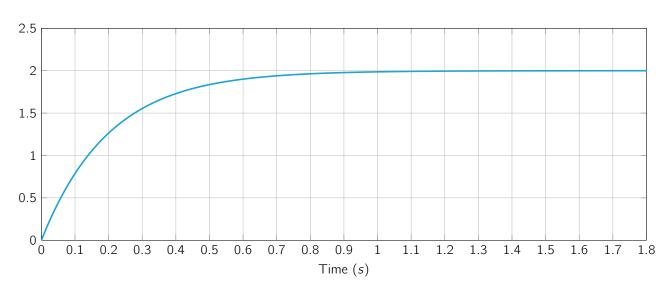
Problem 1.

In this problem, you are to design a PD controller for the system below.

Input
$$U(s)$$
 \longrightarrow $x + b$ $y = 0$ Velocity $y = 0$ Position $y = 0$ Positi

a) The figure below shows a response of the velocity V(s) to a unit step input U(s). Give constants a and b to best fit the model to this data.



b) The control specification is to ensure that the closed-loop poles lie within the shaded region shown in the figure below, where $\alpha = \sin^{-1}(0.96)$ and $\beta = \sin^{-1}(0.8)$.

Give the maximum and minimum for each of the properties below that the specification allows in response to a unit step input

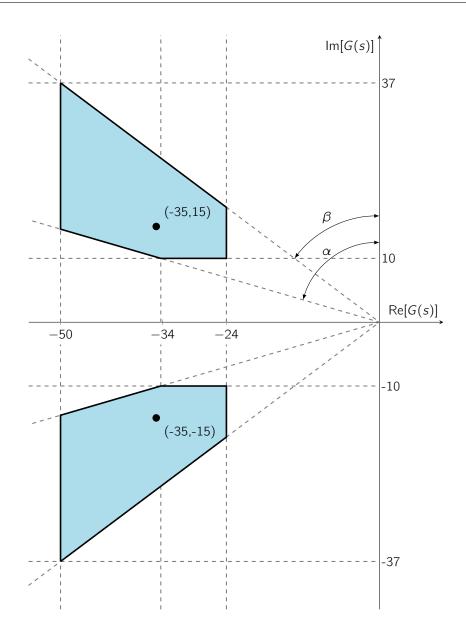
	Min	Max
Percent overshoot	0.002%	1.5%
Settling time (2%)	0.08	0.17
Peak time	0.085	0.31

Reading the plot, we see that

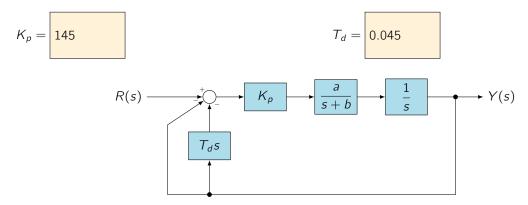
$$24 \le \sigma \le 50 \qquad \rightarrow \qquad \frac{4}{50} = 0.08 \le T_s \le \frac{4}{24} = 0.17$$

$$0.8 \le \zeta \le 0.96 \qquad \rightarrow \qquad 100e^{-\zeta\pi/\sqrt{1-\zeta^2}} = 0.002\% \le PO \le 1.5\%$$

$$10 \le \omega_d \le 37 \qquad \rightarrow \qquad \frac{\pi}{\omega_d} = \frac{\pi}{37} = 0.085 \le T_p \le \frac{\pi}{10} = 0.31$$



c) Design a PD controller such that the specification given in the previous part is met for the system. Choose your controller so that the closed-loop poles are as those marked on the figure $-35 \pm 15i$. The structure of the control system is shown below (recall that we don't take the derivative of the reference).



Compute the closed-loop transfer function

$$Y = \frac{1}{s} \frac{a}{s+b} K_p (R - Y - T_d s Y)$$

$$\frac{Y}{R} = \frac{aK_p}{s^2 + s(b + aK_p T_d) + aK_p}$$

$$= \frac{10K_p}{s^2 + s(5 + 10K_p T_d) + 10K_p}$$

From the points given on the figure, we form the desired characteristic equation

$$(s - (-35 + 15i))(s - (-35 - 15i)) = s^2 - s(-35 - 15i - 35 + 15i) + 35^2 + 15^2$$
$$= s^2 + 70s + 1450$$

Matching the characteristic equations, we solve for $\mathcal{K}_{\textit{p}}$ and $\mathcal{T}_{\textit{d}}$

$$10K_p = 1450$$
 \rightarrow $K_p = 145$ $5 + 10K_pT_d = 70$ \rightarrow $T_d = \frac{6.5}{145} = 0.045$

Problem 2.

