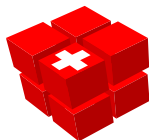


Liquid Lubrication

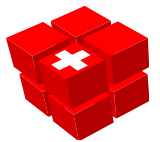
Rowena Crockett, Empa,
Switzerland
CCMX Summer School, Visp 2021



Liquid lubrication



<http://images.google.ch>

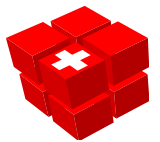


Part 1. Fluid film lubrication

- Base oil
- Stribeck Curve
- The fluid film
- Slip: influence of the surface
- Greases
- Waterbased lubrication

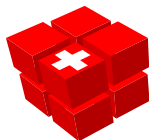
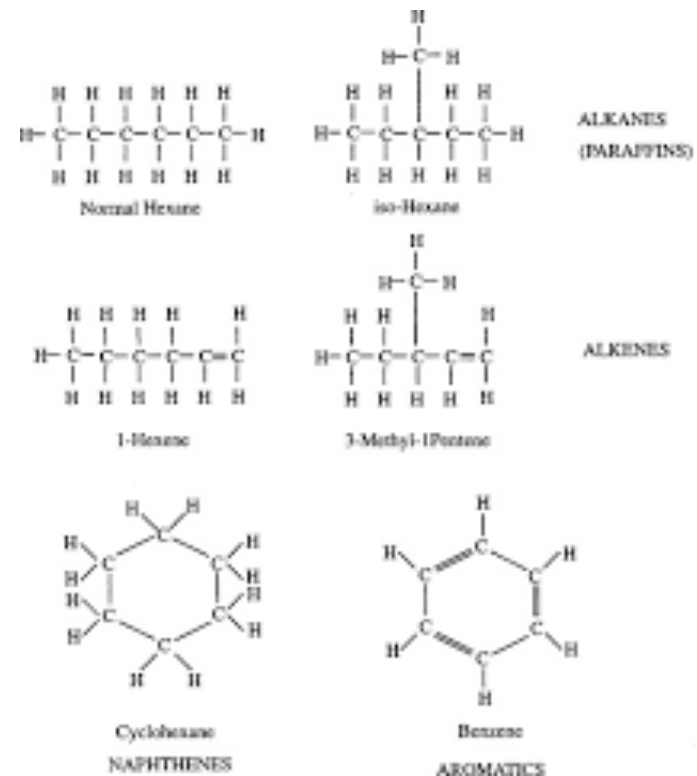
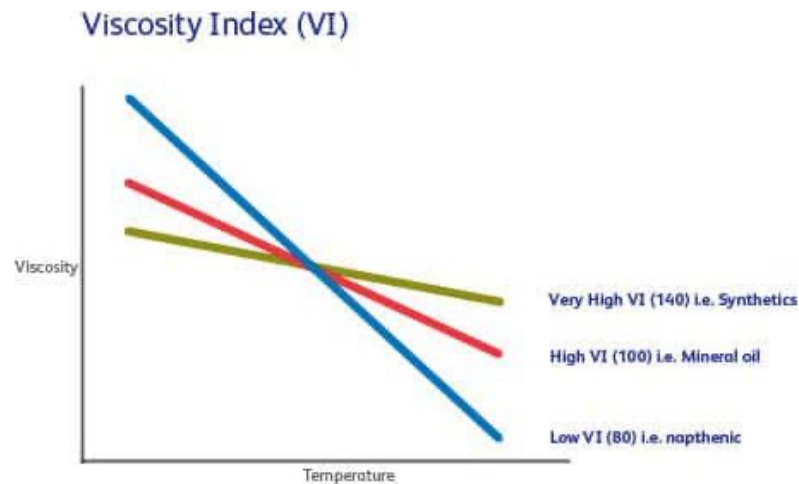
Part 2. Boundary lubrication

- Surface interactions of additives
- Competition for surface coverage
- Stick-slip
- Seals



Mineral base oils

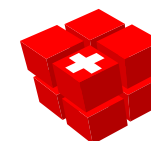
- Paraffinic mineral oil: high viscosity, good viscosity-temperature behaviour (VI = 100)
- Naphthenic mineral oil: Medium viscosity, medium viscosity-temperature behaviour (VI = 0)
- Aromatic mineral oil: Low viscosity, poor viscosity-temperature behaviour



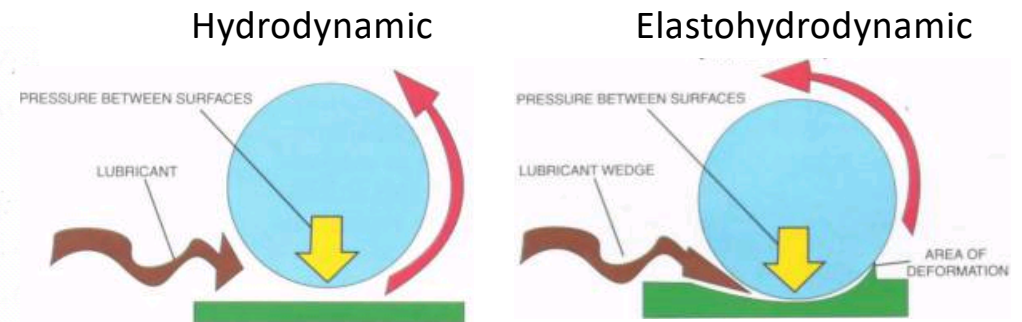
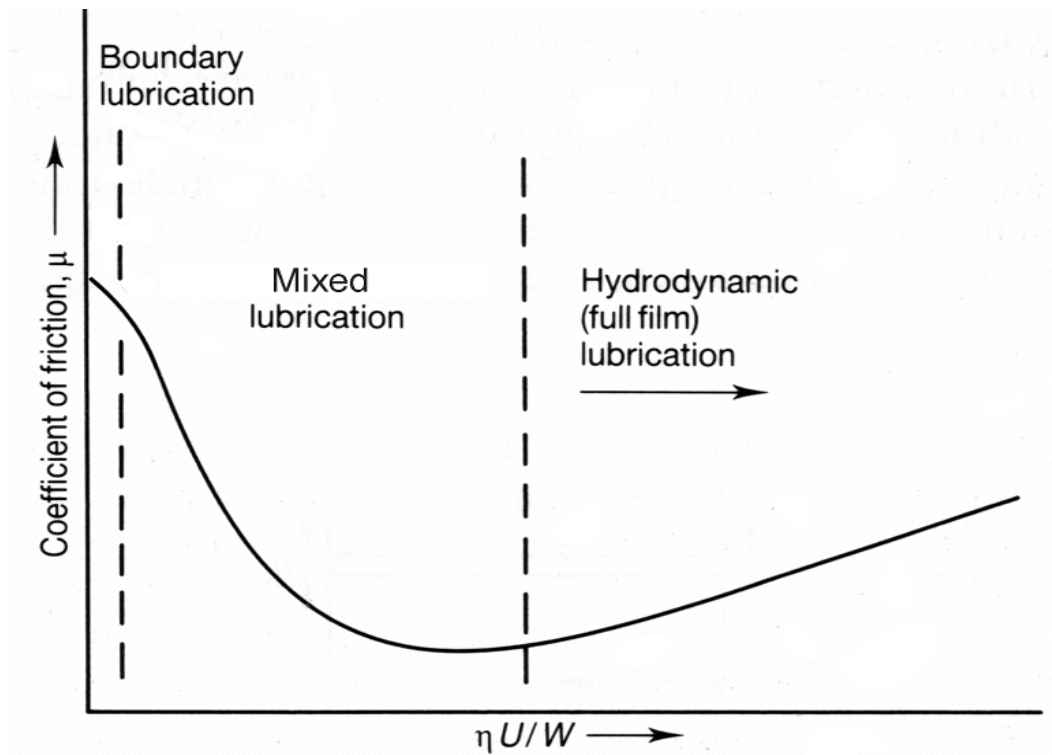
Synthetic Base Oils

SYNTHETIC FLUID	STRENGTHS	WEAKNESSES	APPLICATIONS
POLYALPHAOLEFINS	High VI, high thermal oxidative stability, low volatility, good flow properties at low temperatures, nontoxic	Limited biodegradability, limited additive solubility, seal shrinkage risk	Engine oil, gear oils, bearing oils, compressor oils, high-temperature grease, lube-for-life applications
DIESTERS AND POLYOLESTERS	Nontoxic, biodegradable, high VI, good low-temperature properties, miscible with mineral oils	Low viscosities only, bad hydrolytic stability, limited seal and paint compatibility	Compressors oils, high-temperature grease, a co-base stock with PAOs, bearing oils, gear oils, oil mist, jet engine oils
PHOSPHATE ESTERS	Fire-resistant, biodegrades quickly, excellent wear resistance, scuffing protection	Low VI, limited seal compatibility, not miscible with mineral oils, moderate hydrolytic stability	Fire-resistant hydraulic fluids used in power plants, factories, marine vessels, mining, aircraft, mobile equipment
POLYALKYLENE GLYCOLS	Excellent lubricity, nontoxic, good thermal and oxidative stability, high VI	Additives marginally miscible, not miscible with mineral oils, limited seal and paint compatibility	Refrigeration compressors, brake fluids (water-soluble), gas compressors (low gas solubility), warm and high-temperature gears, chain lube (clean burn off), metal-working and quenchant, H1 food grade
SILICONES	Highest VI, high chemical stability, excellent seal compatibility, very poor thermal and oxidative stability.	Worst mixed and boundary film lubrication properties, not miscible with mineral oils or additives	High-temperature fluids, speciality greases, lubricant-contacting chemical, some brake fluids

<https://www.bharatpetroleum.in/>

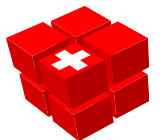


Stribeck Curve: A tool to describe the friction behavior of a liquid lubricant

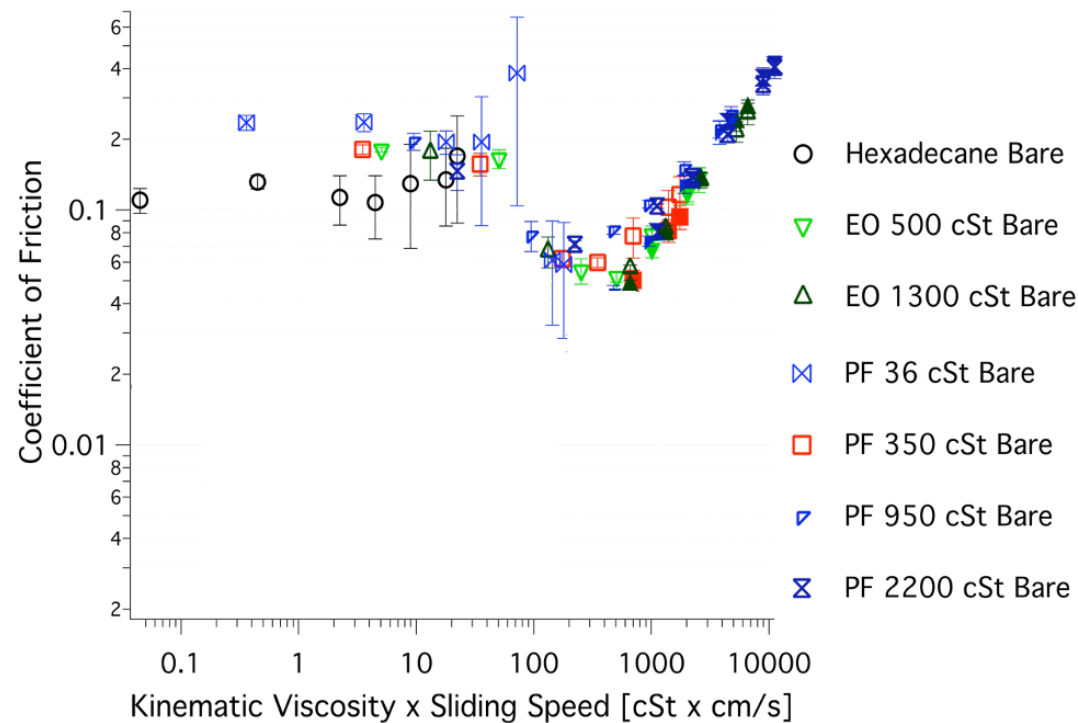


Hersey number
$$\frac{\text{ms}^{-1} * \text{Nsm}^{-2}}{\text{Nm}^{-1}}$$

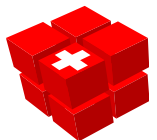
$(\text{Shaft radius/radial clearance}) * (\text{Dynamic Viscosity} \times \text{Speed}) / \text{load (pressure)}$

