

The background image is a composite of two scenes. On the left, a traditional wooden windmill stands on a grassy bank next to a paved path. In the background, several high-voltage electrical transmission towers are visible against a blue sky with light clouds. On the right, a large industrial power plant is shown, featuring a prominent, tall, cylindrical cooling tower. The sky here is a mix of blue and light orange, suggesting a sunset or sunrise. A semi-transparent red rectangular box is overlaid on the right side of the image, containing the course title in white text.

ME-251: Thermodynamics and energetics I

First Law for Open Systems

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Photo Credit: Trougnouf

- Estimate thermodynamic properties in special cases

- Incompressible substance
- Ideal gas

○ Reading: Moran 3.10-3.15

- Liquids and solids are often modeled as incompressible

- $\frac{pv}{RT} = \frac{p\tilde{v}}{\tilde{R}T} = Z \equiv 1$, the dilute limit of real gases

T : temperature [K]

\tilde{R} : universal gas constant [J/K/mol]

R : specific gas constant [J/K/kg]

\tilde{v} : molar volume [m³/mol]

$$c_v = \text{const}, \quad c_p = c_v + R$$

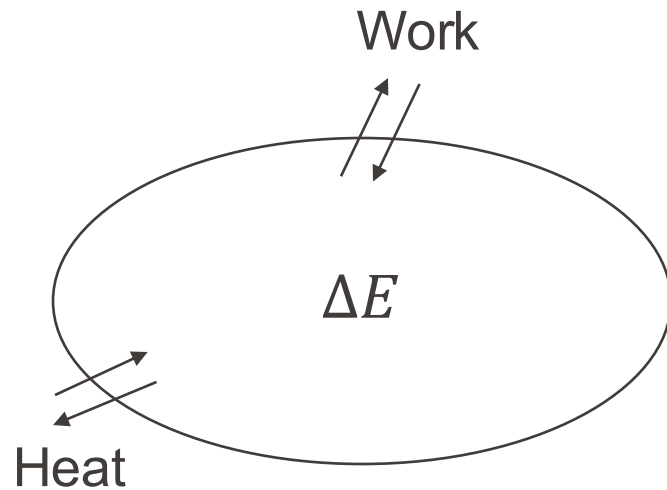
Common approximations:

Monoatomic $c_v = 1.5R$

Diatomic $c_v = 2.5 R$

Polyatomic: $c_v = 3R$

- Understand the difference between the analyses of closed and open systems
- Apply mass balance to open systems
- Apply energy balance to open systems
- Reading: Moran 4.1 - 4.5



Can be intuitively understood as energy conservation

Energy change = Energy going in – Energy going out

Energy flows can take the form of heat, work, ...

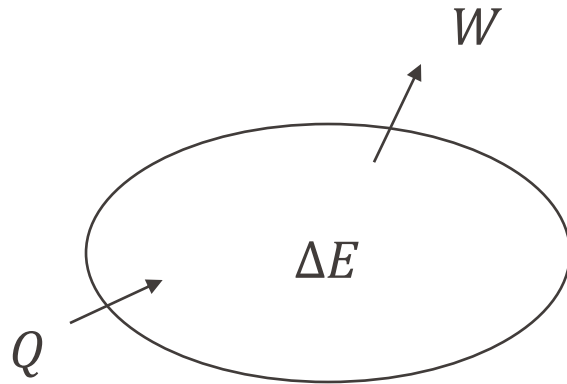
Change in the amount of energy contained within a system during some time interval

=

Net amount of energy transferred **into** the system by heat transfer during the time interval

–

Net amount of energy transferred **out of** the system by work during the time interval



Time Rate Form of First Law for Closed Systems

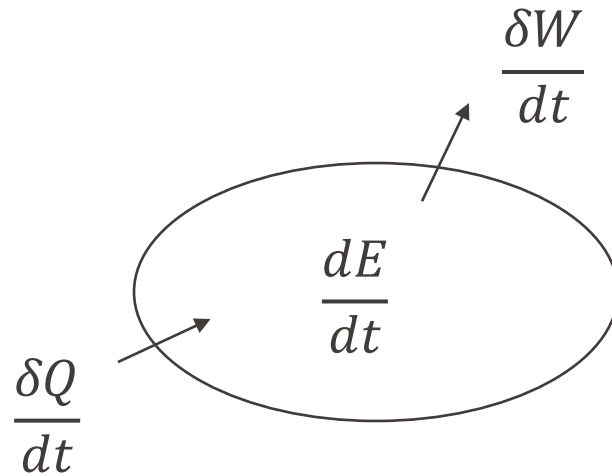
Time rate of change in the amount of energy contained within a system at time t

=

Net rate at which energy is transferred **into** the system by heat transfer at time t

