



# Pumped storage hydropower

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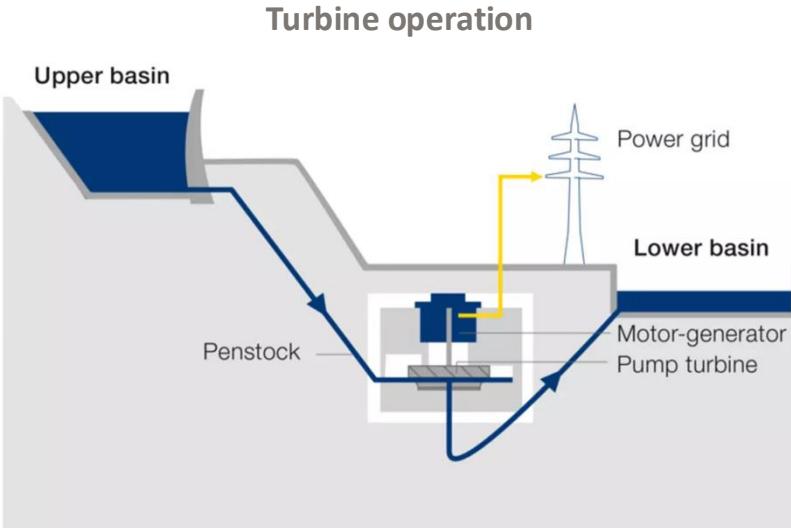
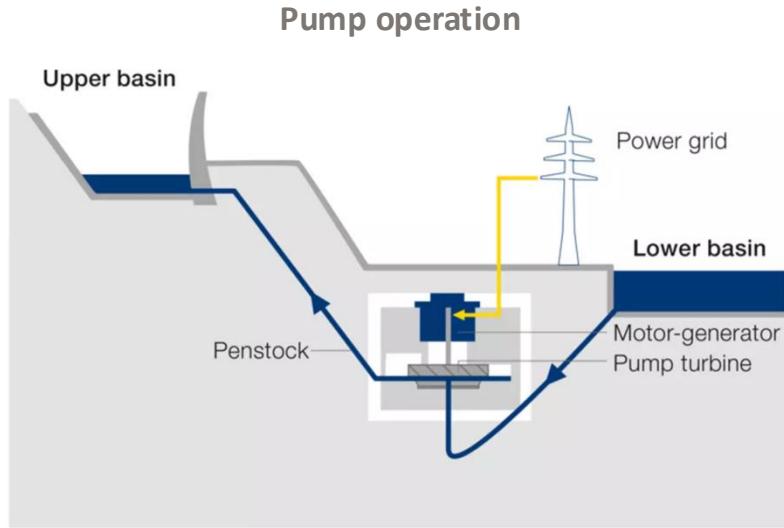
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# Introduction

- Pumped Storage Hydropower (PSH) is a type of hydropower that stores and generates electricity by using gravitational potential energy of the water
- At times of low demand and low electricity prices, the water is pumped to the higher reservoir
- At times of high demand and higher prices, the water is then released to drive a turbine in a powerhouse and supply electricity to the grid
- The energy storage capacity of a pumped hydro facility depends on the size of its two reservoirs, while the amount of power generated is linked to the size of the turbine.

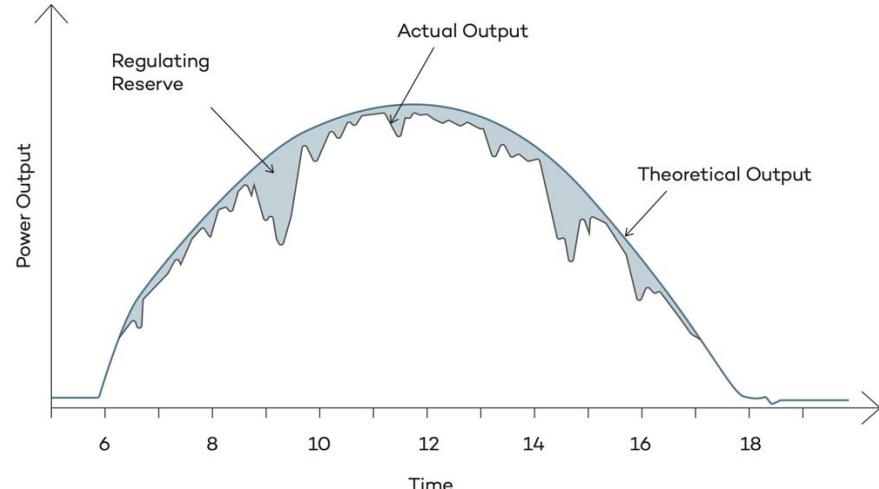
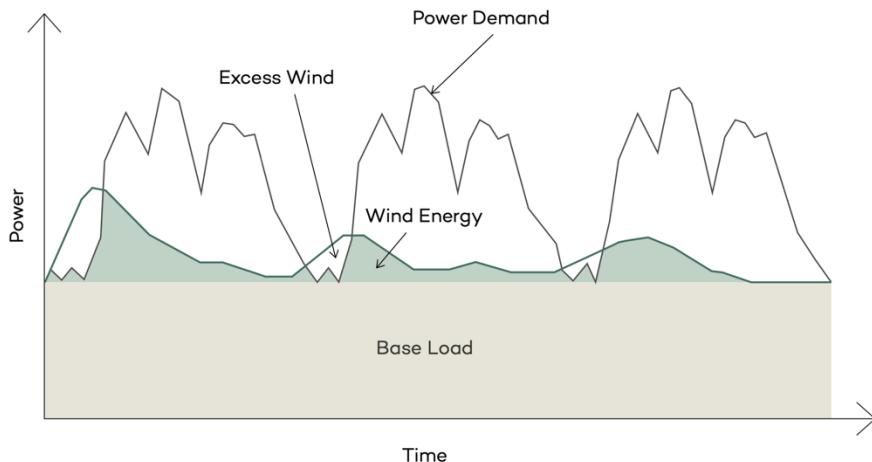
# PSH operating principle



- Power from the grid is used to operate the electric motor
- The electric motor powers the pump turbine
- The water is pumped from the lower basin to the upper basin
- Water flows from the upper basin through a penstock to the turbine
- The turbine powers the generator, producing electricity for the grid
- The water is discharged into the lower basin.

