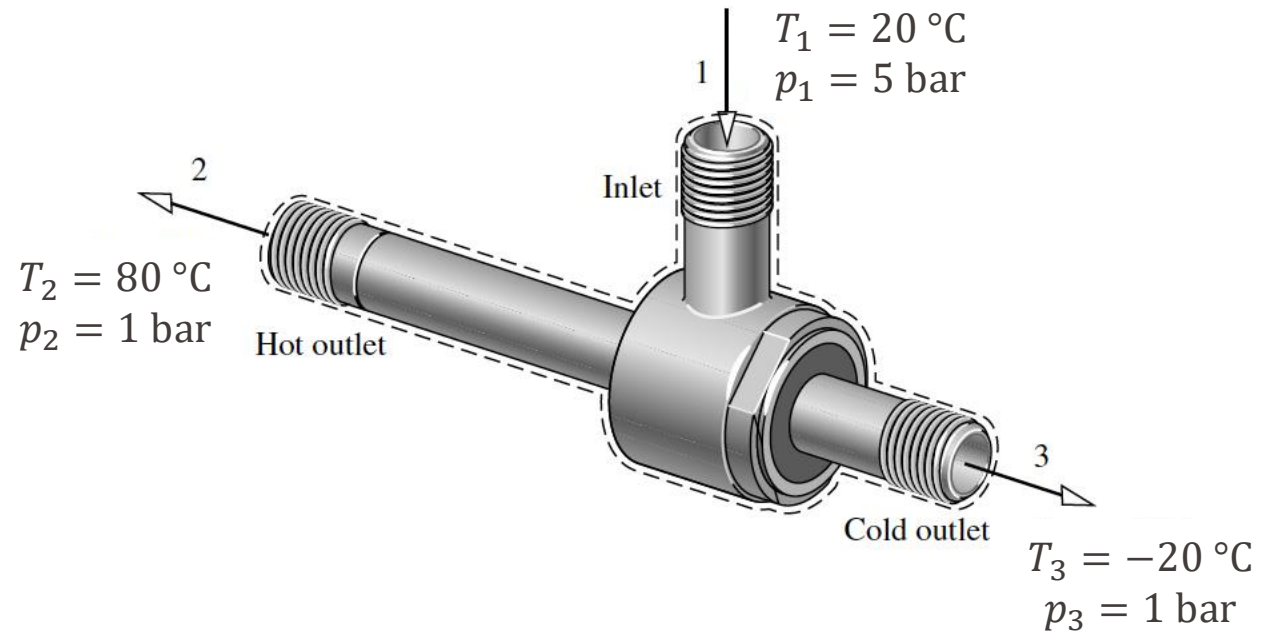
$$\dot{m}_2 : \dot{m}_3 : \dot{m}_1 = 40 : 60 : 100$$



$$0 = \dot{m}_1 s_1 - \dot{m}_2 s_2 - \dot{m}_3 s_3 + \dot{\sigma}_{cv}$$

$$\dot{\sigma}_{cv} = \dot{m}_2 s_2 + \dot{m}_3 s_3 - (\dot{m}_2 + \dot{m}_3) s_1$$

