AEROELASTICITY AND FLUID-STRUCTURE INTERACTION

Chapter 6:

Aeroelasticity Pseudo-Static framework



 $U_R \ll 1$



The solid evolves in an almost still fluid

- → Flow induced stiffness
- → Added mass

 $U_R\gg 1$



The fluid evolves with an almost fixed solid

→ Static instability (Divergence)

Page 2

→ Dynamic Instability (Flutter)

Case of moderate reduced velocity?

What if U_R is not very small and not very large?

$$U_R = \frac{T_{solid}}{T_{fluid}}$$



Moderate reduced velocity:

- The velocity of the solid may not be neglected:
- Far from the interface: $U_f \sim O(1)$
- At the fluid-solid interface:

$$U_f \sim O\left(\frac{\xi_0}{T_{solid}} \frac{1}{U_0}\right) = O\left(\frac{\xi_0}{T_{solid}} \frac{1}{\frac{L}{T_{fluid}}}\right) = O\left(\frac{D}{U_R}\right)$$

The non-dimensional displacement of the interface is of the order of D and evolves in a non-dimensional time scale of U_R

Acceleration:

$$t_f = \frac{t}{T_{fluid}}$$
 and $t \sim T_{solid} \Rightarrow \frac{\partial U_f}{\partial t_f} \sim O\left(\frac{D}{U_R^2}\right)$

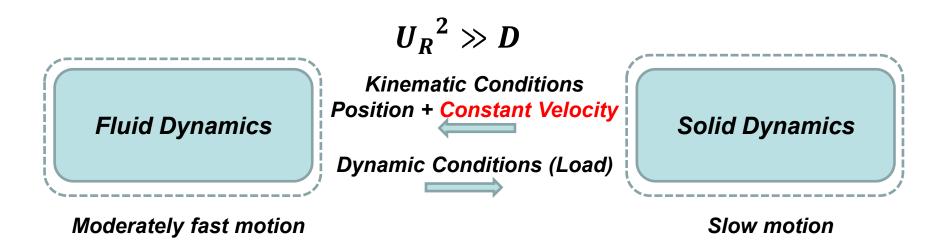


Pseudo-static aeroelasticity:

- Hypothesis: $\frac{D}{U_R^2} \ll 1 \Rightarrow U_R^2 \gg D \Rightarrow \frac{\partial U_f}{\partial t_f} \approx 0$ at the interface
 - The velocity of the solid is supposed constant within the time scale of the fluid. Its variations are neglected.
 - > From fluid viewpoint, the solid moves so slowly that its velocity seems constant (pseudo-static aeroelasticity)
 - Pseudo-static vs quasi-static aeroelasticity: The condition ${U_R}^2\gg D$ is weaker than in quasi-static aeroelasticity ($U_R\gg 1\gg D$), where the solid seems frozen from the fluid viewpoint ($U_f = 0$ at the interface)



Pseudo-Static aeroelasticity approximation



At every time step (on the fluid side):

- The flow may be solved by considering both the actual position and velocity of the solid frozen in time.
- As for quasi-static aeroelasticity approximation, there is no need to solve fluid and solid motions simultaneously. \rightarrow Iterations: fluid computation $\leftarrow \rightarrow$ solid computation



Dimensionless equations of solid motion

In the solid side (single mode approximation):

$$\xi_s(x_s,t_s) = Dq_s(t_s)\phi(x_s)$$

$$U_R^2 \frac{d^2 q_s}{dt_f^2} + q_s = f_s$$

 In the case of stationary flow, we may state that, at the interface and everywhere in the fluid, pressure and velocity depend only on the position and the velocity of the solid:

$$p_f(Dq_s, D\dot{q}_s)$$
 and $U_f(Dq_s, D\dot{q}_s)$



Dimensionless equations of solid motion

The fluid load on the solid (Expansion in D):

$$F_{s} = Cy \int_{I} \left\{ \left[-p_{f}I + \frac{1}{Re} \left(\nabla U_{f} + \nabla^{t}U_{f} \right) \right] \cdot n \right\} \cdot \phi dS$$

$$F_s = C_y F(Re, Dq_s, D\dot{q}_s)$$

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$$m{F}_S = m{F}_S^{\ 0} + m{D}m{f}_S + \cdots \qquad f_S$$
 : fluctuation of the fluid loading on the solid

 F_s^0 only affects the static equilibrium. No influence on the solid dynamics

$$f_{s} = C_{y} \left(\frac{\partial F}{\partial Re} \right) Re + C_{y} \left(\frac{\partial F}{\partial q_{s}} \right) q_{s} + C_{y} \left(\frac{\partial F}{\partial \dot{q}_{s}} \right) \dot{q}_{s} \dots$$