–

Net rate at which energy is transferred **out of** the system by work at time t

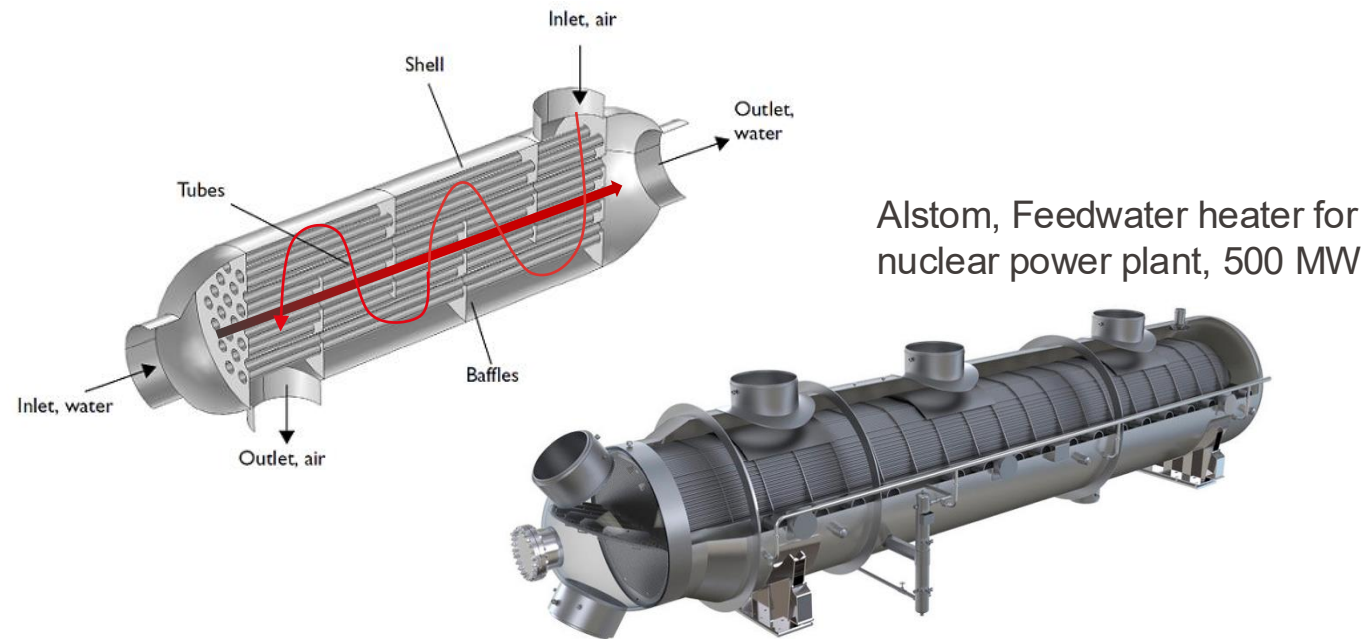


- Type of work transfer
 - Expansion/compression (e.g., polytropic processes)
 - Electrical, spring, shaft,...

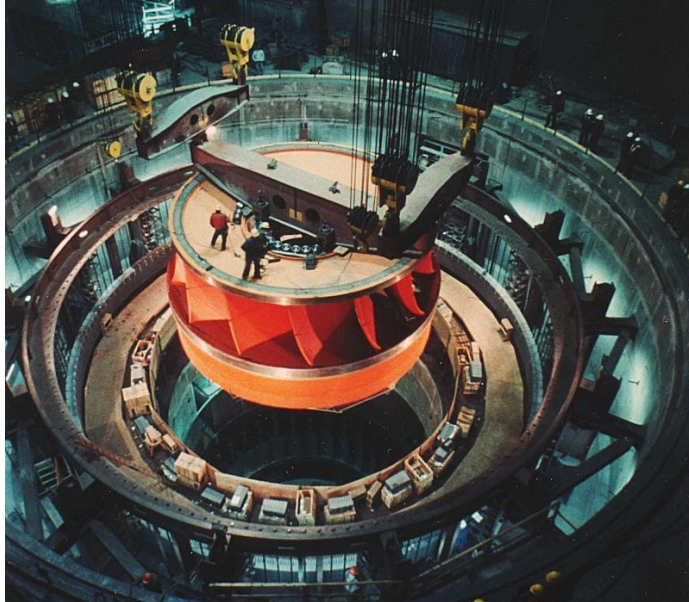
- Type of energy
 - Potential energy
 - Kinetic energy
 - Internal energy (thermal motion, latent, chemical, nuclear,...)

- Type of heat transfer (requires temperature gradient)
 - Conduction, convection, radiation

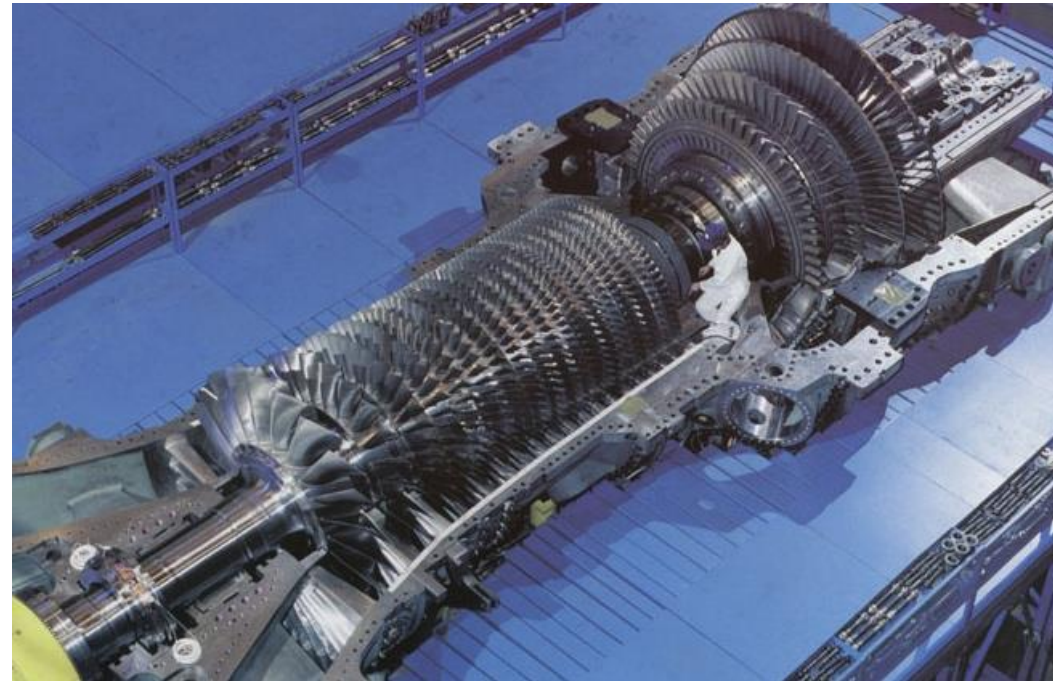
- Open systems: systems with mass transport across the boundary
 - Heat exchangers