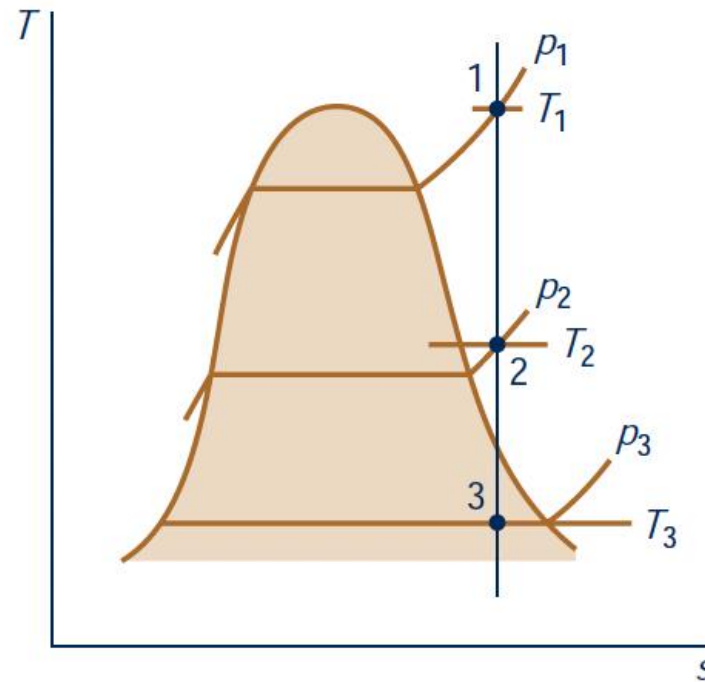
$$\begin{aligned}
 \frac{\dot{\sigma}_{cv}}{\dot{m}_1} &= 0.4(s_2 - s_1) + 0.6(s_3 - s_1) \\
 &= 0.4 \left[ c_p \ln \left( \frac{T_2}{T_1} \right) - R \ln \left( \frac{p_2}{p_1} \right) \right] + 0.6 \left[ c_p \ln \left( \frac{T_3}{T_1} \right) - R \ln \left( \frac{p_3}{p_1} \right) \right] \\
 &= 0.4R \left[ 3.5 \ln \frac{353.15}{293.15} - \ln \frac{1}{5} \right] + 0.6R \left[ 3.5 \ln \frac{253.15}{293.15} - \ln \frac{1}{5} \right] \approx 1.56R > 0
 \end{aligned}$$

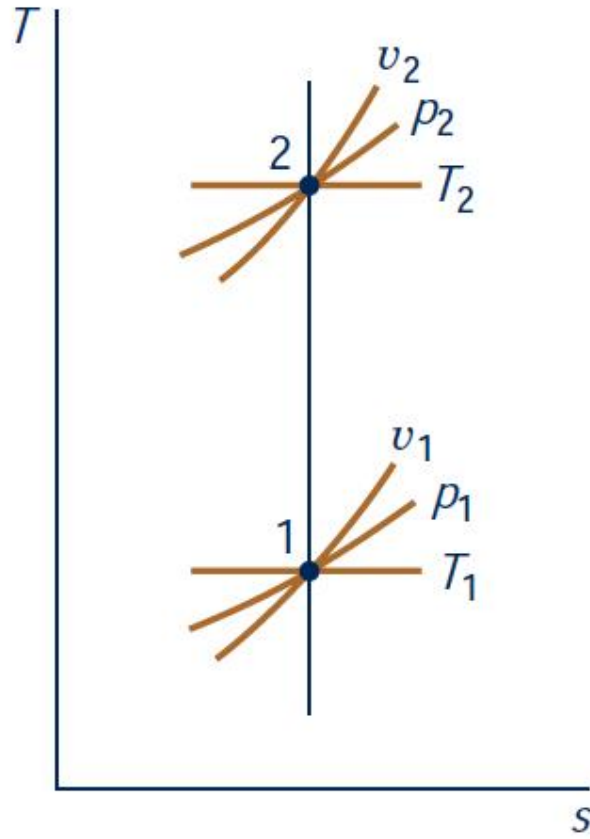


How is this possible for a steady-state device to “de-mix” air flows without additional external heat or work transfer ?

We “sacrificed” the pressure

Isentropic means constant entropy (typically constant specific entropy)