# The role of PSHs in enhancing grid flexibility

- PSHs can help mitigate the variability of renewable energy sources by providing regulating capacity to match power consumption
- Additionally, they balance the load in the system, absorbing energy during off-peak hours and meeting demand in peak times.



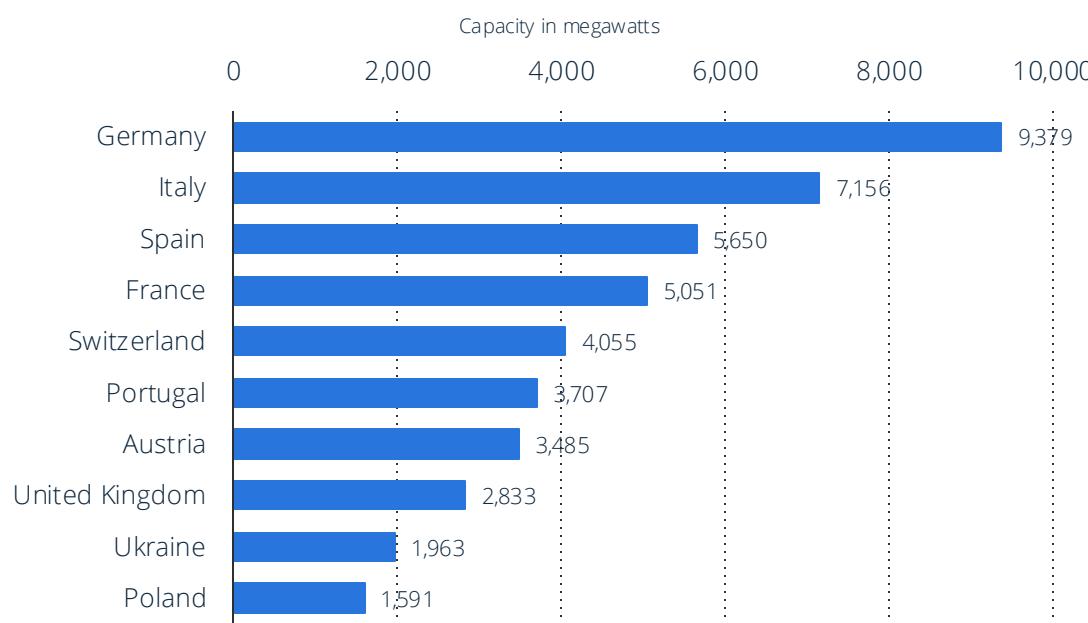
# Global pumped storage installed capacity

- In 2024, PSH accounted for over 90% of installed global energy storage capacity and over 92% of energy stored in grid scale applications.
- The total installed capacity is around 179 GW
- PSH storage capacity is expected to increase by 50% to about 240 GW by 2030
- Locations and data for existing and planned PSH projects: [here](#)



# Cumulative installed PSH capacity in Europe in 2023, by country (in MW)

Installed pumped storage capacity in Europe 2023, by country

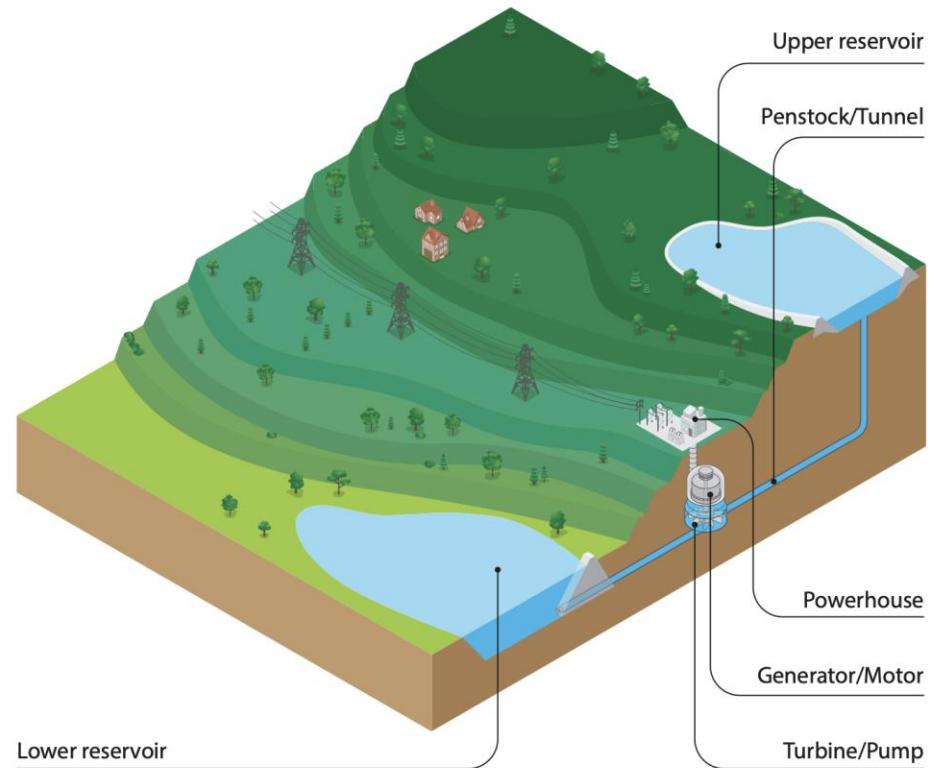


- The EU hosts 46 GW of PHS capacity, which is a quarter of the global installed capacity
- The 1.8 GW Grand' Maison PSH in the French alps is the largest PSH in Europe
- Switzerland is a leader in PSH with several high-capacity plants, including the 900 MW Nant de Drance