Consider the system

$$G(s) = \frac{s+1}{s^2 + 3s + 4}$$

a) Give the system in control canonical form $\dot{x} = Ax + Bu$, y = Cx + Du

$$A = \begin{bmatrix} -3 & -4 \\ 1 & 0 \end{bmatrix}$$

$$C = \begin{bmatrix} 1 \\ 1 \end{bmatrix}$$

$$D = \begin{bmatrix} 0 \\ 0 \end{bmatrix}$$

b) Design a full state feedback controller that places the closed-loop poles at -3 and -4.

$$K = \begin{bmatrix} 4 & 8 \end{bmatrix}$$

$$\det(sI - A + BK) = \begin{vmatrix} s & 0 \\ 0 & s \end{vmatrix} - \begin{bmatrix} -3 & -4 \\ 1 & 0 \end{bmatrix} + \begin{bmatrix} 1 \\ 0 \end{bmatrix} \begin{bmatrix} K_1 & K_2 \end{bmatrix} \end{vmatrix}$$
$$= \begin{vmatrix} s + 3 + K_1 & 4 + K_2 \\ -1 & s \end{vmatrix}$$
$$= s^2 + (3 + K_1)s + 4 + K_2$$
$$= (s + 3)(s + 4) = s^2 + 7s + 12$$

which gives the solution $K = \begin{bmatrix} 4 & 8 \end{bmatrix}$

c) Design an estimator for the system that places the estimator poles 3× faster than the controlled poles

$$L = \begin{bmatrix} 61 \\ -43 \end{bmatrix}$$

$$det(sI - A + LC) = \begin{vmatrix} s & 0 \\ 0 & s \end{vmatrix} - \begin{bmatrix} -3 & -4 \\ 1 & 0 \end{bmatrix} + \begin{bmatrix} L_1 \\ L_2 \end{bmatrix} \begin{bmatrix} 1 & 1 \end{bmatrix} \begin{vmatrix} s + 3 + L_1 & 4 + L_1 \\ -1 + L_2 & s + L_2 \end{vmatrix} = s^2 + s(L_2 + L_1 + 3) + L_2(3 + L_1) - (4 + L_1)(-1 + L_2) = (s + 9)(s + 12) = s^2 + 21s + 108$$

Solving gives $L_1 = 61$ and $L_2 = -43$

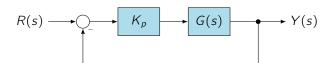
d) What are the poles of the resulting closed-loop system?

$$\begin{bmatrix} -3 & -4 & -9 & -12 \end{bmatrix}$$

Separation principle means that the closed-loop poles are those of the controlled system and the estimator.

Problem 3.

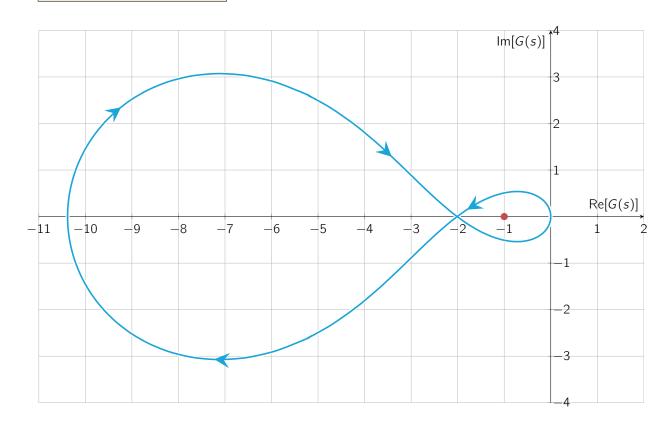
Consider the following closed-loop system with a proportional controller \mathcal{K}_p



a) The Nyquist plot for the system $5.2\frac{(s+3)(s+2)}{(s-3)(s+1)^2}$ is shown below.