Dimensionless equations of solid motion

• The fluid load on the solid is the sum of 2 forces:

$$f_s = C_y \left(\frac{\partial F}{\partial q_s} \right) q_s + C_y \left(\frac{\partial F}{\partial \dot{q}_s} \right) \dot{q}_s \dots$$

- $C_y\left(\frac{\partial F}{\partial q_s}\right)q_s$: proportional to modal displacement \rightarrow fluid induced stiffness force (already obtained in quasi-static aeroelasticity approximation)
- $C_y\left(\frac{\partial F}{\partial \dot{q}_s}\right)\dot{q}_s$: proportional to solid velocity \rightarrow fluid induced damping force



Dimensionless equations of solid motion

• If T_{solid} is taken as reference time $(t_s = t/T_{solid})$, the modal displacement obeys the following equation:

$$\frac{d^2q_s}{dt_s^2} + q_s = C_y \left(\frac{\partial F}{\partial q_s}\right) q_s + \frac{C_Y}{U_R} \left(\frac{\partial F}{\partial \dot{q}_s}\right) \dot{q}_s$$

• We define the fluid stiffness k_f and fluid damping c_f as follows:

$$k_f = -C_y \left(\frac{\partial F}{\partial q_s} \right)$$
 and $c_f = -\frac{C_Y}{U_R} \left(\frac{\partial F}{\partial \dot{q}_s} \right)$
 $\Rightarrow \frac{d^2 q_s}{dt_s^2} + c_f \dot{q}_s + (1 + k_f) q_s = 0$

Here we did not take into account the structural damping



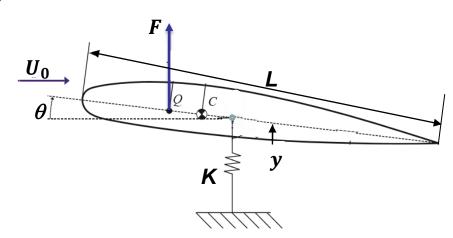
Dimensionless equations of solid motion

- The fluid damping (as the fluid stiffness) is proportional to Cauchy number $\left(Cy=rac{
 ho U_0^2}{E}
 ight)$
- The fluid damping may be positive or negative:
 - $c_f > 0$: The oscillations are damped by the fluid (stable)
 - $c_f < 0$: The oscillations increase without limit (unstable)
- Comparison with the quasi-static instability:
 - Quasi-static instability involves two modes with different frequencies, which come close to each other by the flow
 - For pseudo-static instability, any given mode may become unstable without the need of a frequency coincidence with another mode. Such instability is therefore more likely to occur.



Case of a flow over an airfoil in plunge mode

- We consider an airfoil of mass M, placed in an air stream of upstream velocity U_0 at a fixed incidence θ , attached to a spring (stiffness K) so that it can move only in the vertical direction
- In quasi-static aeroelasticity approximation $(U_R\gg D)$
 - Equation of the airfoil motion ?
 - Is there any fluid induced stiffness and/or damping?
 - Stability?

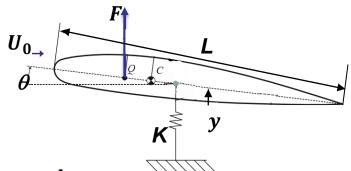




Case of a flow over an airfoil in plunge mode

- In quasi-static aeroelasticity approximation $(U_R\gg D)$:
 - As the airfoil moves in the y direction, its incidence angle remains unchanged and so does the lift force.
 - The equation of solid motion reads:

$$M\ddot{y}+Ky=0$$
 (The lift and stiffness forces cancel out)



- \rightarrow The flow has no effect on the airfoil dynamics (except translational displacement) Increase of flow velocity \rightarrow the spring is more stretched Dynamically, the airfoil behaves as if it is in still fluid ($U_0 = 0$)
- Infinite stiffness in torsion

 there is no risk of divergence



Case of a flow over an airfoil in plunge mode

- In pseudo static aeroelasticity approximation $({U_R}^2\gg D)$
 - The solid displacement (y) and velocity (\dot{y}) are both frozen
 - In the frame of the airfoil, the upstream velocity is V_0 and the effective incidence angle is θ_{eff} such that:
 - The Lift force (F) depends on both θ and \dot{y}

$$\overrightarrow{U_0} = \overrightarrow{V_0} + \overrightarrow{\dot{y}}$$
 $\theta_{eff} = \theta + \alpha$