# Types of pumped-hydro

There are two main types of pumped-hydro:

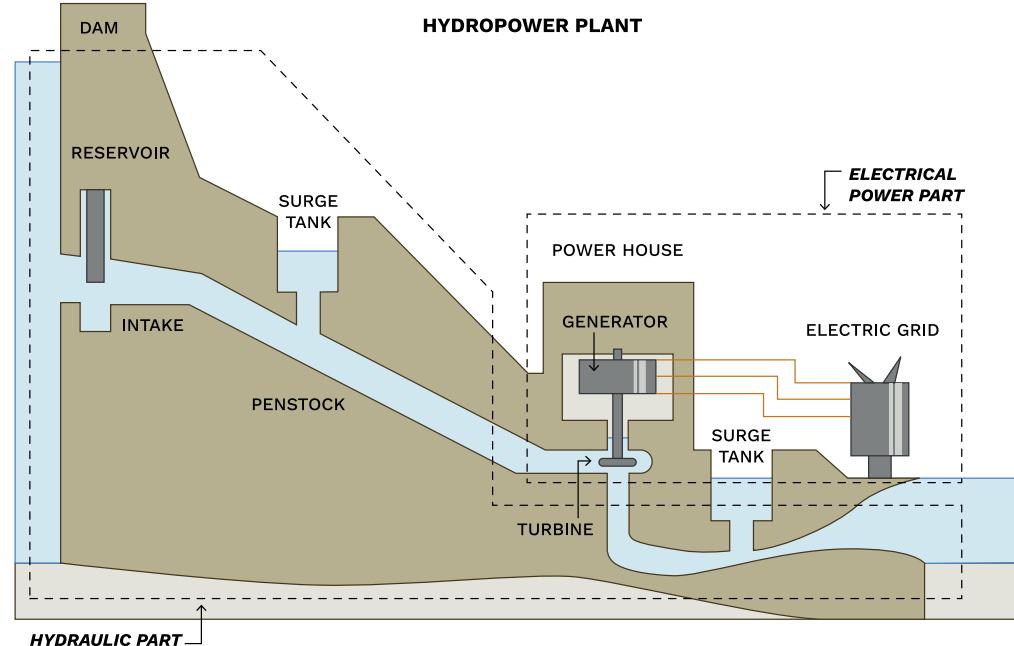
- **Open-loop:** with either an upper or lower reservoir that is continuously connected to a naturally flowing water source such as a river.
- **Closed-loop:** an 'off-river' site that produces power from water pumped to an upper reservoir without a significant natural inflow



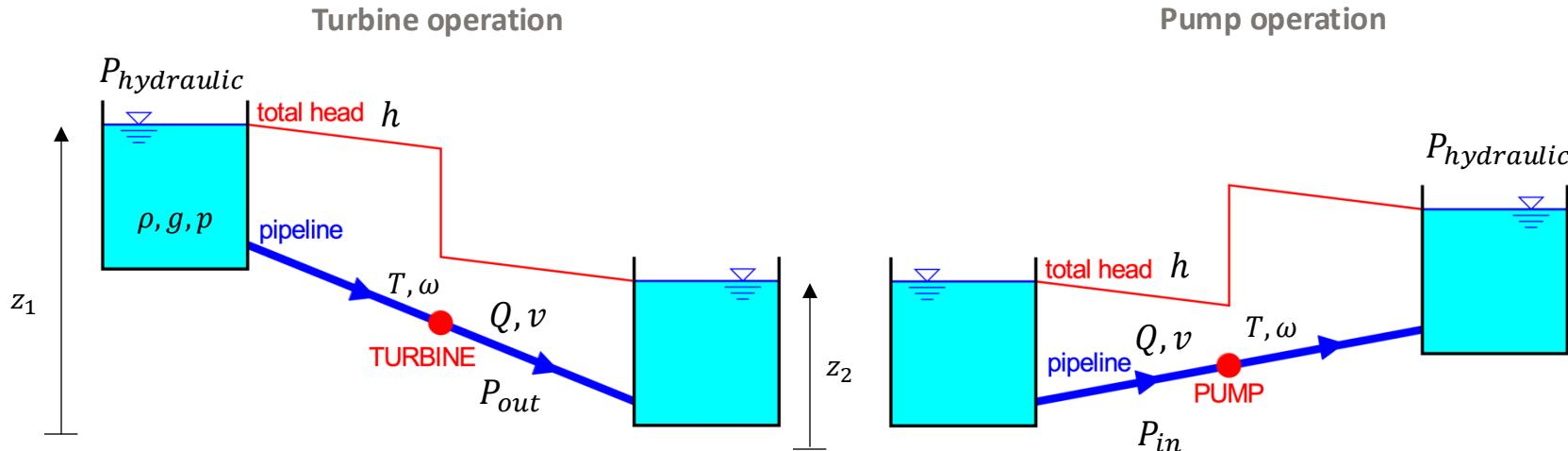
# Components of a PSH

The main components of a PSH are:

- **Dam:** a barrier that stops or restricts the flow of surface
- **Reservoir:** an enlarged lake behind a dam built to store water
- **Penstock:** open or closed channel to flow the water from the reservoir to the turbine
- **Surge tank:** cylindrical tank open at the top to control the pressure in penstock
- **Pump/Turbine:** convert the hydraulic energy of the water into the mechanical energy or pump the water upstream
- **Power house:** building provided to protect the hydraulic and electrical equipment such as the generator



# Hydraulic and mechanical quantities



where:

- $v$  is the fluid flow speed [m/s]
- $g$  is the gravity acceleration [m/s<sup>2</sup>]
- $z$  is the elevation of the point above a reference plane [m]
- $p$  is the pressure at the chosen point [Pa]
- $\rho$  is the density of the fluid at all points in the fluid [kg/m<sup>3</sup>]
- $Q$  is the flow rate = fluid flow speed x section of the tube [m<sup>3</sup>/s]