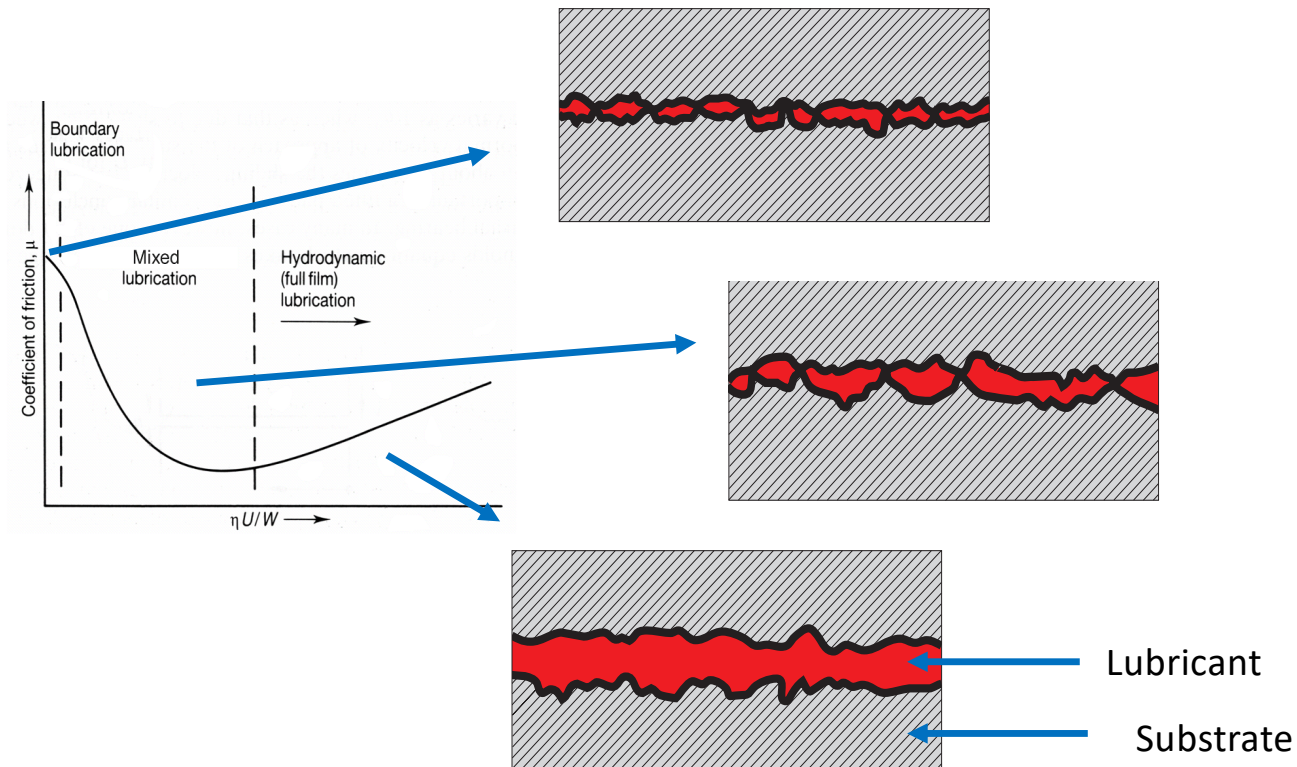


Stribeck Curve: A tool to describe the friction behavior of a liquid lubricant



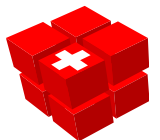
Robert M. Bielecki, Maura Crobu, and Nicholas D. Spencer *Tribol Lett*
(2013) 49:263–272





Friction regimes

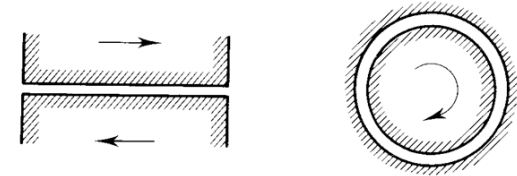
- Boundary: the sliding or rolling surfaces are separated by residual adsorbed oil or a film formed from the additives in the oil
- Mixed: Fewer asperity contacts as surfaces begin to separate
- Hydrodynamic: Full film lubrication with complete separation of surfaces



Hydrodynamic Lubrication

Prerequisites:

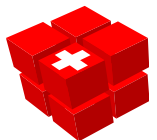
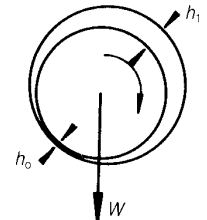
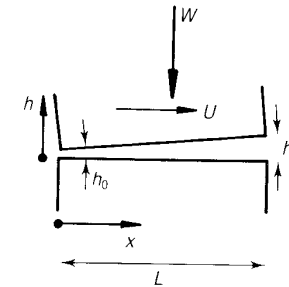
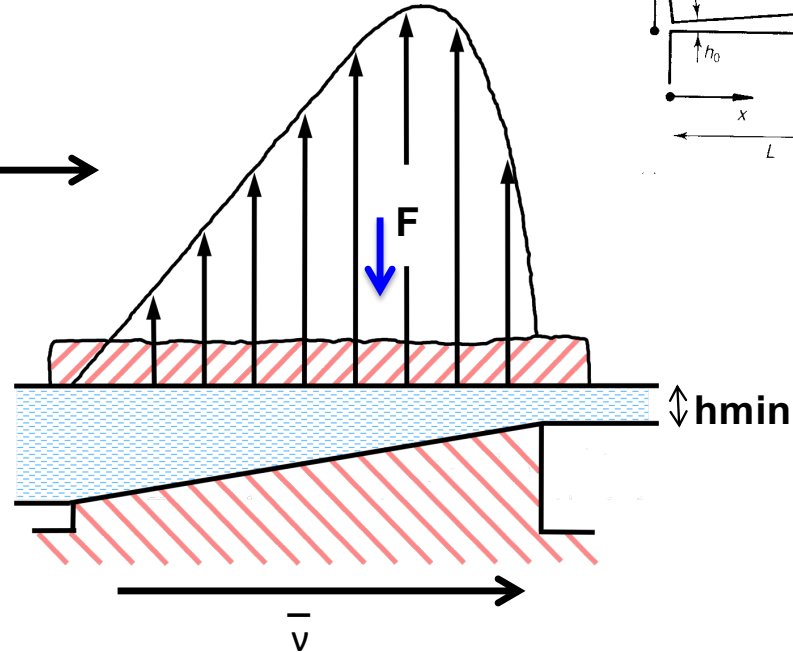
- Conforming surfaces with a narrowing gap
- Relative motion between surfaces
- Sufficient fluid
- Sufficient adhesion of the lubricant to the surface profile



Pressure distribution



Lubricating film



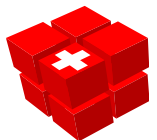
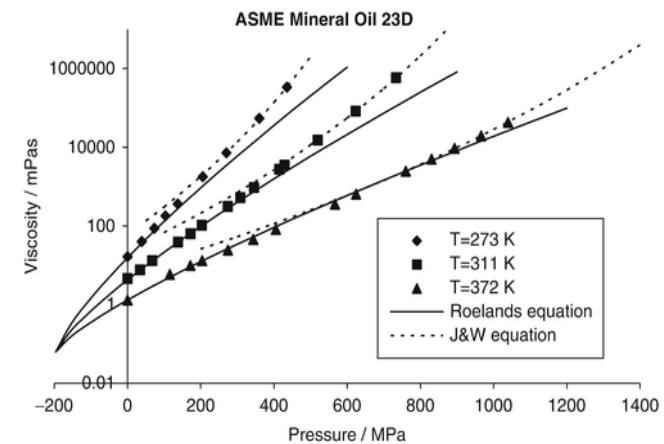
Hydrodynamik

$$h_{\min} = \frac{3.63 \cdot (h \cdot v)^{0.68} \cdot a^{0.49}}{\left(\frac{1}{r_1} + \frac{1}{r_2}\right)^{0.466} \cdot Q^{0.073}} \cdot \frac{E}{1 - \left(\frac{1}{m}\right)^2}^{-0.117} \cdot (1 - e^{-0.68 \cdot k}) \text{ [mm]}$$

$h_{\min}$	[mm]	film thickness in the rolling contact
a	[mm ² /N]	pressure-viscosity coefficient
h	[mPa · s]	dynamic Viscosity
v	[m/s]	$= (v_1 + v_2) / 2 =$ average rolling speed
r_1	[mm]	radius of ball
r_2	[mm]	radius of inner or outer ring
Q	[N]	load
E	[N/mm ²]	elastic modulus
$1/m$		Poisson's ratio

$k = a/b =$ relationship between axis of the contact area

www.tribology-abc.com

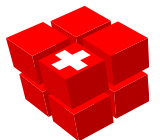
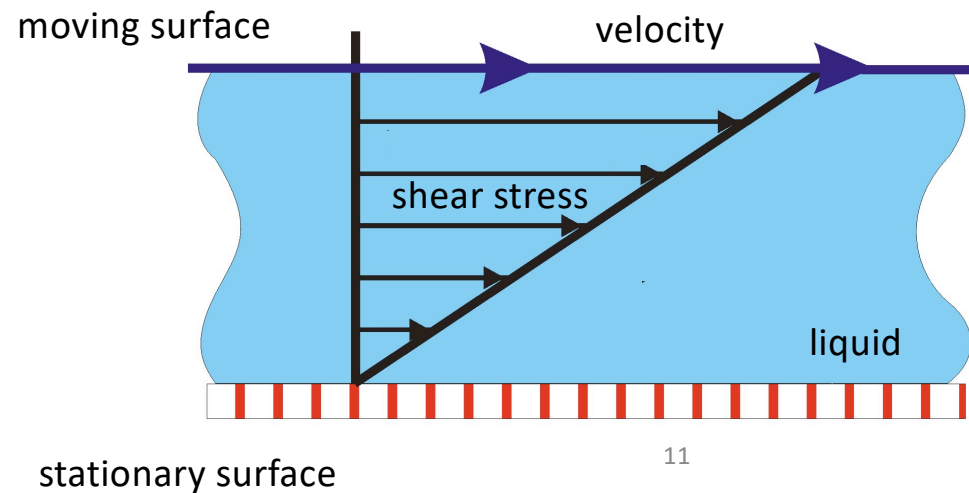


Factors influencing hydrodynamic lubrication

- $h_{min} > 3 * Ra$
- Load and average speed,
- Contact geometry
- Elastic modulus and Poisson's Ratio of contacting materials
- Dynamic viscosity
- Pressure-viscosity coefficient

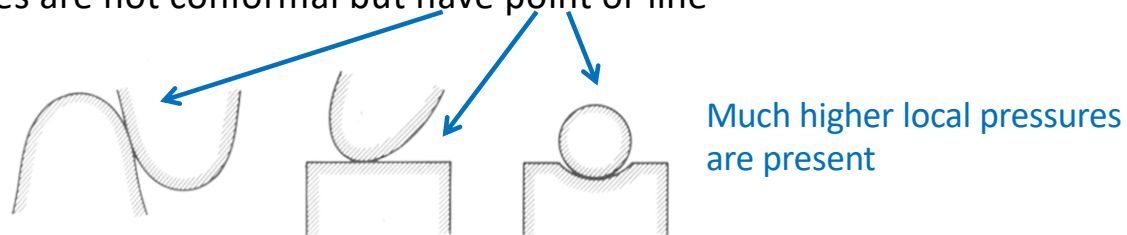
Viscosity is a measure of the resistance of a fluid to deform under shear stress

Dynamic viscosity: shear force at a plane inside the liquid per unit velocity gradient, in a direction perpendicular to the plane (Pascal-second or Centipoise)



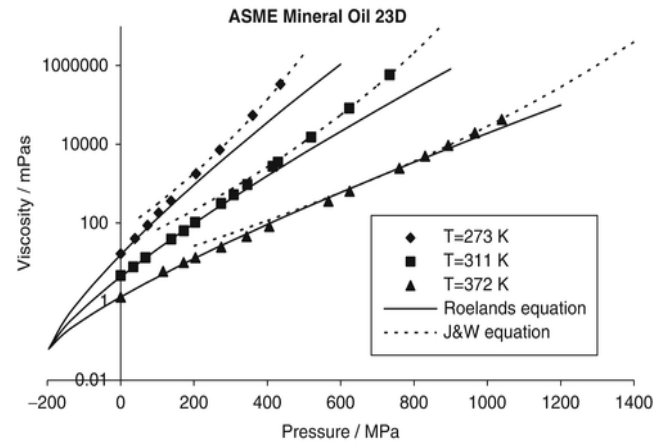
Elastohydrodynamic lubrication

When the sliding surfaces are not conformal but have point or line contact

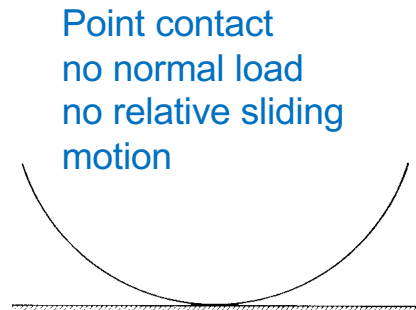


- Important factors: pressure dependence of the lubricant and elastic deformation of the surfaces
- GPa pressures are frequently involved—oil becomes so viscous it is practically solid!