$$an lpha = -rac{\dot{y}}{U_0}$$
 $lpha$ and \dot{y} are of opposite signs

$${V_0}^2 = {U_0}^2 + \dot{y}^2$$



Case of a flow over an airfoil in plunge mode

- In pseudo static aeroelasticity approximation (U_R²>>D)
 - The aerodynamic force (lift):

$$F = \frac{1}{2}\rho V_0^2 LC_L(\theta_{eff}) = \frac{1}{2}\rho V_0^2 LC_L(\theta + \alpha)$$

$$F = \frac{1}{2}\rho U_0^2 \left(1 + (\tan \alpha)^2\right) L \left(C_L(\theta) + \frac{\partial C_L}{\partial \theta}(\theta) \cdot \alpha\right)$$

• For small values of α : $\tan \alpha \approx \alpha \quad \alpha^2 \ll 1$

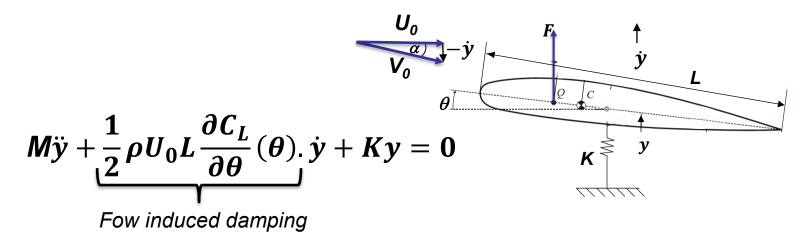
$$\Rightarrow F \approx F_0 + \frac{1}{2}\rho U_0^2 L \frac{\partial C_L}{\partial \theta}(\theta). \alpha = F_0 - \frac{1}{2}\rho U_0 L \frac{\partial C_L}{\partial \theta}(\theta). \dot{y}$$



Case of a flow over an airfoil in plunge mode

- In pseudo static aeroelasticity approximation $(U_R^2 >> D)$
 - Equation of the airfoil motion:

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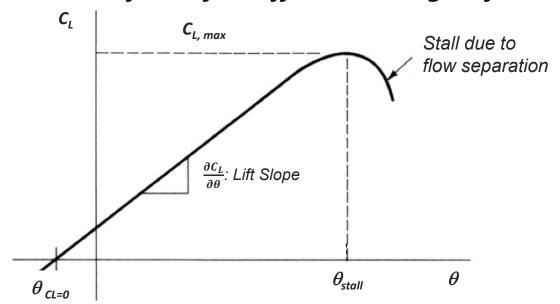


- The sign of lift slope is the key parameter:
 - Positive damping $\left(\frac{\partial C_L}{\partial \theta} > 0\right)$ \Rightarrow Stable
 - Negative damping $\left(\frac{\partial C_L}{\partial \theta} < 0\right)$ \rightarrow Unstable



Case of a flow over an airfoil in plunge mode

- In pseudo static aero-elasticity approximation $(U_R^2 >> D)$
 - Typical curve of the lift coefficient vs angle of attack θ :



$$\theta < \theta_{stall} \Rightarrow \frac{\partial C_L}{\partial \theta} > 0$$
positive damping \rightarrow Stable

$$\theta > \theta_{stall} \Rightarrow \frac{\partial C_L}{\partial \theta} < 0$$

Negative damping → Unstable → <u>Stall flutter</u>



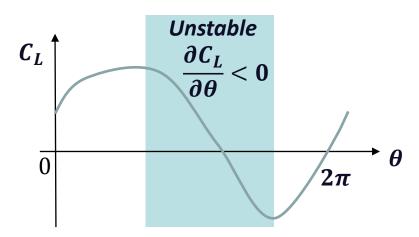
Summary of possible instabilities for a foil

- Fast flows $(U_R >> D)$ quasi-static aero-elasticity approximation
 - → Divergence: Static instability in torsion mode
 - → Flutter: Dynamic instability of the torsion and plunge modes
- Moderate flows $(U_R^2 >> D)$ Pseudo-static aero-elasticity approximation → Stall Flutter: Dynamic instability of the plunge mode
- These instabilities may be encountered in any configuration involving lifting surfaces in a flow



Case of an arbitrary bluff body in plunge mode – Lift Galloping

- In pseudo static aero-elasticity approximation $(U_R^2 >> D)$
 - The $C_L(\theta)$ curve may be of any shape but must be periodic
 - ightharpoonup There must be a portion of the C_L curve where $rac{\partial C_L}{\partial heta} < \mathbf{0}$