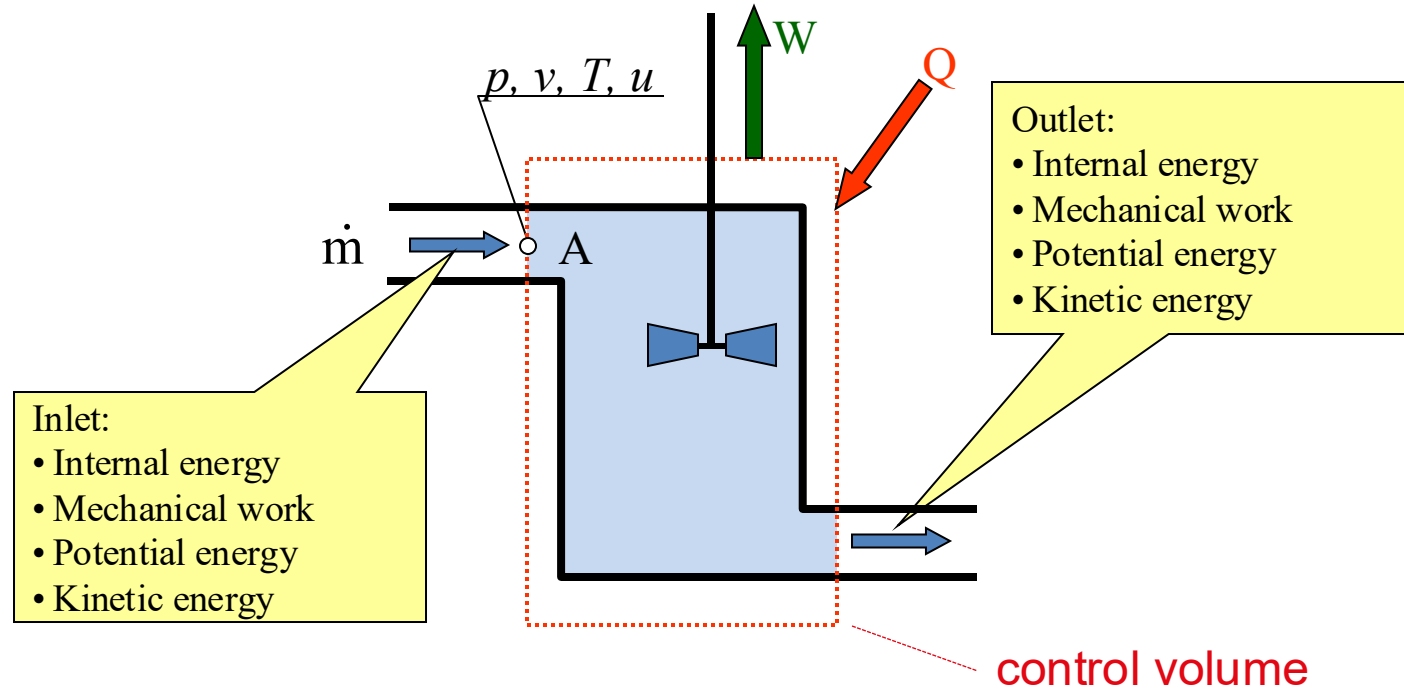
- E.g., turbines



Francis turbine, Grand Coulee Dam,
biggest hydro plant in the US,
capacity 6800 MW, 33 turbines (different sizes)



GE 9H gas turbine, 480 MW



Unlike closed systems, energy transfer can be due to incoming and outgoing mass flow

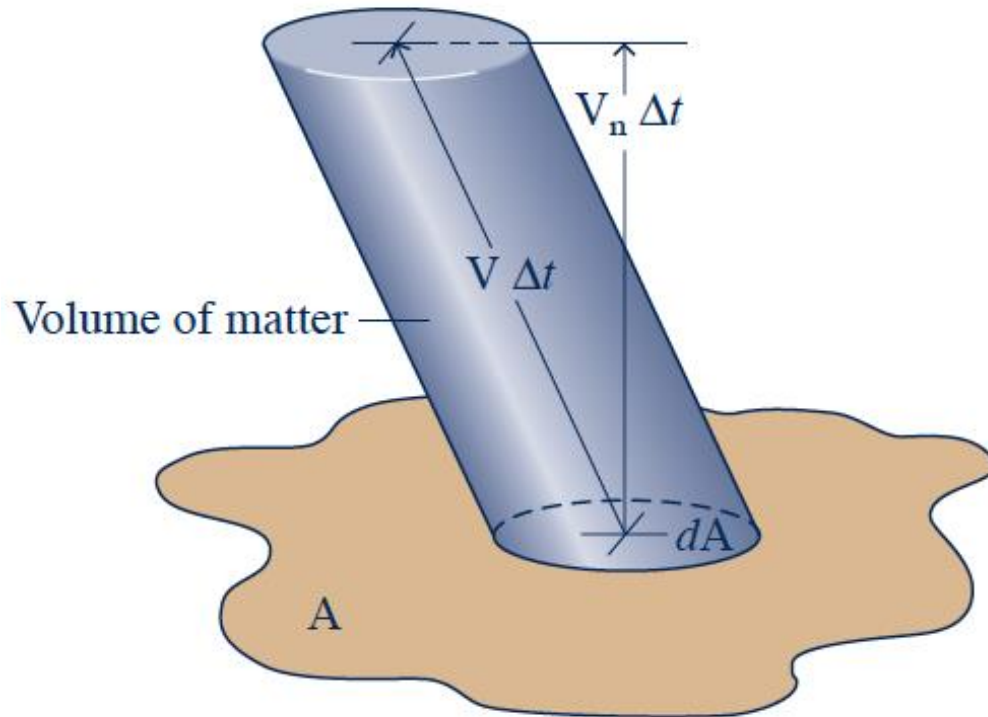
Time rate of change
in the amount of mass
contained within a
control volume at time t