- $T$  is the mechanical torque [Nm]
- $\omega$  angular speed [rpm]

# Energy conversion

## Energy balance

$$E_{tot} = E_{pressure} + E_{potential} + E_{kinetic}$$

**Pressure energy** = work done to lift a fluid column due to pressure

$$E_{pressure} = p \cdot V$$

where  $V$  is the volume

**Potential energy** = energy held by a mass because its elevation

$$E_{potential} = mgz$$

**Kinetic energy** = energy held by a mass due to its motion

$$E_{kinetic} = \frac{1}{2}mv^2$$

In terms of **Head,  $h$**  (energy per unit weight =  $E/mg$ ) in meters:

$$h = \frac{E_{tot}}{mg} = \frac{p}{\rho g} + z + \frac{v^2}{2g}$$

Pressure head

Elevation head

Velocity head

# Power in turbine and pump modes

Hydraulic power = Energy / time

$$P_{hydraulic} = \frac{E_{tot}}{t} = \frac{m}{t} gh = \rho g Q h$$

where  $\frac{m}{t} = \rho Q \left[ \frac{kg}{m^3} \cdot \frac{m^3}{t} \right]$

Power balance in turbine mode:

$$P_{hydraulic} = P_{electric} + P_{losses} = P_{electrical} + P_{penstock,loss} + P_{turbine,loss} + P_{generator,loss}$$

Power balance in pump mode:

$$P_{hydraulic} = P_{electric} + P_{losses} = P_{electrical} + P_{penstock,loss} + P_{pump,loss} + P_{motor,loss}$$

# Efficiency in turbine and pump modes

Overall efficiency in turbine mode,  $\eta$ :

$$\eta = \frac{P_{electric}}{P_{hydraulic}} = 1 - \frac{P_{losses}}{\rho g Q h} = \eta_{penstock} \cdot \eta_{turbine} \cdot \eta_{generator}$$

Overall efficiency in pump mode,  $\eta$ :

$$\eta = \frac{P_{hydraulic}}{P_{electric}} = 1 - \frac{\rho g Q h}{P_{losses}} = \eta_{penstock} \cdot \eta_{pump} \cdot \eta_{motor}$$

**Turbine efficiency,**

$\eta_{turbine}$  :

$$\eta_t = \frac{P_{out}}{P_{in}} = \frac{P_{mech}}{\rho g Q h} = \frac{T \cdot \omega}{\rho g Q h}$$

**Pump efficiency,**

$\eta_{pump}$  :

$$\eta_p = \frac{\rho g Q h}{P_{mech}} = \frac{\rho g Q h}{T \cdot \omega}$$

**Penstock efficiency,**

$\eta_{penstock}$  :

$$\eta_{penstock} = \frac{P_{hydraulic,out}}{P_{hydraulic}}$$

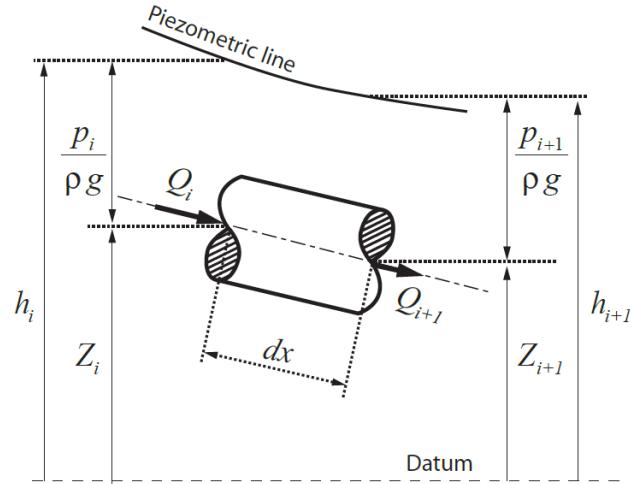
# Penstock losses

Head loss  $h_f$  due to friction (Darcy-Weisbach equation)

$$h_f = f \cdot \frac{L}{D} \cdot \frac{v^2}{2g}$$

where:

- $f$  is the Darcy friction factor
- $D$  is the penstock's diameter
- $L$  is the penstock's length



Penstock efficiency,  $\eta_{penstock}$ :

$$\eta_{penstock} = \frac{P_{hydraulic,out}}{P_{hydraulic}} = \frac{\rho g Q (h - h_f)}{\rho g Q h}$$

# Types of hydraulic machines (1/2)

$$h = \frac{p}{\rho g} + z + \frac{v^2}{2g}$$

## Impulse turbines:

Fluid's kinetic energy converted to mechanical work at the atmospheric pressure

- Change in velocity → Change in head

## Reaction turbines:

Both fluid's kinetic and potential energy is converted to mechanical work

- Change in pressure → Change in head

# Types of hydraulic machines (2/2)

The most common hydraulic machines used in hydropower plants are:

- **Pelton turbine** (impulse turbine)
  - very high head
  - nozzle
- **Francis pump-turbine** (reaction turbine)
  - medium head
  - reversible machines
- **Kaplan turbine** (reaction turbine)
  - low head
  - double control system



# 1-D Modelling of hydraulic circuits (1/2)

The governing equations are the continuity of mass and momentum:

$$\begin{aligned}\frac{\partial h}{\partial t} + \frac{a^2}{gA} \frac{\partial Q}{\partial x} &= 0 \\ \frac{\partial h}{\partial x} + \frac{1}{gA} \frac{\partial Q}{\partial t} + \underbrace{\frac{f|Q|}{2gA^2D}}_{\text{friction losses}} Q &= 0\end{aligned}$$

with state variables

$h$  : Head

$Q$  : Volumetric flow rate

parameters and constant

$A$  : Pipe cross-sectional area

$D$  : Pipe internal diameter

$f$  : Friction factor

$a$  : Speed wave (or celerity)