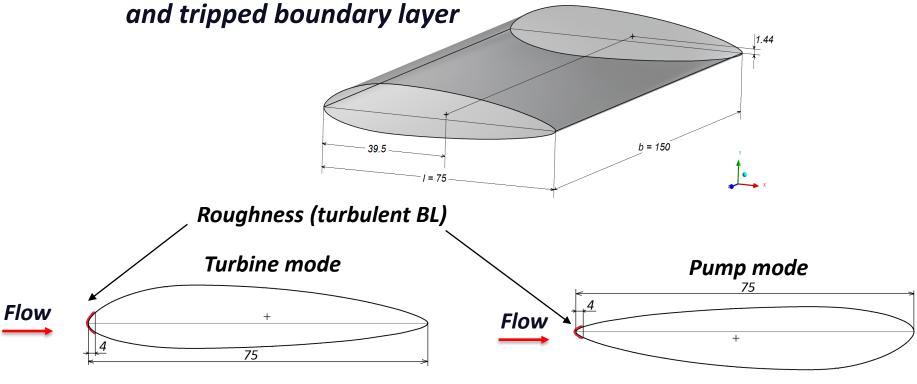


• Such instability is called "Lift Galloping". It is a generalization of the stall flutter



Case of a flow over a wicket gate of a pump turbine

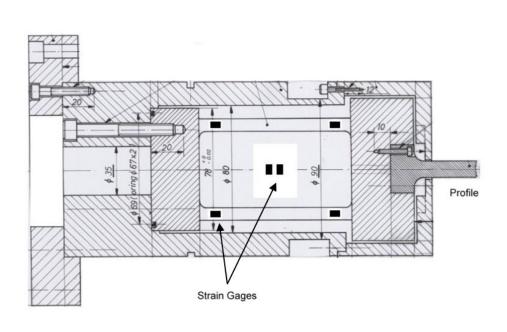
- Designed to operate in both pumping and generating modes
 - Rounded leading and trailing edges
 - Tested in EPFL High-Speed Cavitation Tunnel with smooth





Case of a flow over a wicket gate of a pump turbine

- Measurement of hydrodynamic forces (lift and drag):
 - 5-components load cell

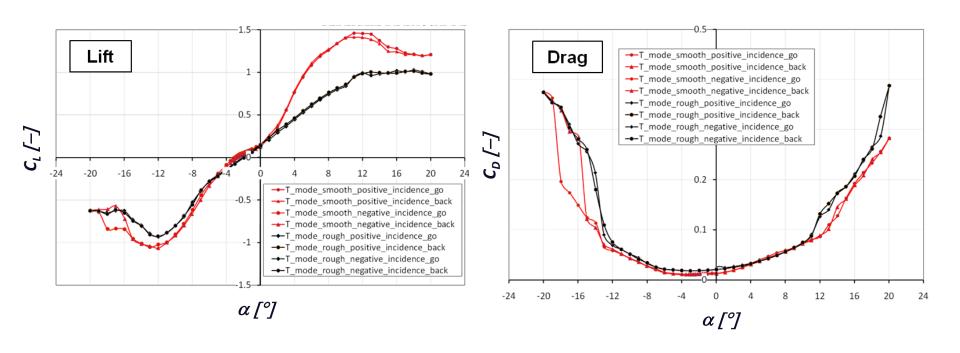






Case of a flow over a wicket gate of a pump turbine

- Lift and drag coefficients vs angle of attack in turbine mode:
 - Natural and tripped transition of the boundary layer
 - Stable for -12 $^{\circ}$ < α < 12 $^{\circ}$

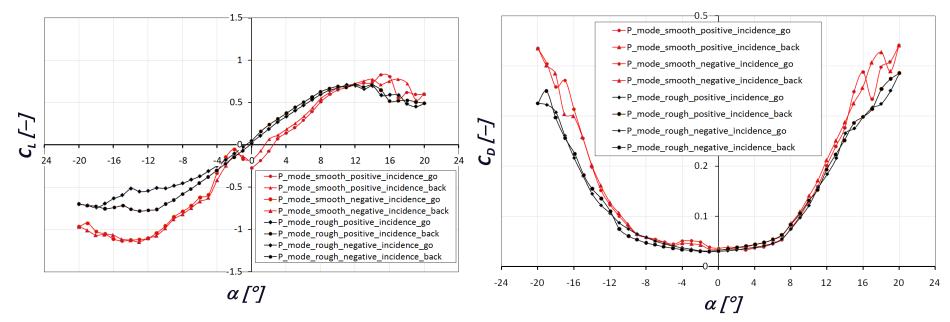




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Case of a flow over a wicket gate of a pump turbine