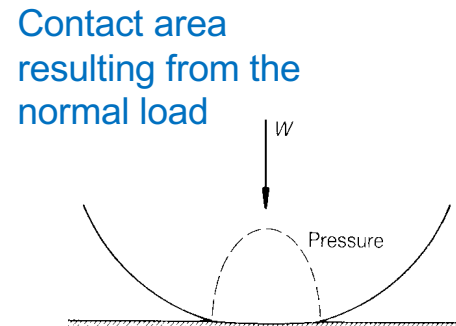
Piezo viscous effect



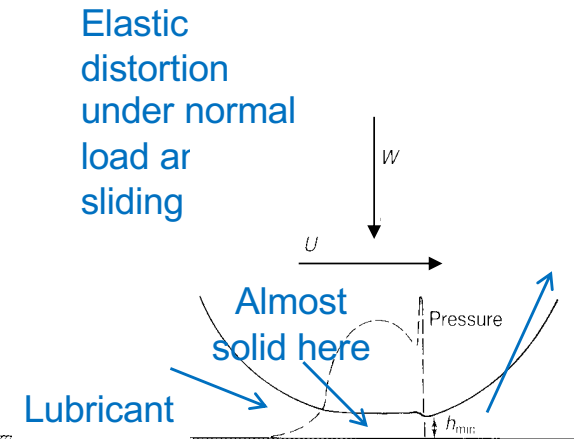
Elastohydrodynamic lubrication



Contact initially occurs at a point



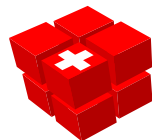
Pressure increases
↓
contact area grows



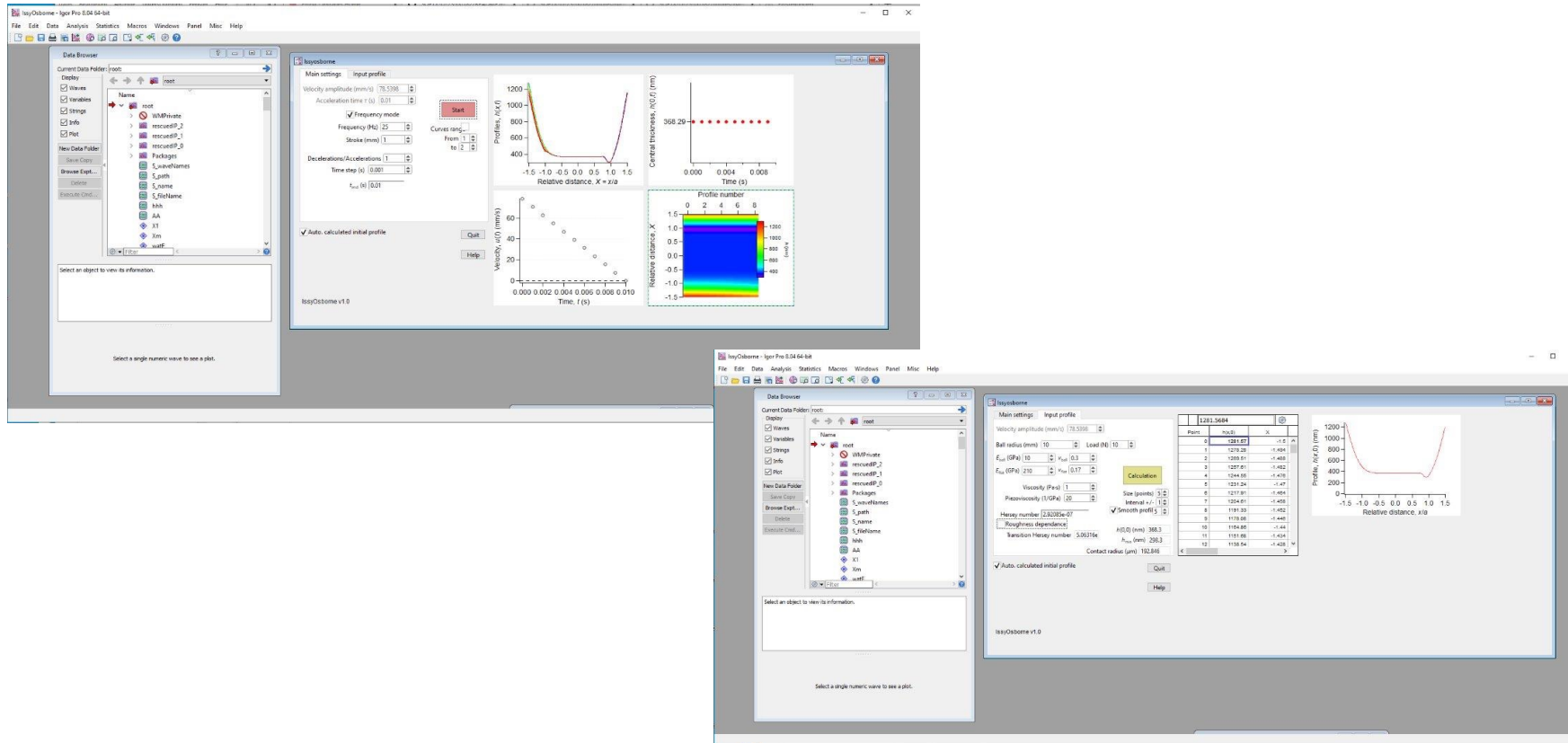
Lubricating film and sphere sliding over the surface

Sudden release of lubricant: elastic modification of one of the solids

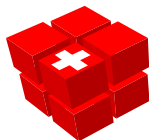
Reynolds equation shows that the gap remains parallel at first with increasing pressure, but then suddenly enlarges on one side



Calculation of fluid film profile



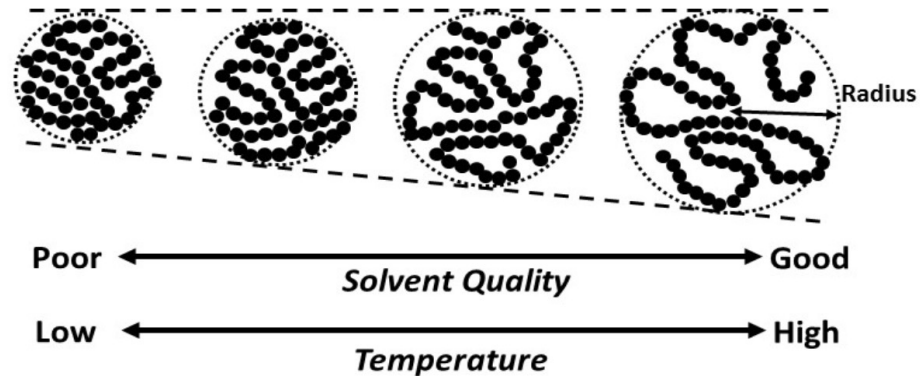
<https://github.com/ElsevierSoftwareX/SOFTX-D-23-00182>
 Malik Yahiaoui, Denis Mazuyer, Juliette Cayer-Barrio



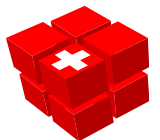
Hydrodynamic Lubrication: Viscosity improvers

VI improvers are polymers: olefin copolymers (OCP), polyalkylmethacrylates (PMA) and hydrogenated poly (styrene-co-conjugated dienes)

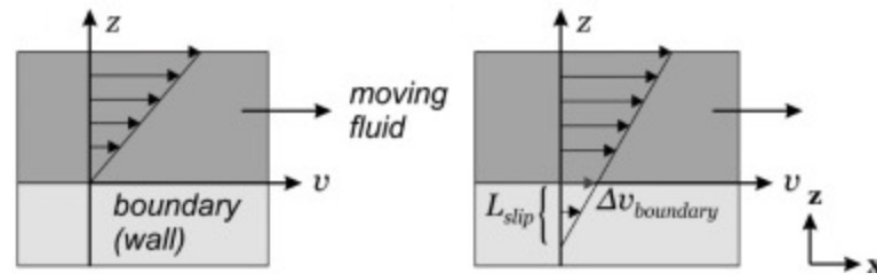
PMAs appear to thicken oils by expansion of the polymer coil with temperature



Olefin copolymers thicken base oils by about the same proportion, regardless of temperature. Therefore the degree of viscosity increase from 100 °C to 40 °C will be less for a polymer-containing low viscosity base oil than a polymer-free higher viscosity base oil of equal 100 °C viscosity.



Slip

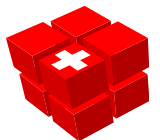


Important factors: liquid-surface interactions, surface roughness, shear rate

No interaction $\longrightarrow$ large slip, no hydrodynamic lubrication

Strong interaction $\longrightarrow$ no slip

Bastian E. Rapp, in Microfluidics: Modelling, Mechanics and Mathematics, 2017

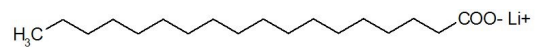


Grease

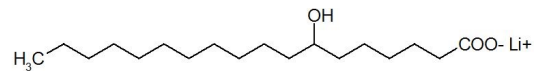
Grease is mostly nasty
Used in extreme conditions
Compositions: Base oil, thickener, additives



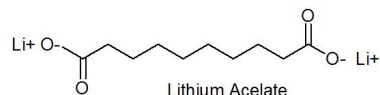
Examples of thickeners



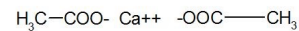
Lithiumstearate



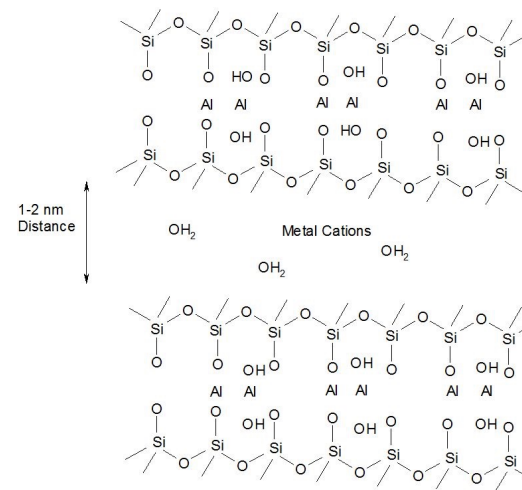
Lithium-12-hydroxystearate



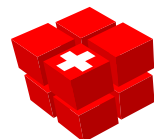
Lithium Acelate



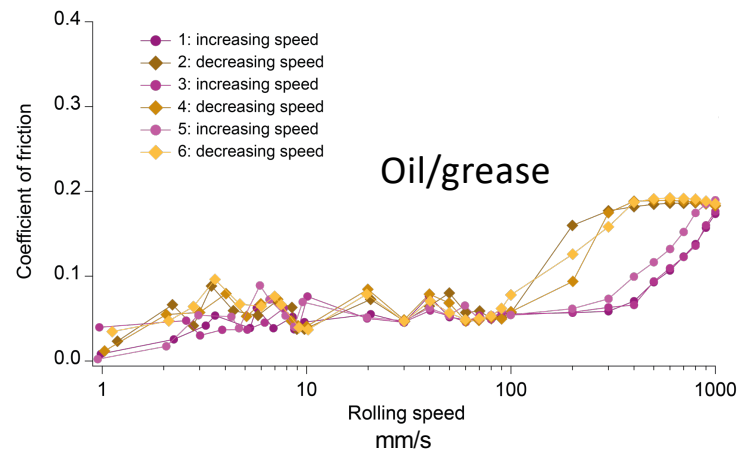
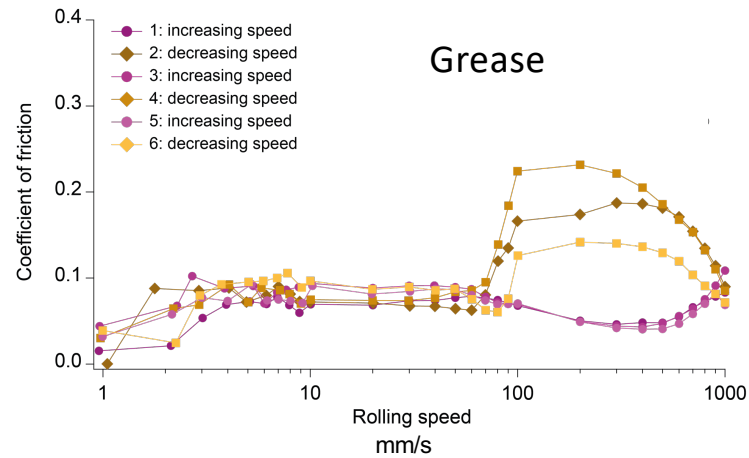
Calciumacetate