=

Rate at which mass
is transferred **into** the
system at time t

–

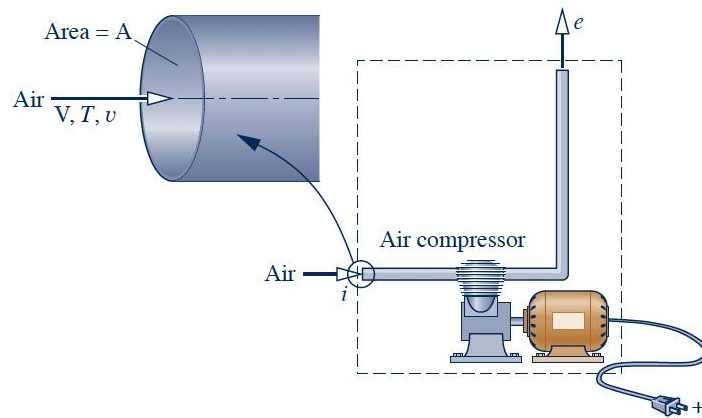
Rate at which mass
is transferred **out of**
the system at time t



- Consider a small quantity of matter flowing with velocity V across an incremental area dA in a small time interval Δt

In Moran, non-italic V represents velocity

- 1D form of the mass rate balance
 - The flow is normal to the boundary at locations where mass enters or exits the control volume ($V = V_n$)
 - Velocity and density are uniform over the specified inlet or exit area



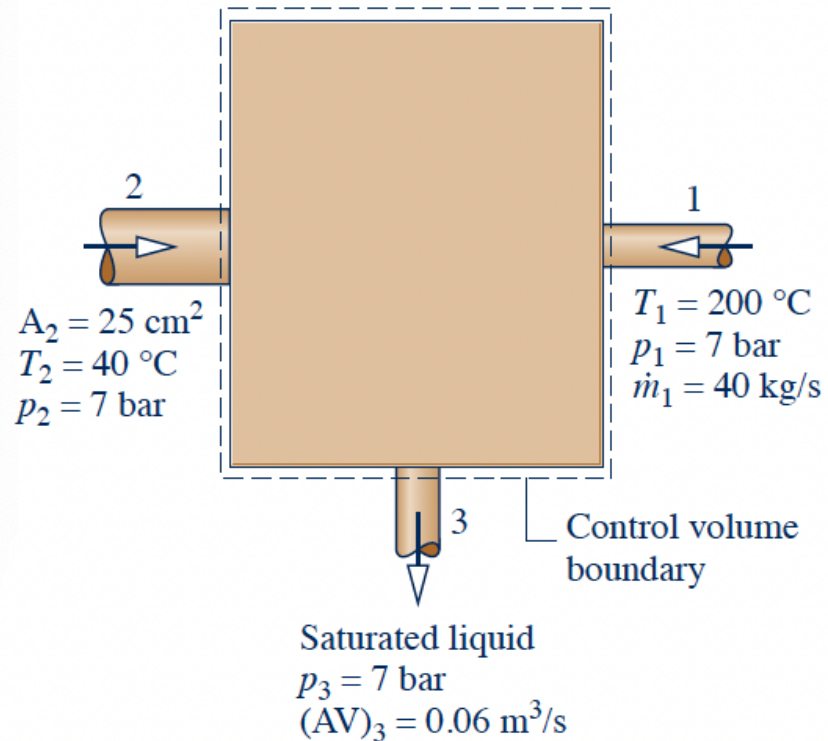
- Steady state: all properties in the control volume are invariant over time

A feedwater heater operating at steady state

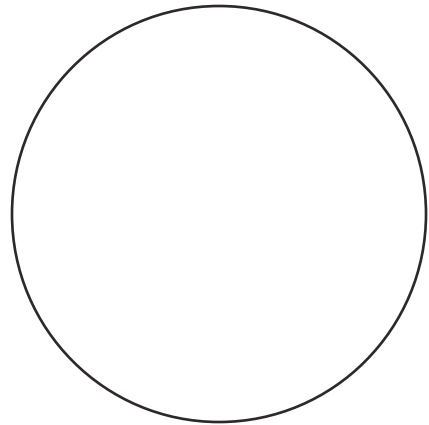
Determine the mass flow rates at inlet 2 and at the exit, and the velocity V_2

$$v_3 = 1.108 \times 10^{-3} \text{ m}^3/\text{kg}$$

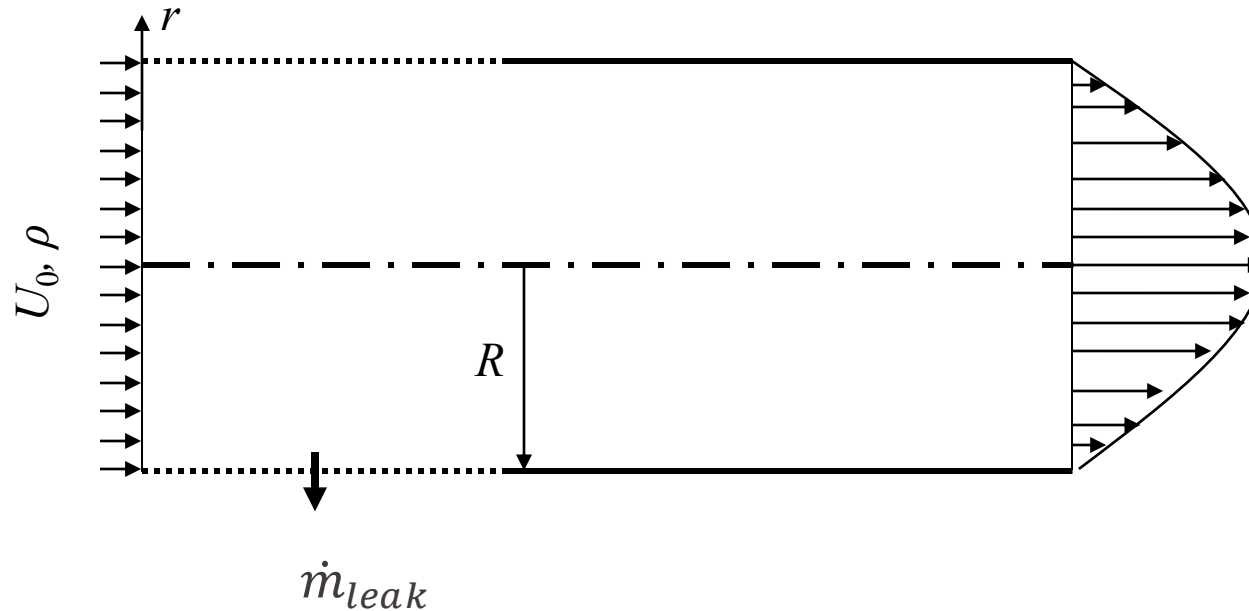
Approximate liquid water as incompressible substance