Give the range of values of \mathcal{K}_{p} for which the closed-loop system will be stable

$$K_p \ge 0.5$$



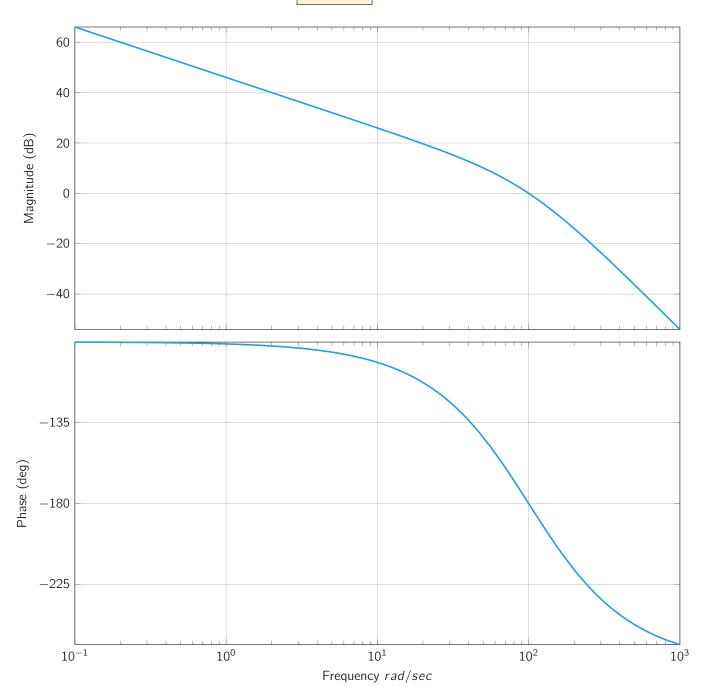
b) Sketch the Bode plot of the system $G(s) = \frac{2,000,000}{s^3 + 200s^2 + 10,000s}$ and estimate the gain and phase margins.

Gain margin \approx 0 (dB)

Phase margin \approx 0 (degrees)

Will the closed-loop system be stable if $K_p = 5$?

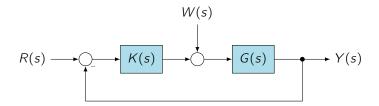
No (yes/no)

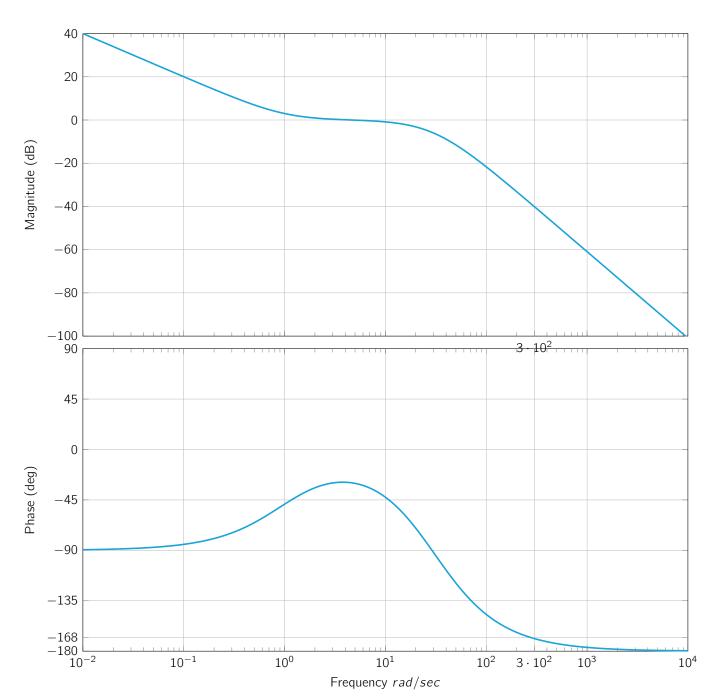


Problem 4.

Shown below is the Bode plot of the system

$$G(s) = \frac{s+1}{s(s/30+1)^2}$$





- a) Design a lead compensator and proportional gain to satisfy the following criteria
 - Crossover frequency of around 300r/s
 - Phase margin of 50 degrees

b) What is the steady-state error of the closed-loop system with your controller in response to:

Unit step of the reference R(s) Unit step of the disturbance W(s)

- c) Add a lag compensator to your controller so that the following criteria are met
 - Crossover frequency of around 300r/s
 - Phase margin of at least 30 degrees
 - Zero steady-state error in response to a unit step input in the disturbance
 - Zero steady-state error in response to a unit ramp input in the reference

K(s) =	
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Problem 5.