- Lift and drag coefficients vs angle of attack in <u>pump mode</u>:
 - Natural and tripped transition of the boundary layer



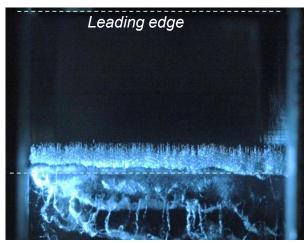
- Natural BL transition: negative lift slope for $-2^{\circ}<\theta<0^{\circ}$
- \rightarrow Risk of lift galloping due to negative slope of $C_L(\alpha)$ Cause: Boundary layer separation/attachment near 0° incidence



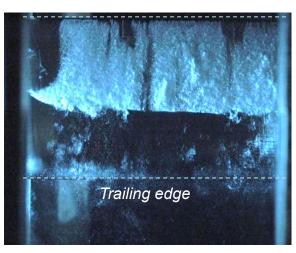
Case of a flow over a wicket gate of a pump turbine

Flow visualization (top view) in pump mode, using cavitation:

 α = -2°, Vortex shedding (Strong vibration)



 α = -8°, attached flow



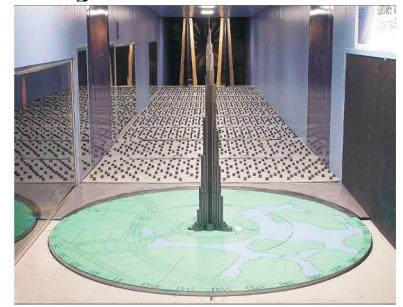
 α = -16°, BL separation at leading edge (stall)



Example: The case of tall buildings

- There always exist an interval for wind direction where the lift slope is negative → Risk of lift galloping
 - Such a risk is seriously taken into account during the design process (numerical simulation and model testing)
 - Sophisticated monitoring of static & dynamic deformations of the building with the help of sensing networks

Reduced scale model of Burj Khalifa placed in 2.4 x 2.0 m test section of a wind tunnel





Lift galloping

- It is well known that for small incidence angles:
 - Elongated shapes in the streamwise direction are stable
 - Elongated shapes in the transverse direction are unstable

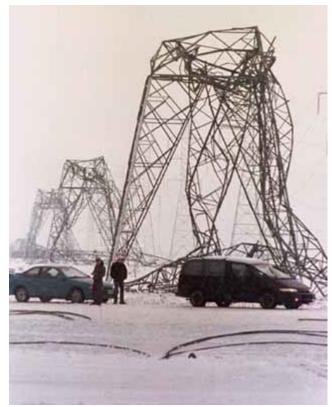




Lift galloping

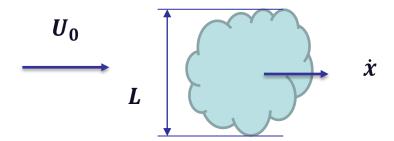
- Example: Electric lines during ice storms (Quebec blackout, 1998)
 - Ice accumulation around the lines
 - → Increased risk of lift galloping





Drag induced instability: Drag Crisis

- Within the pseudo static aeroelasticity approximation $(U_R^2 >> D)$
 - We consider a bluff body oscillating in the direction of the flow:



- The fluid loading (Drag force) ?
- Is there any flow induced stiffness or damping?
- Stability in the case of a cylinder?



Drag induced instability: Drag Crisis

- Within the pseudo static aeroelasticity approximation $(U_R^2 >> D)$
 - We consider a bluff body moving the direction of the flow with a speed frozen in time:
 - The fluid loading (Drag force):

$$F(U_{0} - \dot{x}) = \frac{1}{2}\rho(U_{0} - \dot{x})^{2}LC_{D}(R_{E}) \approx F(U_{0}) - \dot{x}\frac{\partial F}{\partial \dot{x}} + \cdots$$

$$\frac{\partial F}{\partial \dot{x}} = \frac{\partial}{\partial \dot{x}}\left(\frac{1}{2}\rho(U_{0} - \dot{x})^{2}LC_{D}(R_{E})\right) \qquad R_{E} = \frac{\rho(U_{0} - \dot{x})L}{\mu}$$

$$\frac{\partial F}{\partial \dot{x}} \approx -\rho U_{0}L\left(C_{D} + \frac{1}{2}R_{E}\frac{\partial C_{D}}{\partial R_{E}}\right) \quad assuming \quad \dot{x} \ll U_{0}$$



Drag induced instability: Drag Crisis

- In pseudo static aeroelasticity approximation $(U_R^2 >> D)$
 - We consider a bluff body oscillating in the direction of the flow:
 - The fluid loading (Drag force):
 - Damping force