$g$  : Acceleration of gravity

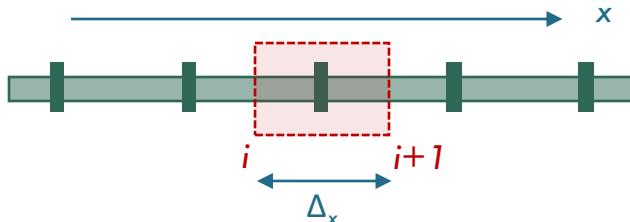
$\rho$  : Water density

# 1-D Modelling of hydraulic circuits (2/2)

Spatial discretization

$$\frac{\partial h}{\partial x} \Big|_{i+\frac{1}{2}} = \frac{h_{i+1} - h_i}{\Delta x}$$

$$\frac{\partial Q}{\partial x} \Big|_{i+\frac{1}{2}} = \frac{Q_{i+1} - Q_i}{\Delta x}$$



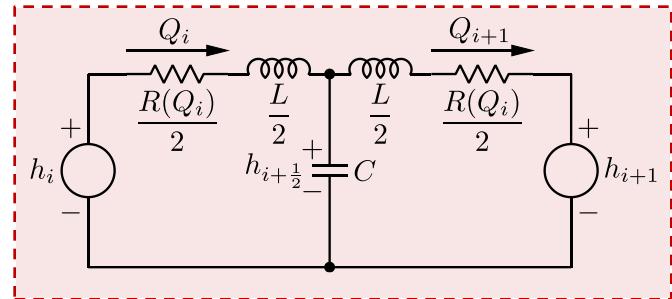
$$\begin{cases} h_{i+\frac{1}{2}} = \frac{1}{C} (Q_i - Q_{i+1}) \\ h_{i+\frac{1}{2}} = \frac{L}{2} \dot{Q}_i + \frac{R}{2} Q_{i+1} - h_i \\ h_{i+\frac{1}{2}} = \frac{L}{2} \dot{Q}_{i+1} + \frac{R}{2} Q_{i+1} + h_{i+1} \end{cases} \quad \text{where:}$$

$$C \triangleq \frac{gA}{a^2} \Delta x$$

$$L \triangleq \frac{1}{gA} \Delta x$$

$$R(Q) \triangleq \frac{\lambda |Q|}{2gA^2 D} \Delta x$$

Electrical equivalent circuit



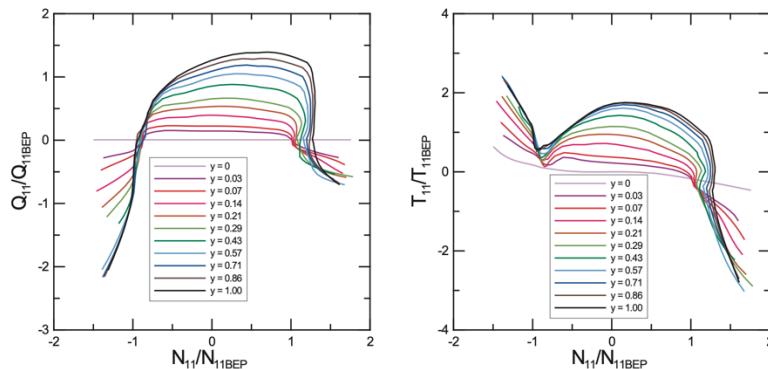
# 1-D Modelling of hydraulic machines (1/2)

- **Quasi-static model:** The transient behavior of the hydraulic machines can be simulated as a succession of different steady-state operating points
- **Characteristic curves:** capture the relation between the machine state variables (e.g., specific energy ( $gh$ ), torque, rotational speed, flow, guide vane opening)

# 1-D Modelling of hydraulic machines (2/2)

To display the characteristic curves it is convenient to eliminate one of the state by means of the **hydraulic machine similitude laws**:

$$N_{11} = \frac{N \cdot D_{ref}}{\sqrt{(E/g)}} \quad ; \quad Q_{11} = \frac{Q}{D_{ref}^2 \cdot \sqrt{(E/g)}} \quad ; \quad T_{11} = \frac{T}{D_{ref}^3 \cdot E/g}$$



Volumetric flow (left) and torque (right) characteristic curves of a Francis pump/turbine



The S-shape between the 1st and 4th quadrant leads to numerical troubles for the numerical tractability

Polar representation

# Polar representation of a Francis machine

The **pressure source H** of the model is directly driven from the turbine characteristic  $W_H(y, \theta(Q, N))$ :

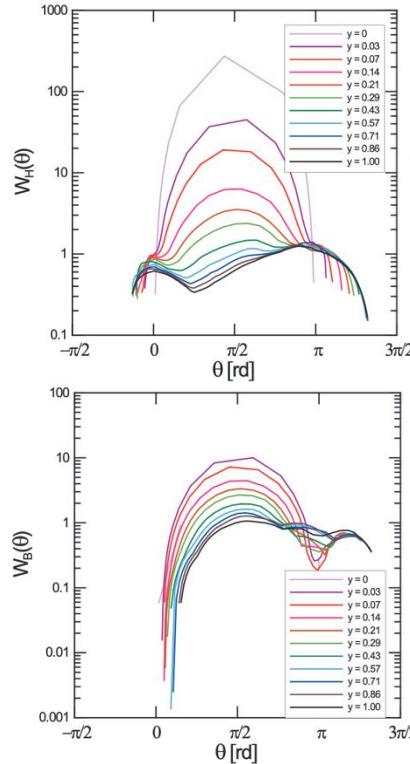
$$W_H(\theta) = \frac{H/H_{BEP}}{(Q/Q_{BEP})^2 + (N/N_{BEP})^2} \rightarrow \text{Best Efficiency Point}$$

The **mechanical torque T** of the pump-turbine is obtained from the torque characteristic  $W_B(y, \theta(Q, N))$ :

$$W_B(\theta) = W_H(\theta) \cdot \frac{T_{11}}{T_{11BEP}} = \frac{T/T_{BEP}}{(Q/Q_{BEP})^2 + (N/N_{BEP})^2}$$