Example: steady-state pipe flow with steady leakage
(inlet and outlet velocities normal to the surface)

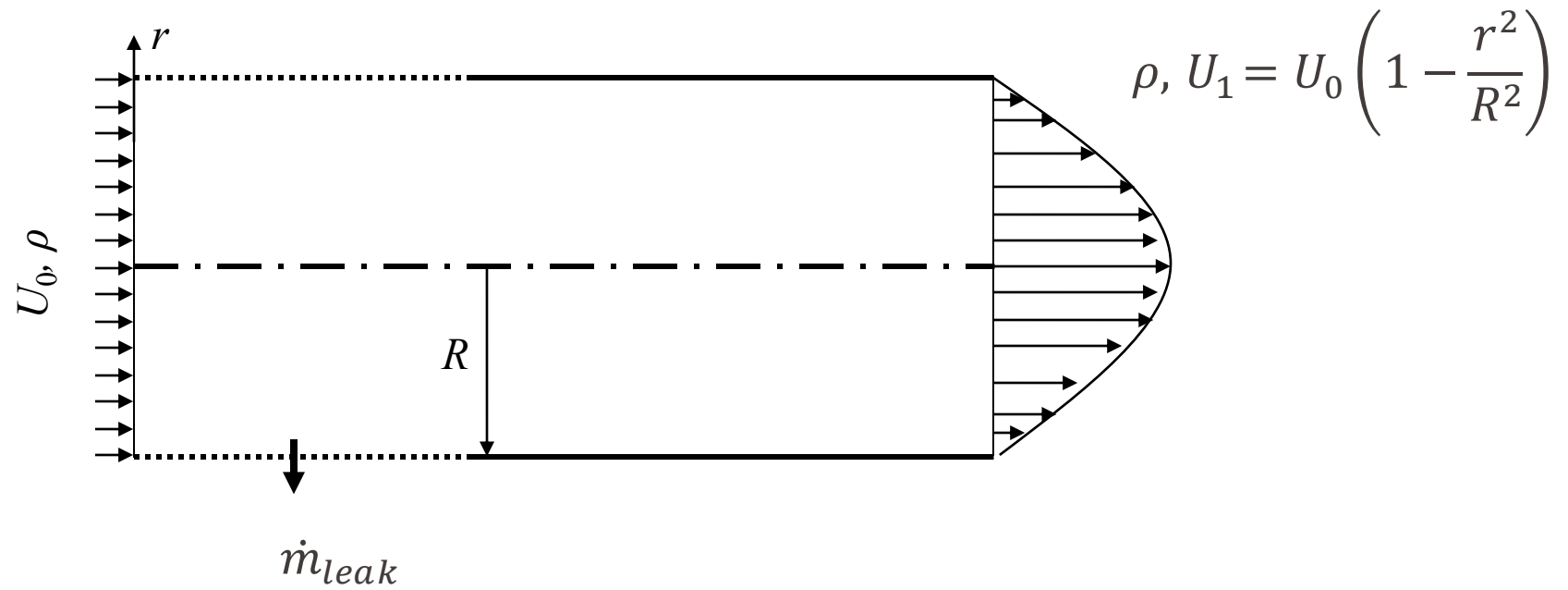
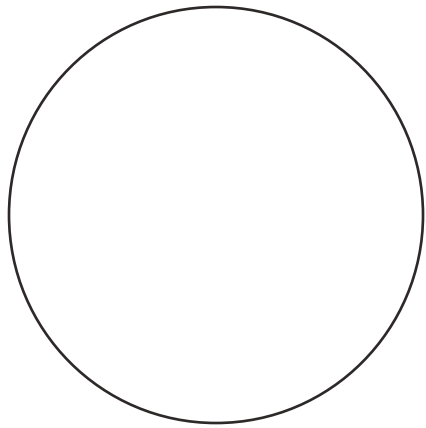