Consider the dynamic system

$$\dot{x} = \begin{bmatrix} -5 & -4 \\ -4 & -5 \end{bmatrix} x + Bu$$
$$y = Cx$$

a) For which of the following matrices is the system controllable and/or observable?

		Controllable? (yes/no)	Observable? (yes/no)
$B = \begin{bmatrix} 1 \\ 1 \end{bmatrix}, C = \begin{bmatrix} 1 \\ 1 \end{bmatrix}$	= [1 1]	No	No
$B = \begin{bmatrix} 1 \\ 2 \end{bmatrix}, C = \begin{bmatrix} 1 \\ 2 \end{bmatrix}$	= [1 1]	Yes	No
$B = \begin{bmatrix} 1 \\ 2 \end{bmatrix}, C = \begin{bmatrix} 1 \\ 2 \end{bmatrix}$	= [2 1]	Yes	Yes
$B = \begin{bmatrix} 1 \\ 1 \end{bmatrix}$, $C = \begin{bmatrix} 1 \\ 1 \end{bmatrix}$	= [2 1]	No	Yes

For the remaining question parts, assume that $B = \begin{bmatrix} 1 \\ 0 \end{bmatrix}$, $C = \begin{bmatrix} 1 & 0 \end{bmatrix}$ and that the controller $u = -\begin{bmatrix} 1 & 2 \end{bmatrix}x + Nr$ is applied to the system, where r is a reference input and N is a constant.

b) Give the transfer function from the reference r to the output of the system y, parameterized by N.

$$\frac{Y(s)}{R(s)} = N \frac{s+5}{s^2 + 11s + 6}$$

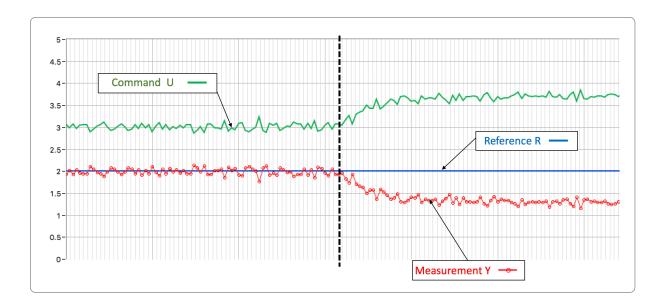
c) Give a value for N so that the output y will equal r in steady-state if r is a step input.

$$N = \boxed{\frac{6}{5}}$$

Problem 6. Travaux Pratiques

Notes:

- The answer and the justification must be correct to get the points.
- Measurements have been made on the electrical drives used during the labs.

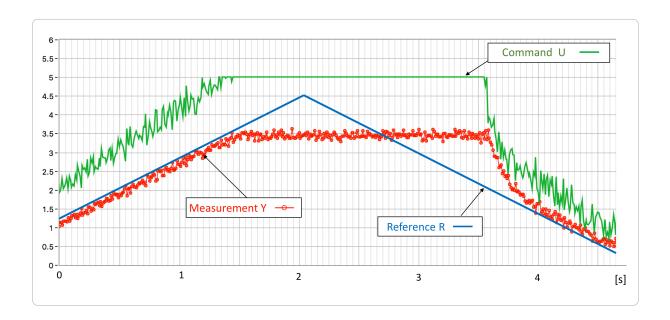


1a) A constant perturbation has been applied to the system at the dashed line. What type of control structure is implemented? Justify.

Speed control \rightarrow PI or P + U0, but since it cannot compensate for the perturbation \rightarrow P + U0

1b) Compute the controller parameters.

U0 = 3v since U = (R-Y)*Kp+U0, 3 = (2-2)*Kp + U0, measured before the perturbation Kp = 1, since U = (R-Y)*Kp+U0, 3.75 = (2-1.25)*Kp + 3, measured after the perturbation



2a) What is the controller structure? Justify.

PI since speed ctrl and permanent err

2b) Between 1.5 [s] and 3.5 [s] the command remains saturated, why? Justify.

The command was saturated between 1.5 and 2.8s, cannot reach the desired value Between 2.8-3.5s still saturated since Ui was increasing internally and needs to be emptied before the output can follow the reference again (- $\dot{\iota}$ ARW not actif)