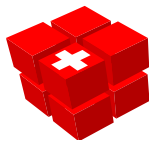
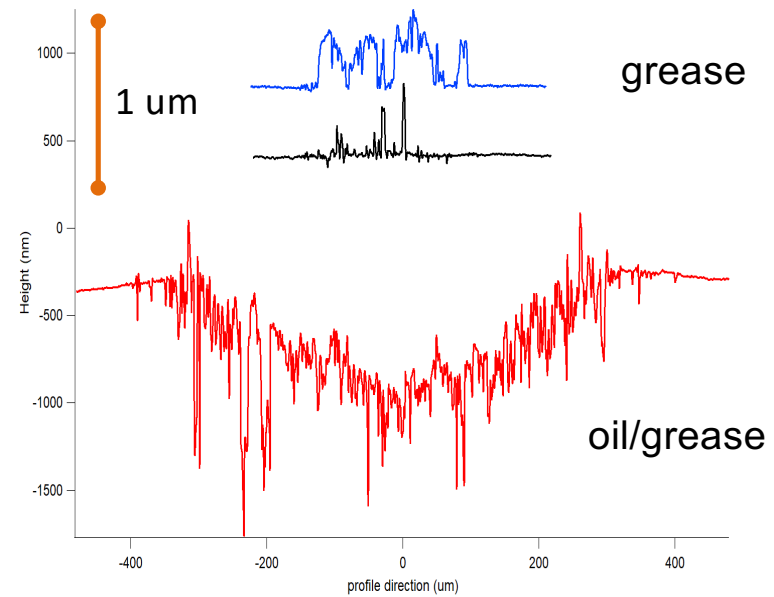
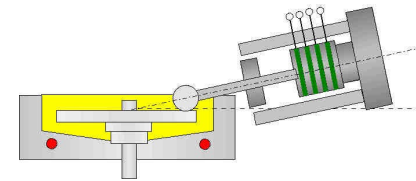
Clays, talc



Bleeding in grease

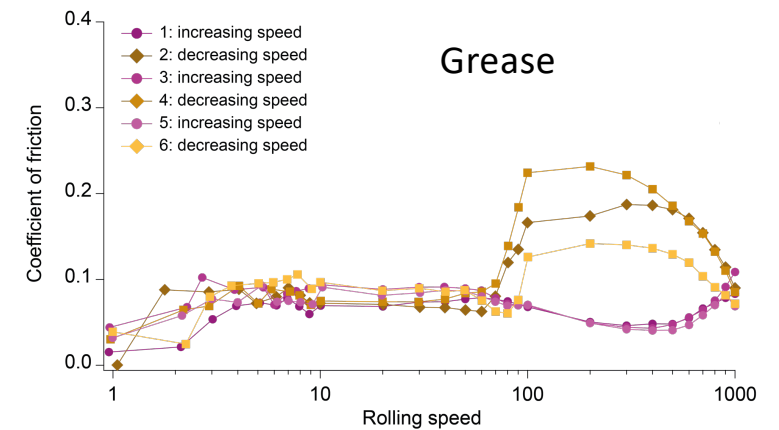
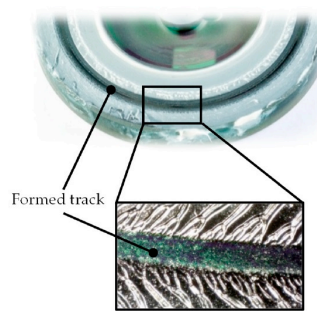


Mini Traction Machine (MTM)

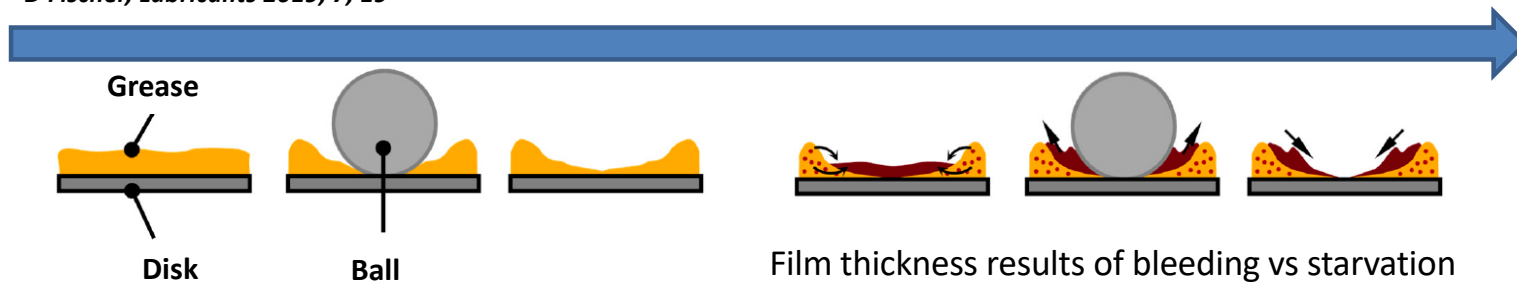


Grease bleeding

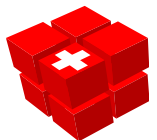
EHL/MX lubrication regime can be established, if a sufficient supply of liquid is available



D Fischer, Lubricants 2019, 7, 19



Film thickness results of bleeding vs starvation
(+ inertial effects at very high speeds)



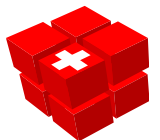
Aqueous-based Lubrication

advantages

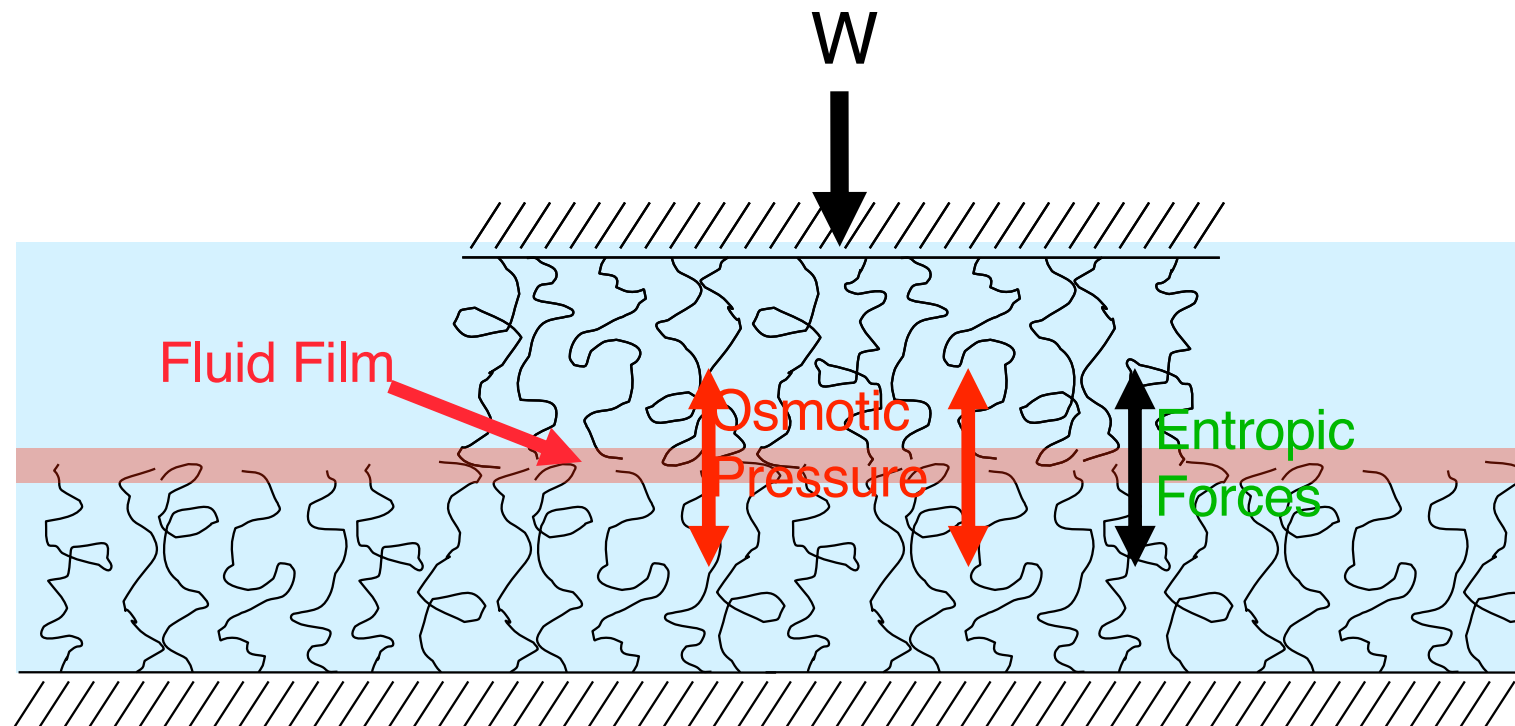
- non flammable
- non toxic, biocompatible
- cheap, readily available

disadvantages

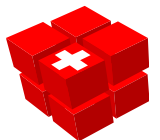
- low viscosity
- viscosity does not increase under pressure (low pressure coefficient of viscosity)



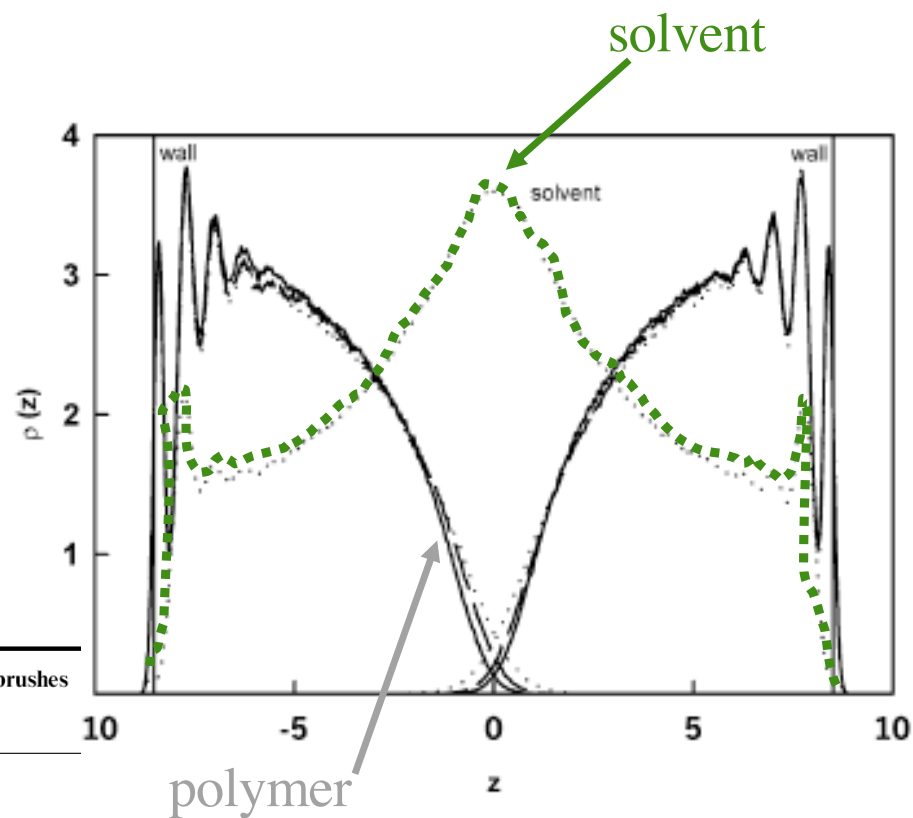
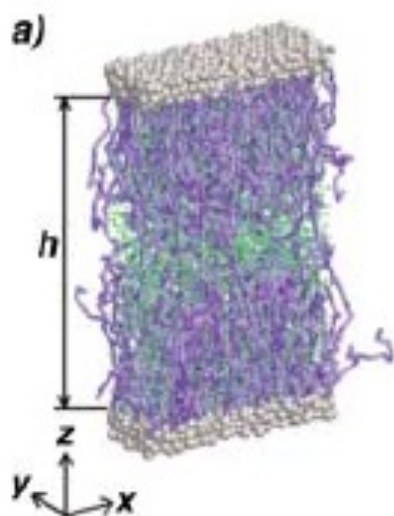
Aqueous based lubrication with polymers