Cross-section

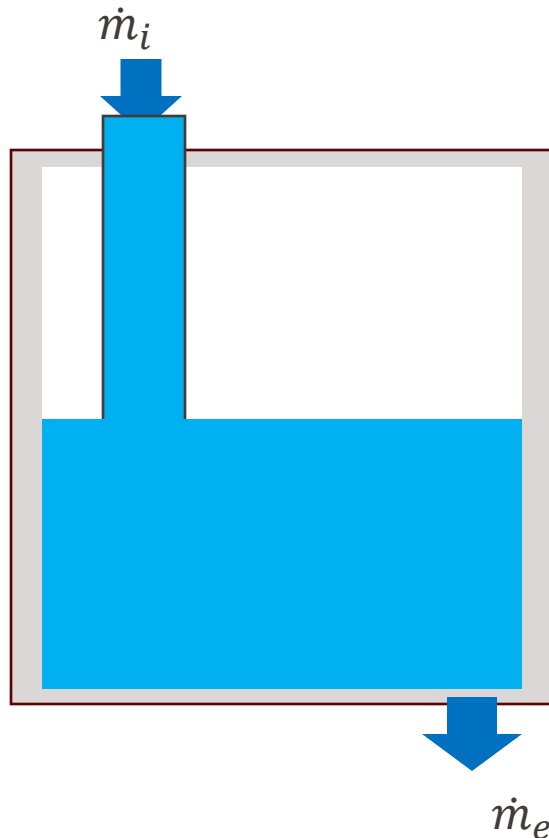


$$\rho, U_1 = U_0 \left(2 - \frac{2r^2}{R^2} \right)$$

Calculate the mass flow rate leaking through the perforated pipe wall



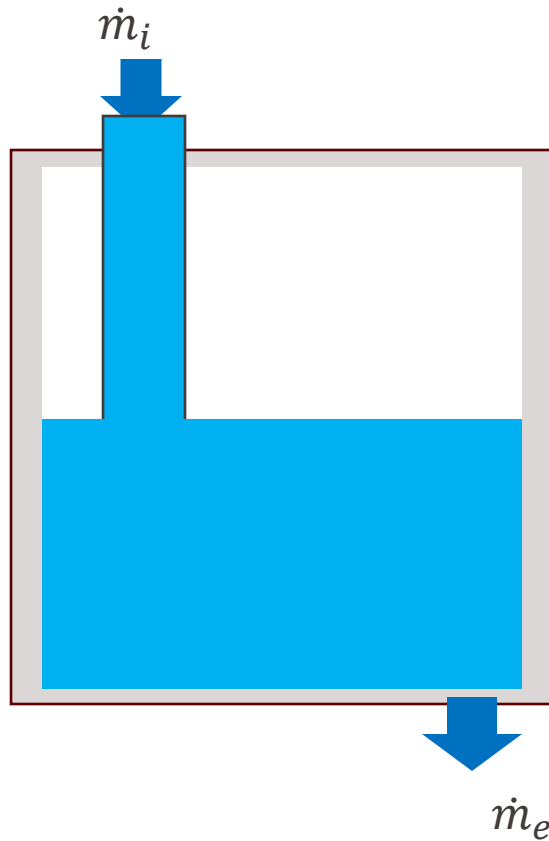
Example 4.2 in Moran (Transient Mass Balance)



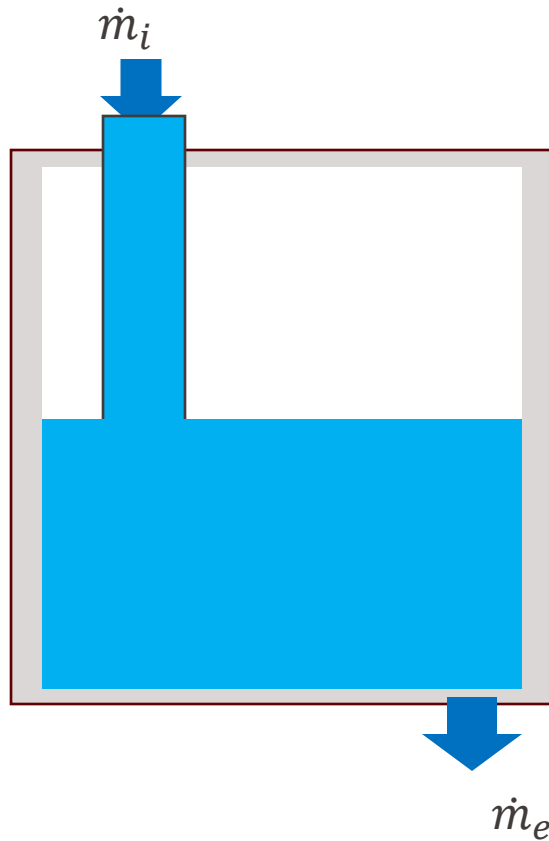
- Water enters and exits an initially empty barrel.
 - Inlet flow rate is $\dot{m}_i = 7 \text{ kg/s}$
 - Outlet flow rate is proportional to the height of liquid inside $\dot{m}_e = 1.4L$, where L is the instantaneous liquid height (SI units)
 - Base area of the barrel is $A = 0.2 \text{ m}^2$ and density of water is $\rho = 1000 \text{ kg/m}^3$

Plot the variation of liquid height L as a function of time t

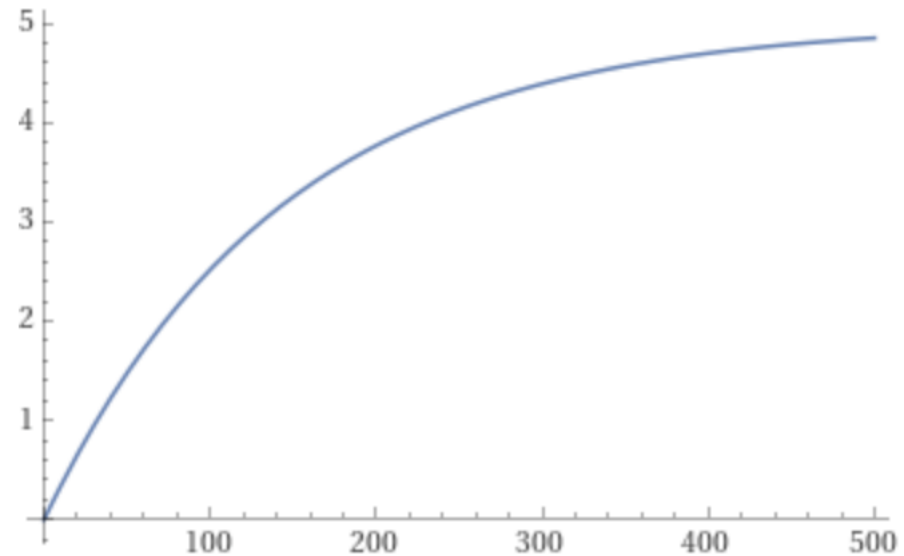
Example 4.2 in Moran (Transient Mass Balance)