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$$\stackrel{U_0}{\longrightarrow} L$$

$$F(U_0 - \dot{x}) \approx F(U_0) - \rho U_0 L \dot{x} \left(C_D + \frac{1}{2} R_E \frac{\partial C_D}{\partial R_E} \right)$$

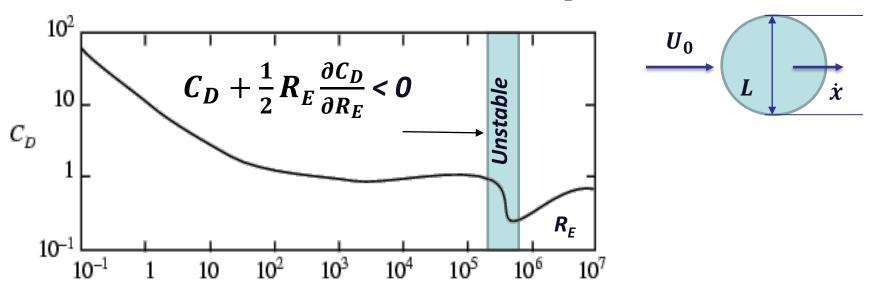
- $C_D + \frac{1}{2}R_E \frac{\partial C_D}{\partial R_E} > 0 \Rightarrow positive damping \Rightarrow Stable$
- $C_D + \frac{1}{2}R_E \frac{\partial C_D}{\partial R_E} < 0 \Rightarrow$ negative damping \Rightarrow Unstable

→ DRAG CRISIS Instability or DRAG Galloping



Drag induced instability: Drag Crisis

- Is it possible to have a negative drag-induced damping?
- Example: A cylinder oscillating in the direction of the flow:
 - The Drag coefficient of a cylinder vs R_E :

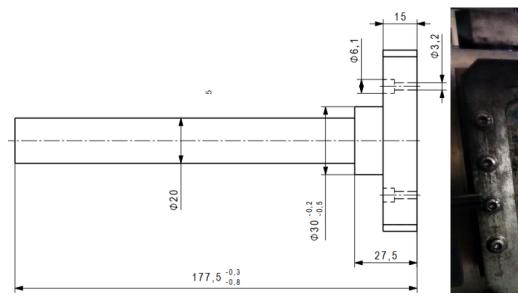


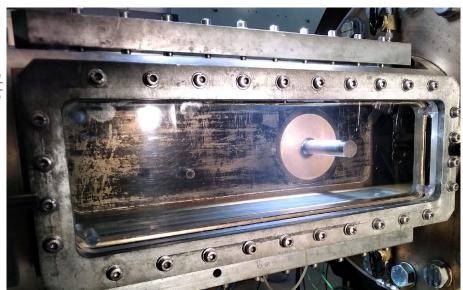
 A negative damping is the result of a profound change in the flow structure (boundary layer separation delayed)



Drag induced instability: Drag Crisis

- Case f a cylinder in EPFL Cavitation Tunnel (semester project 2021)
 - Stainless steel, made from one bloc of metal
 - Several cylinders made of assembly of 2 parts were destroyed because of too much vibration !!
 - 25 mm diameter, 150 mm span
 - Measurement of vibration, Lift&Drag, High-speed visulization