X. Yan, S.S. Perry, N.D. Spencer, S. Pasche, S.M. De
Paul, M. Textor, M.S. Lim, *Langmuir* 2004, 20, 423-428



Density Distribution of Polymer and Solvent



Dissipative particle dynamics simulation of grafted polymer brushes under shear

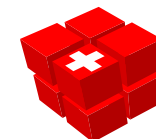
Danial Irfachsyad,^a Dominic Tildesley^a and Patrice Malfreyt^b

^a Unilever Research Port Sunlight, Bebington, Wirral, UK CH63 3JW

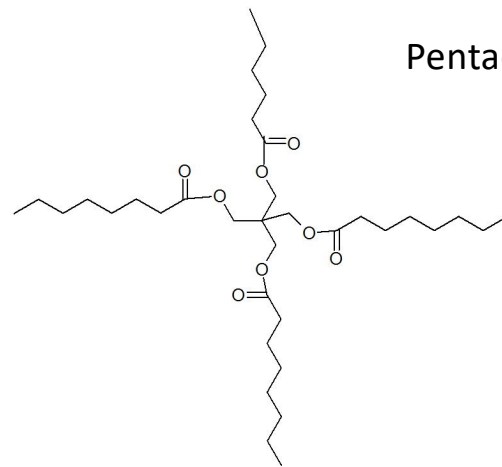
^b Laboratoire de Thermodynamique des Solutions et des Polymères, UMR CNRS 6003, 24 avenue de Landais, Université Blaise-Pascal, 63177, Aubière Cedex, France

Received 22nd November 2001, Accepted 22nd April 2002

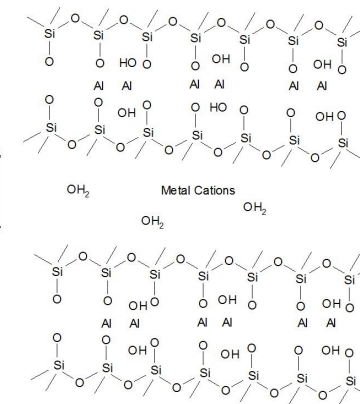
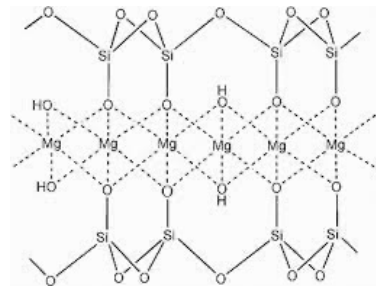
First published as an Advance Article on the web 27th May 2002



Thickeners for water-based hydraulic fluids

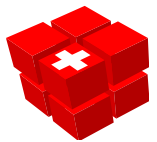
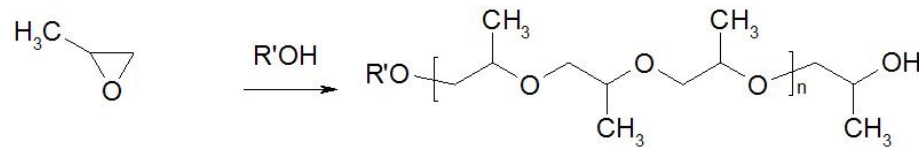


Pentaerythrolesters



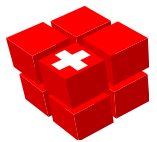
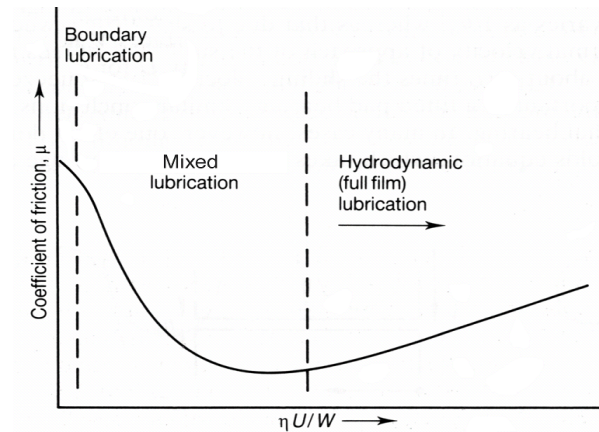
Clays or talc

Polypropylene Glycols



Summary

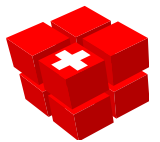
- Ideally, machinery will operate in the hydrodynamic or elastohydrodynamic regimes
- Choice of base oil based on cost, operating conditions, safety
- Additives used to improve fluid film formation of base oil



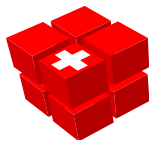
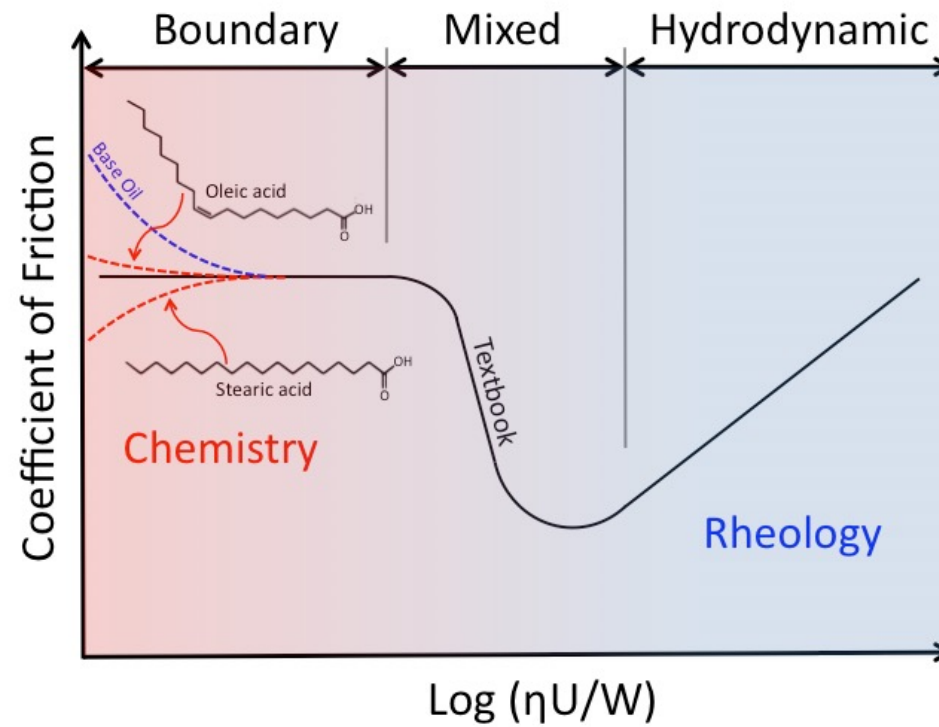
Liquid Lubrication

Part 2. Boundary Lubrication

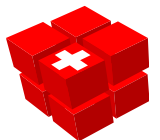
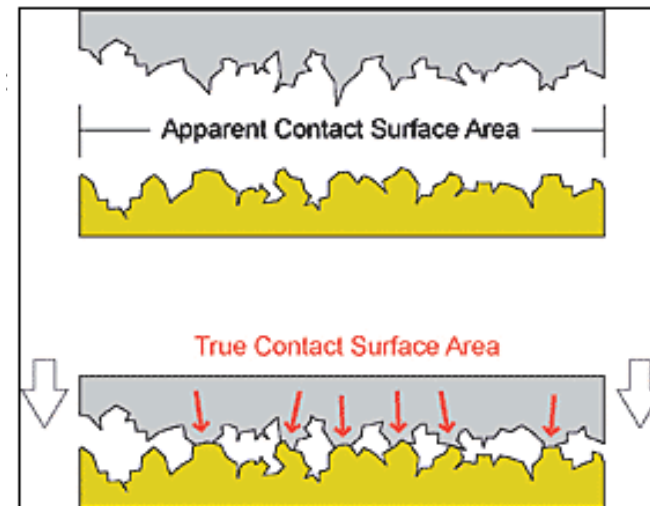
- Surface interactions of additives
- Competition for surface coverage
- Mechanisms of wear protection and friction control
- Stick-slip
- Seals



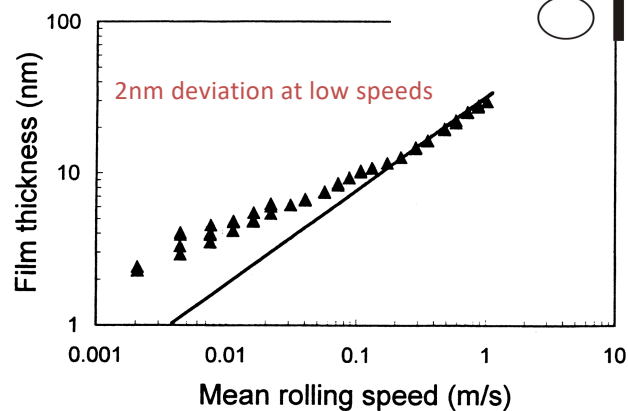
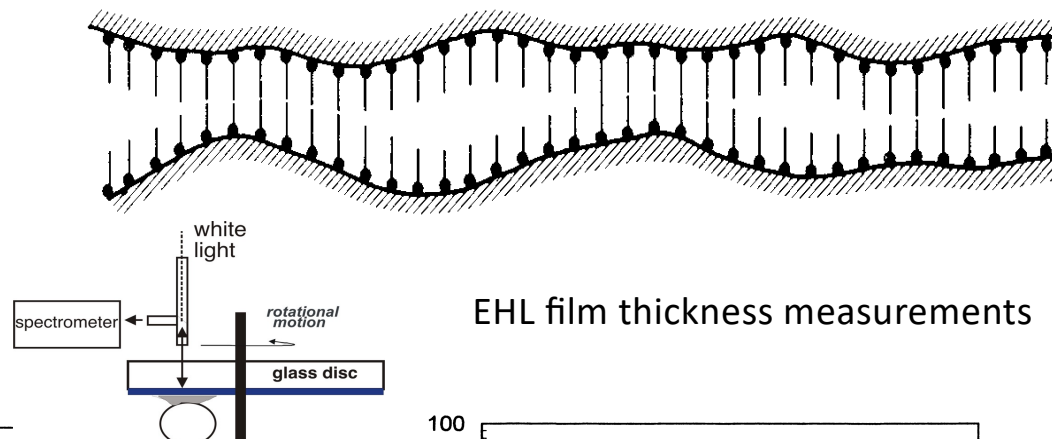
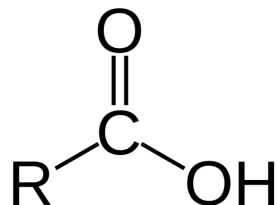
Stribeck Curve



Contact area:
Friction modifiers and antiwear
additives only required to function in
the true contact area