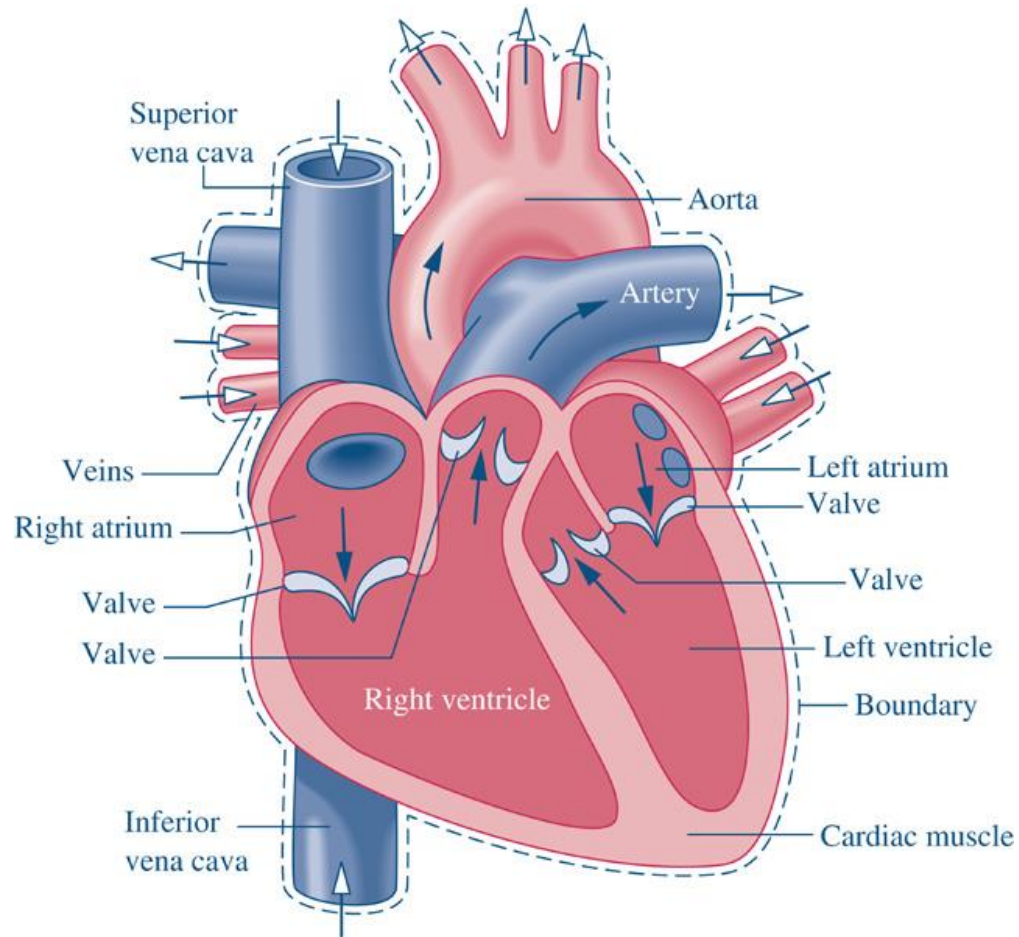


Example 4.2 in Moran (Transient Mass Balance)

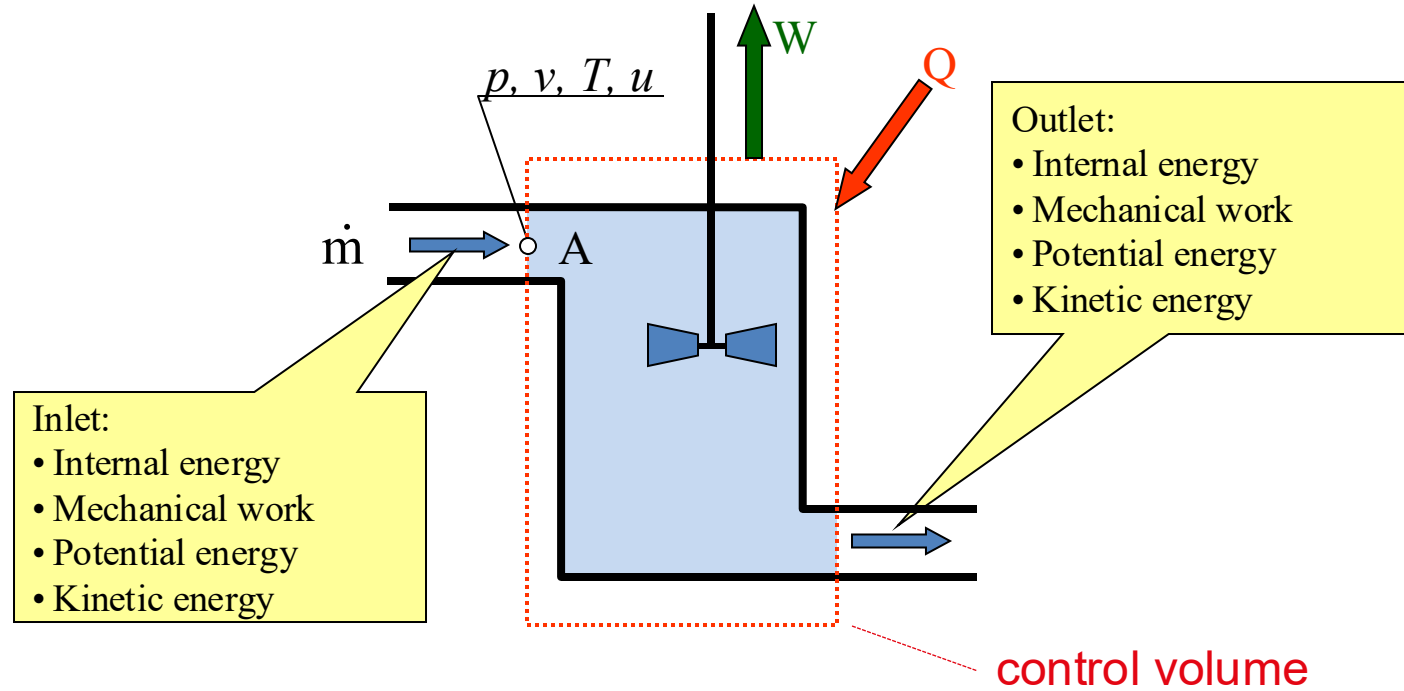


$$L = 5[1 - \exp(-0.007t)]$$





- Flow in a human heart is controlled by valves that intermittently allow blood to enter and exit as the heart muscles pump
- The boundary of this control volume is moving as the heart pulses