The **turbine rotational speed  $\omega$**  is according to the momentum equation applied to the rotational inertias:

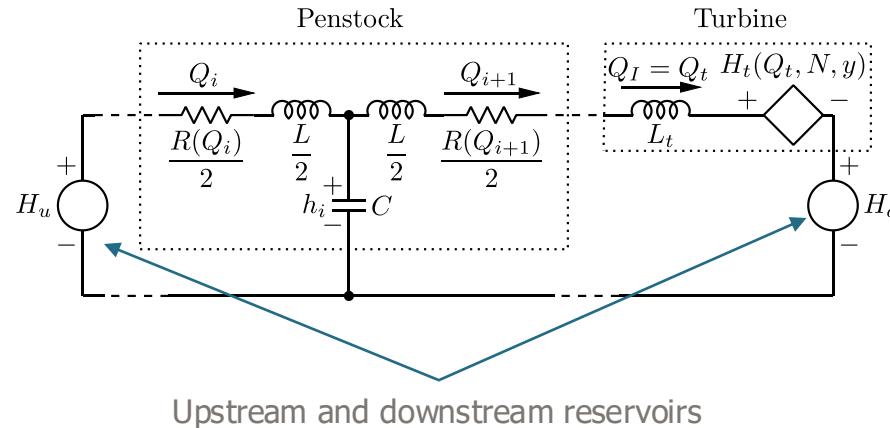
$$J \cdot \frac{d\omega}{dt} = \sum T_{ext} = T_{turb} - T_{elect}$$



Polar representation

# Equivalent circuit model of hydropower plants

**Equivalent circuit model (ECM)** of the complete plant obtained by combining the penstock model, turbine model, and reservoirs

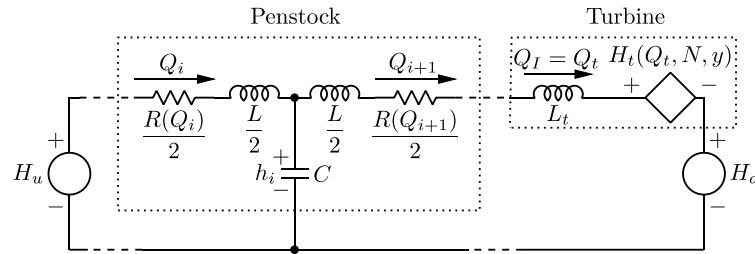


# State-space representation

Equivalent circuit model



State-Space representation



$$\dot{X} = A \cdot X + B \cdot u$$

$$\dot{X} = \begin{bmatrix} -R & 0 & -1 & 0 \\ \frac{L}{2} & -R & \frac{1}{L} & 0 \\ 0 & \frac{1}{L} & 0 & 0 \\ \frac{1}{C} & -1 & 0 & 0 \\ 0 & 0 & 0 & 0 \end{bmatrix} \begin{bmatrix} Q_1 \\ Q_2 \\ H_c \\ \omega \end{bmatrix} + \begin{bmatrix} \frac{1}{L} & 0 & 0 & 0 \\ 0 & -1 & 0 & 0 \\ 0 & \frac{1}{L} & 0 & 0 \\ 0 & 0 & 1 & 0 \\ 0 & 0 & 0 & \frac{1}{J} \end{bmatrix} \begin{bmatrix} H_u \\ H_t - H_d \\ T_t + T_{el} \end{bmatrix}$$



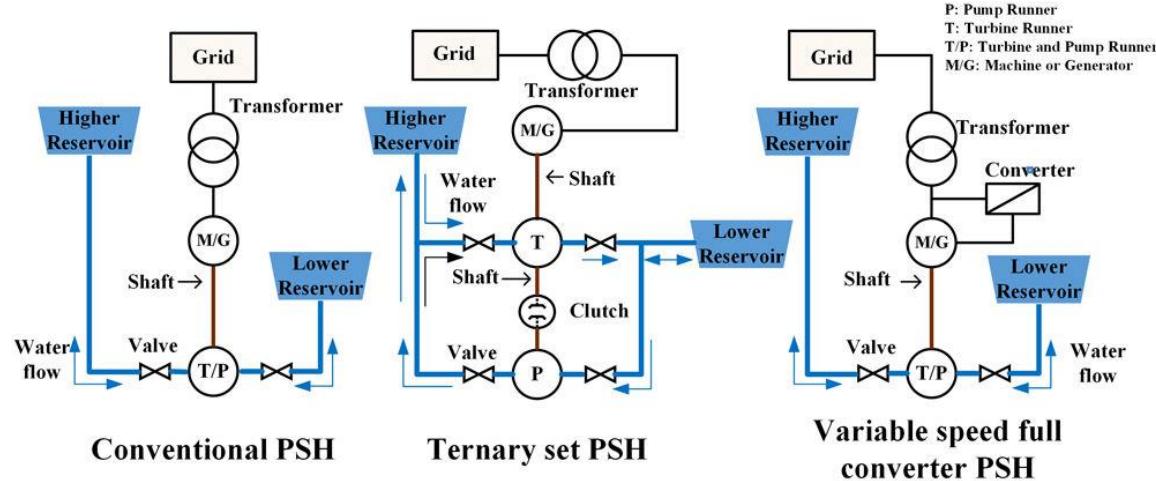
$H_t$  and  $T_t$  are obtained from  $W_H$  and  $W_B$  from the polar representation

Numerical integration: Runge-Kutta 4<sup>th</sup> order

# Pumped storage technologies

The pumped storage technologies can be grouped into three categories:

1. Conventional PSH
2. Tertiary set PSH
3. Variable-speed PSH



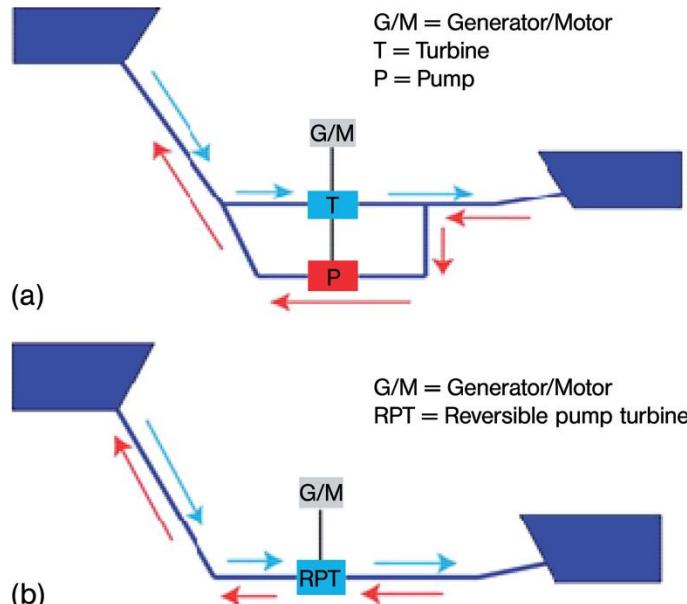
# Conventional PSH

Typical layout consists in:

1. install two separate aggregates, one pump and one turbine, or
2. use a machine that runs both ways (reversible pump turbines (RPTs))

PSH are characterized by:

- long lifetime expectancy (typically between 50 and 100 years)
- a round-trip efficiency of 70–80%
- fast response time in the order of seconds or minutes.



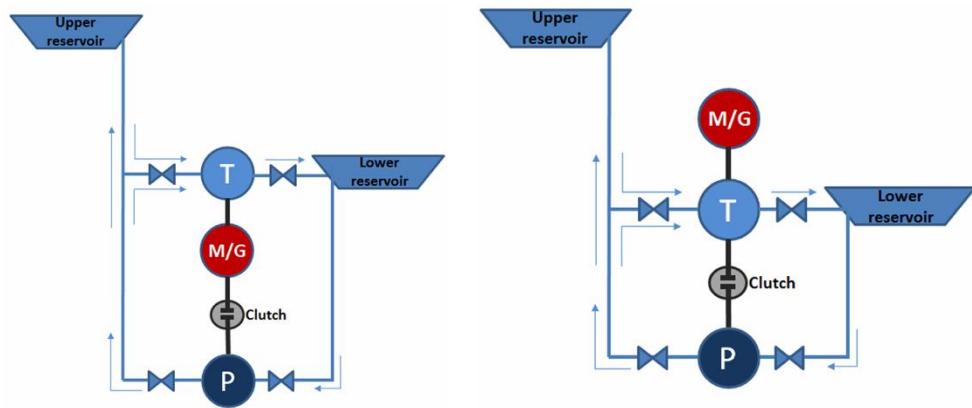
# Tertiary set PSH

Tertiary set consist of a motor-generator, a separate turbine (e.g., Francis or Pelton) and a pump set

Typical layout consists in:

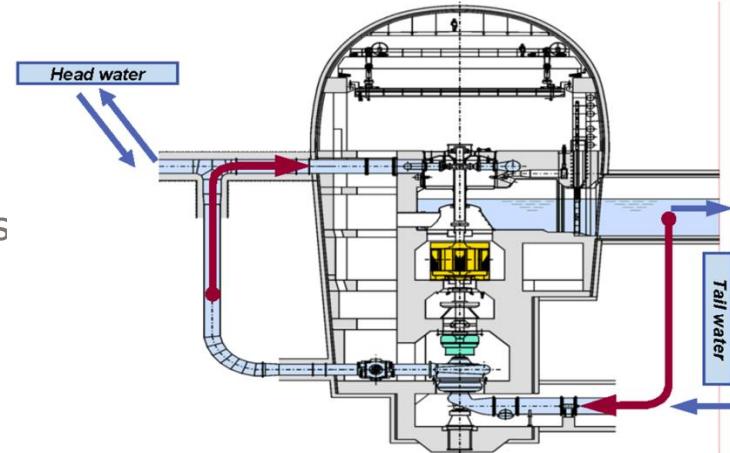
1. vertical shaft and the motor-generator located between the turbine and the pumps
2. motor-generator located at the top, above both the turbine and the pump

A **clutch** is inserted on the shaft above the pump to disconnect the pump during turbine operation to avoid ventilation losses in the pump



# Operating modes of tertiary units

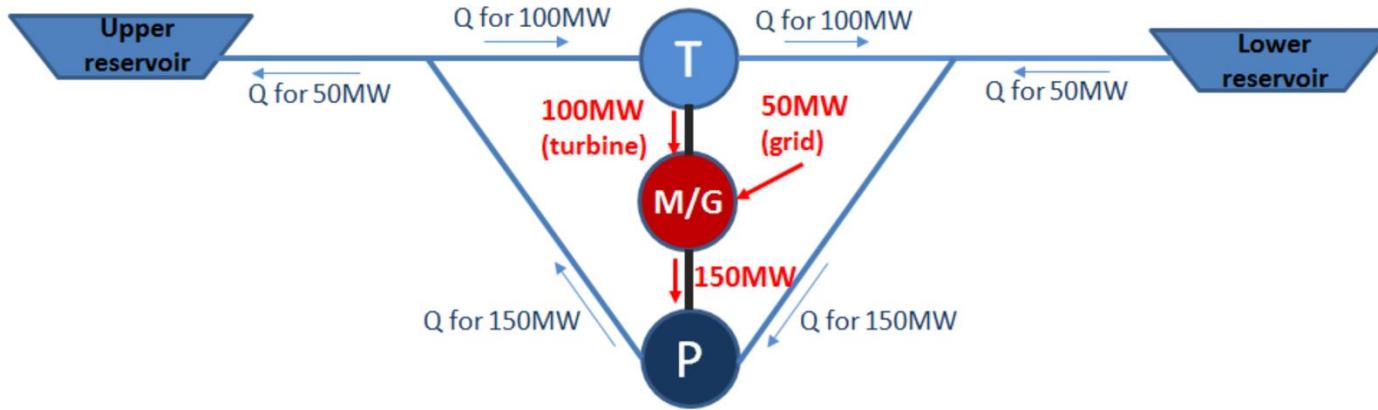
- **Generating mode:** the guide vanes to the pump are closed and the clutch is not engaged. The amount of generation is controlled by the position of the turbine guide vanes.
- **Pumping mode:** the guide vanes to the turbine are closed and the clutch is enabled. The pump guide vanes are wide open and there is no regulation capability.
- **Pumping/generating mode:** the clutch is enabled and both the pump and the turbine operate