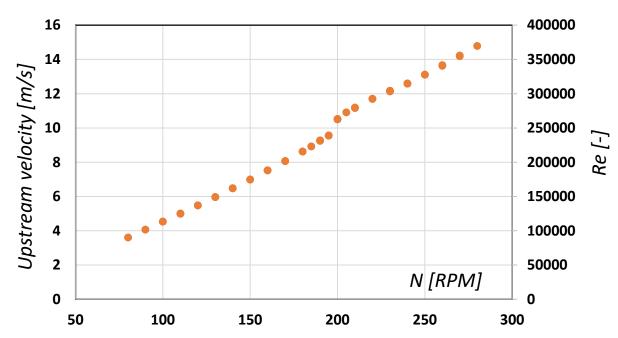






Drag induced instability: Drag Crisis

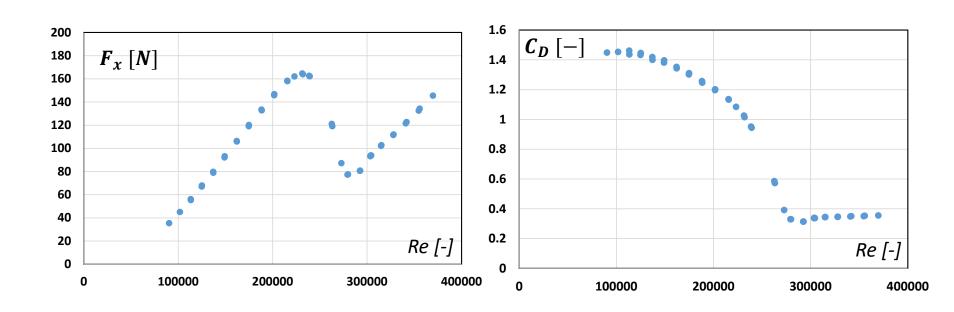
- Case f a cylinder in EPFL Cavitation Tunnel (semester project 2021)
 - Evidence of drag crisis (Cavitation free)
 - Upstream velocity vs Rotation speed of the pump
 - Constant acceleration of the pump rotation during 3 minutes
 - Sudden increase around (200 RPM): Re=250'000
 - Due to a sudden drop of the drag force





Drag induced instability: Drag Crisis

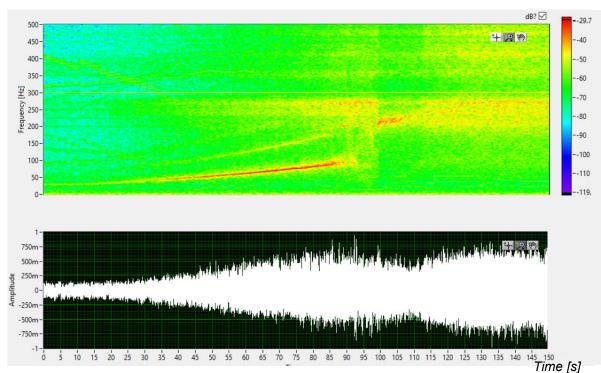
- Case f a cylinder in EPFL Cavitation Tunnel (semester project 2021)
 - Evidence of drag crisis
 - Drag force and drag coefficient vs Reynolds number
 - Significant decrease of the drag force around Re=250'000
 - Due to a delay in boundary layer separation





Drag induced instability: Drag Crisis

- Case f a cylinder in EPFL Cavitation Tunnel (semester project 2021)
 - Flow induced vibration (upstream velocity 3.6 → 13 m/s)
 - No significant increase of vibration at onset or beyond drag crisis
 - Strouhal frequency dominant before drag crisis (St~0.2)
 - Wake less organized beyond drag crisis

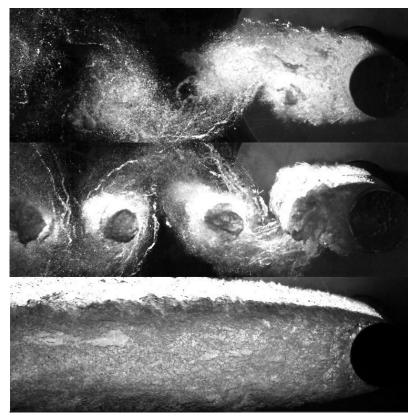




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Drag induced instability: Drag Crisis

- Case f a cylinder in EPFL Cavitation Tunnel (semester project 2021)
 - Effect of cavitation on fluid-structure vibration:
 - Cavitation occurrence in the wake
 - the vortices are more coherent
 - → A tremendous increase of vibration
 - Supercavitation:
 - → No vortex shedding
 - → Minimum vibration
 - → Minimum drag
 - Further research is underway to understand these peculiar effects





"Lift Crisis"

PRL 117, 234501 (2016)

PHYSICAL REVIEW LETTERS

week ending 2 DECEMBER 2016

Sharp Transition in the Lift Force of a Fluid Flowing Past Nonsymmetrical Obstacles: Evidence for a Lift Crisis in the Drag Crisis Regime

Patrick Bot,^{1,*} Marc Rabaud,² Goulven Thomas,¹ Alessandro Lombardi,¹ and Charles Lebret¹

¹Naval Academy Research Institute, IRENAV CC600, 29240 Brest Cedex 9, France

²Laboratoire FAST, Univ. Paris-Sud, CNRS, Université Paris-Saclay, F-91405 Orsay, France

(Received 3 February 2016; revised manuscript received 6 October 2016; published 29 November 2016)

Bluff bodies moving in a fluid experience a drag force which usually increases with velocity. However in a particular velocity range a *drag crisis* is observed, i.e., a sharp and strong decrease of the drag force. This counterintuitive result is well characterized for a sphere or a cylinder. Here we show that, for an object breaking the up-down symmetry, a *lift crisis* is observed simultaneously to the drag crisis. The term lift crisis refers to the fact that at constant incidence the time-averaged transverse force, which remains small or even negative at low velocity, transitions abruptly to large positive values above a critical flow velocity. This transition is characterized from direct force measurements as well as from change in the velocity field around the obstacle.

