INSTITUT D'INGÉNIERIE CIVILE IIC

# Laboratoire des Structures Métalliques Résilientes RESSLAB

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# In-class Exercise – Week #8: Zero length elements for material nonlinearity

#### Exercise #1:

Derive the stiffness matrix for an element that is comprised of a zero-length rotational element and an elastic beam-column element as shown in Figure 1.1 in the basic reference system. Assume that the rotational stiffness of the zero-length element is  $n\frac{3EI_e}{L}$  where  $I_e = \frac{n+1}{L}I$ .



Figure 1.1. Zero length rotational element and elastic beam-column element in series

Hint: By using static condensation, determine the coefficients  $S_{22}$ ,  $S_{23}$ ,  $S_{32}$  and  $S_{33}$  of the stiffness matrix  $\hat{\mathbf{k}}_{mod}$  of the elastic beam-column element given by

elastic beam-column element given 
$$\hat{\mathbf{k}}_{mod} = \begin{bmatrix} \frac{EA}{L} & 0 & 0\\ 0 & \frac{S_{22}EI_e}{L} & \frac{S_{23}EI_e}{L}\\ 0 & \frac{S_{32}EI_e}{L} & \frac{S_{33}EI_e}{L} \end{bmatrix}$$

such that a beam-column member can be modeled with the derived elastic beam-column element and a rotational spring.

Recall that the elastic stiffness matrix,  $\hat{\mathbf{k}}$  of an elastic beam-column element in the basic reference system is as follows:

$$\hat{\mathbf{k}} = \begin{bmatrix} \frac{EA}{L} & 0 & 0\\ 0 & \frac{4EI}{L} & \frac{2EI}{L}\\ 0 & \frac{2EI}{L} & \frac{4EI}{L} \end{bmatrix}$$

## Exercise #2:

Write a script for a zero-length rotational element that utilizes the moment-rotation constitutive relation shown in Figure 2.1. Consider both the loading and unload paths in your model. Validate your implementation for the following input parameters:

- o  $k_e = 400000kN.mm/rad$ o  $\theta_p = \theta_c - \theta_y = 0.02 \ rad$ o  $\theta_{pc} = \theta_u - \theta_c = 0.05 \ rad$ o  $M_y^* = 4000kN.mm \ and \ M_u = 4500kN.mm$
- 1. Load case #1: Rotational monotonic loading,  $\theta = \{0, 0, 08\}^T rad$
- 2. Load case #2: Rotational cyclic loading,  $\theta = \{0, 0, 06, -0, 06, 0\}^T rad$

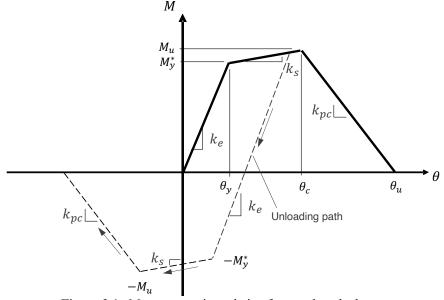


Figure 2.1. Moment-rotation relation for zero length element

### Exercise #3:

Consider the following column from Week #5:

$$A = 1.27 \cdot 10^4 mm^2$$
,  $I = 3.66 \cdot 10^7 mm^2$ ,  $E = 200,000 MPa$ 

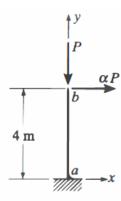


Figure 1. Column under axial and lateral loading

Consider material nonlinearity with the element model you developed in Exercise #2 and the element stiffness matrix you derived in Exercise #1. For a=0.05, determine the loaddisplacement relationship (secondary equilibrium path) of the cantilever member when:

- $\theta_p = \theta_c \theta_y = 0.02 \, rad$
- $\theta_{pc} = \theta_u \dot{\theta_c} = 0.05 \, rad$
- $M_y^* = 4000kN.mm$  and  $M_u = 4500kN.mm$
- 1. Would you use a displacement or load-control scheme for your solution? Explain your answer.
- 2. Compare the computed secondary equilibrium path for the following cases:
  - a. Case #1: Linear elastic analysis (from Week #5)
  - b. Case #2: Nonlinear geometric analysis and linear material (from Week #5)
  - c. Case #3: Nonlinear analysis with material nonlinearity and linear geometric transformation
  - d. Case #4: Nonlinear analysis with both material and geometric nonlinearities
- 3. Calculate the displacement at which the cantilever member reaches zero lateral strength (i.e., collapse) with your program.

### Notes:

The spring should be considered to be n times stiffer than the flexural stiffness of the elastic element:,

$$\circ \quad k_e^{spring} = \frac{n3EI_e}{L}, I_e = \frac{n+1}{n}I$$

The post-yield stiffness of the spring should be adjusted accordingly,
$$o k_s^{spring} = a_s \cdot k_e^{spring} = \frac{a_s^{member}}{1 + n \cdot (1 - a_s^{member})} \cdot k_e^{spring}$$

- O Strain hardening ratio of the member:  $a_s^{member} = \frac{M_u M_y^*}{K_e^{member} \cdot \theta_p}$

O Post-capping hardening ratio of the member:  $a_{pc}^{member} = -\frac{M_u}{K_e^{member} \cdot \theta_{pc}}$