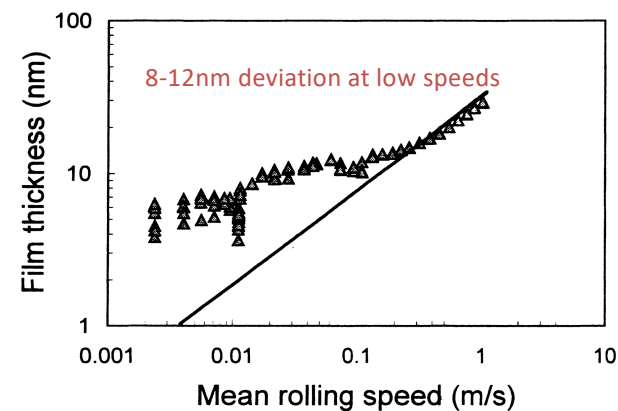
Friction modifiers



octadecanoic acid (1% in hexadecane dry)

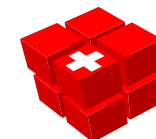
Formation of iron carboxylate salts in the presence of water

EHL film thickness measurements



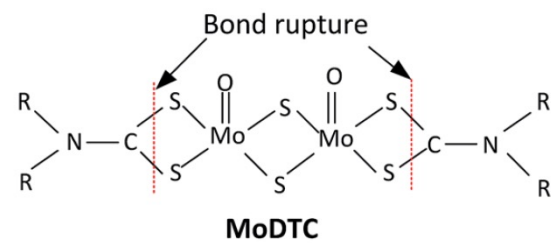
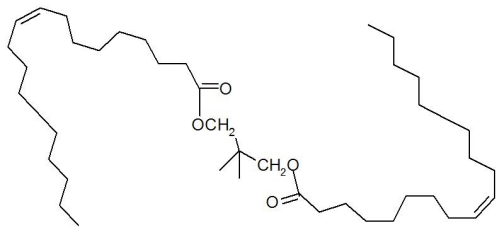
octadecanoic acid (1% in hexadecane, wet)

Hugh Spikes, Tribology International, 2001, 34, 789

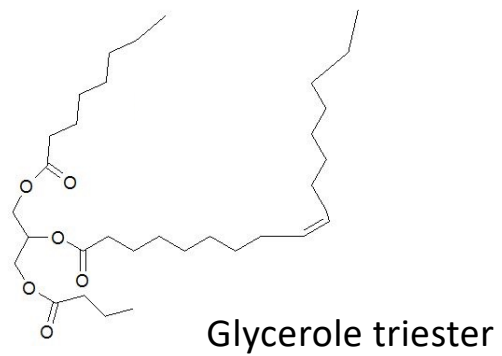
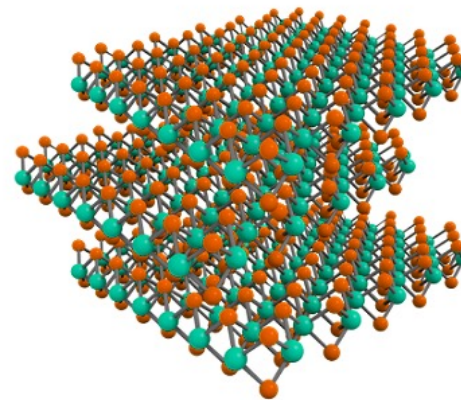


Other friction modifiers

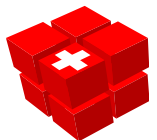
Neopentylglycol dioleate



Molybdenum Dithiocarbamate

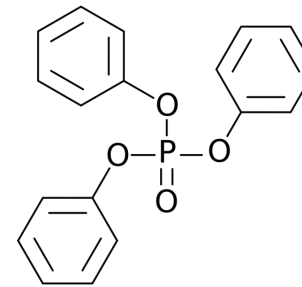
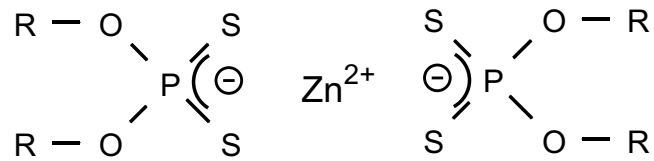


Glycerole triester

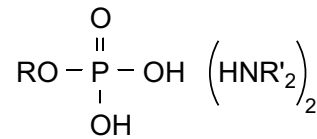


Anti-wear additives

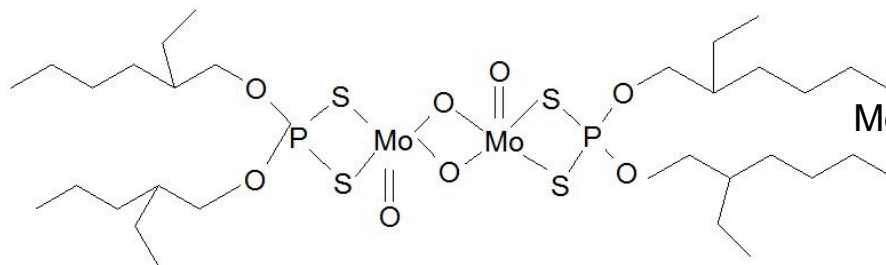
Zinc dialkyl dithiophosphate



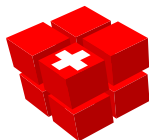
Triphenylphosphate



Amine phosphate

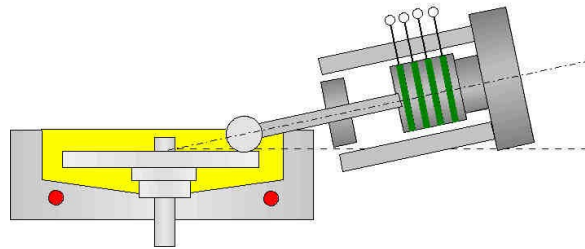


Molybdenum dithiophosphate

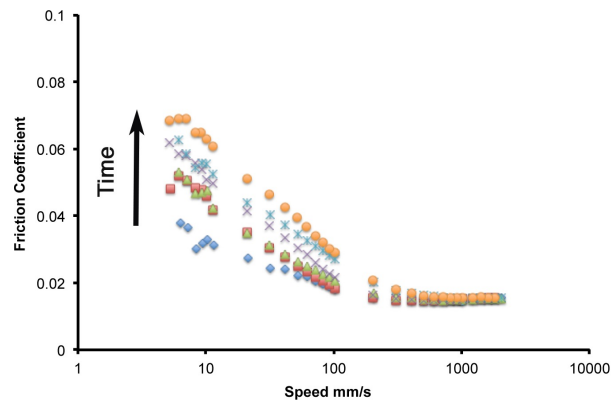


Friction behavior of ZnDTP

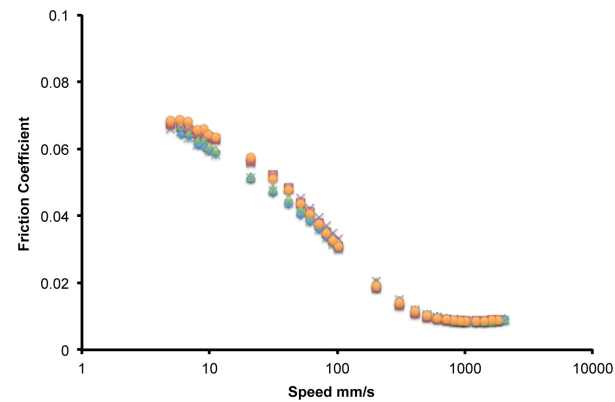
Mini Traction Machine (MTM)



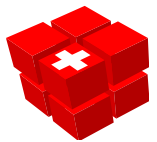
Load: 40 N
Speed: 5 to 1000 mm/s
Slide-roll-ratio: 50%
Temperature 80 °C



First measurements

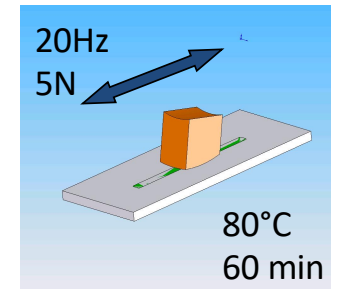
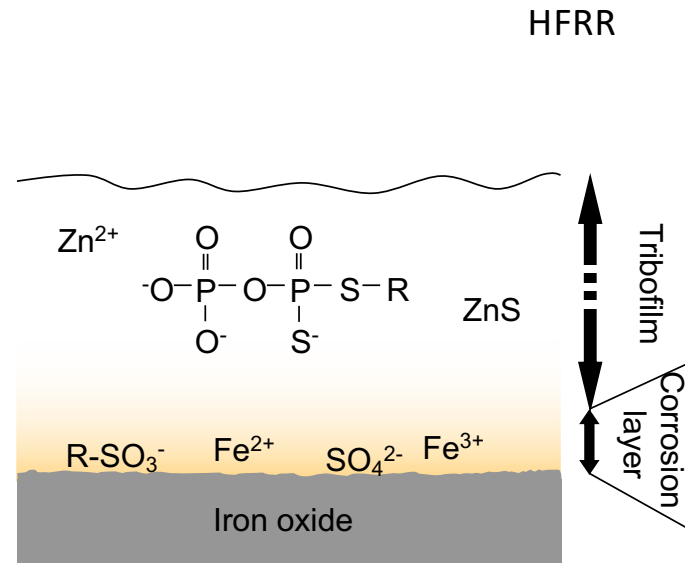
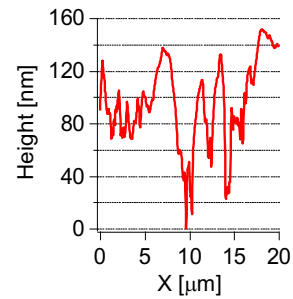
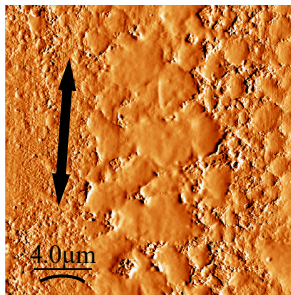


Last measurements

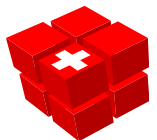
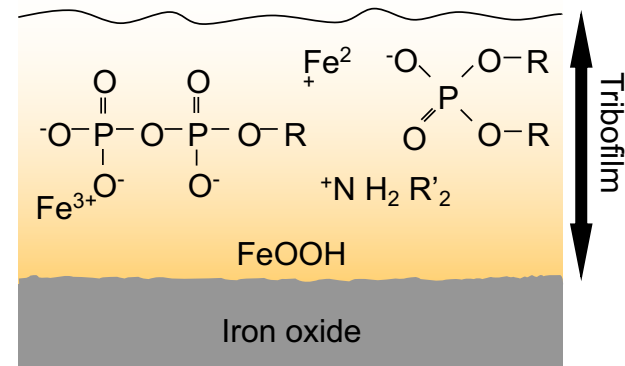
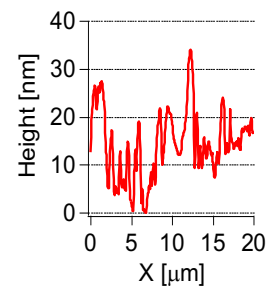
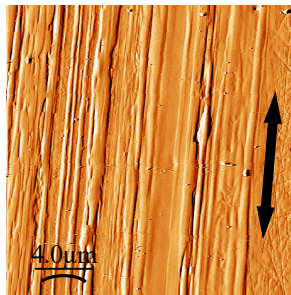


Topography of tribofilm on steel: Comparison of ZnDTP and amine phosphate (AP)

ZnDTP



AP

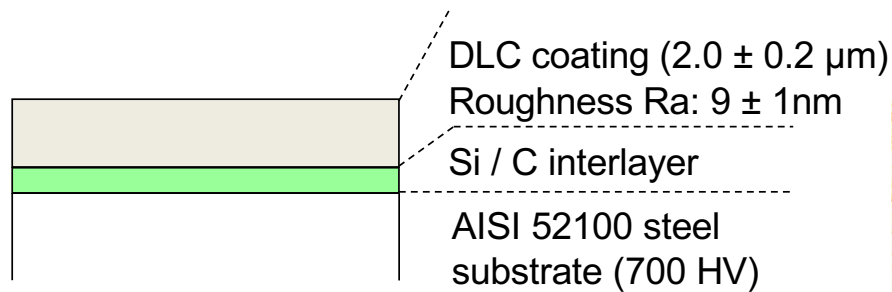


DLC coatings

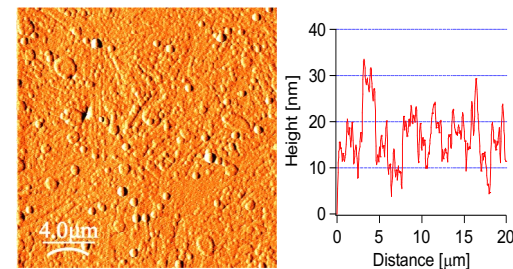
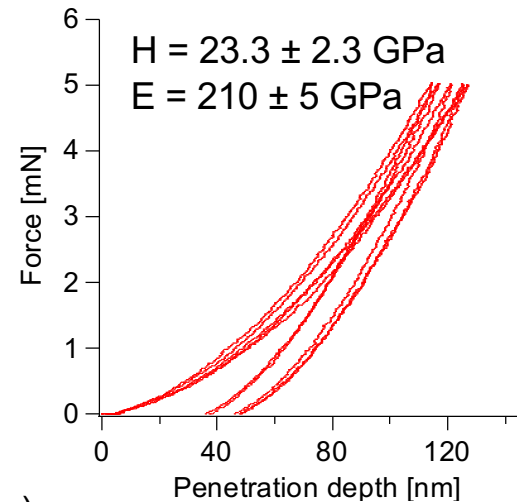
Hydrogenated amorphous carbon coatings (a-C:H) deposited by Plasma-Enhanced Chemical Vapor Deposition (PECVD)

Gases: C_2H_2 , Ar.