$$s_2 - s_1 = c_v \ln \frac{T_2}{T_1} + R \ln \frac{v_2}{v_1} = 0$$

$$\Rightarrow \frac{T_2}{T_1} = \left( \frac{v_1}{v_2} \right)^{k-1}$$

$$s_2 - s_1 = c_p \ln \frac{T_2}{T_1} - R \ln \frac{p_2}{p_1} = 0$$

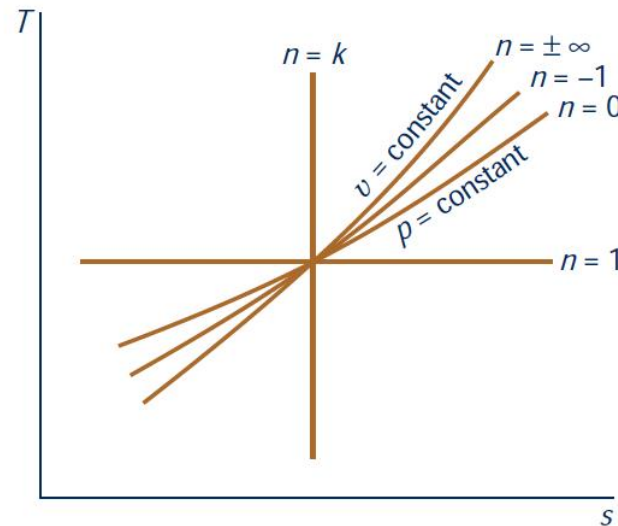
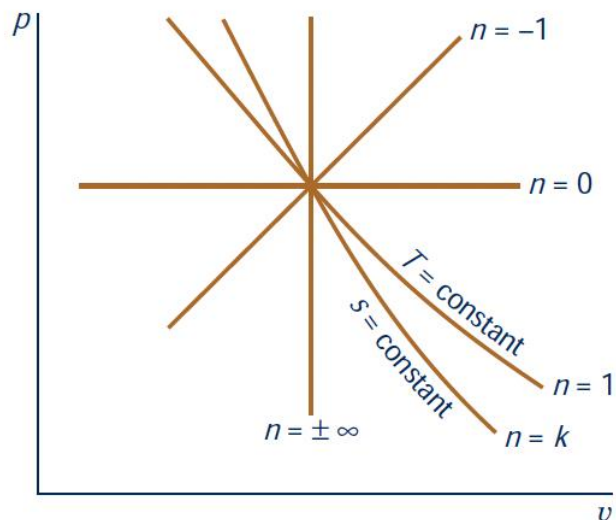
$$\Rightarrow \frac{T_2}{T_1} = \left( \frac{p_2}{p_1} \right)^{\frac{k-1}{k}}$$

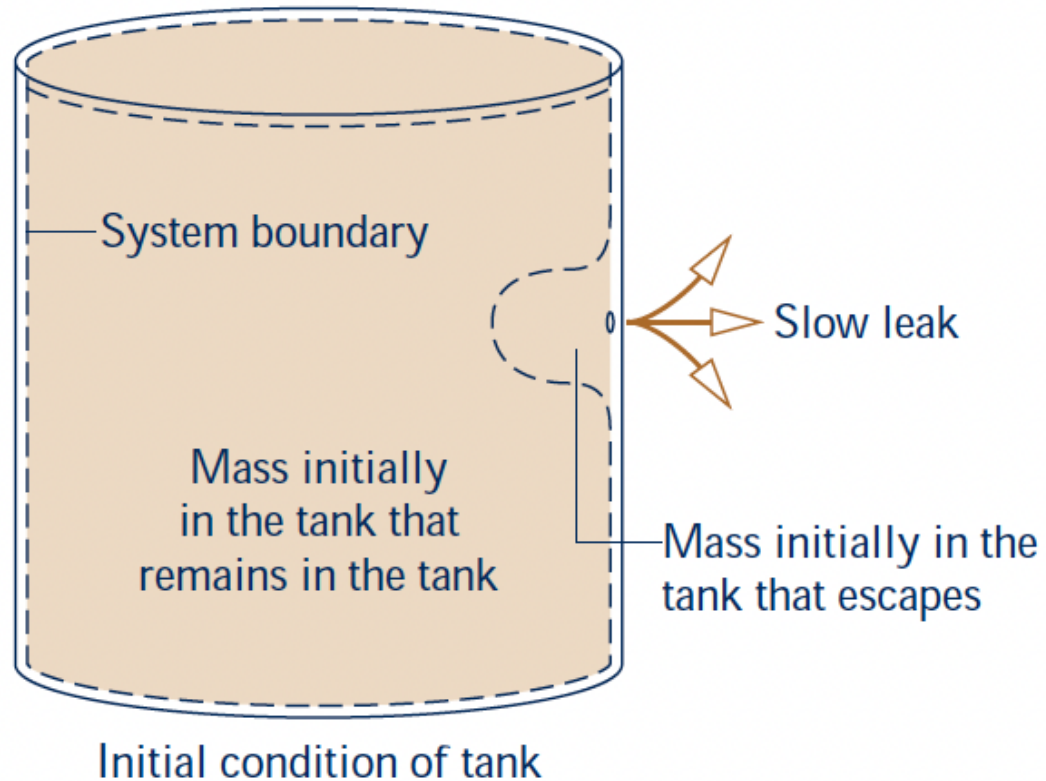
$$p_1 v_1^k = p_2 v_2^k$$

$$p v^k = \text{const}$$

$$k = \frac{c_p}{c_v} = \frac{c_v + R}{c_v}$$

- A process where  $pv^n = \text{const}$ 
  - Isobaric process:  $p = \text{const} \Rightarrow n = 0$
  - Isothermal process:  $pv = RT = \text{const} \Rightarrow n = 1$
  - Isochoric processes:  $v = \text{const} \Rightarrow n = +\infty$  ( $p^{1/n}v = \text{const}$ )
  - Isentropic process:  $pv^k = \text{const} \Rightarrow n = k = c_p/c_v$