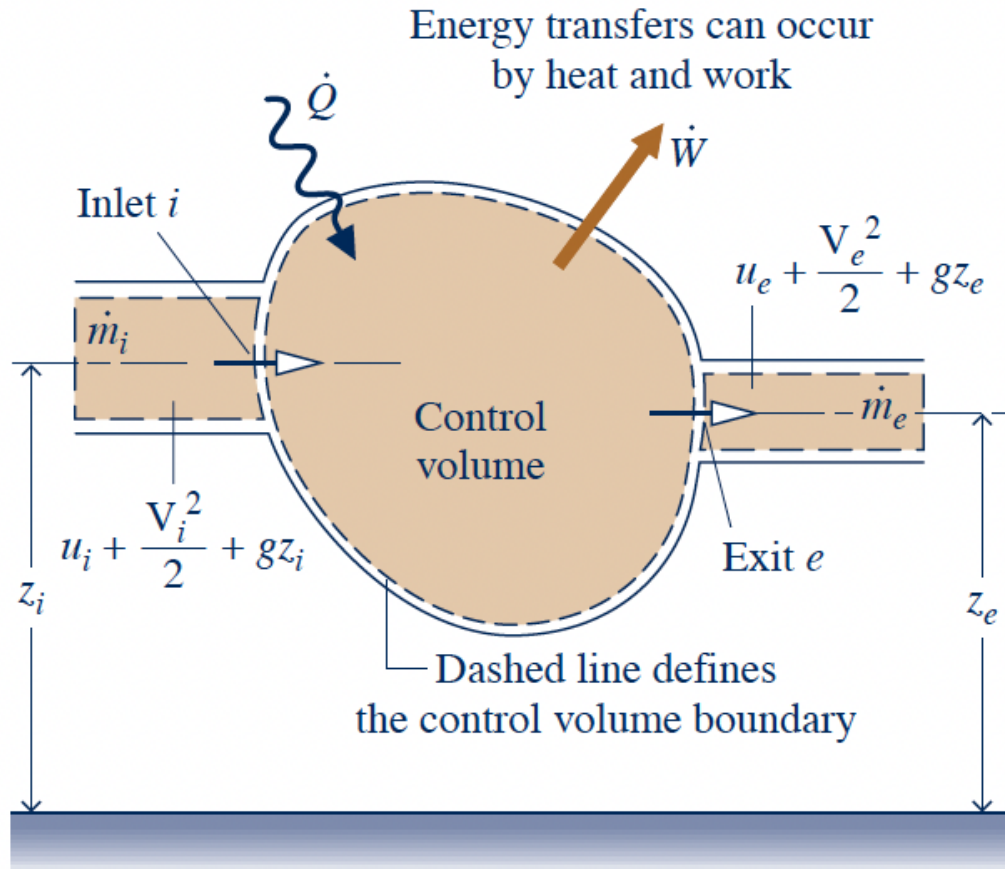


Unlike closed systems, energy transfer can be due to incoming and outgoing mass flow

- Energy rate balance

Time rate of change in the amount of energy contained within a control volume at time t

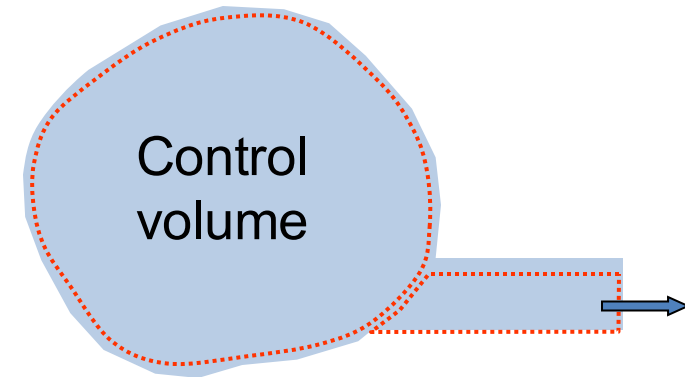
$$= \text{Net rate at which energy is transferred into the system by heat transfer at time } t - \text{Net rate at which energy is transferred out of the system by work at time } t + \text{Net rate at which energy is transferred into the system by accompanying mass flow}$$



- Separate \dot{W} into two contributions
 - Work (power) associated with fluid pressure as mass is introduced at inlets and removed at exits
 - All other works (power) \dot{W}_{cv}

Rate of energy transfer
by work **out of** control
volume at exit

Rate of energy transfer
by work **into** control
volume at inlet



Energy Balance Rewritten



$$E_{cv} = \int_{\mathcal{V}} \rho \left(u + \frac{\vec{V}^2}{2} + gz \right) d\mathcal{V}$$

$$\begin{aligned} \frac{d}{dt} \int_{\mathcal{V}} \rho \left(u + \frac{\vec{V}^2}{2} + gz \right) d\mathcal{V} &= \dot{Q}_{cv} - \dot{W}_{cv} + \sum_i \left[\int_A \left(h + \frac{\vec{V}^2}{2} + gz \right) \rho \vec{V} \cdot d\vec{A} \right]_i \\ &\quad - \sum_e \left[\int_A \left(h + \frac{\vec{V}^2}{2} + gz \right) \rho \vec{V} \cdot d\vec{A} \right]_e \end{aligned}$$

Compare this to
$$\frac{dE_{cv}}{dt} = \dot{Q}_{cv} - \dot{W}_{cv} + \sum_i \dot{m}_i \left(h_i + \frac{\vec{V}_i^2}{2} + gz_i \right) - \sum_e \dot{m}_e \left(h_e + \frac{\vec{V}_e^2}{2} + gz_e \right)$$

Mass + Energy Conservation for CV