This is referred to as a “hydraulic short circuit”

# Hydraulic short circuit

Example of operation in pumping mode with regulation capability using the Hydraulic Short-Circuit concept



- The power applied to the shaft from the pump is 150 MW drawn from the power system
- The turbine guide vanes are adjusted so that the turbine supplies 100 MW to the power system
- The net result is that 50 MW is drawn from the power system, and the flow pumped up to the reservoir is equivalent to 50 MW

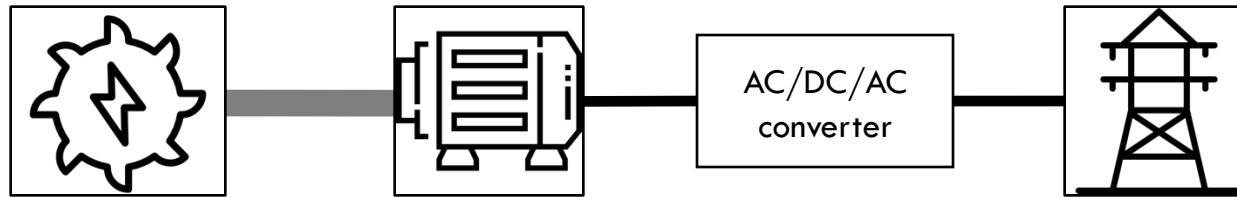
# Advantages of tertiary units

## Main advantages:

- Ternary plant can simultaneously operate both the pump and turbine
- The shaft rotates in the same direction in both operational mode reducing hydraulic transients and increasing the time to switch from turbine to pump mode
- Round-trip efficiency of up to 82%

# Variable-speed pump turbines

The variable-speed technology allows to decouple the grid and generator frequency



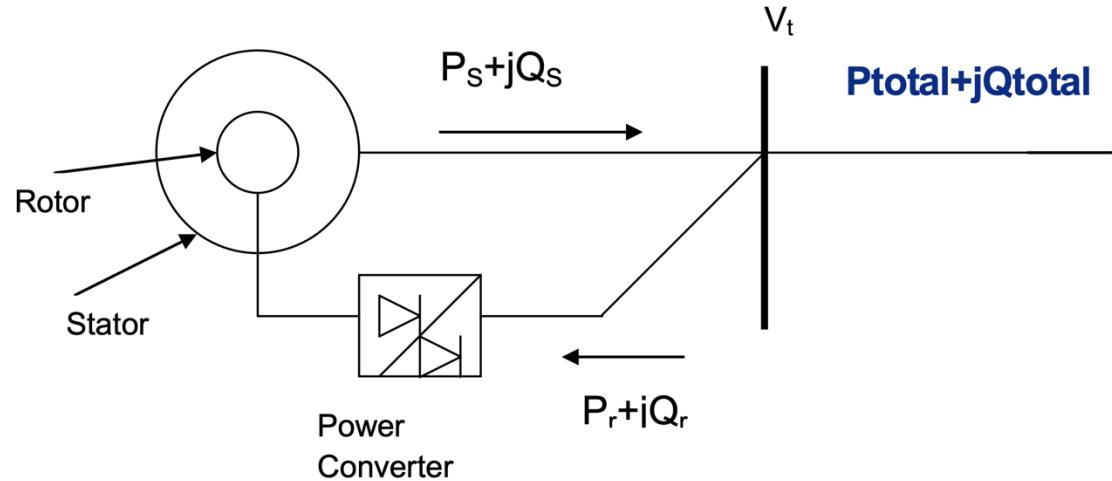
The main procedures adopted to obtain variable speed are:

- Synchronous Generator with back-to-back frequency converter
- Double-Fed Induction Generator (DFIG)

**Main Advantage:** the recovery of efficiency of the hydraulic turbine (almost to its rated efficiency) when working at a head lower than the design head.

# DFIG working principle

- The rotor of the machine is connected to the machine's terminal through a power converter
- The power converter can control the voltage, current and frequency in the rotor circuit
- The DFIG rotor must be excited with a complement frequency to achieve the rated frequency in the stator generated voltage



# Grand' Maison PSH, France

- **Nominal power:** 1.8 GW with 4x 158.5 MW multi-jet Pelton turbines and 8x 154MW Francis pump-turbines
- **Energy capacity:** 1420 GWh
- **Storage capacity:** 137 millions of m<sup>3</sup> of water in the upper reservoir
- **Gross head:** 918 m
- **Technologies tested:** Hydraulic short circuit



# Nant de Derance PSH, Switzerland

- **Nominal power:** 900 MW with 6x150 MW Francis pump-turbines
- **Energy capacity:** 200 GWh
- **Storage capacity:** 25 millions of m<sup>3</sup> of water in the upper reservoir
- **Turbine to pump switching time:** less than 5 minutes
- **Net head:** 71.1 m



# References

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- [12] Nant de Derance website: <https://www.nant-de-drance.ch>

# To do

1 cluch disaccoppiamento

2 turbine

3 set up grand maison

4 tutte le tecnologie disponibili

5 esercizio in matlab: didattico per spiegare bene le basi metodologiche: trovare la Potenza a partire dalla discharge etc (come il mio modello)

6 definizione di energia

7 efficienza? numeri