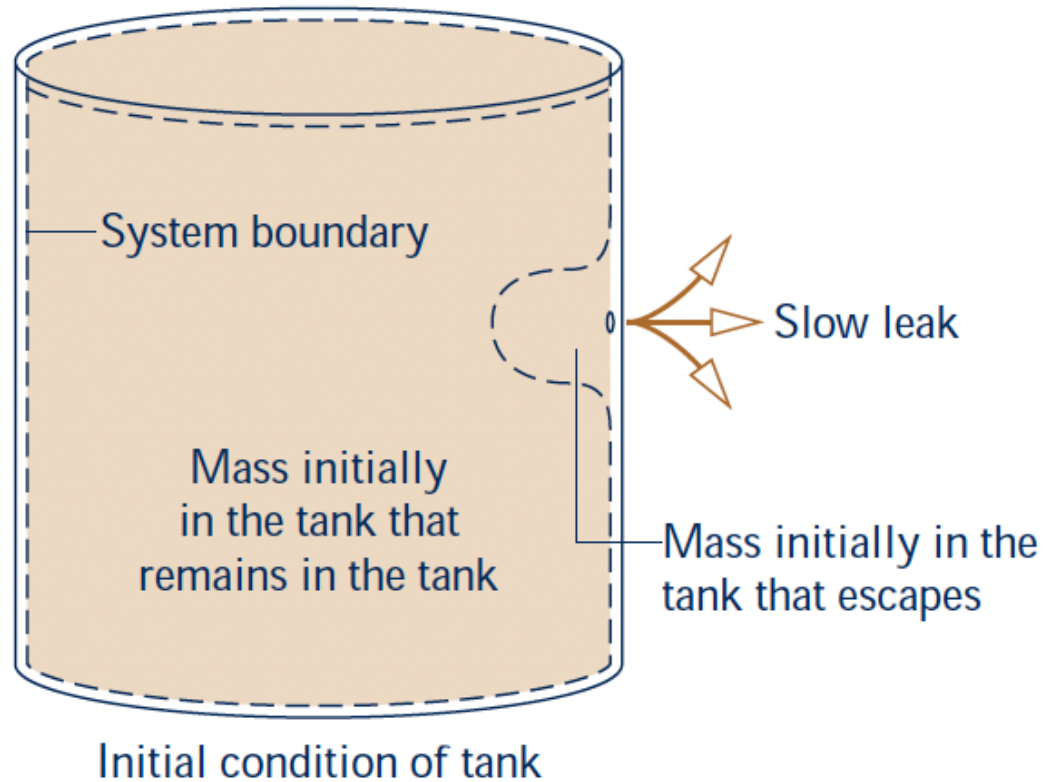


Air leaking from a rigid tank insulated from the surrounding (initially 5 kg, 5 bar, 500 K)

Air slowly escapes with **no internal irreversibility** inside the tank, until reaching 1 bar

Assume perfect gas behavior,  $c_p = 3.5R$

**Determine the mass of the remaining air in the tank and its temperature**



$$\frac{dS}{dt} = \sum_j \frac{\dot{Q}_j}{T_j} + \dot{\sigma}$$

$$k = \frac{c_p}{c_v} = \frac{c_p}{c_p - R} = 1.4$$

$$\frac{T_2}{T_1} = \left( \frac{p_2}{p_1} \right)^{\frac{k-1}{k}} \Rightarrow T_2 = 316 \text{ K}$$

$$p_1 v_1^k = p_2 v_2^k$$

$$\Rightarrow \frac{m_2}{m_1} = \frac{V_{\text{tank}}/v_2}{V_{\text{tank}}/v_1} = \frac{v_1}{v_2} = \left( \frac{p_2}{p_1} \right)^{\frac{1}{k}} \Rightarrow m_2 = 1.58 \text{ kg}$$