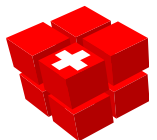
The AISI 52100 steel is also used for the steel/steel tribological tests.



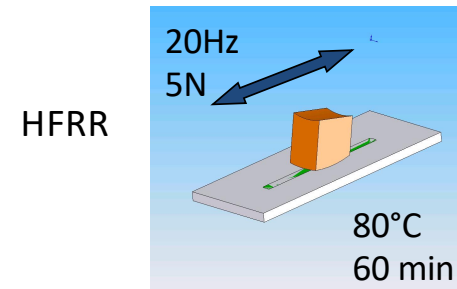
Hardness of the DLC coating measured by nanoindentation



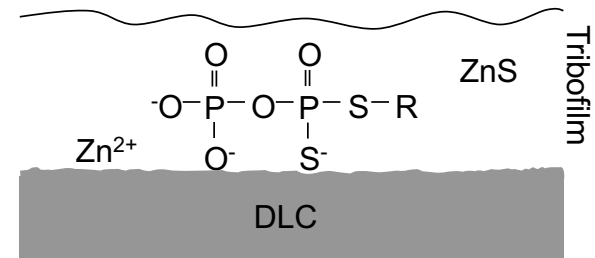
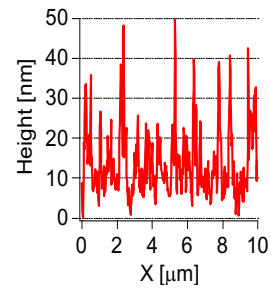
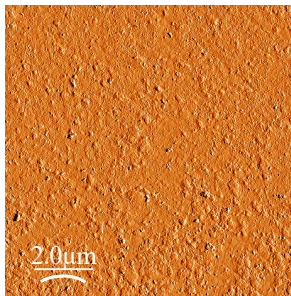
AFM observation of a DLC coating as-deposited surface



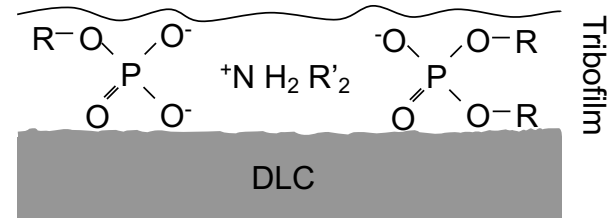
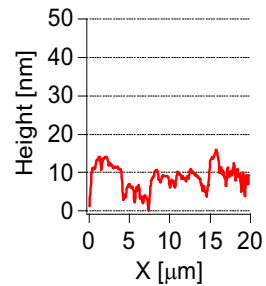
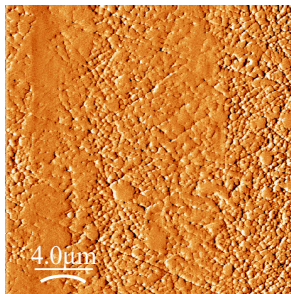
ZnDTP and AP on DLC



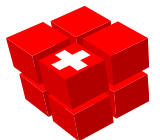
ZnDTP tribofilm on DLC



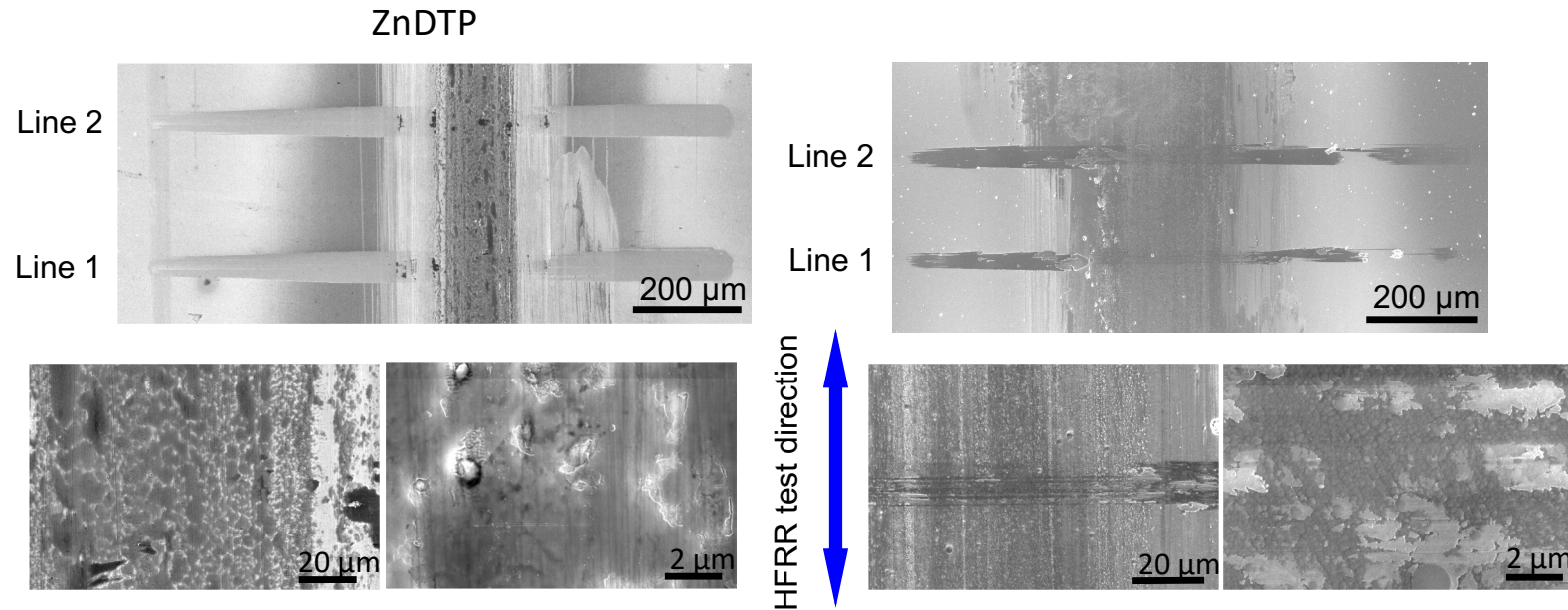
AP tribofilm on DLC



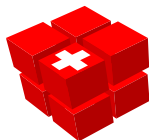
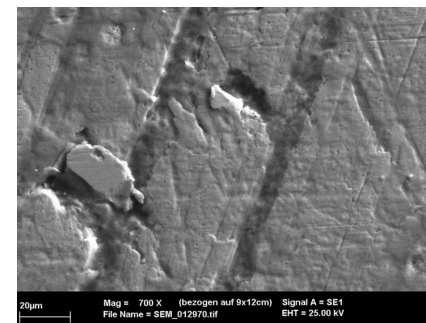
ZnDTP provided some wear protection but AP accelerated wear



Comparison of ZnDTP on steel and DLC surfaces

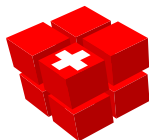


Corrosion layer results in good
adhesion and eventually to damage
to oil lubricated surfaces

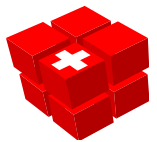
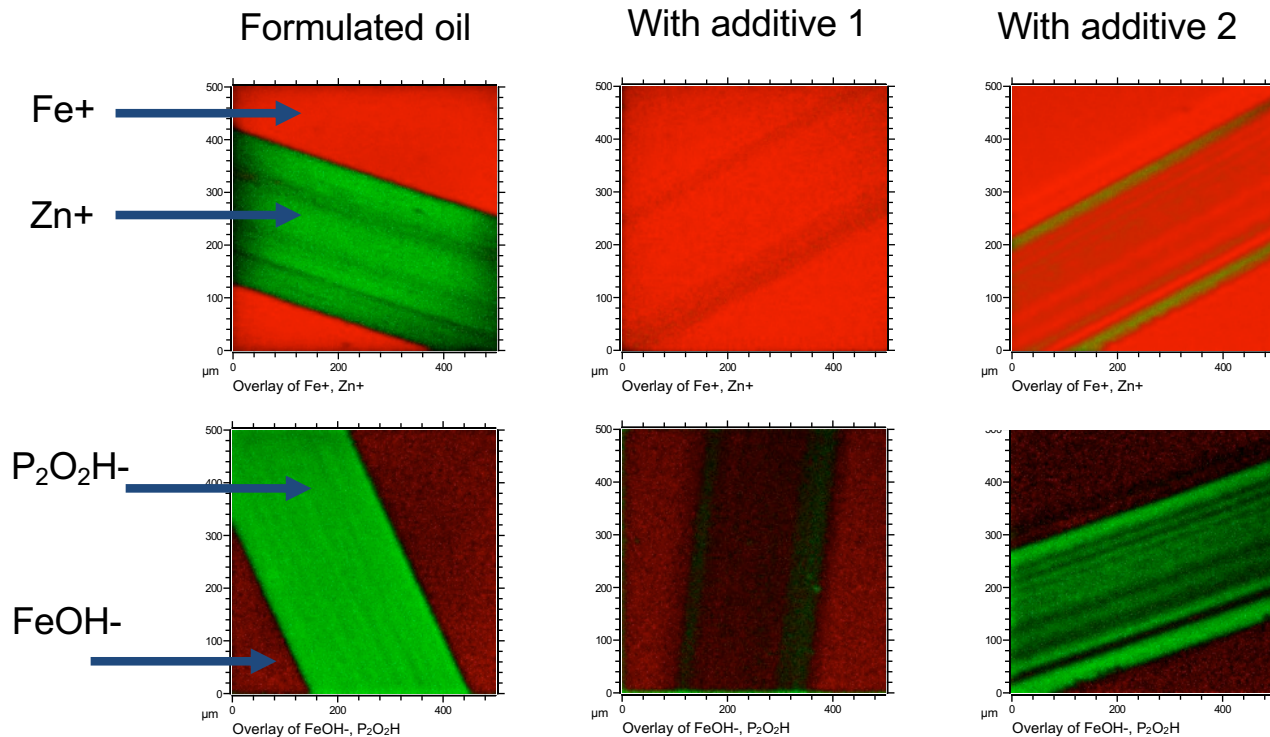