DOI: 10.1103/PhysRevLett.117.234501



"Lift Crisis"

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PHYSICAL REVIEW LETTERS

week ending 2 DECEMBER 2016

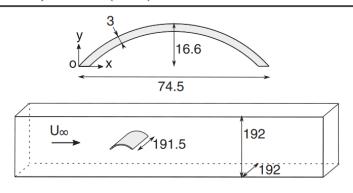
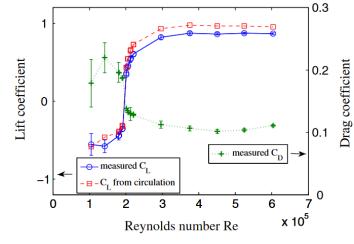
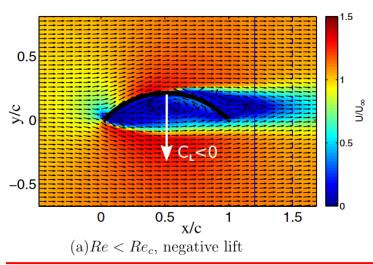
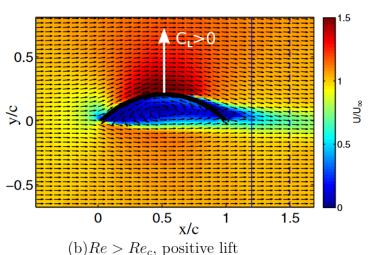
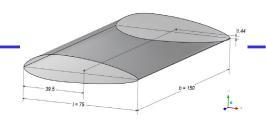


FIG. 1. Curved plate section: chord length c = 74.5 mm and camber t = 16.6 mm (top) and tunnel test setup (bottom). All dimensions are in mm.





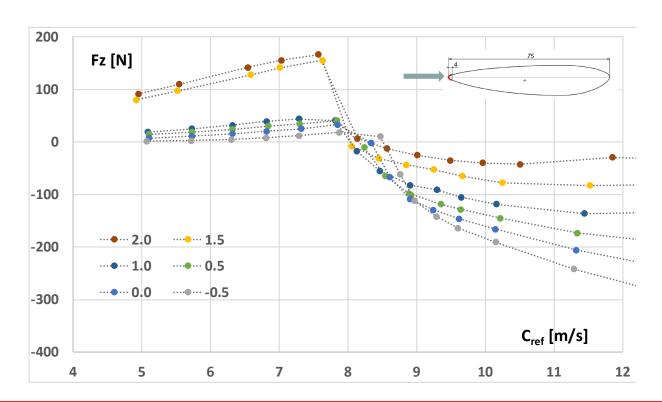




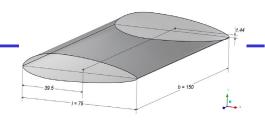
"Lift Crisis"

Case of a flow over a wicket gate of a pump turbine

- Lift force variation with upstream velocity
 - Pump mode, smooth leading edge (<u>natural BL transition</u>)



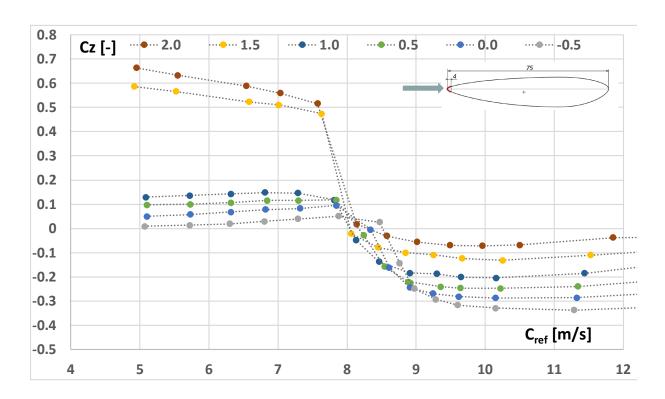




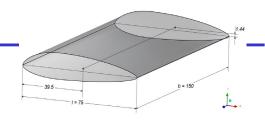
"Lift Crisis"

Case of a flow over a wicket gate of a pump turbine

- Lift coefficient variation with upstream velocity
 - Pump mode, smooth leading edge (natural BL transition)







"Lift Crisis"

Case of a flow over a wicket gate of a pump turbine

- Lift coefficient variation with Reynold Number
 - Pump mode, smooth leading edge (<u>natural BL transition</u>)