Competition for the surface

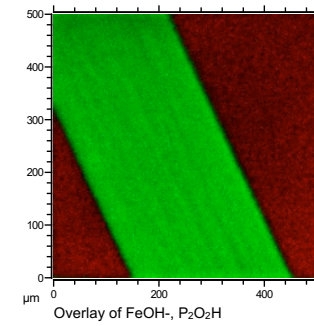
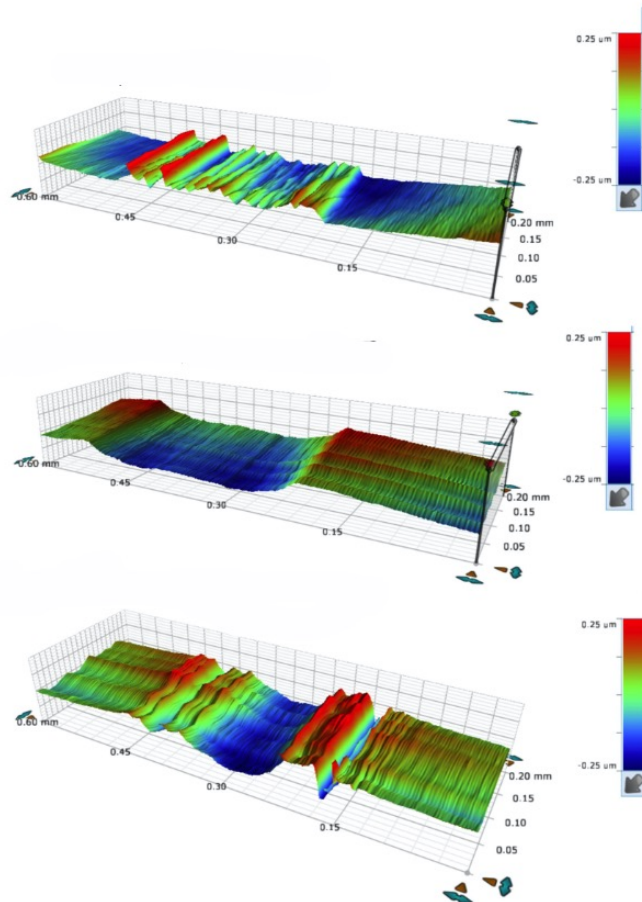
- For Viscosity
 - *Viscosity-index improvers*
 - *Pour-point depressants*
- For Lubricity
 - *Friction modifiers*
 - *Extreme-pressure and antiwear additives*
- Controlling Chemical Breakdown
 - *Detergents to prevent sludge*
 - *Corrosion inhibitors*
 - *Antioxidants*
- For Contaminant Control
 - *Dispersants (to suspend soot)*
 - *Antifoam agents*
 - *Antimisting agents*



Competition for the surface: will an antiwear additive still work when a new friction modifier is added?
 Analysis of surface with Time-of-Flight Secondary Ion Mass Spectrometry (ToF-SIMS)

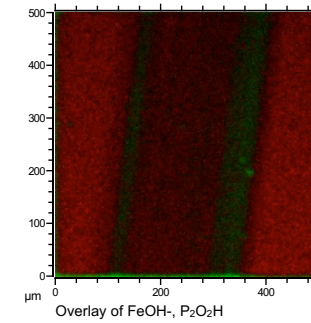


Track wear

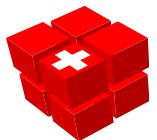
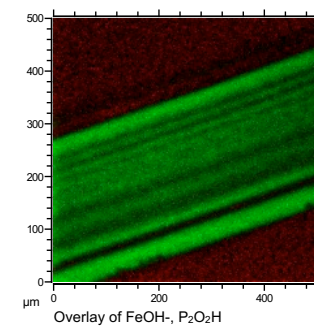


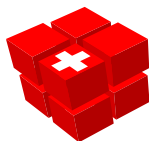
Oil

With additive 1

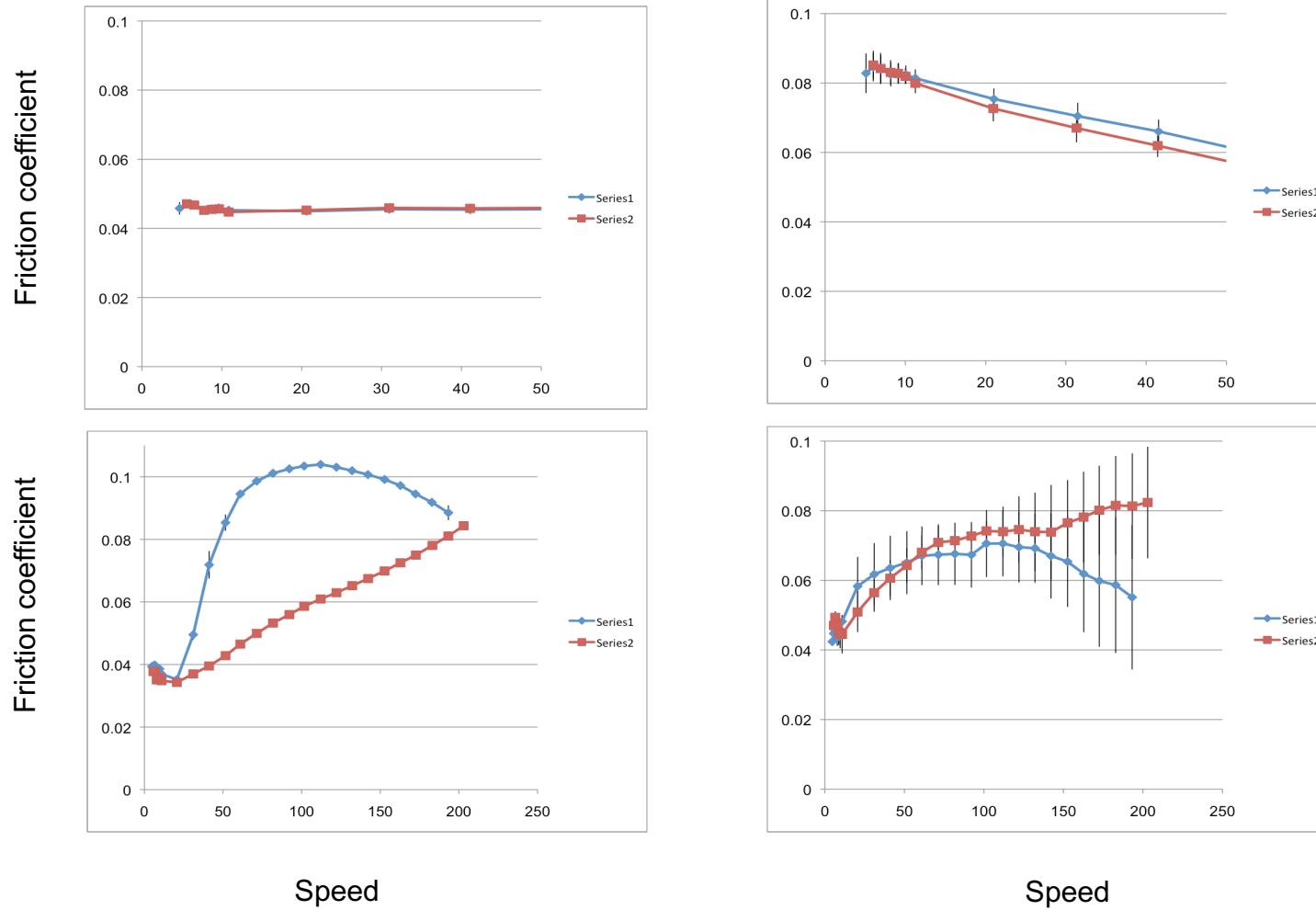


With additive 2

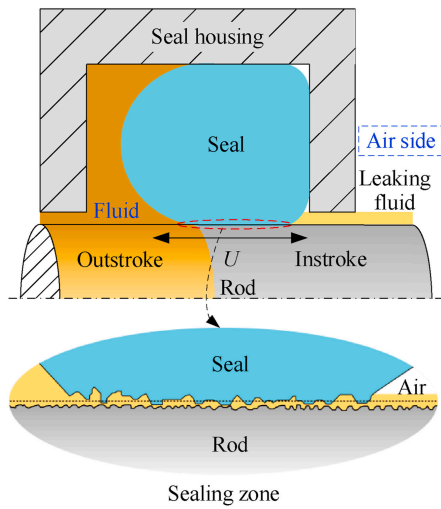




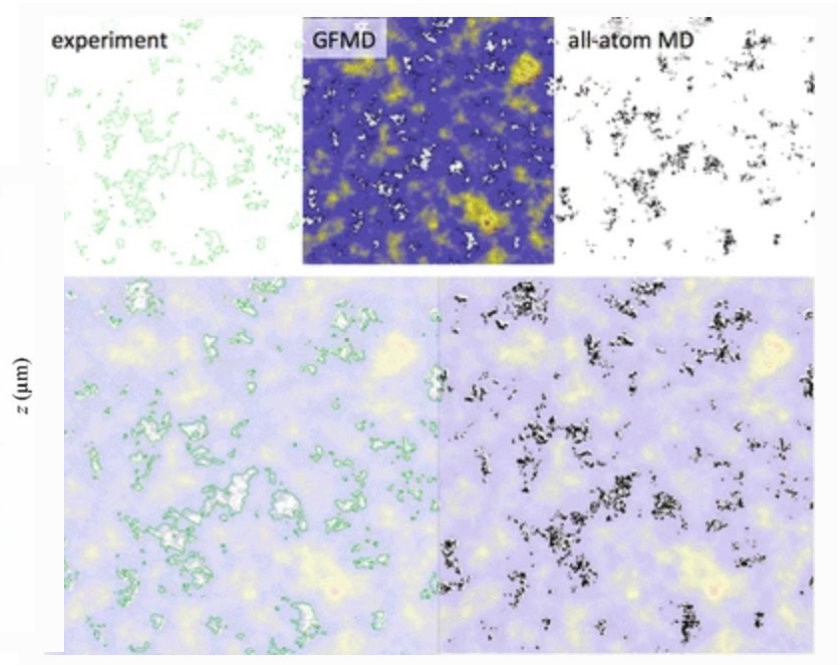
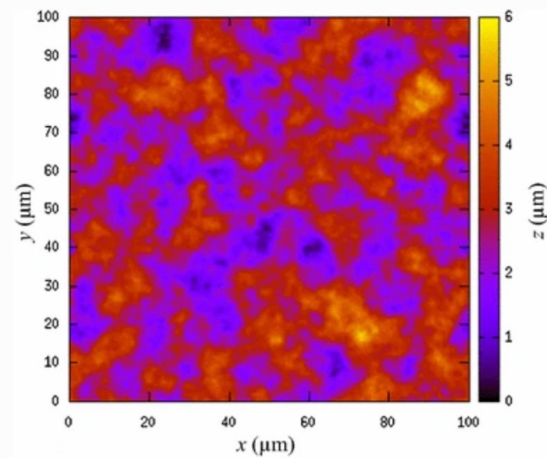
Friction instabilities – Kinetic stick-slip



Seals

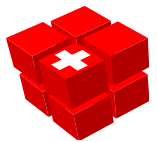


Theoretical countersurface

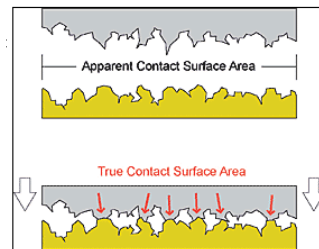


B. Wang, X. Meng, X. Peng, Y. Chen, *Tribology International*, 2021, 156, 106791

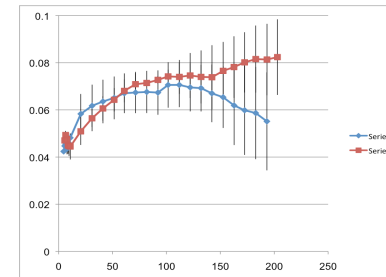
M. H. Müser et al (35 authors), *Tribology Letters*, 2017, 65, 118



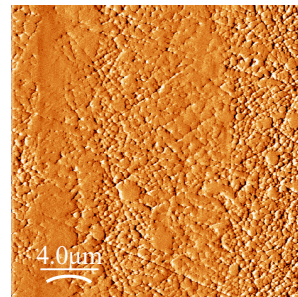
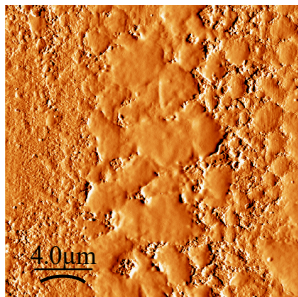
Summary



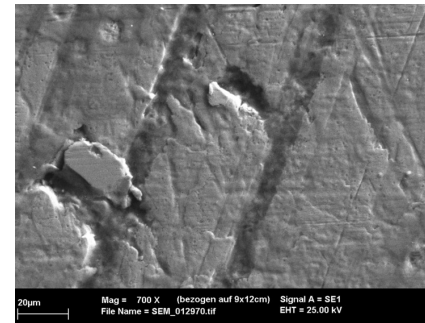
Reduction of friction
and prevention of
wear



Stabilisation of friction



Additives mainly designed for use with steel



Good adhesion has negative
consequences

